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THESIS

PRELIMINARY PROPELLER SELECTION USING THE
WAGENINGEN B-SCREW SERIES AND A
GENERAL PURPOSE NON-LINEAR OPTIMIZER

by

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June 1983

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(20. ABSTRACT Continued)

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- 1) diameter limitation
- 2) cavitation limit on expanded area ratio using Keller's criterion
- 3) strength requirement determined by an empirical relation and by a method developed by Schoenherr with modifications by the author.

Objective functions considered are maximized open water efficiency and minimized propeller blade weight. Optimized solutions to specific problems previously presented by other authors are obtained and results are compared.

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Preliminary Propeller Selection Using the
Wageningen B-Screw Series and a
General Purpose Non-Linear Optimizer

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from the

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June 1983

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This thesis presents the use of a general purpose non-linear optimization program in the preliminary stage of ship design for the selection of a propeller based on methodical series propeller test data. The propeller series utilized is the well-known Wageningen B-Series. Three (3) "Design Cases", representing the thrust, power and matching approaches to powering problems, are formulated as FORTRAN subprogram analysis codes for solution by the synthesis/optimization program COPES/CONMIN. Designer constraints considered are:

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If someone says it's impossible, then, quite obviously, he has never done it.

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I. INTRODUCTION

A. BACKGROUND

The ship design process, in its most rudimentary form, has been formulated and tracked by the utilization of the classical design spiral (see Figure 1.1). The design follows a convergent helical path past each major milestone "spoke" until, after numerous iterative cycles, the final configuration is "centered" upon. Whether one attempts to segregate the principal phases of Preliminary, Advanced and Contract Design into separate spirals or combine these phases in series along the entire path to the center, it is not long before the designer's roughed-out sketches give way to serious "number crunching", specifically that of propulsion power estimation.

To estimate the power required to drive the ship through the water at its design speed, a decision must first be made as to what type of propulsor (i.e., propeller, water jet, paddle wheel, etc.) will be used. For the average case, and for the discussion that follows, the marine propeller is chosen to be the propulsion device. Since

a ship propeller may be regarded as a transducer that converts the rotational power transmitted through the shaft into the translational power to propel the ship, [Ref. 18: p. 10]

the selection and design of this device is obviously an important factor in the eventual size (weight and power) of the ship's propulsion plant. While hydrodynamicists provide

TRADITIONAL DESIGN SPIRAL

LBP, L/B , B/H , C_P , C_X

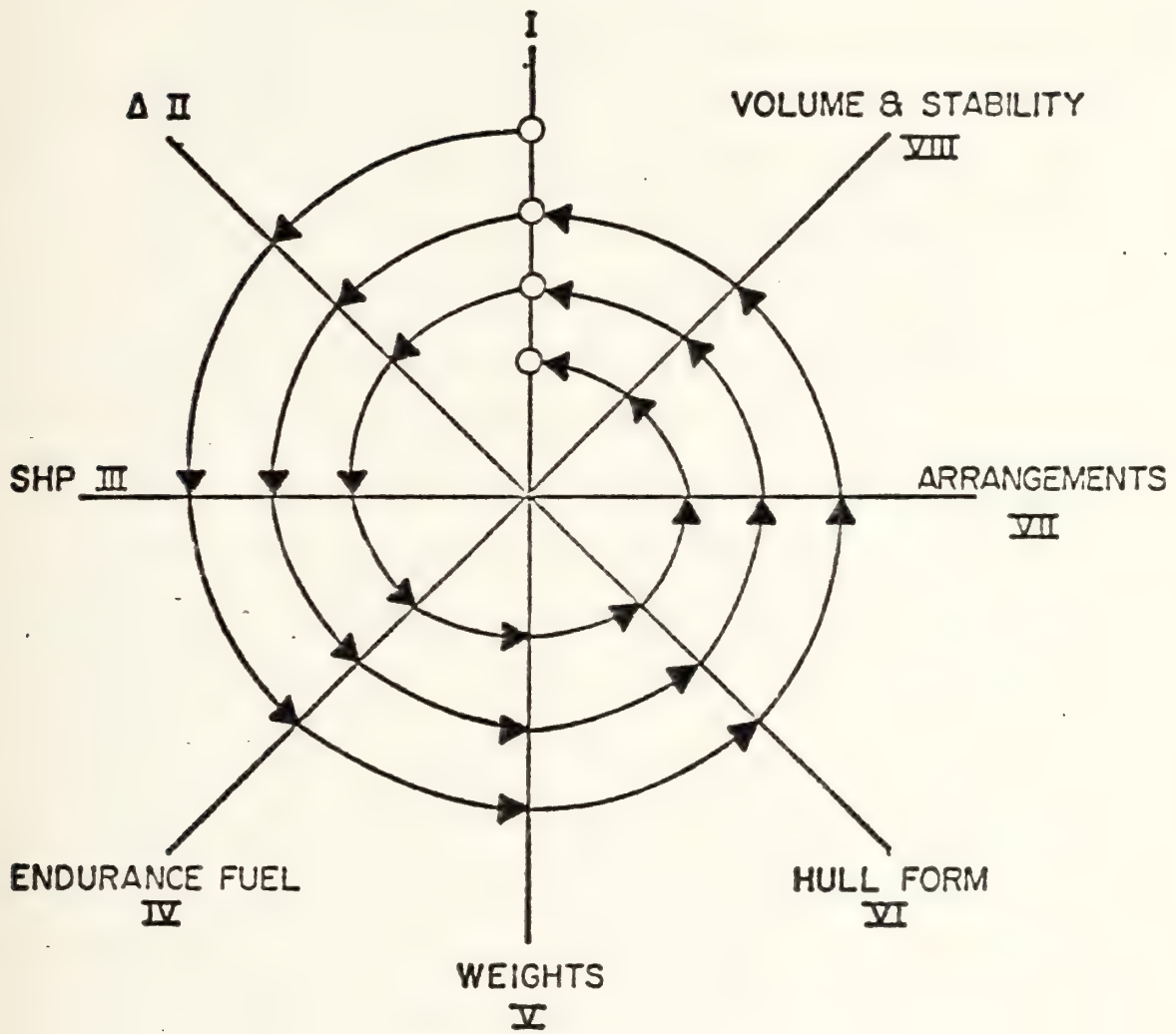


Figure 1.1 Traditional Design Spiral

a myriad of theories and techniques to generate a "custom built" (i.e., wake adapted) propeller for the ship under consideration, their expertise is usually not required in the early stages of preliminary design simply because the design has not been refined enough beyond gross estimates. At this stage, the designer strives to formulate what is possible based on previous experience. For preliminary power estimation, previous propeller designs (i.e., "stock" propellers) and results from methodical series of model propellers are analyzed by the designer in order to select the "best" available propeller under various conditions posed by the problem under consideration. Three examples of typical problems encountered in preliminary ship design are:

- 1) Given the ship's effective horsepower at a specific speed and estimates of hull performance parameters, which propeller, as determined by certain principal characteristics, will require the least amount of delivered power from the propulsion plant?

- 2) Given the delivered power from a specific propulsion system in terms of torque and revolution rate at the propeller/shaft interface and estimates of hull performance parameters, which propeller will generate the largest effective horsepower and speed parameters?

- 3) Given a ship's effective horsepower and speed, various hull performance parameters, and the propulsion plant's delivered power characteristics, which propeller will "match"

these requirements at a minimum amount of weight for a specified material?

(Author's Note: For the sake of brevity, the three selection problems just cited will, henceforth, be referred to as "Design Case No. 1", "Design Case No. 2" and "Design Case No. 3", respectively.)

For this study, the methodical propeller series method is viewed as the designer's choice for preliminary powering analysis. One of the most-widely used methodical series data on model propellers is the Wageningen B-Screw Series. Initially, the results of the series were presented as tabulations of non-dimensional thrust and torque coefficients (K_T and K_Q respectively) versus the non-dimensional advance ratio (J) for analytical work and as the familiar " $B_p-\delta$ " and " $B_u-\delta$ " diagrams for design purposes. As "trial & error" design methods performed by hand in all engineering disciplines gradually transcended to numerical manipulation by the modern digital computer, the necessity for the adaption of the Series results to a format suitable for use in computer-aided design methods became obvious. This was accomplished through multiple regression analysis of the original open-water test data of the 120 propeller models in the Series and presented in the form of polynomial expressions for " K_T " and " K_Q " [Refs. 1,2].

The adaptation of the Wageningen B-Screw Series polynomials to various types of propeller selection problems formulated

for computer solution has been implemented recently by two authors. Triantafyllou [Ref. 3] and, of late, Markussen [Ref. 4] presented different propeller selection problems and proposed different schemes for computer-aided "optimized" solutions. In short, specific expressions for the constraints imposed and the objective (optimality condition) to be maximized, expressed in terms of a number of design variables and parameters, were developed. Then, each system of equations was solved by a Newton-Raphson method to give a solution set of the design variables which maximized the objective and met all constraints.

Rather than formulating and coding a different optimization scheme each time a propeller selection problem presents a different combination and number of design parameters, variables and constraints, a better approach would involve formulating the problem (constraints and objective function) once in terms of all design parameters and variables and utilizing a general purpose optimization scheme which can handle any combination and number of constraints and design variables. This alternative certainly allows the designer more flexibility in solving his problem. Moreover, it eliminates repetitive coding and debugging associated with the implementation of a computer-sided solution for each particular design problem.

B. PROBLEM STATEMENT

The problem, then, is that the previously cited computer-aided "optimized" solutions to the propeller selection problem are not broad enough in capability to handle variations in the problem formulation. The objective of this thesis is to apply an available general purpose optimization computer code to the solution of various propeller selection problems encountered in Preliminary Ship Design in order to enhance the flexibility of the selection procedure.

C. SCOPE

To achieve the stated objective, the general purpose non-linear optimization code CONMIN [Refs. 5,6] together with the engineering synthesis code COPES [Ref. 7] (hereafter referred to collectively as COPES/CONMIN) is utilized in the solution of the three previously cited preliminary design propeller selection problems. Using the Wageningen B-Screw Series propeller characteristics expressed in polynomial expressions of various design variables, three "analysis" codes, required by COPES/CONMIN, are developed in such a way that various combinations of design variables and constraints are used, thereby demonstrating the applicability of the COPES/CONMIN optimization program in the solution of propeller selection problems.

D. THESIS ORGANIZATION

The remainder of the thesis is organized in the following manner.

Chapter II presents a short description of the optimization problem in general terms and a follow-on discussion of the COPES/CONMIN optimization program and the mathematical techniques employed therein.

Chapter III introduces definitions and concepts applicable to the propeller selection problem. A subsequent discussion on the Wageningen B-Screw Series is followed by final comments on constraints imposed on the propeller selection problem.

Chapter IV presents the formulation of the propeller selection problem as a design optimization problem which can be solved using COPES/CONMIN.

Chapter V discusses the background, formulation and programming utilized in estimating a propeller blade's weight for subsequent consideration as an objective function.

Chapter VI reviews the author's modifications to the propeller strength analysis developed by Schoenherr [Ref. 8] in the early 1960's for the American Bureau of Shipping. A subsequent discussion on the programming details of FORTRAN codes, which are utilized for the determination of adequate propeller blade strength, completes the chapter.

Chapter VII reviews the formulation and programming for the analysis code which is used in solving propeller selection problems represented by Design Case No. 1. Sample solutions are presented and compared to those presented previously by other authors.

Chapters VIII and IX consider Design Case No. 2 and Design Case No. 3 selection problems, respectively, in a similar fashion to Chapter VII.

Chapter X, the final chapter, presents the author's conclusions and recommendations.

As a final note, all computer coding presented in this thesis is done in FORTRAN IV, the language used by COPES/CONMIN. For the reader's convenience, Appendix A provides a cross-reference of the symbols presented throughout the thesis to appropriate FORTRAN variable names appearing in the author's codes.

II. OPTIMIZATION

A. INTRODUCTION

The purpose of this chapter is to introduce definitions and concepts used in the formulation and solution of the general optimization problem. Then, a short discussion on the theory and implementation details of COPES/CONMIN is presented.

For further study on the theory and methods of optimization, the reader is directed to the texts by Fox [Ref. 9], Fiacco and McCormick [Ref. 10], and Himmelblau [Ref. 11].

B. DEFINITIONS

Before discussing the techniques of optimization and their application to engineering problems, some preliminary definitions of basic terminology should be stated. Terms which have relevant significance are:

1) Parameters--The numerical quantities for which values are assigned to produce a design are called parameters. From this, it follows that a design may be specified by a vector \bar{D} containing "p" components, each of which is associated with a parameter. That is:

$$\bar{D} = \begin{pmatrix} D_1 \\ \vdots \\ D_p \end{pmatrix} \quad (2.1)$$

However, in a design process, the parameters are determined by some logical procedure through analysis of some kind. Some might take on fixed values to become "preassigned" parameters. Interrelationships among other parameters might exist so that only some of the parameters are changed when one design is compared to another. This consequence leads to the definition of "design variable".

2) Design Variables--The parameters for which values are chosen in some fashion to produce a design are called design variables. They represent an ordered collection of components which is a subset of the design vector \bar{D} . This subset is unique in that its components are "variable", i.e., they may take on different values in the design process. Having "preassigned" or fixed some of the design's parameters and only allowing the remaining "design variables" to change, leads to the conclusion that a design is now uniquely specified by a vector \bar{X} containing "n" components ($n \leq p$), each of which is associated with a design variable. That is:

$$\bar{X} = \begin{Bmatrix} X_1 \\ \vdots \\ X_n \end{Bmatrix} \quad (2.2)$$

3) Objective Function--The computable function of all or some of the design's preassigned parameters and/or design variables, with respect to which the design is to be optimized, is called the objective function. Single valued in

quantitative terms, the objective function's minimum or maximum value represents the "best" obtainable or "optimized" design. It is expressed as $F(\bar{D})$ to show its dependence on the design's parameters. But, since a design can be uniquely defined by \bar{X} alone, then clearly $F(\bar{X})$ suffices as an expression for the objective function.

4) Constraints--Restrictions on the design which must be satisfied in order to produce an acceptable design are called constraints. A constraint may be classified as a "side" or a "behavior" constraint. A side constraint restricts or bounds the range of the design for reasons other than direct consideration of performance. The side constraint on the "i"th design variable may be expressed as:

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n \quad (2.3)$$

A constraint derived from those performance or behavior requirements that are explicitly considered is called a behavior constraint. Most often, it appears as a computable functional relation involving the design's parameters, both preassigned and variable alike. The relation may be an inequality so that the "j"th of "m" inequality constraints can be expressed as:

$$G_j(\bar{D}) \leq 0 \quad j = 1, \dots, m \quad (2.4)$$

Alternatively, the relation may be an equality on the "k"th of " ℓ " equality constraints expressed as:

$$H_k(\bar{D}) = 0 \quad k = 1, \dots, \ell \quad (2.5)$$

Of noteworthy importance here is that, as before, if some of the design's parameters are preassigned, then the resulting design is that defined by \bar{X} which contains only the parameters that can be varied in the design process, i.e., the design variables. Therefore, constraints imposed upon the design may be expressed under one equation as:

$$\begin{aligned} G_j(\bar{X}) &\leq 0 & j &= 1, \dots, m \\ H_k(\bar{X}) &= 0 & k &= 1, \dots, \ell \\ x_i^{\text{lower}} &\leq x_i \leq x_i^{\text{upper}} & i &= 1, \dots, n \end{aligned} \quad (2.6)$$

A final form for constraints is that of the discrete-valued design variable.

5) Feasible Design--A design in which specified constraints are satisfied is called a feasible or "acceptable" design.

6) Infeasible Design--A design in which constraints are violated is called an infeasible or "unacceptable" design.

C. PROBLEM STATEMENT

If one presupposes that a range of designs exists within a selected design concept, then it follows that different

methodologies also exist by which one may choose the parameters which describe the design. One such method is optimization where parameters are chosen in a way that the design will satisfy all of the limitations and restrictions imposed upon it and will be "best" in some sense. In view of the foregoing definitions, optimization is then a selection method applied to a design problem by which an objective function $F(\bar{D})$ is minimized to produce an acceptable design which satisfies a certain set of requirements called constraints.

Formulated mathematically, the general, non-linear, constrained optimization problem may be stated under one equation as:

$$\text{Minimize: } F(\bar{D}) = \text{OBJ}$$

$$\text{Subject to: } G_j(\bar{D}) \leq 0 \quad j = 1, \dots, m \quad (2.7)$$

$$H_k(\bar{D}) \leq 0 \quad k = 1, \dots, \ell$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Again, as pointed out in the previous section, the design may be uniquely defined by just its design variables as specified by \bar{X} when some parameters are preassigned. Thus, the general, non-linear, constrained optimization problem can now be stated under one equation as:

$$\text{Minimize: } F(\bar{X}) = \text{OBJ}$$

$$\text{Subject to: } G(\bar{X}) \leq 0 \quad j = 1, \dots, m \quad (2.8)$$

$$H_k(\bar{X}) = 0 \quad k = 1, \dots, \ell$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Solutions methods for this optimization problem are abundant. Those pertaining to the linear and quadratic optimization problems involving a few design variables are most often presented in graphical or analytic form, although numerical schemes are, by no means, a dormant form. Structural and thermal problem solutions are most prevalent. However, as the optimization problem becomes more complex in terms of non-linear relationships among an increasing number of design variables and of an increased number of design constraints, numerical or mathematical programming techniques dominate the solution methods.

To limit the scope of this discussion, only the numerical techniques relevant to COPES/CONMIN will be considered. For more background on optimization techniques and applications, the reader is directed to a recent paper by Vanderplaats [Ref. 12] which presents a concise, but thorough, qualitative review of optimization. Although this paper deals exclusively with the application of design optimization to structural problems, it also contains a very extensive and current list of references on general techniques and applications of optimization.

D. COPES/CONMIN

As previously stated in Chapter I, COPES/CONMIN is the collective acronym for the FORTRAN program utilizing the optimization code CONMIN and the synthesis code COPES. COPES stands for Control Program for Engineering Synthesis; CONMIN is an acronym for CONstrained function MINimization.

1. CONMIN

CONMIN is a FORTRAN program, in subroutine form, which solves the general non-linear constrained optimization problem as stated:

$$\text{Minimize: } F(\bar{X}) = \text{OBJ}$$

$$\text{Subject to: } G_j(\bar{X}) \leq 0 \quad j = 1, \dots, m \quad (2.9)$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, n$$

Equation (2.9) applies to the entire statement. Observe that equation (2.9) differs from equation (2.8) in that the equality constraint set, given by $H_k(\bar{X}) = 0$, is not specified. This is because the version of COPES/CONMIN used in this study does not consider these types of constraints. However, this will not pose any difficulty in solving the propeller selection problems previously cited.

Again, $F(\bar{X})$ is the objective function (OBJ). The vector \bar{X} contains the "n" design variables (NDV). $G_j(\bar{X})$ are the "m" inequality behavior constraints (NCON) imposed on the optimization problem; x_i^{lower} and x_i^{upper} are the

respective lower and upper side constraints which bound the "design space" over which $F(\bar{X})$ and $G_j(\bar{X})$ are defined. As functional relationships involving \bar{X} , $F(\bar{X})$ and $G_j(\bar{X})$ may be implicit or explicit, but, in any event, must be continuous and have finite numerical values.

When the inequality condition of equation (2.9) is not satisfied, i.e., $G_j(\bar{X}) > 0$ for any constraint, the constraint is said to be violated. If the equality condition is met, i.e., $G_j(\bar{X}) = 0$ for any constraint, the constraint is said to be active. And, finally, if the inequality condition is satisfied, i.e., $G_j(\bar{X}) < 0$, for any constraint, that constraint is termed inactive. Any design, defined by \bar{X} , which satisfies the inequalities of equation (2.9) is designated as a feasible design. Likewise, any one which violates these inequalities is termed an infeasible design. The feasible design with the minimum objective function value, often referred to as the "minimum feasible design", will, therefore, be the optimum design.

During the optimization process, CONMIN employs the Fletcher-Reeves algorithm [Ref. 13] for locally unconstrained problems, and Zoutendijk's method of deasible directions [Refs. 14,15] for locally constrained problems, in a numerical procedure which attempts to minimize the objective function, $F(\bar{X}) = \text{OBJ}$, until one or more of the constraints, $G_j(\bar{X})$, becomes active. The numerical search procedure begins with an initial \bar{X} vector which may or may not specify a feasible

design. Modifications are included in CONMIN so that, if the initial design is infeasible, a feasible solution will be obtained with minimal increase in $F(\bar{X})$. By iteratively updating the design vector \bar{X} by the following relation:

$$\bar{X}^{(q+1)} = \bar{X}^{(q)} + \alpha^* \bar{S}^{(q)} \quad (2.10)$$

the optimization process continues by following the constraint boundaries in a direction of search \bar{S} so that the value of $F(\bar{X})$ decreases with each iteration q . The scalar α^* defines the distance of travel in the direction of search \bar{S} . The process terminates when a vector \bar{X} is found such that no further decrease in $F(\bar{X})$ can be made. The vector \bar{X} is considered to be optimal and, at least, a local minimum.

CONMIN can be used alone as a subroutine in any FORTRAN program where numerical optimization is desired. However, in order to make the optimization process more "user-friendly", CONMIN has been coupled to COPES in order to simplify its application to various types of problems. Further information on CONMIN can be found in previously cited references [5] and [6].

2. COPES

COPES is a FORTRAN program that provides automated design and trade-off capability to the design engineer. It utilizes the optimizer CONMIN to provide the following six specific capabilities:

- 1) simple analysis
- 2) optimization
- 3) sensitivity analysis
- 4) two variable function space analysis
- 5) optimum sensitivity
- 6) optimization using approximation techniques

During the execution of COPES, say for optimization, three principal tasks are performed:

- 1) data management on the design variables and constraints through location assignments in a FORTRAN common block called GLOBCM.
- 2) decision process control on the attainment of an optimal design vector \bar{X} through multiple calls to the optimizer until a minimum or maximum value of OBJ is achieved and all $G_j(\bar{X})$ are satisfied.
- 3) evaluation of OBJ and $G_j(\bar{X})$ at each \bar{X}^q and \bar{X}^{q+1} when ICALC = 2 through multiple calls to the user-provided analysis subprogram, SUBROUTINE ANALIZ.

For the application under consideration in this study, only the optimization capability will be used. Therefore, further elaboration on the other capabilities is not warranted.

Reference [7] is the user's manual for COPES/CONMIN. Details on the mechanics of user implementation are presented with subsequent illustration by example. The reader is, therefore, encouraged to familiarize himself with the reference. However, at this point, it is sufficient to be aware of the fact that a user of COPES/CONMIN is required to:

- 1) provide a FORTRAN subroutine called ANALIZ which performs the input of preassigned parameters, the evaluation of the objective function and constraints during the analysis phase of the optimization search and the output of the results.

- 2) provide an assembled deck of control cards required by COPES.

E. CONCLUDING NOTE

The field of optimization is both extensive and complex and, therefore, the foregoing presentation is, by no means, complete in every detail. However, it is felt that the preceding overview, in conjunction with the cited references, covers the necessary prerequisites that will enable the reader to follow the application of COPES/CONMIN to the various propeller selection problems in the chapters that follow.

III. POWERING, PERFORMANCE AND PROPELLERS

A. INTRODUCTION

The purpose of this chapter is to present an overview of the terminology and concepts that pertain to ship propulsion, propeller selection and the use of model propeller test data. Initially, fundamental definitions used in ship powering problems are presented. This is followed by a discussion of the "classic" types of propeller design/selection problems encountered by the naval architect and marine and naval engineers. Propeller model testing and propeller performance characteristics are reviewed next. The chapter is completed with a discussion of the Wageningen B-Screw Series.

The goal here is brevity. The reader is, therefore, encouraged to investigate the references cited for further details.

B. DEFINITIONS

Some fundamental terms associated with most propeller design/selection problems are:

1) Effective Horsepower (P_E)--power required to tow the "bare" hull (without propeller; rudder and appendage allowance assumed included) that generates a given resistance (R_T) at a given speed (V). It is determined by:

$$P_E = \frac{R_T V}{550} \quad (3.1)$$

2) Thrust Horsepower (P_T)--power delivered to water by a propeller developing a thrust force (T) and moving at a speed of advance (V_A) without the influence of a hull form ahead of it. P_T is determined by:

$$P_T = \frac{T V_A}{550} \quad (3.2)$$

3) Delivered Horsepower (P_D)--power delivered by shaft to propeller, normally specified at the outboard side of the stern tube. Q_S is the torque delivered to the propeller; n_P is the revolution rate of the shaft and, consequently, the propeller. P_D is determined by:

$$P_D = \frac{2\pi Q_S n_P}{550} \quad (3.3)$$

4) Shaft Horsepower (P_S)--power delivered to the inboard side of the stern tube having a transmission efficiency of η_S . P_S is determined by

$$P_S = \frac{P_D}{\eta_S} \quad (3.4)$$

5) Brake Horsepower (P_B)--power delivered by the prime mover at connection flange to the power train. While P_B is normally associated with the prime mover's rated power at this connection (BHP), it can also be specified from the power train/propeller side as:

$$P_B = \frac{P_S}{\eta_B \eta_G} \quad (3.5)$$

where η_B and η_G are, respectively, the bearing system and reduction gear transmission efficiencies.

6) Thrust deduction factor (1-td)--ratio of the tow resistance (R_T) to the thrust (T) provided by the propeller. It is determined by

$$(1-td) = \frac{R_T}{T} \quad (3.6)$$

7) Wake Factor (Taylor's) (1-wt)--ratio of the speed of advance (V_A) to the ship's speed (V). It is given by:

$$(1-wt) = \frac{V_A}{V} \quad (3.7)$$

8) Advance Ratio (J)--a non-dimensional value, associated with propeller test data presentation (see figure (3.1), given by the following relation:

$$J = \frac{V(1-wt)}{n_P D} = \frac{V_A}{n_P D} \quad (3.8)$$

9) Thrust Coefficient (K_T)--a non-dimensional value associated with the thrust force (T) developed by a propeller of diameter D which is turning at a rate n_P and operating in a fluid of density ρ . It is defined by the following expression:

$$K_T = \frac{T}{\rho n_P^2 D^4} \quad (3.9)$$

10) Torque Coefficient (K_Q)--a non-dimensional value associated with the torque (Q_P) absorbed by a propeller of diameter D which is turning at a rate n_P and operating in a fluid of density ρ . It is defined by the following expression:

$$K_Q = \frac{Q_P}{\rho n_P^2 D^5} \quad (3.10)$$

11) Open Water Efficiency (η_O)--the ratio of P_T to P_D for a propeller in open water conditions, with a uniform inflow velocity field at a speed of advance V_A . It is expressed as:

$$\eta_O = \frac{P_T}{P_D} = \frac{T V_A}{2\pi n_P Q_S} = \frac{J K_T}{2\pi K_Q} \quad (3.11)$$

12) Hull Efficiency (η_H)--a ratio of work done on the ship to that done by the propeller expressed as:

$$\eta_H = \frac{P_E}{P_T} = \frac{R_T V}{T V_A} = \frac{(1-td)}{(1-wt)} \quad (3.12)$$

13) Relative Rotative Efficiency (η_R)--the ratio of the actual, behind-hull efficiency to the open water efficiency. The value of η_R does not, in general, depart from the value

of 1.0. Most often, η_R varies between 0.95 and 1.0 for twin-screw ships and between 1.0 and 1.1 for single screw ships.

Applicable units for the terms in the expressions above are:

- 1) horsepower (hp)-- P_E, P_T, P_S, P_D and P_B
- 2) pounds (lbf)-- R_T, T
- 3) feet/second (ft/sec)-- V, V_A
- 4) foot-pounds (ft-lbf)-- Q_S, Q_P
- 5) feet (ft)-- D
- 6) revolutions/second (rps)-- n_P
- 7) revolutions/minute (rpm)-- $N_P = n_P/60.0$

The quantities T, V_A, D, Q_P and n_P are obtained from the propeller test data results. The quantities R_T and V are specified from the design point on the R - V curve for the hull under study. The quantity ρ is a property of the fluid in which the hull and propeller operate. And, finally, η_B, η_G and η_S are characteristics of the bearing, gear and stern tube systems. In preliminary design studies, nominal values, based on previous designs, are usually assumed unless, of course, these systems have been selected and actual values can be specified.

C. POWERING CONCEPTS

1. Basic Relations

Simply stated, the fundamental powering relationship to be solved in ship propulsion and powering problems is:

$$P_E = \frac{(1-td)}{(1-wt)} \cdot \eta_R \eta_O P_D \quad (3.13)$$

Utilizing the definitions just presented, equation (3.13) can be rewritten as:

$$\frac{R_T V}{550} = \frac{(1-td)}{(1-wt)} \cdot \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.14)$$

Rearranging terms of equation (3.14) gives:

$$\frac{R_T}{(1-td)} \cdot \frac{(1-wt)V}{550} = \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.15)$$

And, finally, when substitutions are made, equation (3.15) becomes:

$$\frac{T V_A}{550} = \frac{T(1-wt)V}{550} = \eta_R \eta_O \cdot \frac{2\pi Q_S n_P}{550} \quad (3.16)$$

Equations (3.14), (3.15), and (3.16) provide the basis for different approaches to the solution of a typical powering problem. More background and information on the definitions and equations presented above may be found in Chapter VI, Sections 10-16 in the text by Comstock [Ref. 16] and O'Brien's book [Ref. 17].

2. Approaches to the Powering Problem

From equation (3.16), three types of propeller selection problems are discernible. In the first instance, the propeller thrust T and the propeller's speed of advance V_A

are taken as known quantities. The fact that T is known substantiates the "Thrust Approach" nomenclature given to this type of selection problem. In the preliminary (or, in some circles, conceptual) ship design phase, the specification of T is based upon the requirement imposed by the resistance of the ship (R_T) at its design speed (V) (or, the effective horsepower (P_E) at V) and estimates of w_t and t_d in the absence of wake surveys and self-propulsion data from model tests. Essentially, the thrust delivered by a selected propeller must provide, at least, the thrust required for the ship hull under study. The objective in the "Thrust Approach" selection problem is to determine, by logical means, the appropriate values of Q_S and n_p when the open water efficiency (η_o) is set by the selected propeller and its performance characteristics.

In the second instance, the delivered torque (Q_S) and the propeller shaft speed (n_p) are taken to be known. The "Power Approach" nomenclature is given to propeller selection problems of this type because P_D is known. Here, with the shaft and propeller speeds being equal, the torque absorbed by the propeller (Q_p) must be, at least, equivalent to the delivered torque (Q_S). The corresponding objective in the "Power Approach" selection problem is to determine, by logical means, the expected ship speed (V) (or, the speed of advance (V_A)) and the associated thrust (T) that can be developed when the open water efficiency (η_o) is, again, set by the selected propeller and its performance characteristics.

The final, and most familiar, types of propeller selection problem occurs when T , V_A or V , Q_S and n_P are all known. From equation (3.16), the open water efficiency (η_O) is now established as a requirement to be met. The objective is, simply, to select a propeller whose open water efficiency (η_O), developed thrust and absorbed torque are equivalent to or "match" the requirements imposed. Obviously, this approach on the selection problem has been designated as a "matching problem".

The reader is directed to the paper by Vassilopoulos [Ref. 18] for further information.

D. PROPELLER PERFORMANCE CHARACTERISTICS

Up until the late 1950's, much of the knowledge about the performance of propellers has been gained from experience with models. To study the relationships governing their behavior, a model propeller is built and run in a towing tank without any hull ahead of it. This is done by running the propeller on a long shaft projecting well ahead of a narrow, hydrodynamically shaped pod or "propeller boat" which contains the driving mechanism and recording apparatus and is attached to the towing carriage. The propeller advances into undisturbed fluid (usually water of density ρ and kinematic viscosity ν) so that the speed of advance (V_A) is known and the flow into the "disc" swept by the turning blades is uniform. For the model propeller of diameter (D) under test, readings of thrust (T), torque (Q_S) and shaft revolutions (n_P)

are recorded over a range of values for speed of advance (V_A) in this "open water" condition.

Using the laws of similitude, the collected data is reduced and scaled appropriately into the familiar functional relationships between the advance ratio (J) and the non-dimensional coefficients of propeller performance. These coefficients or performance characteristics, defined previously, are:

- 1) Thrust Coefficient (K_T)
- 2) Torque Coefficient (K_Q)
- 3) Open Water Efficiency (η_O)

Figure (3.1) graphically depicts the relationship between J and K_T and K_Q derived from test data for a propeller defined by a specific expanded area ratio (A_E/A_O), pitch-diameter ratio (P/D), number of blades (Z) and thickness-to-chord ratio (t/c).

Definitions of these terms with graphical illustrations pertaining to various aspects of propeller geometry can be found in Section 15 of references [16], [17] and in van Manen's publication [Ref. 19].

More recently, highly analytical theories (lifting line, modified lifting line, lifting surface, etc.) for use with high-speed digital computers have been formulated and subsequently used in "modeling", in a mathematical sense, the propeller and its behavior in the "wake adapted" (or, behind hull) condition as well as the "open water" condition.

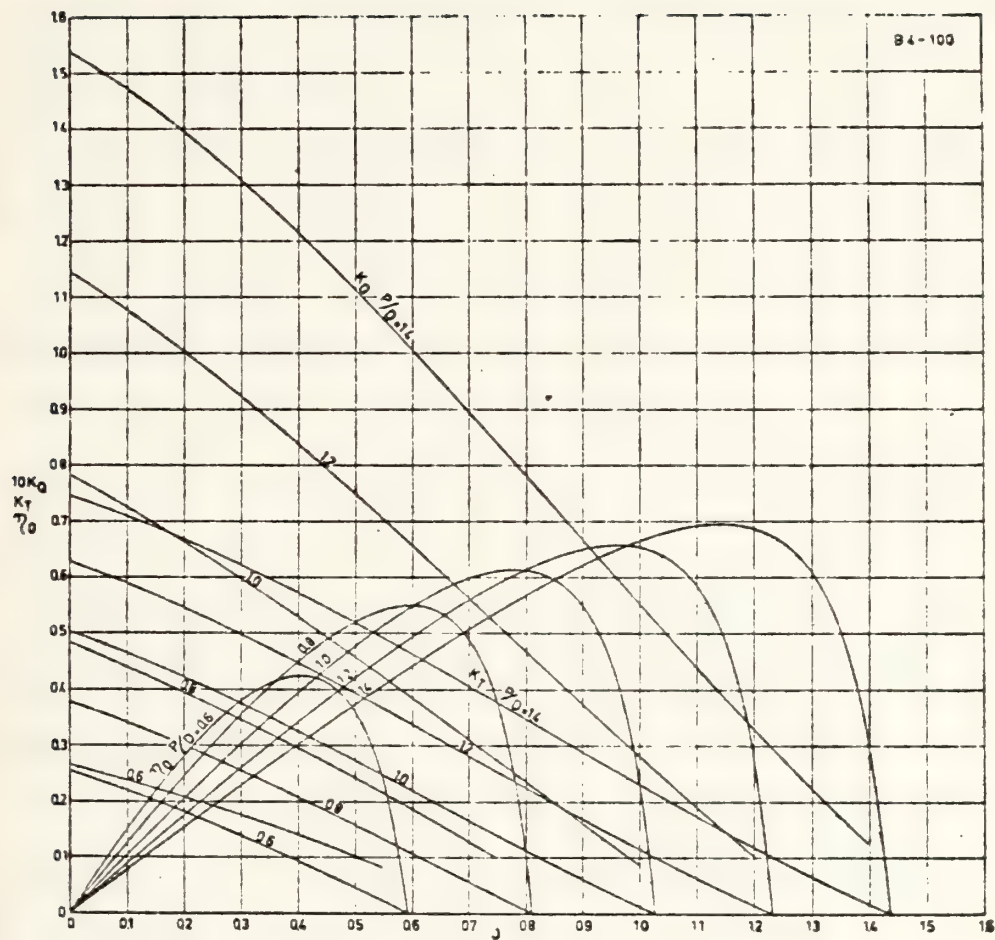


Figure 3.1 Open Water Test Results--B 4-100 Series Propeller

Additional benefits derived from this approach to propeller performance analysis include:

- 1) determination of blade section profiles along the propeller's blade radius (R) to achieve uniform lift and internal stress distributions;
- 2) computation of "off-design" performance characteristics in all quadrants;
- 3) subsequent determination of hull surface forces, bearing loads and spindle torques induced by the propeller;
- 4) prediction of steady and unsteady stress distributions in the propeller blade using the finite element method on the blade of the propeller under study.

Obviously, this approach to propeller performance analysis serves to:

- 1) eliminate the time-consuming and expensive model construction and testing of propellers in tow tanks and cavitation tunnels;
- 2) eliminate the "scaling" discrepancies which inhibit the reliability of design charts and model propeller data;
- 3) eliminate those design charts altogether.

As in the case with model experiments, however, the ultimate objective remains the same, i.e., establishing the performance characteristics of the propeller in terms of K_T , K_Q and as functions of J . Having these relationships enables the ship designer to proceed in solving the power equation (equation (3.13)) through any of the approaches previously discussed.

E. THE WAGENINGEN B-SCREW SERIES

1. Background

The model test data of the Wageningen B-Screw Series have been selected for use in the powering problems to be solved utilizing COPES/CONMIN. The choice was driven by the following considerations:

- 1) the Series is widely known and, despite its growing obsolescence, is still used in preliminary ship design studies.
- 2) the availability of previous investigations [Refs. 3,4] which utilized the series, for comparative analysis of optimization results.
- 3) the applicability of the polynomial expressions for K_T and K_Q to computer-aided analysis.

The Series tests were conducted from 1940 through 1960 and, therefore, represent propeller designs (principally naval and merchant applications) and design philosophy of that era.

Specifically, the Series consists of 120 model propellers. As is customary in methodical or systematic model propeller series testing, the number of blades (Z), expanded area ratio (A_E/A_O) and pitch-diameter ratio (P/D) are varied systematically, while the blade outline, the profile of the blade's cross section along the blade radius, blade cross section maximum thickness (t), blade section chord length (c), diameter (D) and propeller hub-to-diameter ratio (d/D) were kept constant for given values of A_E/A_O and Z .

Table (I) summarizes the variations in Z and A_E/A_O for each set of model propellers having pitch-diameter ratios (P/D) of 0.4, 0.6, 0.8, 1.0, 1.2 and 1.4.

TABLE I
Summary of the Wageningen B-Screw Series

Blade number	Blade area ratio A_E/A_O														
2	0.30														
3		0.35			0.50			0.65			0.80				
4			0.40			0.55			0.70			0.85		1.00	
5				0.45			0.60			0.75					1.05
6					0.50			0.65			0.80				
7						0.55			0.70			0.85			

2. Series Results

The test results of the Wageningen B-Screw Series were originally presented in the form of B_p - δ , B_u - δ and K_T , K_Q , and $-J$ diagrams. As stated in Chapter I, multiple regression analysis was performed (again, [Refs. 1,2]) on the results to produce the polynomial expressions for K_T and K_Q . The open water efficiency (η_o), as a function of J , follows from equation (3.11). The correction for "scale effects" was achieved by using Lerb's method of equivalent profiles [Ref. 20]. Although Triantafyllou's thesis [Ref. 21] suggests an improved method for scale correction, the results of Reference [2] will be used in this study.

In propeller selection problems which use this Series, values for K_T and K_Q are defined as:

$$K_T = f_1(J, P/D, A_E/A_O, Z, t^*/c_{.75R}) \quad (3.17)$$

$$K_Q = f_2(J, P/D, A_E/A_O, Z, t^*/c_{.75R})$$

To compute K_T and K_Q , the following equations are used:

$$K_T = K'_T + \Delta K_T \quad (3.18)$$

$$K_Q = K'_Q + \Delta K_Q$$

The polynomial expressions found in Tables (5) and (6) of Reference [2] are then used to evaluate the components K'_T , K'_Q , ΔK_T and ΔK_Q .

Table (5) in Reference [2] lists the coefficients used in the polynomial expressions for K'_T and K'_Q at an equivalent Reynolds number ($Rn_{.75R}$) of 2×10^6 . It is defined as:

$$Rn_{.75R} = \frac{c_{.75R} \sqrt{(V_A)^2 + (0.75\pi n_P D^2)}}{\nu} \quad (3.19)$$

where:

- ν = kinematic viscosity of the fluid (ft^2/sec);
- $c_{.75R}$ = blade section chord length at 3/4 propeller radius (.75R) in feet (ft).

To account for "other effects", coefficients ΔK_T and ΔK_Q are introduced. Table (6) in Reference [2] lists the coefficients used in the polynomial expressions for these coefficients. "Other effects" include the operation of the propeller at an equivalent Reynolds Number different from 2×10^6 . Also, variations in other parameters which define the propeller's geometry, specifically t/c values different from the ones fixed by the Wageningen propellers, are taken into account by corrections to the equivalent Reynolds number. By keeping the blade section's chord length (c) at the value of the Wageningen propeller, a change in a blade section's maximum thickness from the standard one defined by the Series (t) to one preferred in the selection (t^*) produces a new equivalent Reynolds number ($Rn^*_{.75R}$) given by:

$$Rn^*_{.75R} = \exp \left[4.6052 + \left\{ \sqrt{\frac{1+2(t/c)_{.75R}}{1+2(t^*/c)_{.75R}}} (\ln Rn_{.75R}^{-4.6052}) \right\} \right] \quad (3.20)$$

where:

- $Rn^*_{.75R}$ = the new equivalent Reynolds number;
- $(t^*/c)_{.75R}$ = new equivalent t/c at 3/4 propeller radius;
- $Rn_{.75R}$ = the Reynolds number computed by equation (3.19);
- $(t/c)_{.75R}$ = standard equivalent t/c at 3/4 propeller radius for Wageningen propellers.

For the Wageningen B-Screw Series, the standard equivalent t/c is given by:

$$t/c_{.75R} = \frac{(0.0185 - 0.00125Z)Z}{2.073 A_E/A_O} \quad (3.21)$$

Further details on blade section geometry will be addressed in Chapter V. Reference [2] contains background and other information on the equations above.

3. Limitations on Series Data

In utilizing the Wageningen B-Screw Series in any propeller selection problem, the following restrictions apply to the Series data:

1) Number of Propeller Blades (Z)--The Series considers only propellers with numbers of blades as shown in Table (I). Therefore,

$$Z = 2, 3, 4, 5, 6 \text{ or } 7 \quad (3.22)$$

However, the two bladed propeller, i.e., $Z = 2$, is not very common in conventional merchant and naval ship designs and, therefore, is not included in this study.

2) Equivalent Reynolds Number--The Series data, as published, is valid only in the range of equivalent Reynolds numbers given by:

$$2 \times 10^6 \leq Rn_{.75R} \leq 2 \times 10^9 \quad (3.23)$$

If the equivalent t/c is varied from the standard equivalent value $(t/c)_{.75R}$, then the new equivalent Reynolds number $(Rn^*_{.75R})$, which results from this variation, must lie within the same limits. That is,

$$2 \times 10^6 \leq Rn^*_{.75R} \leq 2 \times 10^9 \quad (3.24)$$

These limits are appropriate for full-size propellers. For example, given the following:

- a) $wt = .22$
- b) $V = 20$ (knots)
- c) $N_p = 104$ (rpms)
- d) $D = 25$ (ft)
- e) $c_{.75R} = 4.0$ (ft)
- f) $v = 1.2285 \times 10$ (ft²/sec)

the value for $Rn_{.75R}$ is equal to 3.619×10^7 .

3) Pitch-diameter Ratio (P/D)--The series data considers only pitch diameter ratios in the range given by:

$$0.4 \leq P/D \leq 1.4 \quad (3.25)$$

4) Advance Ratio (J)--An inspection of the Series results in graphical format shows that J varies over a range given by:

$$0 < J \leq 1.6 \quad (3.26)$$

5) Expanded Area Ratio (A_E/A_O)--Using Table (I), A_E/A_O varies over certain ranges depending on Z . This is stated as:

$$0.35 \leq A_E/A_O \leq 0.8 \quad Z = 3 \quad (3.27)$$

$$0.40 \leq A_E/A_O \leq 1.0 \quad Z = 4 \quad (3.28)$$

$$0.45 \leq A_E/A_O \leq 1.05 \quad Z = 5 \quad (3.29)$$

$$0.50 \leq A_E/A_O \leq 0.8 \quad Z = 6 \quad (3.30)$$

$$0.55 \leq A_E/A_O \leq 0.85 \quad Z = 7 \quad (3.31)$$

6) Hub diameter-to-Propeller Diameter Ratio (d/D)--From Table 37, Section 17 of Reference [16], the Series data requires that:

$$d/D = 0.18 \quad Z = 3,7 \quad (3.32)$$

$$d/D = 0.167 \quad Z = 4,5,6 \quad (3.33)$$

F. SUMMARY

From the preceding discussions, the following observations can be made:

1) the "Design Cases", defined in Chapter I, are examples of the powering equation solution approaches. That is, Design

Case No. 1 constitutes a "Thrust Approach" problem; Design Case No. 2, a "Power Approach" one; Design Case No. 3, a "Matching" problem.

2) equations (3.17) and (3.8) imply that an optimization solution to the "Design Cases" will involve P_E , V , wt , D , P/D , A_E/A_O , $(t^*/c)_{.75R}$, Q_S and n_P as possible design variables.

3) when viewed from the concepts on optimization presented in Chapter II, equations (3.25) through (3.31) constitute side constraints to an optimized solution of a propeller selection problem which uses the Wageningen B-Screw Series. Having noted these points, the propeller selection problem can now be formulated as a general, non-linear, constrained optimization problem.

IV. PROPELLER SELECTION--AN OPTIMIZATION PROBLEM

A. INTRODUCTION

The purpose of this chapter is to present the formulation of the propeller selection problem as an optimization problem that can be solved using COPES/CONMIN. Three usual restrictions considered by the designer in any propeller selection analysis are stated as constraints. Then, the components of \bar{D} and \bar{X} are assembled based on requirements from previously cited relationships. The restrictions considered by the designer and the limitations imposed by the use of the Wageningen B-Screw Series are presented in inequality constraint format. A formal statement of the propeller selection problem as an optimization problem is followed by a review of the GLOBCM common block format and the basic subprograms used in all three versions of SUBROUTINE ANALIZ that pertain to each Design Case.

The FORTRAN subprogram listings are found in Appendix B. Comment cards have been used extensively in the coding development to assist the reader.

B. DESIGNER'S CONSIDERATIONS

1. Propeller Size

The first restriction on the selection of any propeller is size. That is, the propeller race in the stern of the hull under consideration will only accommodate a

propeller of some given maximum diameter (D_{lim}). As a constraint on a selected propeller of diameter D , this may be written as:

$$D \leq D_{lim} \quad (4.1)$$

or, alternatively, as:

$$G_9(\bar{X}) = \frac{D}{D_{lim}} - 1 \leq 0 \quad (4.2)$$

2. Cavitation

Another item of importance in propeller selection is the cavitation phenomenon. When a propeller of given diameter D and expanded area ratio A_E/A_O is operating to produce a thrust T , the formation and subsequent collapse of water vapor bubbles on the blade surface, i.e., cavitation, is likely to occur if the localized surface pressures, usually on the "back" side of the blade, drop below the pressure at which the fluid would boil (p_{watvap}) in the surrounding environment. Avoidance of cavitation can be reasonably assured by selecting a propeller having certain geometric characteristics. A good empirical relationship that establishes these characteristics for propellers typified by the Wageningen B-Screw Series is the Keller Cavitation criterion [Ref. 2: p. 259]. It specifies the minimum required expanded area ratio ($A_E/A_O \min$) to avoid cavitation and is given by:

$$(A_E/A_O)_{\min} = \frac{(1.3 + 0.3Z)}{(p_{\text{atm}} + \rho \cdot \text{acg} h_{\text{cl}} - p_{\text{watvap}})} \cdot \frac{T}{D^2} + b \quad (4.3)$$

where:

- Z = number of blades;
- T = developed thrust (lbf);
- D = propeller diameter (ft);
- p_{atm} = atmospheric pressure (psia);
- p_{watvap} = fluid vaporization pressure (psia)
- ρ = fluid density (lbf sec²/ft⁴)
- acg = 32.174 (ft/sec²)
- h_{cl} = depth to shaft centerline (ft)
- b = constant: 0.1 for Z = 2, 0.2 for Z = 1.

As a constraint on the propeller selection, this requirement is written as:

$$(A_E/A_O)_{\min} \leq A_E/A_O \quad (4.4)$$

or

$$G_{10}(\bar{X}) = \frac{(A_E/A_O)_{\min}}{A_E/A_O} - 1 \leq 0 \quad (4.5)$$

3. Strength

The final designer's consideration (for this study), included in the selection of a propeller, is that of strength.

Given the propeller's material (promat), selected from Table (II), and the loadings (T and Q_S) imposed, it is important to ensure that the blade's cross sections have proper dimensions (in an ideal sense, maximum blade section thickness (t^*) and chord length (c)) to ensure adequate strength. Since the use of the B-Screw Series requires that the chord length (c) vary as a prescribed function of D, Z and A_E/A_O , as given in Table 1 of Reference [2], the adequacy for strength can be determined by an appropriately selected value for blade section maximum thickness-to-chord ratio (t^*/c) alone. So, if t_{\min}^* is the established minimum blade section maximum thickness, then the strength requirement follows from the constraint given by:

$$t_{\min}^*/c = (t^*/c)_{\min} \leq t^*/c \quad (4.6)$$

The fact that blade section maximum thickness for the B-Screw Series varies linearly with the propeller radius (R) allows the strength constraint (equation (4.6)) to be evaluated at one section along the radius. This point is chosen to be at the 3/4 radius (.75R). Therefore, equation (4.6) becomes:

$$(t^*/c)_{.75R \min} \leq (t^*/c)_{.75R} \quad (4.7)$$

or

$$G_{11}(\bar{X}) = \frac{(t^*/c)_{.75R \min}}{(t^*/c)_{.75}} - 1 \leq 0 \quad (4.8)$$

Reference [2] suggests the following empirical relation for the minimum required equivalent blade section maximum thickness-to-chord ratio $(t^*/c)_{.75R \min}$:

$$(t^*/c)_{.75R \min} = \frac{z \left\{ 0.0028 + 0.21 \sqrt[3]{\frac{(2375 - 1125P/D) P_D}{4.123 N_P D^3 (S_c + \frac{D^2 N_P^2}{12.788})}} \right\}}{2.073 A_E/A_O} \quad (4.9)$$

where:

- D = propeller diameter (ft);
- P_D = delivered power (hp);
- N_P = propeller revolution rate (rpm);
- S_c = propeller material allowable stress (psi);
- P/D = pitch-diameter ratio.

However, in Chapter VI of this thesis, an algorithm which employs the Schoenherr formulation [Ref. 8] with some modifications, is presented as an alternative to equation (4.9).

C. THE DESIGN VECTOR

In view of the preceding presentations on optimization and powering, the design vector \bar{D} can be assembled for the general propeller selection problem utilizing the B-Screw Series. This vector is composed of preassigned parameters relating to environmental conditions, hull characteristics

and the propeller which are required for various equations and the design variables.

1. Parameters

a. Environmental

These parameters pertain primarily to the fluid conditions in which the propeller operates and to the atmosphere. Required for various calculations, they are:

- 1) fluid temperature ($^{\circ}\text{F}$)--Temp
- 2) fluid density ($\text{lbf sec}^2/\text{ft}^4$)-- ρ
- 3) fluid viscosity (ft^2/sec)-- ν
- 4) fluid vaporization pressure (psia)-- p_{watvap}
- 5) atmospheric pressure (psia)-- p_{atm}

b. Hull Characteristics

These parameters pertain to certain details prescribed for the hull under study in the powering analysis.

They are:

- 1) wake fraction--wt
- 2) thrust deduction--td
- 3) relative rotative efficiency -- η_R
- 4) number of propellers--noscrw
- 5) shaft centerline depth (ft)-- h_{cl}
- 6) propeller diameter limit (ft)-- D_{lim}

c. Propeller

These parameters are specified in view of their discrete-valued nature. They are:

- 1) number of blades--Z
- 2) material--promat

Table (II) lists materials and properties considered in this study. These values are taken from Table (35), Section 15 of Reference [16].

TABLE II
Material Identifier Reference

promat	Material	Allowable Stress--Sc (psi)	Density--wd (lbf/in ³)
1	Cast Iron	3600--3950	.260
2	Cast Steel	5915--6265	.289
3	Type 2 Bronze	7200-7585	.305
4	Type 4 Ni-Al Bronze	8910--9430	.278
5	Stainless Steel	5400--5500	.283

2. Design Variables

In view of equations (3.13) through (3.17), the design variables common to all selection approaches are:

- 1) P_E
- 2) V
- 3) D
- 4) P/D
- 5) A_E/A_O
- 6) $(t^*/c) .75R$
- 7) N_P
- 8) Q_S

The vectors \bar{D} and \bar{X} are shown schematically in figure (4.1).

$$\begin{array}{c}
 \bar{X} = \left\{ \begin{array}{l} P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\}
 \end{array}
 \begin{array}{c}
 \text{-----} \\
 \bar{D} = \left\{ \begin{array}{l} \text{Temp} \\ \rho \\ v \\ P_{\text{watvap}} \\ P_{\text{atm}} \\ \text{wt} \\ \text{td} \\ \eta_R \\ \text{noscw} \\ h_{cl} \\ D_{lim} \\ z \\ \text{promat} \\ P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\}
 \end{array}$$

Figure 4.1 Design Vectors \bar{D} and \bar{X}

D. CONSTRAINTS

Besides the constraints imposed by equations (4.2), (4.5) and (4.8), equations (3.25) through (3.31) are rearranged to the format of constraints in equation (2.9). They are listed as follows:

1) Equivalent Reynolds Number--Equation (3.25) becomes:

$$1 \leq \frac{Rn^* .75R}{2 \times 10^6} \leq 1000 \quad (4.10)$$

Two constraints are derived:

$$G_3(\bar{X}) = 1 - \frac{Rn^* .75R}{2 \times 10^6} \leq 0 \quad (4.11)$$

$$G_4(\bar{X}) = \frac{Rn^* .75R}{2 \times 10^6} - 1000 \leq 0 \quad (4.12)$$

2) Expanded Area Ratio--Equations (3.27) through (3.31) become:

$$(A_E/A_O)_{\text{lower}}(Z) \leq A_E/A_O \leq (A_E/A_O)_{\text{upper}}(Z) \quad (4.13)$$

Two constraints are derived:

$$G_5(\bar{X}) = (A_E/A_O)_{\text{lower}}(Z) - A_E/A_O \leq 0 \quad (4.14)$$

$$G_6(\bar{X}) = A_E/A_O - (A_E/A_O)_{\text{upper}}(Z) \leq 0 \quad (4.15)$$

3) Advance Ratio--Equation (3.22) becomes:

$$0 \leq \frac{J}{1.6} \leq 1 \quad (4.16)$$

Two constraints are derived:

$$G_1(\bar{X}) = \frac{-J}{1.6} \leq 0 \quad (4.17)$$

$$G_2(\bar{X}) = \frac{J}{1.6} - 1 \leq 0 \quad (4.18)$$

4) Equivalent Blade Section Maximum Thickness-to-Chord Ratio--Using equation (3.19), boundaries on the range of $(t^*/c)_{.75R}$ are defined by:

$$\frac{1}{2}(t/c)_{.75R} \leq (t^*/c)_{.75R} \leq 4(t/c)_{.75R} \quad (4.19)$$

Two constraints are derived:

$$G_7(\bar{X}) = \frac{1}{2}(t/c)_{.75R} - (t^*/c)_{.75R} \leq 0 \quad (4.20)$$

$$G_8(\bar{X}) = (t^*/c)_{.75R} - 4(t/c)_{.75R} \leq 0 \quad (4.21)$$

E. OBJECTIVE FUNCTIONS

Upon consideration of equation (3.13), Design Case No. 1 and Design Case No. 2 require that the open water efficiency (η_o) , given by equation (3.11), be maximized. In terminology related to optimization, this is stated as:

$$\text{OBJ}_{1,2} = -\eta_0 \quad (4.22)$$

Design Case No. 3, the "matching" problem, requires that the blade weight (bldwt) be minimized. This is stated as:

$$\text{OBJ}_3 = \text{bldwt} \quad (4.23)$$

F. PROPELLER SELECTION OPTIMIZATION PROBLEM STATEMENT

As a general, non-linear constrained optimization problem to be solved by COPES/CONMIN, the propeller selection problem for all Design Cases may be stated as one equation given by:

$$\text{Minimize: } F(\bar{X}) = \text{OBJ}_{1,2} \text{ or } \text{OBJ}_3 \quad (4.24)$$

$$\text{Subject to: } G_j(\bar{X}) \leq 0 \quad j = 1, \dots, 12$$

$$x_i^{\text{lower}} \leq x_i \leq x_i^{\text{upper}} \quad i = 1, \dots, 8$$

The constraint $G_{12}(\bar{X})$ and the values for x_i^{lower} and x_i^{upper} will be specified according to each Design Case.

G. CODING FUNDAMENTALS

1. GLOBCM Common Block

The GLOBCM common block, required by COPES/CONMIN, is now assembled. Table (III) specifies the assignment locations for the FORTRAN variables which define objective functions, design variables and constraints.

2. SUBROUTINE ANALIZ

While each Design Case uses a different approach, all analyses are very similar. Therefore, each SUBROUTINE

TABLE III

Global Common (GLOBCM) Catalog

Global Location	FORTTRAN Name	DEFINITION
1	ETAO	η_O
2	WEIGHT	bldwt
3	AEDVAO	A_E/A_O
4	DIA	D
5	N	$N_P = 60 \cdot n_P$
6	PE	P_E
7	PDIVD	P/D
8	QS	Q_S
9	TC75R	$(t^*/c) \cdot .75R$
10	V	V (ft/sec)
11	RJCHL	$G_1(\bar{X})$ --eqn (4.17)
12	RJCNU	$G_2(\bar{X})$ --eqn (4.18)
13	R75RCL	$G_3(\bar{X})$ --eqn (4.11)
14	R75RCU	$G_4(\bar{X})$ --eqn (4.12)
15	AEAOCCL	$G_5(\bar{X})$ --eqn (4.14)
16	AEAOCU	$G_6(\bar{X})$ --eqn (4.15)
17	TC75CL	$G_7(\bar{X})$ --eqn (4.20)
18	TC75CU	$G_8(\bar{X})$ --eqn (4.21)
19	POWBAL	$G_{12}(\bar{X})$ --eqn (7.10) or (8.11) or (9.4)
20	DIACNU	$G_9(\bar{X})$ --eqn (4.2) or (9.6)
21	AEAOCV	$G_{10}(\bar{X})$ --eqn (4.5)
22	TCSTRS	$G_{11}(\bar{X})$ --eqn (4.8)
23	RJ	J

ANALIZ shares a common structure and other common subroutines which perform calculations required in all cases. Appendices C, F and I contain, respectively, the source listings of SUBROUTINE ANALIZ for Design Case No. 1, Design Case No. 2 and Design Case No. 3.

a. Structure

The structure common to all cases follows accordingly:

- 1) all initialization of environmental, hull and propeller parameters is accomplished in the input section (ICALC = 1).
- 2) evaluation of K_T and K_Q , all constraints and appropriate objective functions ($-\eta_0$ or bldwt) are accomplished in the execution section (ICALC = 2).
- 3) output of results for each optimization problem is accomplished in the output section (ICALC = 3).

b. Basic Subprograms

The following FORTRAN subprograms are used in all three SUBROUTINE ANALIZ codes:

- 1) SUBROUTINE CH75RA--calculates the equivalent blade section chord length ($c_{.75R}$) for the propeller using Table 1 [Ref. 2, p. 252].
- 2) SUBROUTINE REY75R--calculates the equivalent Reynolds number ($Rn_{.75R}$) using equations (3.19) and (3.20).
- 3) SUBROUTINE COEFSA--calculates the thrust and torque coefficients (K_T and K_Q) through sequential calls to SUBROUTINE CALCKT and SUBROUTINE CALCKQ. The polynomial expressions

(Tables (5) and (6), [Ref. 2]) for these coefficients are contained in SUBROUTINE CALCKT and SUBROUTINE CALCKQ respectively.

4) SUBROUTINE OPWEFF--calculates the open water efficiency (η_o) using equation (3.11).

5) SUBROUTINE JCNA--calculates the constraints on the advance ratio (J) given by equations (4.17) and (4.18).

6) SUBROUTINE REYCNA--calculates the equivalent Reynolds number constraints given by equations (4.11) and (4.12).

7) SUBROUTINE EXTCCN--calculates the constraints on expanded area ratio (A_E/A_O) and equivalent blade section maximum thickness-to-chord ratio ($t^*/c_{.75R}$) given by equations (4.14), (4.15), (4.20) and (4.21).

8) SUBROUTINE DICNUA--calculates the constraint on the propeller diameter (D) given by equation (4.2) using the hull parameter on maximum diameter (D_{lim}).

9) SUBROUTINE CAVCNA--calculates the constraint for cavitation given by equation (4.5) using equation (4.3).

10) SUBROUTINE STRCNA--calculates the constraint for strength given by equation (4.8) using equation (4.9).

H. SUMMARY

The propeller selection problem has now been formulated as a constrained optimization problem which can be solved by COPES/CONMIN. Two items remain for discussion before proceeding to specify the final details pertaining to each SUBROUTINE ANALIZ code and to present numerical examples. These items are:

1) the theory and coding relating to the computation of the propeller's blade weight for the evaluation of the objective function in Design Case No. 3 (OBJ_3).

2) the theory and coding relating to the computation of the minimum required equivalent blade section maximum thickness-to-chord ratio ($t^*/c_{.75 \text{ min}}$) for use in the alternative evaluation of the strength constraint given by equation (4.8).

V. PROPELLER BLADE WEIGHT--AN OBJECTIVE FUNCTION

A. INTRODUCTION

In this chapter, the method for the computation of the propeller's blade weight (bldwt), the objective function OBJ, is examined. First, a brief overview on the steps in the computational procedure is presented. The FORTRAN subprogram SUBROUTINE WGTAL developed from the algorithm is then described. Again, Appendix B contains all subprogram listings.

B. THEORY AND PROCEDURE

Given the material of a propeller, the calculation of the weight of one blade involves nothing more than a volume calculation, a relatively routine task performed by most naval architects/marine engineers. Analogous to the determination of the underwater volume of a ship's hull, the calculation is an integration of blade section profiles' cross-sectional areas over the propeller radius (R).

1. Limits of Integration

Figure (5.1) depicts a side elevation view of a blade and hub, parallel to the propeller shaft axis. The cross-hatched area indicates the trace of the volume to be calculated. In view of equations (3.32) and (3.33), limits of integration are from $r = .167R$ to $r = R$ for $Z = 4, 5, 6$ and $r = .18R$ to $r = R$ for $Z = 3, 7$. For convenience, a non-dimensional variable "x" will be defined as:

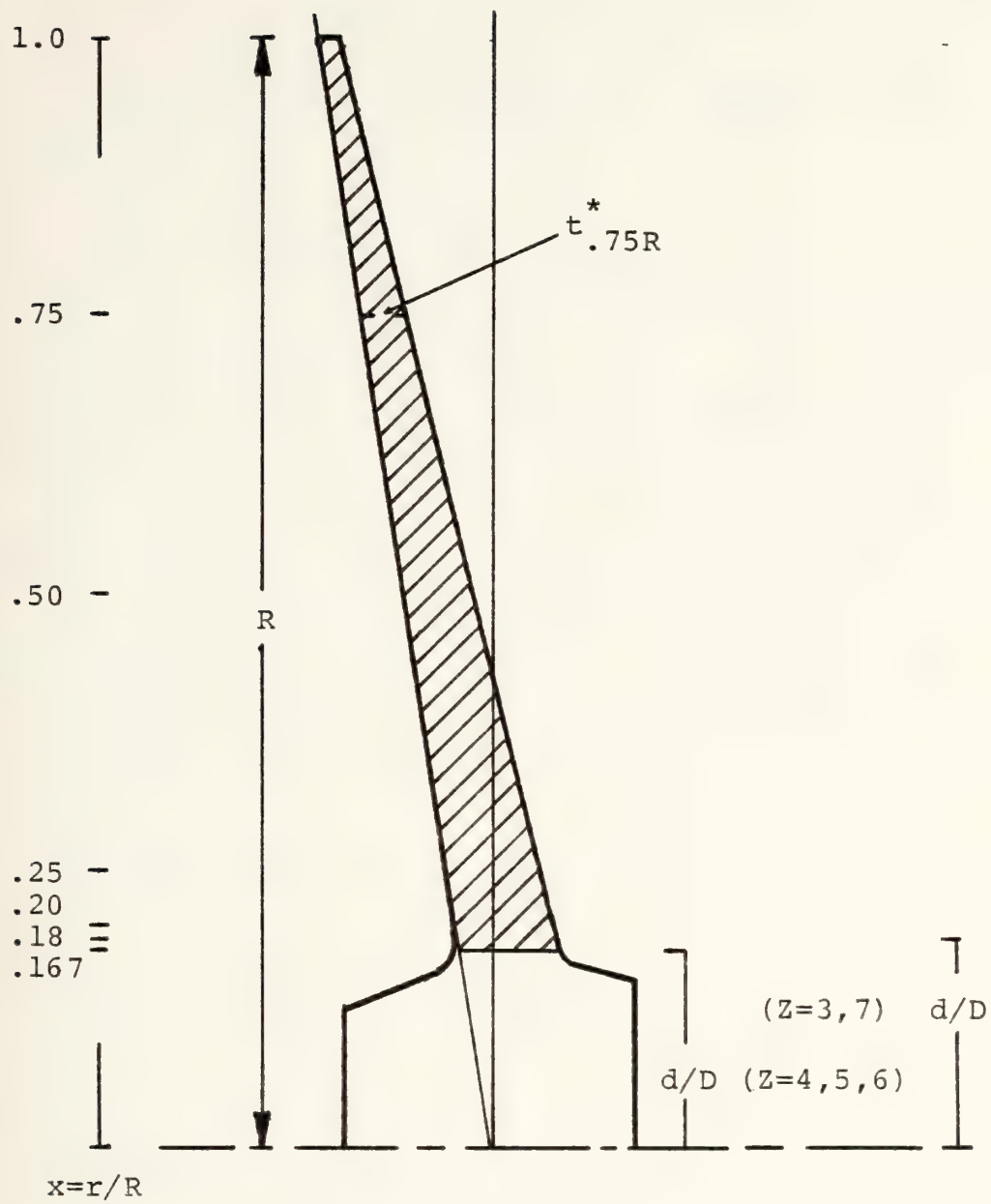


Figure 5.1 Propeller Blade & Hub--Side View

$$x = \frac{r}{R} \quad (5.1)$$

Limits are now expressed as $x = .167$ or $.18$ to $x = 1.0$.

Quite obviously, $R = (D/2.0)$.

2. Blade Section Profile

Figure (5.2) depicts an expanded cylindrical blade section in profile view at a given r or x . For the Wageningen B-Screw Series, the profile is defined, geometrically, by a succession of vertical ordinates which specify points along the blade section's profile on the "face" (y_f) and on the back (y_b) with respect to the pitch reference line. At any $r = xR$, vertical ordinates for "aft" ($P < 0$) and "fwd" ($P > 0$) portions of the blade section are determined by:

$$y_{fa} = V_1(t^* - t_{te}^*) \quad P \leq 0 \quad (5.2)$$

$$y_{ba} = (V_1 + V_2)(t^* - t_{te}^*) + t_{te}^* \quad P \leq 0 \quad (5.3)$$

and

$$y_{ff} = V_1(t^* - t_{le}^*) \quad P > 0 \quad (5.4)$$

$$y_{bf} = (V_1 + V_2)(t^* - t_{le}^*) + t_{le}^* \quad P > 0 \quad (5.5)$$

where:

V_1, V_2 = tabulated values depending on x and P
(see Tables (2) and (3), Ref. [2]);

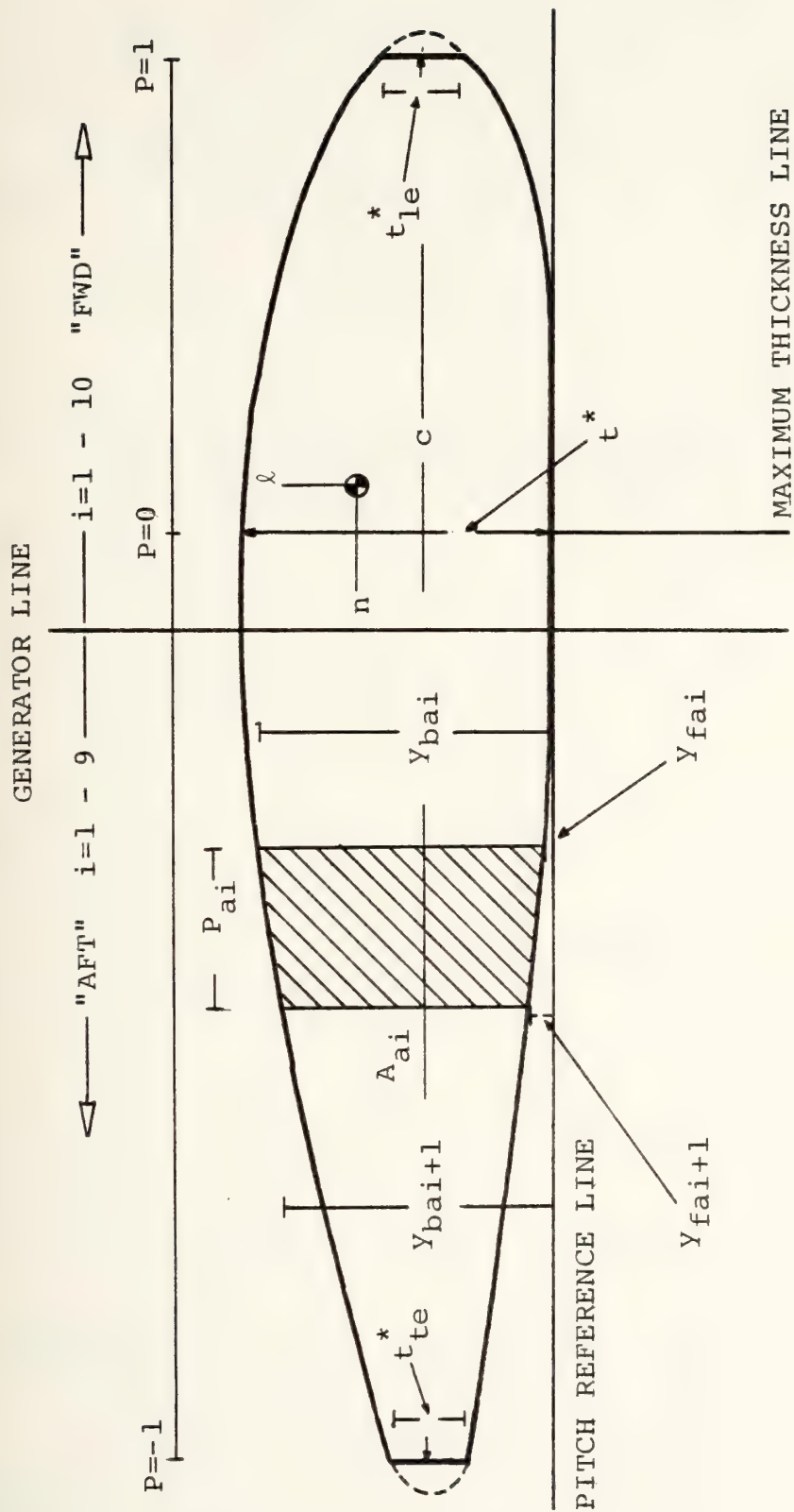


Figure 5.2 Expanded Cylindrical Blade Section--Profile View

t_{le}^* = blade section leading edge thickness (ft);

t_{te}^* = blade section trailing edge thickness (ft).

Units for y_{fa} , y_{ba} , y_{ff} and y_{bf} are feet (ft). For this study, a reasonable assumption is made in that:

$$t_{le}^* = t_{te}^* = \left(\frac{1}{10}\right) t^* \quad (5.6)$$

3. Blade Section Cross-Sectional Area

The cross-sectional area at each $x = r/R$ ($A(x)$) is determined by:

$$A(x) = \sum_{i=1}^9 A_{ai} + \sum_{i=1}^{10} A_{fi} \quad (5.7)$$

where

$$A_{ai} = \Delta P_{ai} \left\{ \frac{h_{ai+1} + h_{ai}}{2} \right\} \quad (5.8)$$

$$h_{ai} = y_{bai} - y_{fai} \quad (5.9)$$

$$h_{ai+1} = y_{bai+1} - y_{fai+1} \quad (5.10)$$

Expressions for A_{fi} , h_{fi} and h_{fi+1} follow in similar fashion.

Values for y_{fai} and y_{bai} are determined at 9 points along the "aft" portion of a given blade section's chord (c): values of y_{ffi} and y_{bfi} are determined at 10 points along the

"fwd" portion. The values for ΔP_{ai} and ΔP_{fi} are fractional values of the blade section's chord length (c) at radius $r = xR$ as determined from Tables (2) and (3) in Reference [2]. The units for $A(x)$ are square feet (ft²). Units for h_{ai} , h_{ai+1} , h_{fi} , h_{fi+1} , c, ΔP_{ai} and ΔP_{fi} are feet (ft).

4. Volume Integration

The blade volume (bldvol) is finally determined by using Simpson's Rule for integration of $A(x)$ along the non-dimensional radius x using appropriate limits.

5. Blade Weight

Once the blade volume (bldvol) is calculated, the weight (bldwt) is determined by:

$$\text{bldwt} = \text{bldvol} \cdot \text{wd} \cdot 1728 \quad (5.11)$$

where:

bldvol = volume of one blade (ft³);

wd = material weight density (lbf/in³).

Weight Density (wd) depends on blade material (promat).

Table (II) lists appropriate values.

C. CODING

SUBROUTINE WGTCAL is the main subprogram for the blade weight calculation. It, in turn, calls the following FORTRAN subprogram for various calculations:

1) SUBROUTINE TDIST--generates, at specified radius values, a distribution of blade section maximum thicknesses (t^*).

2) SUBROUTINE BLDPRP--generates, at specified radius values (i.e., $r = .167R$ or $.18R$, $.2R$, $.3R$, $.4R$, ..., $.9R$, $1.0R$), various "blade section properties", one of which is a blade section's cross-sectional area given by equation (5.7). Other properties which are determined (for later use in direct stress computations) include blade section chord lengths and centroids and "critical point" locations as defined in Chapter VI.

3) SUBROUTINE BLDVOL --performs a Simpson's Rule integration for the propeller blade volume (bldvol) using blade section cross-sectional areas generated in SUBROUTINE BLDPRP. The blade weight (bldwt) is computed as a final step in the main subprogram SUBROUTINE WGTAL.

Examination of the codes in Appendix B reveals extensive use of common blocks for passing data from one subprogram to another. Comment cards provide a full definition of all common blocks as well as a description of the task being performed at various points in a given subprogram.

D. SUMMARY

The coding developed for this study is, admittedly, not very compact and efficient. However, the intention has been to write all codes with sufficient documentation in order to facilitate the reader's understanding of the algorithms employed as well as to make the author's debugging work easier.

VI. THICKNESS-TO-CHORD RATIO--A DESIGN CONSTRAINT

A. INTRODUCTION

The purpose of this chapter is to examine the development of an algorithm that will be used to determine the minimum required equivalent blade section maximum thickness-to-chord ratio used in equation (4.8). The formulation is based upon the method developed by Dr. Karl E. Schoenherr [Ref. 8] in 1963. After a review of the past and present methods employed in propeller strength analysis is conducted, a description of the Schoenherr model and a list of the assumptions used with that model is presented. Then, a brief restatement of his model's equations which are used in the algorithm is followed by a derivation of the author's modifications to the Schoenherr method. The chapter is completed by conducting a review of the theory and coding employed by the algorithm.

The principal reference which is cited throughout this chapter is, again, Reference [8]. The reader is encouraged to review this reference for further details.

B. PROPELLER STRENGTH ANALYSIS--A HISTORICAL REVIEW

Marine propeller blades present a special class of structural problem. That problem lies in the difficulty of describing a blade design in simple mathematical terms for subsequent analysis through various means. Until the "finite element method era", analytical methods, including the one

[Ref. 8] adapted for this study, relied heavily on practical experience of the propeller designer and semi-theoretical considerations. Analysis by these methods provided a criterion of stress rather than actual computation of stresses. These methods for predicting blade stresses were developed by using "beam" theory or "shell" theory.

The use of elementary beam theory in propeller strength analysis was first adopted by Taylor [Ref. 22]. He treated a blade as a cantilever beam attached to the propeller hub and loaded by thrust and torque forces distributed linearly over the propeller radius. His approach is often deemed a "modified beam theory" because he chose to calculate the direct stresses using the moment of inertia properties of expanded cylindrical blade sections with neutral axis parallel to the nose-tail (pitch-reference) line or chord of that expanded section. Reasonable estimates of stresses along the blade surface were achieved for the unraked, unskewed and narrow-bladed propellers of his time.

As propellers "modernized" and became skewed and wider with increasing rake (mostly aft), modifications, improvements and alternatives to Taylor's theory were developed. Principally, modifications by Rosingh [Ref. 23] and Hancock [Ref. 24] proposed using moment of inertia properties of a blade section that was normal to the generating line of the axially projected blade outline. Romson [Ref. 25] later improved Taylor's theory for application to wide-bladed

propellers. Morgan [Ref. 26] provided an improved method for calculating the geometric properties of "modern" airfoil-shaped blade sections. Aernoldus and Keyser's [Ref. 27] "quasi-static" modeling of the propeller blade allowed for additional consideration to stresses induced by centrifugal loading of the raked and skewed blade. The beam theory approaches to propeller blade stress analysis culminated, for all practical purposes, with Schoenherr's work [Ref. 8] in 1963.

Alternatives to Taylor's beam theory approach, prior to 1963, consisted of the application of "shell" theory to the propeller blade strength problem. This approach was first proposed by Conn [Ref. 28] and subsequently formulated by Cohen [Ref. 29] who modeled the blade as a helicoidal shell with variable thickness and infinite width. "Shell Theory" was utilized again in experimental studies by Connolly [Ref. 30] who, like his predecessors, was also forced into making an assumption about the behavior of the displacements of the blade sections (i.e., constant displacements normal to the constant pitch blade at each fractional radius distance from the hub) beyond usual assumptions of shell theory. Essentially, his experimental results on one specific propeller contradicted the computational values. Attempts at a generalized numerical solution to Connolly's equations appeared in 1963 [Ref. 31] and 1964 [Ref. 32]. In 1968, Atkinson [Ref. 33], compared Connolly's results with currently

adopted cantilever beam methods and found, based on the inconsistency of results, that it was not possible to recommend one method over the other in the blade strength design procedure; another approach was needed.

Commenting on Atkinson's paper at that time, Sontvedt [Ref. 34] pointed out that, in view of these inconsistencies, the only approach to blade strength analysis which did not require very broad assumptions was the finite element technique. Developments in the method at that time were providing a new and powerful tool for structural analysis. The propeller blade was just another application. Genalis [Ref. 35] developed codes for the determination of displacements and stresses in a blade under hydrodynamic loads using the FEM technique and modeling the blade as a shell, a 3-D element mesh of tetrahedrons and rectangular prisms and, finally, a composite of shell and 3-D elements. As an aside, a finite "difference" solution to Connolly's analytical equations was proposed in 1972 [Ref. 36]. In 1973, Atkinson [Ref. 37] reported the application of both hydrodynamic and centrifugal loads to a blade modeled by a thin-shell triangular mesh and a thick-shell parabolic and cubic curved element mesh. The results of the triangular element were considered unsatisfactory. Another use of the thin-shell triangular element was reported by Sontvedt [Ref. 34] in 1974 using the SESAM-69 code [Ref. 38].

The need to model the blade correctly near the hub, where root stresses are usually critical, necessitated the

consideration of 3-D elements in lieu of thick/thin-shell elements [Refs. 39,40]. The use of the 4- and 10-noded tetrahedral element (i.e., TET-4, TET-10 respectively) to construct a blade mesh was conducted by Beek [Refs. 41,42]. He observed that improved accuracy of stress values, achieved by the use of these meshes, were overshadowed somewhat by the extensive storage capacity required for each analysis. Another "natural" improvement, from a geometrical standpoint, appeared in 1978. A general 3-D curved isoparametric element was incorporated in a computer code [Ref. 43] developed by Ma based on his previous formulation work [Ref. 44].

The finite element approach will continue to grow in use in propeller blade strength analysis with each successive improvement made to the basic elements which are used in the mesh generation of the blade. But, until the computer storage problem is resolved to the point where one mesh generation and subsequent stress analysis of one particular blade becomes a minor processing task, the basic analytical techniques will continue to be a meritable "check" [Ref. 45] in the preliminary (or conceptual) phase of propeller selection/design. In this context, the method formulated by Schoenherr and his colleagues twenty years ago is considered for adoption in determining a minimum required blade section maximum thickness-to-chord ratio.

C. SCHOENHERR'S METHOD

1. Background

Schoenherr's method is applicable to preliminary (or, conceptual) propeller selection problems because it employs an assumed thrust and torque force loading distribution for the propeller blade. This assumption is made at this stage of the ship's design because the exact wake velocity distribution at the propeller race is generally not known.

Also, his method is applicable to propellers represented by the Wageningen B-Screw Series because the blades of these propellers meet Schoenherr's criteria for the blade types covered by his formulation. Specifically, B-Screw Series blades have:

- 1) a small constant rake angle of 15° over the entire blade radius;
- 2) a constant pitch distribution over the propeller radius with the exception of the $Z = 4$ propeller whose pitch is slightly reduced near the hub;
- 3) mild skew;
- 4) linear distribution of blade section maximum thickness over the radius of the blade;
- 5) aerfoil profile qualities where the nose-tail line or the chord of the blade section is approximately parallel to the pitch reference line.

2. The Blade Model

Schoenherr models the propeller blade as a cantilever beam with unsymmetrical and variable area cross sections subjected to loading distributions of hydrodynamic and centrifugal forces. The following additional assumptions apply:

1) Flexure theory applies. This subsequently implies the following: a) plane cross sections remain plane under load, b) Hooke's Law is valid, c) the blade material is homogeneous and isotropic, d) fibers are free to extend and contract independently of adjacent fibers, and e) stresses at a point arising from various forces superimpose.

2) Shearing stresses and their effects are neglected. Only the direct stresses on a strength section are taken into account.

3) The strength sections are taken to be the expanded cylindrical blade sections at various radial locations.

4) The neutral axes of a strength section are straight lines passing through the centroid of the expanded cylindrical blade section and are parallel and normal, respectively, to the pitch reference line, and therefore, the chord, at each blade section.

5) Bending Moments are applied in two planes which are mutually perpendicular to each other. One plane is normal to the pitch-reference line (and chord) of the strength section; the other is parallel.

6) The angle between the principal axes of inertia and the neutral axes is zero.

Using this model and assumptions, Schoenherr applies the following formula for the evaluation of the direct fiber stress ($[\sigma]_o$) in a blade section at a radius $r = r_o$:

$$[\sigma]_o = \frac{[M]_{no} u_o}{I_{no}} + \frac{[M]_{lo} w_o}{I_{lo}} + \frac{[F_c]_o}{A(x_o)} \quad (6.1)$$

where:

$[M]_{no}$, $[M]_{lo}$ = resultant bending moments in planes normal ($[M]_{no}$) and parallel ($[M]_{lo}$) to the strength section's chord at $r = r_o$ (ft-lbf);

$[F_c]_o$ = centrifugal force acting normal to the plane of the strength section at $r = r_o$ and resulting from the centrifugal acceleration of the remaining blade element mass above that strength section (lbf);

u_o , w_o = coordinates of a point on the strength section's periphery with respect to that section's neutral axes system (l - n system) (ft);

$A(x_o)$ = strength section's cross-sectional area (ft²);

I_{lo} = moment of inertia of the strength section with respect to the " l " axis (ft⁴);

I_{no} = moment of inertia of the strength section with respect to the " n " axis (ft⁴);

x_o = non-dimensional radius given by $x_o = r_o/R$.

Since equation (6.1) indicates that the direct fiber stress is greatest at points on the periphery of the strength section, Schoenherr selects to examine four "critical points"

on the periphery where the fiber stress is likely to be a maximum. These points are designated as (1), (2), (3) and (4) on Figure (6.1) and are specified by coordinates (u_1, w_1) , (u_2, w_2) , (u_3, w_3) and (u_4, w_4) respectively in the "l-n" reference system.

The values for $[M]_{no}$ and $[M]_{\ell o}$ are determined by the following relation:

$$[M]_{no} = [M_P]_{no} + [M_{cb}]_{no} \quad (6.2)$$

$$[M]_{\ell o} = [M_P]_{\ell o} + [M_{cb}]_{\ell o} \quad (6.3)$$

where:

$[M_P]_{no}$ = total bending moment due to hydrodynamic loading acting in a plane normal to a strength section's chord at $r = r_o$ (ftlbft);

$[M_P]_{\ell o}$ = total bending moment due to hydrodynamic loading acting in a plane parallel to a strength section's chord at $r = r_o$ (ftlbft);

$[M_{cb}]_{no}$ = total bending moment due to centrifugal loading acting in a plane normal to a strength section's chord at $r = r_o$ (ftlbft);

$[M_{cb}]_{\ell o}$ = total bending moment due to centrifugal loading acting in a plane parallel to a strength section's chord at $r = r_o$ (ftlbft).

3. Bending Moments Due to Hydrodynamic Loading

The derivations of $[M_P]_{no}$ and $[M_P]_{\ell o}$ follow directly from Part I of Schoenherr's paper [Ref. 8: p. 83-89] and,

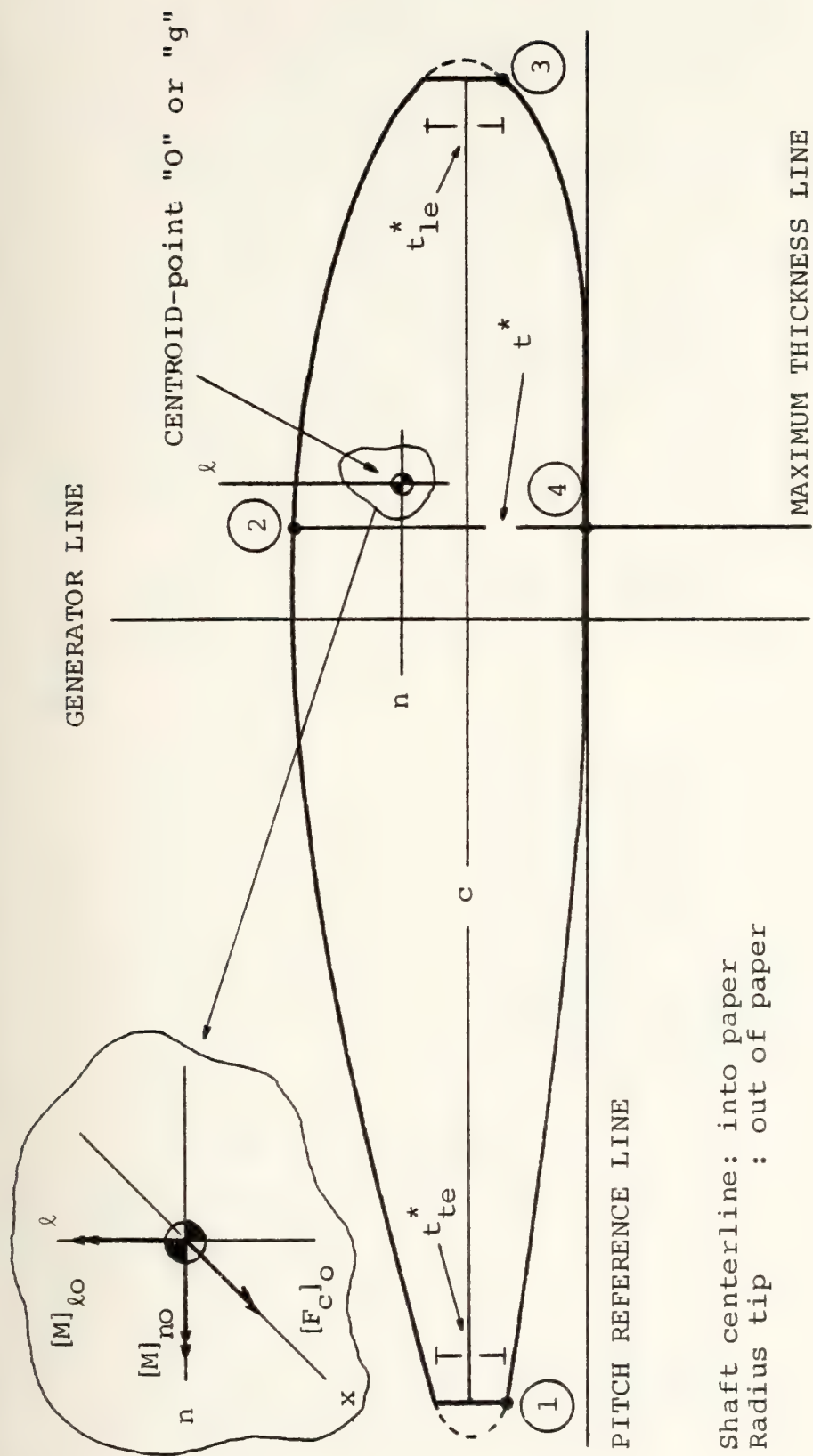


Figure 6.1 Strength Section at $r = r_O$

therefore, only key equations will be restated. References to equations which contain no decimal point apply to equations as numbered in his paper.

Thrust and torque force are components of the hydrodynamic "lifting" force acting on a blade. Using an assumed non-linear distribution of thrust along the blade radius given by equation (2), Schoenherr derives the following expression for the bending moment due to thrust ($[M_t]_o$) which acts at a blade section located at radius $r = r_o$:

$$[M_t]_o = \frac{TR}{Z} \cdot \frac{\phi_2(x_o)}{\phi_1(x_h)} \quad (6.4)$$

where:

- T = propeller thrust (lbf);
- R = propeller radius (ft);
- Z = no. of blades;
- $\phi_2(x_o), \phi_1(x_h)$ = functions of non-dimensional radius x evaluated at $x_o = r_o/R$ and $x_h = 0.2$ and given by equations (4) and (9).

For the bending moment due to torque ($[M_q]_o$) which acts at a blade section located at radius $r = r_o$, Schoenherr derives the following:

$$[M_q]_o = \frac{Q_P}{Z} \cdot \frac{\psi_2(x_o)}{\phi_1(x_h)} \quad (6.5)$$

where:

Q_p = propeller torque (ft-lbf);
 z = no. of blades;
 $\psi_2(x_o), \phi_1(x_h)$ = functions of non-dimensional radius x
 evaluated at $x_o = r_o/R$ and $x_h = 0.2$
 and given by equations (19) and (4).

Figure (6.2) depicts the component resolution for $[M_p]_{no}$ and $[M_p]_{lo}$ which results when equations (6.4) and (6.5) are imposed at a strength section at $r = r_o$ which has a pitch angle β_o . The following relations are derived as equations (42) and (43) in Schoenherr's paper:

$$[M_p]_{no} = [M_t]_o \cos \beta_o + [M_q]_o \sin \beta_o \quad (6.6)$$

$$[M_p]_{lo} = [M_t]_o \sin \beta_o - [M_q]_o \cos \beta_o \quad (6.7)$$

4. Force and Bending Moments Due to Centrifugal Loading

The derivation and expressions contained in this section constitute the author's modifications to the formulation in Part II of Schoenherr's paper. In Part II, Schoenherr's derivations for $[M_{cb}]_{no}$ and $[M_{cb}]_{lo}$ are formulated for computation using a propeller drawing. This follows from the fact that his method, which was funded by the American Bureau of Shipping, was intended to be used as that classification society's "designer's check" on adherence to the Bureau's strength criteria from a propeller blueprint. To evaluate the direct fiber stresses from equation (6.1) for the Wageningen B-Screw Series in accordance with Schoenherr's

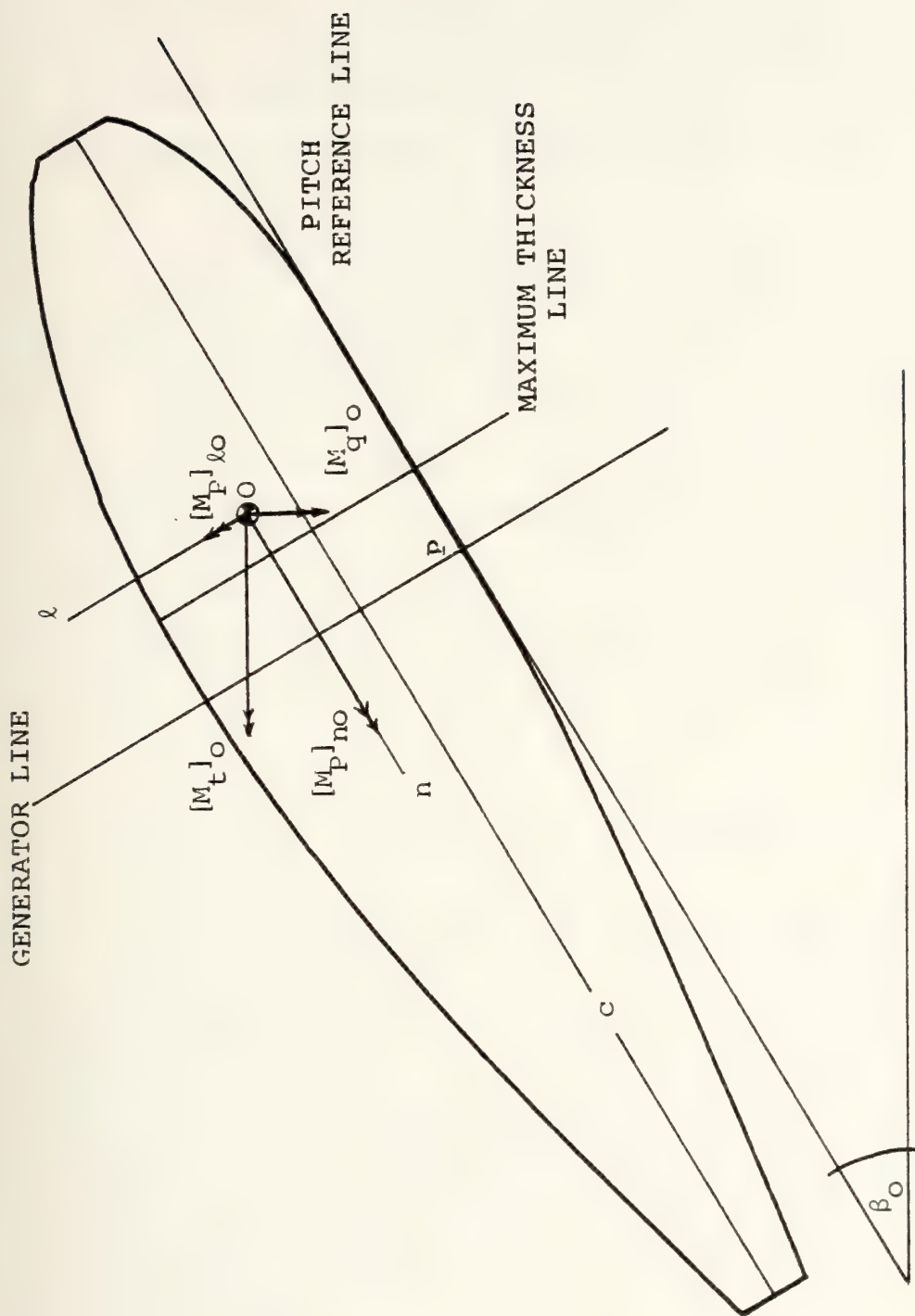


Figure 6.2 Bending Moments due to Thrust and Torque

method, $[M_{cb}]_{no}$ and $[M_{cb}]_{lo}$ must be evaluated from expressions derived from the available information on blade section profiles and other geometric characteristics which are contained in Reference [2] and previously used in Chapter V.

Consider Figure (6.3) where the centrifugal force $[C_F]_O$ of a blade element above a blade section at $x = x_O = r_O/R$ acts in a radial direction from the shaft centerline. Its line of action passes through point "N", which is on the same cylindrical surface as the blade section at $x = x_O = r_O/R$, and through point "G", which is the center of gravity of the blade element above the blade section at $x = x_O = r_O/R$. From the figure, the following expression is derived:

$$[C_F]_O = [C_F]_O \cos \zeta(x_O) + [C_F]_O \sin \zeta(x_O) \quad (6.8)$$

$[C_F]_O$ is shifted to point "N" and is decomposed into components $[C_F]_O \cos \zeta(x_O)$ and $[C_F]_O \sin \zeta(x_O)$. The entire cylindrical surface in which the blade section at $x = x_O = r_O/R$ and point "N" lie is now expanded into a flat plane for further consideration (see Figure (6.4)). In this configuration, $[C_F]_O \cos \zeta(x_O)$ is normal to this flat plane while $[C_F]_O \sin \zeta(x_O)$ lies in this plane.

Let point "O" be the location of the blade section's neutral axes system (i.e., the ℓ -n system). Then, the forces and moments due to the centrifugal reaction of the blade element above this section act at point "O" and are given by:

PITCH REFERENCE LINE

GENERATOR LINE

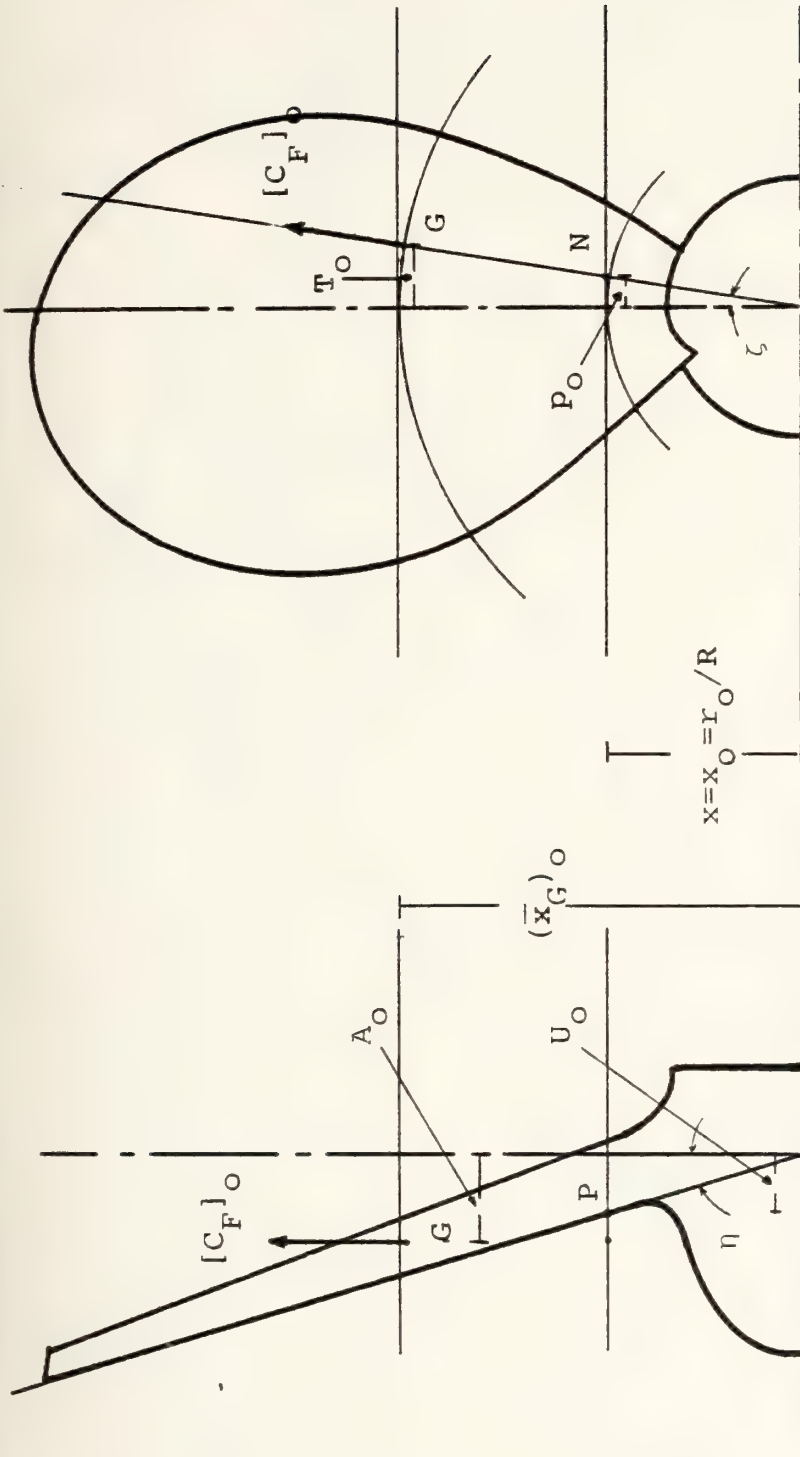


Figure 6.3 Center of gravity "G" and intersection point "N"

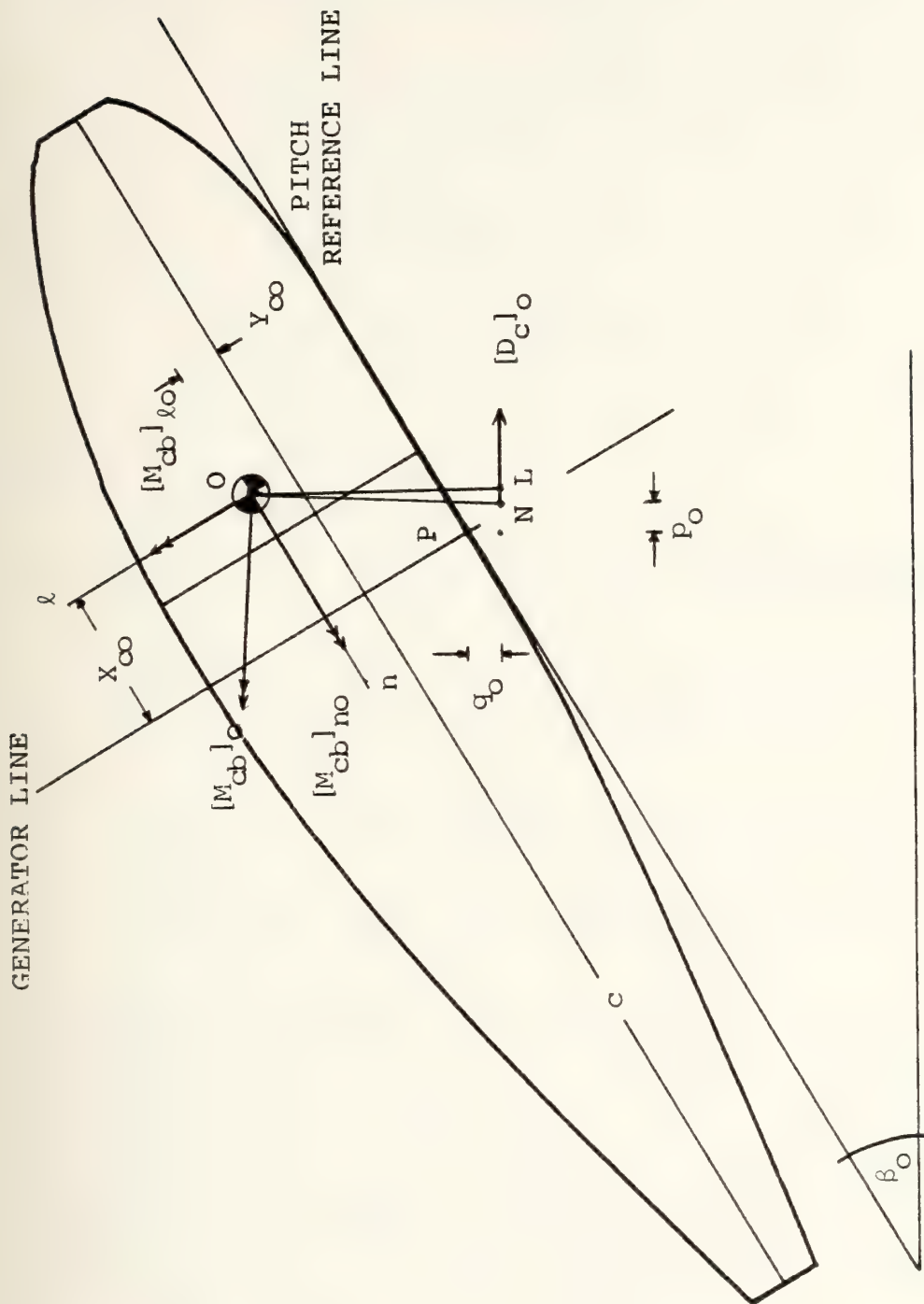


Figure 6.4 Internal Loading on a Blade Section at $x = x_O = r_O/R$

$$[F_c]_O = [C_F]_O \cos \zeta(x_O) \quad (6.9)$$

$$[D_c]_O = [C_F]_O \sin \zeta(x_O) \quad (6.10)$$

$$[M_{cb}]_O = [F_c]_O \cdot \overline{NO} \quad (6.11)$$

$$[M_{cw}]_O = [D_c]_O \cdot \overline{LO} \quad (6.12)$$

where:

$[F_c]_O$ = direct force, due to centrifugal action, acting on the blade section located at $x = x_O = r_O/R$;

$[D_c]_O$ = shear force, due to centrifugal action, acting on the blade section located at $x = x_O = r_O/R$;

$[M_{cb}]_O$ = bending moment at point "O" imposed by $[F_c]_O$ acting through point "N";

$[M_{cw}]_O$ = torsional moment at point "O" imposed by $[D_c]_O$ acting through point "N".

Since Schoenherr's method does not consider shear forces and their effects, $[D_c]_O$ and $[M_{cw}]_O$ will not be considered in this modification. However, $[F_c]_O$ and $[M_{cb}]_O$ must now be computed for each blade section along the propeller's radius in order to account for their contributions to equations (6.2), (6.3) and, finally, in equation (6.1).

The computation is derived as follows. Consider Figure (6.4). Again, $[F_c]_O$ acts through point "N" and is normal (outward) to the plane of the figure. $[M_{cb}]_O$ is now resolved into components of the ℓ -n axes system as follows:

$$[M_{cb}]_{no} = [C_F]_o \cos \zeta(x_o) \{p_o \sin \beta_o + q_o \cos \beta_o + Y_{co}\} \quad (6.13)$$

$$[M_{cb}]_{lo} = [C_F]_o \cos \zeta(x_o) \{q_o \sin \beta_o - p_o \cos \beta_o + X_{co}\} \quad (6.14)$$

where:

β_o = pitch angle of the blade section at
 $x = x_o = r_o/R$;

X_{co} = distance of the blade sections centroid from
the generator line (ft);

Y_{co} = distance of the blade section's centroid from
the pitch reference line (ft);

q_o = distance to point "N" from point "P"
parallel to the shaft axis at $x = x_o = r_o/R$
(ft);

p_o = distance to point "N" from point "P"
perpendicular to the shaft axis at
 $x = x_o = r_o/R$ (ft).

The quantity β_o is found by the relation:

$$\tan \beta_o = \frac{1}{\pi} \cdot (P/D)_o \quad (6.15)$$

For the Wageningen B-Screw Series, $(P/D)_o$ is a constant along R
except for propellers with $Z = 4$.

The quantity $[C_F]_o$ is computed from the relation:

$$[C_F]_o = 1728 \cdot \frac{w d V_o}{acg} \cdot (2\pi n_p)^2 \cdot R \cdot (\bar{x}_G)_o \quad (6.16)$$

where:

- V_o = volume of the blade element above the blade section at $x = x_o = r_o/R$ (ft³);
 acg = 32.174 ft/sec²;
 wd = material weight density (lbf/in³);
 n_p = propeller revolution rate (rps);
 $(\bar{x}_G)_o$ = non-dimensional radial position of "G" for the blade element above the shaft axis;
 R = propeller radius (ft).

At this point, only five quantities remain to be determined for the evaluation of the expressions of equations (6.10), (6.13) and (6.14). They are:

- 1) $\cos \zeta(x_o)$
- 2) p_o
- 3) q_o
- 4) $(\bar{x}_G)_o$
- 5) V_o

These quantities are determined by integration over the blade element above the blade section, located at $x = x_o = r_o/R$, from $x = x_o$ to $x = 1.0$.

The values for p_o and q_o will vary with the location of "G" (from Figure (6.3)) which depends on x . Consider a radially thin slice of the blade element above the blade section, located at $x = x_o = r_o/R$ (see Figure (6.5)). This thin "slice" is located at a non-dimensional distance x from the shaft centerline where $x_o < x < 1.0$. Figure (6.6) depicts this section expanded onto a plane. Let "g" be the centroid of that "slice". If x_g is the distance from the generator

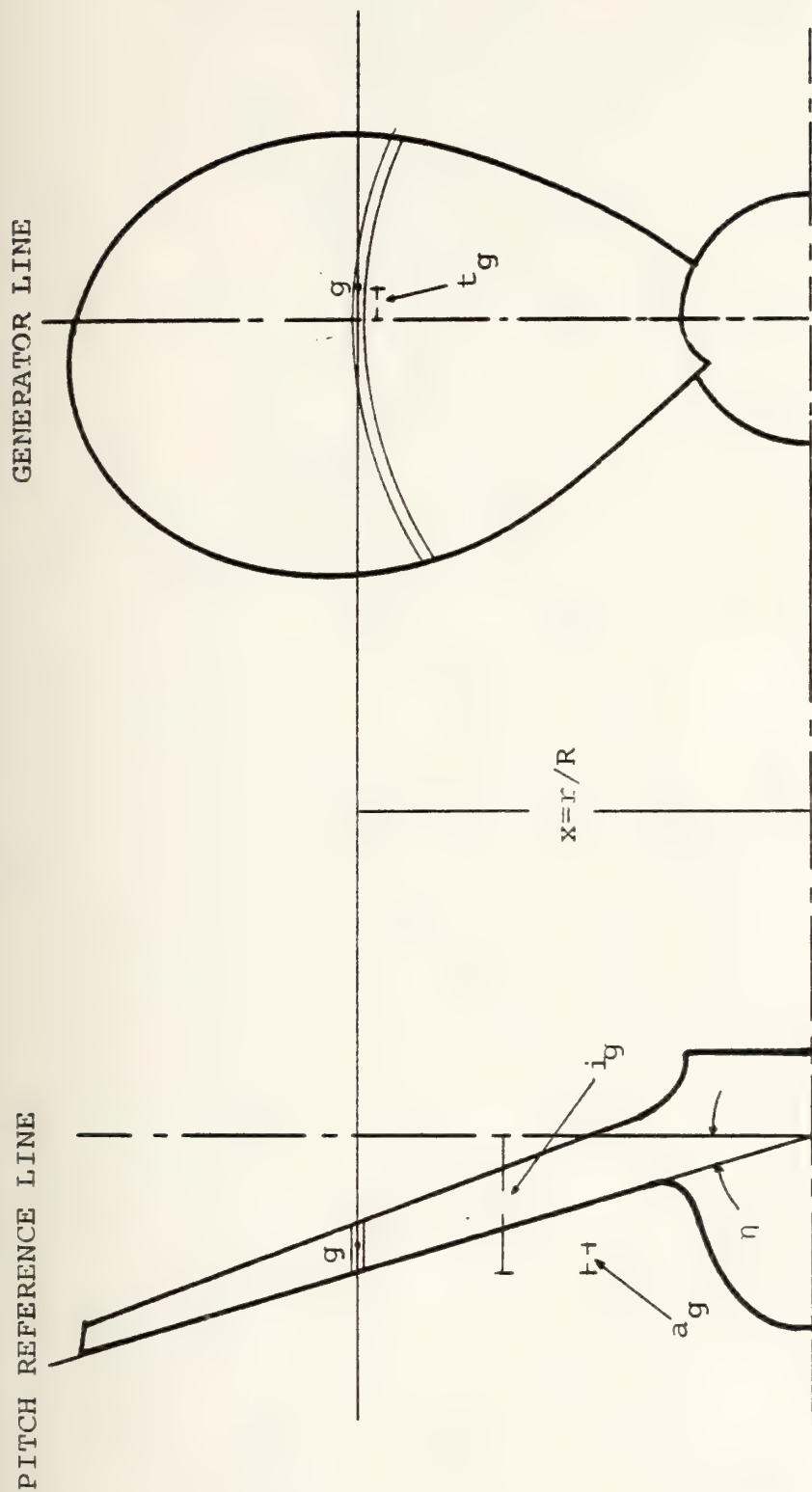


Figure 6.5 Position of "g" of a Blade Section "Slice" at $x = r/R$

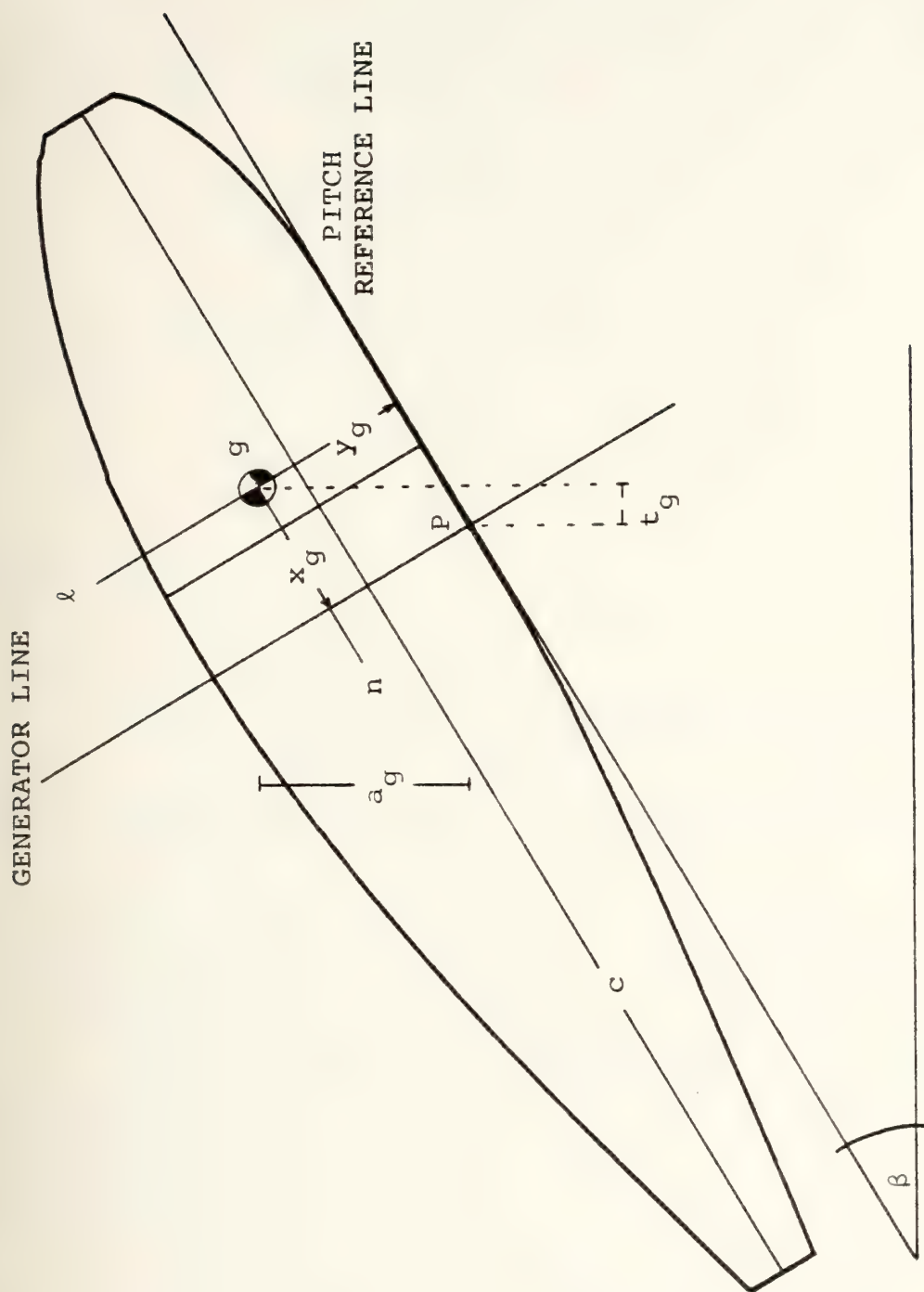


Figure 6.6 Coordinates of centroid "G" of any Blade Section

line to "g" in feet and y_g is the distance from the pitch reference line in feet, then, from Figures (6.5) and (6.6), the following relations apply:

$$i_g = x R \tan \eta \quad (6.17)$$

$$i_g - a_g = i_g - x_g \sin \beta_g - y_g \cos \beta_g \quad (6.18)$$

$$t_g = x_g \cos \beta_g - y_g \sin \beta_g \quad (6.19)$$

where:

η = rake angle at x ;

x_g = distance of "g" from point "P" parallel to the shaft axis (ft);

y_g = distance of "g" from point "P" perpendicular to the shaft axis (ft).

For the Wageningen B-Screw Series, rake angle η is a constant 15° everywhere along the radius R .

Now, to compute the volume of the blade element above the blade section located at $x = x_o = r_o/R$, the following expression is used:

$$V_o = \int_{x=x_o}^1 RA(x) dx \quad (6.20)$$

To compute the non-dimensional radial position of "G" for the blade element above a blade section located at $x = x_o = r_o/R$, the following expression is used:

$$(\bar{x}_G)_O = \frac{\int_{x=x_O}^1 A(x) x dx}{\int_{x=x_O}^1 A(x) dx} \quad (6.21)$$

To compute the tangential position of "G" for the blade element above a blade section located at $x = x_O = r_O/R$, the following expression is used:

$$T_O = \frac{\int_{x=x_O}^1 A(x) t_g dx}{\int_{x=x_O}^1 A(x) dx} \quad (6.22)$$

And, finally, to compute the axial position of "G" for the blade element above a blade section located at $x = x_O = r_O/R$, the following expression is used:

$$A_O = \frac{\int_{x_O}^1 A(x) (i_g - a_g) dx}{\int_{x_O}^1 A(x) dx} \quad (6.23)$$

Using the values just determined for $(\bar{x}_G)_O$, T_O and A_O , the following expressions are used to evaluate p_O and q_O :

$$p_O = \frac{x_O}{(\bar{x}_G)_O} T_O \quad (6.24)$$

$$q_o = A_o - x_o \tan \eta \quad (6.25)$$

The expression for $\cos \zeta(x_o)$ follows:

$$\cos \zeta(x_o) = \frac{p_o}{x_o R} \quad (6.26)$$

The formulation is now complete. Equations (6.10), (6.13) and (6.14) can now be evaluated for any blade section located at $x = x_o = r_o/R$. From here, equations (6.2) and (6.3) are evaluated. Finally, using the results from these equations and equation (6.9), equation (6.1) can be evaluated for the four points, specified by coordinates (u_1, w_1) , (u_2, w_2) , (u_3, w_3) and (u_4, w_4) , at any location $x = x_o = r_o/R$.

From the development discussed in Chapter V, the values for $A(x)$, x_g , y_g , I_{no} and I_{lo} are readily available.

D. ALGORITHM FOR THE CONSTRAINT

1. Theory

Schoenherr's formulation with the modifications just derived can be used in determining the minimum required equivalent blade section maximum thickness-to-chord ratio $((t^*/c)_{.75R \min})$ for use in the constraint $G_{11}(\bar{X}) \leq 0$ given by equation (4.8). The procedure employed is as follows:

- 1) assume an initial value for $(t^*/c)_{.75R \min}$ using equation (3.21);
- 2) increase this value by a small amount;

3) using $(t^*/c)_{.75R \text{ min}}$ obtained from step (2), generate a distribution of minimum required blade section thicknesses (t^*_{min}) for blade sections at specified points along the propeller radius, say at $r = .2R, .3R, .4R, .5R, .6R, .7R, .8R$ and $.9R$;

4) determine all blade section properties to include:
a) cross-sectional area, b) chord length, c) centroid location, d) moments of inertia with respect to the principal axes system (i.e., ℓ -n system), e) coordinate values for the four critical points defined in the previous section;

5) compute the hydrodynamic bending moment components $[M_P]_{n0}$ and $[M_P]_{\ell 0}$ at radius locations just specified;

6) compute the values of the centrifugal force $[F_c]_0$ and the bending moment components $[M_{cb}]_{n0}$ and $[M_{cb}]_{\ell 0}$ acting on blade sections at radius locations just specified;

7) calculate the direct fiber stresses at all four critical points for all radius locations specified in step (3);

8) check the following condition on the calculated fiber stress at all four critical points at all specified radius locations using:

$$[\sigma]_0 \leq 144 \cdot S_c \quad (6.27)$$

9) if the maximum allowable stress (S_c) for the material is exceeded, then return to step (2) and repeat. Otherwise, proceed to next step.

10) since the minimum required equivalent blade section maximum thickness-to-chord ratio assumed in step (2) has produced blade sections of adequate strength, evaluate the constraint given by equation (4.8).

2. Coding Details

The algorithm just outlined is incorporated into the main FORTRAN subprogram SUBROUTINE STRCNK. This subprogram, in turn, executes the algorithm through sequential calls to other key FORTRAN subprograms. These subprograms are listed as follows:

1) SUBROUTINE TDIST--accomplishes step (3); generates, at specified radius values, a distribution of minimum required blade section maximum thicknesses (t_{\min}^*) for the assumed value of $(t^*/c)_{.75R \min}$;

2) SUBROUTINE BLDPRP--accomplishes step (4); described previously in Chapter V;

3) SUBROUTINE HYDLD--accomplishes step (5); computes the hydrodynamic bending moment components, given by equations (6.6) and (6.7), at specified radius locations;

4) SUBROUTINE CNFGLD--accomplishes step (6); computes the centrifugal force and bending moments, given by equations (6.9), (6.13) and (6.14) respectively, at specified radius locations;

5) SUBROUTINE SIGNDS--accomplishes step (7); computes direct fiber stresses, given by equation (6.1), for all four critical points at every specified radius location.

During the remaining steps of SUBROUTINE STRCNK, the condition on allowable stress, given by equation (6.27), is checked at all critical points of blade sections located at specified radius locations (again, $r = .2R, .3R, .4R, .5R, .6R, .7R, .8R$ and $.9R$). The final calculation made is that for the constraint given by equation (4.8).

Again, extensive use of common blocks, for passing data from one subprogram to another, is apparent upon examination of the codes just cited. Comment cards are used throughout.

E. SUMMARY

The end of this chapter marks the completion of all prerequisite background and formulation discussions on the application of COPES/CONMIN to propeller selection problems involving the Wageningen B-Screw Series. From this point, each specific Design Case can now be solved as an optimization problem.

VII. DESIGN CASE NO. 1--PROGRAMMING AND COMPARISONS

A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of propeller selection problems which use the "thrust" approach. First, the thrust approach to the propeller selection problem is formulated. Then, a review of a previous author's solution to this problem is presented. Four variations to this propeller selection problem are solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results from the four variations.

B. THRUST APPROACH FORMULATION

1. Design Vector \bar{X}

As previously pointed out at the conclusion of Chapter III, Design Case No. 1 constitutes a propeller selection problem which is solved by the thrust approach. In this approach, the effective horsepower (P_E) and the ship's speed (V) are specified by the designer. From the viewpoint of optimization, the quantities P_E and V become preassigned parameters. This reduces the design vector \bar{X} (see Figure 4.1) to:

$$\bar{X} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ N_P \\ Q_S \end{array} \right\} \quad (7.1)$$

Having specified P_E and V , all of the design variables, as listed in equation (7.1), are not independent. Recalling equations (3.3) and (3.13), the following relationship results:

$$P_E = \frac{(1-t_d)}{(1-wt)} \cdot \eta_R \cdot \eta_O \cdot \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (7.2)$$

Rearranging terms, this equation becomes:

$$\eta_O Q_S N_P = \frac{(1-wt)}{(1-t_d)} \cdot \frac{P_E}{\eta_R} \cdot \frac{550 \cdot 60}{2\pi} \quad (7.3)$$

Considering that the open water efficiency (η_O) is evaluated prior to the computation of ($-\eta_O$), or $OBJ_{1,2}$, then both N_P and Q_S are not independent design variables. One must be selected as the independent design variable. Then, the other variable becomes dependent on the one just selected.

For this study, N_P is selected as the independent design variable. This choice will reduce the design vector \bar{X} for propeller selection problems using the thrust approach to the following:

$$\bar{X} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) \cdot .75R \\ N_P \end{array} \right\} \quad (7.4)$$

Finally, equation (3.8) implies an alternative definition of \bar{X} as given in equation (7.4). The design vector for Design Case No. 1 propeller selection problems is, therefore, defined as:

$$\bar{X1} = \left\{ \begin{array}{c} D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \\ J \end{array} \right\} \quad (7.5)$$

2. Powering Constraint

Having determined the design vector $\bar{X1}$, a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint $G_{12}(\bar{X})$ mentioned in Chapter IV.

Simply stated, the selected propeller, as defined by $\bar{X1}$, must develop enough thrust (T) so that the powering requirement, specified by P_E and V, is met. Using equation (3.2), the thrust developed by the propeller can be specified in terms of thrust horsepower (P_T) as:

$$(P_T)_{dev} = \frac{T V(1-wt)}{550} \quad (7.6)$$

From equation (3.9), it follows that:

$$(P_T)_{dev} = \frac{\rho n_P^2 D^4 K_T}{550} \cdot V(1-wt) \quad (7.7)$$

Using equation (3.12), the developed thrust horsepower can be defined in terms of developed effective horsepower given by:

$$(P_E)_{dev} = \frac{(1-td)}{(1-wt)} \cdot (P_T)_{dev} \quad (7.8)$$

The restriction imposed by the thrust approach method, where P_E and V are specified, can now be stated as:

$$P_E \leq (P_E)_{dev} \quad (7.9)$$

Rearranging equation (7.9), the constraint $G_{12}(\overline{X1})$ follows:

$$G_{12}(\overline{X1}) = 1 - \frac{(P_E)_{dev}}{P_E} \leq 0 \quad (7.10)$$

With the design vector $\overline{X1}$ and $G_{12}(\overline{X1})$ defined, the propeller selection problem represented by Design Case No. 1 can be stated under one equation as:

$$\begin{aligned} \text{Maximize:} \quad & F(\overline{X1}) = OBJ_{1,2} \\ \text{Subject to:} \quad & G_j(\overline{X1}) \leq 0 \quad j = 1, \dots, 12 \\ & x1_i^{lower} \leq x1_i \leq x1_i^{upper} \quad i = 1, \dots, 5 \end{aligned} \quad (7.11)$$

C. PREVIOUS SOLUTIONS

Triantafyllou [Refs. 3,21] considered a propeller selection problem represented by Design Case No. 1. In his example problem, the following parameters were specified:

- 1) $v = 1.139 \times 10^{-6} \text{ (m}^2\text{/sec)} = 1.22613 \times 10^{-5} \text{ (ft}^2\text{/sec)}$
- 2) $wt = .22$
- 3) $td = .19$
- 4) $\eta_R = 1.025$
- 5) $noscrw = 1$
- 6) $Z = 5$
- 7) $P_E = 18153 \text{ (hp)}$
- 8) $V = 24 \text{ (knots)}$
- 9) $D = 22.0 \text{ (ft)}$
- 10) $A_E/A_O = .85$

The hull under study in his example had the following dimensions:

- 1) Length = 710 (ft)
- 2) Draft = 30 (ft)
- 3) Beam = 100 (ft)

For his analysis, the design vector contained two variables and was specified as:

$$\overline{XT} = \begin{pmatrix} J \\ P/D \end{pmatrix}$$

Using an iterative scheme [Ref. 21: p. 79] to solve two equations in two unknowns, he maximized the open water efficiency (η_O) to obtain the following results:

$$P/D = 1.1651$$

$$N_P = 104 \text{ (rpm)}$$

$$P_D = 25544 \text{ (hp)}$$

$$\eta_o = .6676$$

For future comparisons, equations (3.8) and (3.3) give:

$$J = .8286$$

$$Q_S = 1290000.0 \text{ (ft-lbf)}$$

Triantafyllou's results are summarized in Table (IV).

D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (7.11), is now solved by COPES/CONMIN. Four solution variations are considered.

The first and second variations attempt to reproduce the solution given by Triantafyllou. The design vector \overline{XT} (NDV = 2) is used in both cases. One variation uses SUBROUTINE STRCNA to evaluate the constraint $G_{12}(\overline{XT})$ given by equation (4.8). The other uses SUBROUTINE STRCNK to determine $G_{12}(\overline{XT})$.

The remaining two variations will solve the propeller selection problem using the design vector \overline{XI} (NDV = 5) defined in equation (7.5). Again, one variation uses SUBROUTINE STRCNA; the other, SUBROUTINE STRCNK.

In all variations, the following parameters are used:

- 1) Temp = 59 (°F)
- 2) $\rho = 1.9384 \text{ (lbf-sec}^2/\text{ft}^4\text{)}$
- 3) $v = 1.2285 \cdot 10 \text{ (ft}^2/\text{sec)}$

- 4) $P_{\text{watvap}} = .247$ (psia)
- 5) $P_{\text{atm}} = 14.7$ (psia)
- 6) $wt = .22$
- 7) $td = .19$
- 8) $\eta_R = 1.025$
- 9) $noscrw = 1$
- 10) $h_{c_l} = 19.0$ (ft)
- 11) $D_{\text{lim}} = 22.0$ (ft)
- 12) $Z = 5$
- 13) $\text{promat} = 5$ (stainless steel; see Table (II))
- 14) $P_E = 18153$ (hp)
- 15) $V = 24.0$ (knots)

All of the above are initialized in the input phase (ICALC = 1) of each SUBROUTINE ANALIZ pertaining to each variation.

The constraint $G_{12}(\overline{X_I})$ or $G_{12}(\overline{X_T})$ is evaluated by SUBROUTINE BLPOW1 which appears in the execution section of each SUBROUTINE ANALIZ.

1. Variation 1

a. Programming Details

Since this variation uses the design vector $\overline{X_T}$, the following design variables of $\overline{X_I}$ become parameters and are specified in the input section of SUBROUTINE ANALIZ (ICALC = 1) as:

- 1) $D = 22.0$ (ft)
- 2) $A_E/A_O = .85$
- 3) $(t^*/c)_{.75R} = .0348$ (from equation 3.21).

For constraints, the following are used:

$$G_j(\overline{XT}) \leq 0 \quad j = 1, \dots, 8, 12$$

Only nine of twelve constraints are evaluated (NCON = 9).

Obviously, some of the twelve constraints are redundant since

D , A_E/A_O , and $(t^*/c)_{.75R}$ have been specified.

The upper (XT_i^{upper}) and lower (XT_i^{lower}) limits on the design variables J and P/D are set to be:

$$.01 \leq J \leq 1.6$$

$$.4 \leq P/D \leq 1.4$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable (XT_i) is also assigned on card image F under the field labeled X. The first list of card images in Appendix D lists all of the COPES control cards used for this variation and variation 2. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed first in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

2. Variation 2

a. Programming Details

Everything discussed above for the first variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the second variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNK.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed second in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

3. Variation 3

a. Programming Details

This variation uses the design vector $\overline{X1}$. For constraints, the following are used:

$$G_j(\overline{X1}) \leq 0 \quad j = 1, \dots, 12$$

All twelve constraints are evaluated (NCON = 12).

The upper ($X1_i^{\text{upper}}$) and lower ($X1_i^{\text{lower}}$) limits on the design variables D , P/D , A_E/A_O , $(t^*/c)_{.75R}$, and J are set as:

$$1.0 \leq D \leq 50.0 \quad (\text{ft})$$

$$.4 \leq P/D \leq 1.4$$

$$.2 \leq A_E/A_O \leq 1.1$$

$$.003 \leq (t^*/c)_{.75R} \leq .50$$

$$.01 \leq J \leq 1.6$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable (X_{1_i}) is also assigned on card image F under the field labeled X. The second list of card images in Appendix D lists all of the COPES control cards used for this variation and variation 4. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed third in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

4. Variation 4

a. Programming Details

Everything discussed above for the third variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the fourth variation, found in Appendix C, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed last in Appendix E. Results for this variation of the propeller selection problem are tabulated in Table (IV).

E. DISCUSSION

Overall, the results achieved in all variations compare reasonably well to the solution obtained by Triantafyllou. However, the following points can be made.

Variations 1 and 2 give the same results. This was expected in view of the fact that, even though constraints $G_9(\overline{XT})$ through $G_{11}(\overline{XT})$ were evaluated, these constraints were not considered in the optimization search conducted by CONMIN.

In variations 3 and 4, the diameter (D) was driven to the limit (D_{lim}). This bears out a fundamental rule in propeller design, i.e., the larger the propeller diameter (D), the greater the open water efficiency (η_o).

The minimum required equivalent blade section maximum thickness-to-chord ratio ($(t^*/c)_{.75R_{min}}$), computed in

variation 3, is substantially smaller than the one computed for variation 4. As pointed out in Reference [2], the empirical relation, expressed by equation (4.9) and derived from equation (70.x) [Ref. 46: p. 620], does not take into account the effects of centrifugal loading. These effects include, specifically, the direct stresses imposed by the inertia load of the blade and the bending moments which result from rake and skew of the blade. Therefore, the algorithm developed in Chapter VI should, and does, produce a larger value for $(t^*/c) .75R$.

A final observation on the results concerns the values of the open water efficiency. The "optimum" open water efficiency (η_o) achieved by Triantafyllou is lower than those achieved in variations 1 and 2. A possible reason for this might be the neglect of the term " dRe/dJ " in Triantafyllou's formulation of the analytical expressions [Ref. 21: p. 71] that he used in his analysis. The difference in the open water efficiencies subsequently accounts for the differences in the propeller revolution rate (N_p) and the delivered torque (Q_S) when the relation in equation (3.3) is considered.

TABLE IV

Design Case No. 1--Results

GROUP	ITEM	TRIANITA- FYLLLOU	1	2	3	4
Given	P_E	18153.0	18153.0	18153.0	18153.0	18153.0
	V	24.0	24.0	24.0	24.0	24.0
Design Variable Speci- fied	D	22.0	22.0	22.0		
	A_E/A_O	.85	.85	.85		
	(t^*/c) .75R		.0348	.0348		
Design Variables	D				21.9991	21.9659
	P/D	1.1651	1.0036	1.0036	.9981	1.0071
	A_E/A_O				.8205	.8149
	(t^*/c) .75R				.0330	.0642
	J	.8286	.7371	.7371	.7343	.7394
Maximize	η_O	.6676	.7091	.7091	.7109	.7066
Restric- tions	D_{lim}		(22.0)	(22.0)	22.0	22.0
	A_E/A_{Omin}		(0.5258)	(0.5258)	.5269	.5267
	(t^*/c) .75Rmin		(0.21266)	(0.50761)	.021998	.053259
Other	N_P	104	116.9	116.9	117.4	116.7
	Q_S	1290000	1080451.	1080451	1064574	1084003
	P_D	25544.0	24051.3	24051.3	24030.3	24094.8

VIII. DESIGN CASE NO. 2--PROGRAMMING AND COMPARISONS

A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of propeller selection problems which use the "power" approach. First, the power approach to the propeller selection problem is formulated. Then, a review of a previous author's solution to this problem is presented. Four variations to this propeller selection problem are solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results from the four variations.

B. POWER APPROACH FORMULATION

1. Design Vector \bar{X}_2

As previously pointed out at the conclusion of Chapter III, Design Case No. 2 constitutes a propeller selection problem which is solved by the power approach. In this approach, the delivered torque (Q_S) and the propeller revolution rate (N_P) are specified by the designer. From the viewpoint of optimization, the quantities Q_S and N_P become pre-assigned parameters. This reduces the design vector \bar{X} (see Figure (4.1)) to:

$$\bar{X} = \left\{ \begin{array}{c} P_E \\ V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (8.1)$$

Having specified Q_S and N_P , all of the design variables, as listed in equation (8.1), are not independent. Recalling equations (3.3), (3.11) and (3.13), the following relationship results:

$$P_E = \frac{(1-t_d)}{(1-w_t)} \cdot \eta_R \cdot \frac{J K_T}{2\pi K_Q} \cdot \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (8.2)$$

Rearranging terms, this equation becomes:

$$\frac{P_E}{V} = \frac{(1-t_d)}{(1-w_t)} \cdot \eta_R \cdot \frac{(1-w_t)}{n_P D} \cdot \frac{K_T}{K_Q} \cdot \frac{Q_S}{550} \cdot \frac{N_P}{60} \quad (8.3)$$

Considering the relations for K_T and K_Q in equation (3.17), then both P_E and V are not independent design variables. One must be selected as independent, while the other becomes dependent on the one selected.

For this study, V is selected as the independent design variable. This choice will reduce the design vector \bar{X} for propeller selection problems using the power approach to the following:

$$\bar{X} = \left(\begin{array}{c} V \\ D \\ P/D \\ A_E/A_O \\ (t^*/c)_{.75R} \end{array} \right) \quad (8.4)$$

Finally, equation (3.8) implies an alternative definition of \bar{X} as given in equation (8.4). The design vector for

Design Case No. 2 propeller selection problems is, therefore, defined as:

$$\overline{X2} = \left\{ \begin{array}{c} V \\ J \\ P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (8.5)$$

2. Powering Constraint

Having determined the design vector $\overline{X2}$, a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint $G_{12}(\overline{X})$ mentioned in Chapter IV.

Simply stated, the selected propeller, as defined by $\overline{X2}$, must absorb at least all of the power delivered to it (P_D) which is specified in terms of Q_S and N_P . Using equation (3.3), the power absorbed by the propeller can be specified in terms of delivered horsepower (P_D) as:

$$(P_D)_{\text{absorb}} = \frac{2\pi Q_P}{550} \cdot \frac{N_P}{60} \quad (8.6)$$

From equation (3.10), it follows that:

$$(P_D)_{\text{absorb}} = \frac{K_Q \rho n_P^2 D^5}{550} \cdot \frac{2\pi N_P}{60} \quad (8.7)$$

But, equation (3.3) also defines the power delivered to the propeller as:

$$P_D = \frac{2\pi Q_S}{550} \cdot \frac{N_P}{60} \quad (8.8)$$

The restriction imposed by the power approach method, where Q_S and N_P are specified, can now be stated as:

$$P_D \leq (P_D)_{\text{absorb}} \quad (8.9)$$

Rearranging equation (8.9), the constraint $G_{12}(\overline{X2})$ follows:

$$G_{12}(\overline{X2}) = 1 - \frac{(P_D)_{\text{absorb}}}{P_D} \leq 0 \quad (8.10)$$

Further simplification of equation (8.10) gives:

$$G_{12}(\overline{X2}) = 1 - \frac{Q_P}{Q_S} \leq 0 \quad (8.11)$$

With the design vector $\overline{X2}$ and $G_{12}(\overline{X2})$ defined, the propeller selection problem represented by Design Case No. 2 can be stated under one equation as:

$$\begin{aligned} \text{Minimize: } & F(\overline{X2}) = \text{OBJ}_{1,2} \\ \text{Subject to: } & G_j(\overline{X2}) \leq 0 \quad j = 1, \dots, 12 \\ & x2_i^{\text{lower}} \leq x2_i \leq x2_i^{\text{upper}} \quad i = 1, \dots, 5 \end{aligned} \quad (8.12)$$

C. PREVIOUS SOLUTIONS

Markussen [Ref. 4] considered a propeller selection problem represented by Design Case No. 2. In his example problem,

the following parameters were specified:

- 1) Temp = 18 (°C) = 64.4 (°F)
- 2) $P_{\text{watvap}} = 0.0206411 \text{ (bars)} = .29943921 \text{ (psia)}$
- 3) $P_{\text{atm}} = 1.01312856 \text{ (bars)} = 14.6974 \text{ (psia)}$
- 4) noscrw = 1
- 5) $h_{\text{cl}} = 6.7 \text{ (meters)} = 21.9827 \text{ (ft)}$
- 6) Z = 6
- 7) $P_D = 18.9 \text{ (MegaWatts)} = 25344.9 \text{ (hp)}$
- 8) $N_P = 110 \text{ (rpm)}$
- 9) $V_A = 15.65 \text{ (knots)}$

For his analysis, the design vector contained three variables and was specified as:

$$\overline{\text{XM}} = \begin{pmatrix} J \\ P/D \\ A_E/A_O \end{pmatrix}$$

A restriction for the minimum required expanded area ratio $((A_E/A_O)_{\text{min}})$, given by equation (4.3), was also considered. This imposed a constraint given by equation (4.4).

Using an iterative scheme [Ref. 4: p. 110] to solve three equations in three unknowns, Markussen maximized the open water efficiency (η_o) to obtain the following results:

$$J = .61095$$

$$P/D = .864380$$

$$A_E/A_O = 36.1861/40.6123 \text{ (m}^2\text{/m}^2\text{)}$$

$$= .891012$$

$$\eta_o = .654391$$

For future comparisons, equations (3.8) and (3.3) give:

$$D = 7.19091 \text{ (meters)} = 23.593375 \text{ (ft)}$$

$$Q_s = 1210130.0 \text{ (ft-lbf)}$$

Markussen's results are summarized in Table (V).

D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (8.12), is now solved by COPES/CONMIN. Four solution variations are considered.

The first and second variations attempt to reproduce the solution given by Markussen. The design vector $\overline{X_M}$ (NDV = 3) is used in both cases. One variation uses SUBROUTINE STRCNA to evaluate the constraint $G_{12}(\overline{X_M})$ given by equation (4.8). The other uses SUBROUTINE STRCNK to determine $G_{12}(\overline{X_M})$.

The remaining two variations will solve the propeller selection problem using the design vector $\overline{X_2}$ (NDV = 5) defined in equation (8.5). Again, one variation uses SUBROUTINE STRCNA; the other, SUBROUTINE STRCNK.

In all variations, the following parameters are used:

- 1) Temp = 64.4 (°F)
- 2) $\rho = 1.9892 \text{ (lbf-sec}^2/\text{ft}^4)$
- 3) $\nu = 1.1900 \times 10^{-5} \text{ (ft}^2/\text{sec)}$
- 4) $p_{\text{watvap}} = .2994 \text{ (psia)}$

- 5) $p_{atm} = 14.697$ (psia)
- 6) $wt = .22$
- 7) $td = .19$
- 8) $\eta_R = 1.025$
- 9) $n_{scrw} = 1$
- 10) $h_{cl} = 21.9827$ (ft)
- 11) $D_{lim} = 30.0$ (ft)
- 12) $z = 6$
- 13) $promat = 5$ (stainless steel; see Table (II))
- 14) $Q_S = 1210130$ (ft-lbf)
- 15) $N_P = 110$ (rpm)

All of the above are initialized in the input phase (ICALC = 1) of each SUBROUTINE ANALIZ pertaining to each variation.

The constraint $G_{12}(\overline{X2})$ or $G_{12}(\overline{XM})$ is evaluated by SUBROUTINE BLPOW2 which appears in the execution section of each SUBROUTINE ANALIZ.

1. Variation 1

a. Programming Details

Since this variation uses the design vector \overline{XM} , the following design variables of $\overline{X2}$ become parameters and are specified in the input section of SUBROUTINE ANALIZ (ICALC = 1). The ship's speed (V) is specified as:

$$\begin{aligned}
 V &= V_A / (1 - wt) \\
 &= 15.65 / (1 - .22) \\
 &= 20.0641 \text{ (knots)}
 \end{aligned}$$

Markussen elected to use the standard Wageningen blade section maximum thickness-to-chord ratios. Since the equivalent t/c is given as a function of Z and A_E/A_O (see equation (3.21)), then $(t^*/c)_{.75R}$ is calculated during each analysis (ICALC = 2) by the following relation:

$$(t^*/c)_{.75R} = (t/c)_{.75R}$$

For constraints, the following are used:

$$G_j(\overline{XT}) \leq 0 \quad j = 1, \dots, 8, 10, 12$$

Only ten of twelve constraints are considered (NCON = 10). Constraints $G_9(\overline{XM})$ and $G_{11}(\overline{XM})$ are redundant since no limit on the propeller diameter (D_{lim}) appears as a parameter in Markussen's formulation and $(t^*/c)_{.75R}$ was taken to be the Wageningen standard.

The upper (XM_i^{upper}) and lower (XM_i^{lower}) limits on the design variables J , P/D and A_E/A_O are set to be:

$$.01 \leq J \leq 1.1$$

$$.4 \leq P/D \leq 1.4$$

$$.4 \leq A_E/A_O \leq 1.1$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VIB. The initial value for each design variable (XM_i) is

also assigned on card image F under the field labeled X. The first list of card images in Appendix G lists all of the COPES control cards used for this variation and variation 2. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed first in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

2. Variation 2

a. Programming Details

Everything discussed above for the first variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the second variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed second in Appendix H.

Results for this variation of the propeller selection problem are tabulated in Table (V).

3. Variation 3

a. Programming Details

This variation uses the design vector $\overline{X2}$. For constraints, the following are used:

$$G_j(\overline{X2}) \leq 0 \quad j = 1, \dots, 12$$

All twelve constraints are evaluated (NCON = 12).

The upper ($X2_i^{\text{upper}}$) and lower ($X2_i^{\text{lower}}$) limits on the design variables V , P/D , A_E/A_O , $(t^*/c)_{.75R}$, and J are set as:

$$\begin{array}{rcccl} 10.0 & \leq & V & \leq & 100.0 \quad (\text{ft/sec}) \\ .4 & \leq & P/D & \leq & 1.4 \\ .4 & \leq & A_E/A_O & \leq & 1.1 \\ .003 & \leq & (t^*/c)_{.75R} & \leq & .50 \\ .01 & \leq & J & \leq & 1.1 \end{array}$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ($X2_i$) is also assigned on card image F under the field labeled X. The second list of card images in Appendix G lists all of the COPES control cards used for this variation and variation 4. These cards also specify the locations of the design variables

in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

An examination of SUBROUTINE ANALIZ for this variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed third in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

4. Variation 4

a. Programming Details

Everything discussed above for the third variation applies here with one exception. An examination of SUBROUTINE ANALIZ for the fourth variation, found in Appendix F, shows the calling statement made to SUBROUTINE STRCNK instead of SUBROUTINE STRCNA.

b. Results

The output from the optimization/analysis, performed by COPES/CONMIN, is listed last in Appendix H. Results for this variation of the propeller selection problem are tabulated in Table (V).

E. DISCUSSION

The results achieved in variations 1 and 2 compare extremely well to the solution obtained by Markussen. As

pointed out in the discussion in Chapter VII, variations 1 and 2 are expected to give the same results for the vector \overline{XM} . Obviously, the values obtained for J and P/D, as well as those for D, and $Rn_{.75R}^*$, are very close to the values generated in Markussen's example. However, the values for A_E/A_O are somewhat different. It is interesting to note that the value obtained in variations 1 and 2 (and, for that matter, variations 3 and 4) is, essentially, the limiting value for A_E/A_O , as given in Table (I), for $Z = 6$. Markussen's value for A_E/A_O (i.e., .891012) exceeds the limit (i.e., .80) in this table.

As pointed out at the end of Chapter VII, the minimum required blade section maximum thickness-to-chord ratio $((t^*/c)_{.75R_{min}})$, computed in variations 1 and 3, is substantially smaller than the one computed for variations 2 and 4. Again, the same explanation applies here as well.

The results of variations 3 and 4 differ somewhat from Markussen's results. The reason for this is simply that V_A (or V) has not been specified as a parameter. Consequently, a higher value for the advance ratio (J), which corresponds to a higher open water efficiency (η_o), has been found in the optimization search. This result can be interpreted in the following way. Given:

- 1) a six-bladed Wageningen propeller ($Z = 6$) which is made out of stainless steel (promat = 5);
- 2) a power train delivering 25344.9 (hp) at a rate of 110 (rpm);

3) a hull with a wake fraction (wt) equal to .22, a thrust deduction (td) equal to .19 and a shaft centerline depth (h_{cl}) of 21.98 (ft), then, the selected propeller, as defined by $\overline{X2}$, can drive this hull at a maximum speed of V when the hull has a maximum resistance given by P_E .

TABLE V

Design Case No. 2--Results

GROUP	ITEM	MARKUSSEN	VARIATIONS			
			1	2	3	4
Given	P_D	25344.9	25344.9	25344.9	25344.9	25344.9
	Q_S	1210130	1210130	1210130	1210130	1210130
	N_P	110	110	110	110	110
Design Variable Specified	V_A	15.65	15.65	15.65		
	V		20.0641	20.0641		
Design Variables	J	.61095	.6475	.6475	.9927	.8753
	V				30.9138	29.5355
	V_A				24.1127	23.0377
	P/D	.864380	.9036	.9036	1.1986	1.0308
	A_E/A_O	.891012	.8018	.8018	.7946	.7986
	$(t^*/c)_{.75R}$.0397	.0397	.0499	.0638
Maximize	η_O	.654391	.6660	.6660	.7643	.7330
Restrictions	D_{lim}		(30.0)	(30.0)	30.0	30.0
	A_E/A_{Omin}	.574729	.5070	.5070	.4622	.4266
	$(t^*/c)_{.75Rmin}$		(.02729)	(.0647)	.02706	.0638
Other	D	23.53	22.25	22.25	22.36	24.23
	P_E		(14057.3)	(14057.3)	19945.5	20653.7
	$Rn^*_{.75R}$	6.478×10^7	5×10^7	5×10^7	5×10^7	5×10^7

IX. DESIGN CASE NO. 3--PROGRAMMING AND COMPARISONS

A. INTRODUCTION

In this chapter, COPES/CONMIN is used in the solution of a propeller selection problem where "matching" is desired. First, the "matching" approach to the propeller selection problem is formulated. Then, a review of a previous author's solution is presented. One variation to this propeller selection problem is solved by COPES/CONMIN. The chapter is completed with a presentation and discussion of the results.

B. "MATCHING" FORMULATION

1. Design Vector \bar{X}

Design Case No. 3, the final powering problem considered in this study, constitutes a propeller selection problem solved by the "matching" approach. In this approach, the hull's effective horsepower (P_E) and speed (V), the delivered torque (Q_S) and the propeller revolution rate (N_p) are specified by the designer. This reduces the design vector \bar{X} (see Figure (4.1)) to:

$$\bar{X} = \left\{ \begin{array}{l} D \\ P/D \\ A_E/A_O \\ (t^*/c)_{.75R} \end{array} \right\} \quad (9.1)$$

For this study, the design vector \bar{X} is reduced further by eliminating the propeller diameter (D) as a design variable.

That is, D will also be specified by the designer so that the design vector for Design Case No. 3 is defined as:

$$\overline{X3} = \left\{ \begin{array}{c} P/D \\ A_E/A_O \\ (t^*/c) .75R \end{array} \right\} \quad (9.2)$$

2. Powering Constraint(s)

Having determined the design vector $\overline{X3}$, a final restriction to the general propeller selection problem, as stated by equation (4.24), remains for consideration. This restriction constitutes the remaining constraint $G_{12}(\overline{X})$ mentioned in Chapter IV as well as an additional constraint.

In the "matching" problem, the selected propeller, as defined by $\overline{X3}$, must satisfy two conditions. First, it must develop, as a minimum, the effective horsepower (P_E) as imposed by the design specification. Citing the formulation previously derived in Chapter VII, this condition can be stated as:

$$P_E \leq (P_E)_{dev} \quad (9.3)$$

The constraint $G_{12}(\overline{X3})$ follows accordingly as:

$$G_{12}(\overline{X3}) = 1 - \frac{(P_E)_{dev}}{P_E} \leq 0 \quad (9.4)$$

For the second condition, the selected propeller can only absorb, as a maximum, the delivered power (P_D) as

specified by the designer. The formulation is the same as that in Chapter VIII except that the inequality signs are reversed. The condition is stated as:

$$(P_D)_{\text{absorb}} \leq P_D \quad (9.5)$$

In defining a constraint $G_{13}(\overline{X3})$, another location, say location 24, in the GLOBCM block (see Table (III)) would be assigned. But, considering the fact that constraint $G_9(\overline{X})$ will not be used because the propeller diameter (D) is specified, there is no reason why $G_9(\overline{X3})$ cannot be redefined, for this Design Case only, as:

$$G_9(\overline{X3}) = \frac{(P_D)_{\text{absorb}}}{P_D} - 1 \leq 0 \quad (9.6)$$

Further simplification of equation (9.6) gives:

$$G_9(\overline{X3}) = \frac{Q_P}{Q_S} - 1 \leq 0 \quad (9.7)$$

In reality, the constraints just defined should be equality constraints. The word "match" does infer equality in some sense. However, as previously stated in Chapter II, the version of COPES/CONMIN used in this study does not directly handle equality constraints. But, since CONMIN attempts to minimize constraints in the optimization search, it will be assumed that a "match" can be achieved.

With the design vector $\overline{X3}$ and the constraints $G_{12}(\overline{X3})$ and $G_9(\overline{X3})$ defined, the propeller selection problem represented by Design Case No. 3 can now be stated under one equation as:

$$\begin{aligned} \text{Minimize:} \quad & F(\overline{X3}) = \text{OBJ}_3 \\ \text{Subject to:} \quad & G_j(\overline{X3}) \leq 0 \quad j = 1, \dots, 12 \quad (9.8) \\ & x3_i^{\text{lower}} \leq x_i \leq x3_i^{\text{upper}} \quad i = 1, \dots, 3 \end{aligned}$$

C. PREVIOUS SOLUTIONS

The propeller selection problem considered by Vassilopoulos [Ref. 18] actually represents a propeller "design" problem using the "power" approach. The example problem which he elected to solve is taken from that posed by the International Towing Tank Conference (ITTC) Propeller Committee. This problem is concerned with the determination of propeller thrust (T), diameter (D) and speed of advance (V_A) (or ship speed (V)) for a single-screw cargo ship where:

- 1) power available to the propeller (i.e., P_D) is 30,000 (hp)
- 2) $Z = 6$
- 3) $N = 105\text{--}110$ (rpm)
- 4) $D_{\text{lim}} = 23$ (ft)
- 5) $h_{c\ell} = 19$ (ft)

The variation of ship speed (V) and of hull effective power (P_E), thrust deduction factor (td) and the wake fraction (wt) is also given [Ref. 18: p. 20].

The results from Vassilopoulos' propeller design exercise produced a propeller that is "matched" at the following values:

- 1) $P_E = 21292.6$ (hp)
- 2) $V = 24.24$ (knots)
- 3) $Q_S = 1500606.75$ (ft-lbf)
- 4) $N_P = 105$ (rpm)
- 5) $P_D = 30000.0$ (hp)

His propeller "design" was based upon the following specified parameters:

- 1) Temp = 59 ($^{\circ}\text{F}$)
- 2) $\rho = 1.9905$ (lbf-sec²/ft⁴)
- 3) $p_{\text{watvap}} = .247$ (psia)
- 4) wt = .22
- 5) td = .1725
- 6) noscrw = 1
- 7) $h_{cl} = 19.0$ (ft)
- 8) Z = 6
- 9) promat = 5 (stainless teel, see Table (II))
- 10) D = 22.0 (ft)
- 11) $N_P = 105$ (rpm)
- 12) $P_D = 30000.0$ (hp)

Using an optimization scheme incorporated in his MVAPDP computer program, Vassilopoulos maximized the open water efficiency (η_o) and designed a propeller with the following characteristics:

- 1) J = .852
- 2) $K_T = .242$

- 3) $K_Q = .0478$
- 4) $\eta_o = .691$
- 5) $A_E/A_O = .767$
- 6) $bldwt = 7617.2 \text{ (lbf)}$

By utilizing both the lifting line and lifting surface methods in his design procedure, Vassilopoulos' MVAPDP program evolved a "constant stress" propeller blade. Consequently, the values for (t^*/c) and P/D varied non-linearly along the propeller radius (R). According to Vassilopoulos, this resulted in a minimum weight propeller. The values for P/D and (t^*/c) are listed in Tables (8) and (10) of his paper. From these values, $(t^*/c)_{.75R}$ is approximately .040.

While the propeller represented by Vassilopoulos' design is different, in many aspects (rake, skew, blade section aerfoil shape, etc.), from the Wageningen B-Screw Series propeller, it does represent a minimum weight propeller that has been "matched" to specific design values. Appropriate results are summarized in Table (VI).

D. SOLUTIONS BY COPES/CONMIN

The propeller selection problem, as stated by equation (9.8), is now solved by COPES/CONMIN. One solution variation is considered. The following parameters are used:

- 1) $Temp = 59 \text{ (}^\circ\text{F)}$
- 2) $\rho = 1.9905 \text{ (lbf-sec}^2\text{/ft}^4\text{)}$
- 3) $v = 1.2817 \times 10^{-5} \text{ (ft}^2\text{/sec)}$
- 4) $p_{watvap} = .247 \text{ (psia)}$

- 5) $p_{\text{atm}} = 14.7$ (psia)
- 6) $wt = .22$
- 7) $\eta_R = 1.025$
- 8) $\text{noscrw} = 1$
- 9) $h_{c\ell} = 19.0$ (ft)
- 10) $Z = 6$
- 11) $\text{promat} = 5$ (stainless steel, see Table (II))
- 12) $D = 22.0$ (ft)

Two problems are examined. Problem 1 specifies the following additional parameters:

- 1) $td = .1725$
- 2) $P_E = 21292.6$ (hp)
- 3) $V = 24.24$ (knots)
- 4) $Q_S = 1500606.75$ (ft-lbf)
- 5) $N_P = 105$ (rpm)

Problem 2 specifies the same parameters as:

- 1) $td = .171$
- 2) $P_E = 17630.0$ (hp)
- 3) $V = 23.0$ (knots)
- 4) $Q_S = 1500606.75$ (ft-lbf)
- 5) $N_P = 105$ (rpm)

All of the above are initialized in the input section (ICALC = 1) of similar versions of SUBROUTINE ANALIZ. Therefore, only one version is included in Appendix I.

The constraints for $G_9(\overline{X3})$ and $G_{12}(\overline{X3})$ are evaluated by SUBROUTINE BLPOW3 which appears in the execution section of

SUBROUTINE ANALIZ. Also, note that SUBROUTINE DICNUA has been deleted from the execution section, while SUBROUTINE WGTAL has been added.

1. Programming Details

All twelve constraints are evaluated ($NCON = 12$). The upper ($X3_i^{upper}$) and lower ($X3_i^{lower}$) limits on the design variables P/D , A_E/A_O and $(t^*/c)_{.75R}$ are set to be:

$$\begin{aligned} .4 &\leq P/D \leq 1.4 \\ .4 &\leq A_E/A_O \leq 1.1 \\ .003 &\leq (t^*/c)_{.75R} \leq .50 \end{aligned}$$

These upper and lower limits are specified in the COPES control card deck on card image F under respective fields VUB and VLB. The initial value for each design variable ($X3_i$) is also assigned on card image F under the field labeled X. The list of card images in Appendix J lists all of the COPES control cards used for both problems. These cards also specify the locations of the design variables in the common block GLOBCM (see Table (III)) as well as the locations of the constraints and their boundaries. Further details on the COPES control card requirements and the format of each card are contained in Reference [7].

2. Results

The outputs from the optimization/analysis, performed by COPES/CONMIN, are listed in Appendix K. Results of both problems are tabulated in Table (VI).

E. DISCUSSION

Table (VI) presents the results of problems 1 and 2 along with relevant information from Vassilopoulos' "design". Problem 1 attempted to "match" a Wageningen propeller at the design point found by Vassilopoulos. The first COPES/CONMIN printout in Appendix K indicates that the "match" was achieved at P_E equal to 21.168.1 (hp) and P_D equal to 28,150.0 (hp) (or, $Q_S = 1500607$ (ft-lbf) and $N_P = 105$ (rpm)). These values are judged to be close enough to the "Given" values in Table (VI).

It is apparent that the Wageningen propeller does not require all of the 30,000 (hp) of delivered horsepower. The propeller characteristics (i.e., J , K_T , K_Q and η_O) for problem 1 compare very well to Vassilopoulos' values. The expanded area ratios (A_E/A_O) are, also, very similar. Of course, the obvious difference is the blade weight (bldwt). The Wageningen propeller blade is over five thousand pounds heavier. Does this make sense for a minimum blade weight?

The answer is yes.

All one has to do is consider the values of (t^*/c) for problem 1 and Vassilopoulos' design. Vassilopoulos' "constant stress" blade was designed to "absorb" stress up to the allowable design limit of 5,400 (psi) (for stainless steel) all along the entire propeller radius (R). Table (12) in Reference [18] gives further details. The Wageningen propeller blade, however, represents an "older" type of blade which was designed with a linear blade section maximum thickness (t^*)

distribution. Consequently, it was "overdesigned" for strength beyond the $3/10$ -- $4/10$ radius (i.e., $.3R$ -- $.4R$) and contains excess material. A heavier blade, therefore, results. Note, also, that the optimizer did not drive the value of $(t^*/c)_{.75R}$ to the minimum acceptable value, $(t^*/c)_{.75R \min}$.

The results of problem 2 show the effect on blade weight (bldwt) for a Wageningen propeller when the hull's powering requirements (i.e., P_E at V) have been reduced. The weight reduction of 2000 pounds is significant. The complete results are listed in the second COPES/CONMIN printout in Appendix K.

TABLE VI
Design Case No. 3--Results

GROUP	ITEM	VASSILOPOULOS	PROBLEM	
			1	2
Given	P_E	21292.6	21292.6	17630.0
	V	24.24	24.24	23.0
	Q_S	1500607	1500607	1500607
	N_P	105	105	105
Design Variable Specified	D	22.0	22.0	22.0
Design Variables	P/D	*	1.1813	1.0906
	A_E/A_O	.767	.7944	.7742
	$(t^*/c)_{.75R}$.040	.0794	.0681
Minimize	bldwt	—	12842.6	10464.7
Maximize	η_O	.691	—	—
Restric- tions	D_{lim}	—	—	—
	A_E/A_{Omin}	—	.8515	.7722
	$(t^*/c)_{.75Rmin}$	—	.0691	.0681
Other	J	.852	.8290	.7866
	K_T	.242	.2349	.2063
	K_Q	.0478	.0448	.0372
	η_O	—	.6915	.6950
	bldwt	7617.2	—	—

* P/D varies with R

X. CONCLUSIONS AND RECOMMENDATIONS

A. CONCLUSIONS

The general purpose non-linear optimizer/synthesizer COPES/CONMIN has been successfully applied to three typical preliminary ship design propeller selection problems in which the Wageningen B-Screw Series is used. The formulation and programming of each required analysis code (i.e., SUBROUTINE ANALIZ) have been made as general as possible to allow the designer a broad variety of solution options for solving propeller selection problems which can be classified under any of the three Design Cases that were considered. The analysis codes have been "modularized" to the extent that methodical series data from other propeller series, which are available in the polynomial expression format of the B-Screw Series, can be easily adapted for powering analysis utilizing design optimization methods.

Further flexibility in the solution to the propeller selection problem has been achieved by using COPES/CONMIN as the optimizer/synthesizer. The designer has now been afforded the additional capability of specifying the design variables, the objective functions and the constraints of his choice. By solving propeller selection problems in the way presented in this thesis, repetitive problem formulation and coding have been eliminated.

There are other advantages to solving propeller selection problems specifically with COPES/CONMIN which have not been

directly addressed in this study. As stated in Chapter II, COPES/CONMIN is capable of performing optimization analyses, sensitivity studies, optimum sensitivity studies and optimization using approximation techniques. The designer, therefore, can select and perform any of these options, using the same analysis codes which have been presented in this thesis.

While the utilization of a general purpose non-linear optimizer in solving propeller selection problems allows the designer greater flexibility in the selection procedure, there is one important limitation that should be stressed at this point. This concerns the question whether or not the solution vector, determined by the optimizer, is a "global" optimum. As stated in Chapter II, COPES/CONMIN assures that, if a feasible solution vector is found, it is, at least, a "local" minimum (or maximum). This implies that, for two different initial design vectors which are specified in the COPES Control Card deck on card image F, the same optimum solution may not be determined by the optimizer. Both solutions would correspond to minimums (or maximums) of the objective function and are, therefore, correct. But, does one or the other correspond to the minimum (or maximum) of the entire vector design space, i.e., the "global" optimum? For the moment, at least, there is no definitive answer to the question.

Despite this uncertainty, progress in the field of design optimization continues to be made. Current developments [Ref. 47] will soon allow the designer to have a choice in

selecting a specific optimization algorithm from a "library" of proven optimization programs which employ the latest state-of-the-art numerical techniques. Again, using one analysis code, the designer will be able to generate any number of optimized solutions for the problem under study.

B. RECOMMENDATIONS

For future consideration, it is recommended that the automated design and trade-off capability, provided by a general purpose non-linear optimizer/synthesizer such as COPES/CONMIN, be applied to the more difficult problem of propeller design.

As pointed out in Chapter I, the use of the Wageningen B-Screw Series represents a "selection" procedure rather than a "design" process. Today, analytical propeller design procedures, utilizing lifting line and lifting surface theory, are becoming increasingly popular among propeller designers. The propeller design, which results from the utilization of these analytical methods, is, unquestionably, more efficient than the standard series propeller. However, these methods require consideration of many more design variables in the design process. This appears to be a natural application for the use of a general purpose non-linear optimizer/synthesizer.

Here, an analysis code, much larger than those which have been presented in this study, could be developed which would incorporate the lifting line/lifting surface theory

for the determination of the propeller performance characteristics, the local cavitation numbers and also the calculation of the pressure distributions over the blade. These pressure distributions would be utilized in the strength analysis of the blade. This analysis would utilize the finite element technique on an appropriately generated mesh model of the blade. Having defined the steps for this design procedure in the analysis code, the propeller designer now "couples" his analysis to the optimizer/synthesizer for determination of the optimum design. A massive amount of computer storage would certainly be required, but this concept is feasible and, in the author's view, is worthy of future consideration.

C. A FINAL NOTE

In conclusion, this thesis has demonstrated, in effect, another interesting application of the method of design optimization. The author, in no way, wishes to leave the reader with the impression that the techniques of design optimization are the "be all--end all" for engineering analysis. Design optimization techniques are useful and powerful tools that stand to relieve the engineer of the mundane tasks of numerical calculations and subsequent graphic plotting. But, they are just tools. In the final "analysis", good engineering judgment is paramount in their application and use.

APPENDIX A

FORTTRAN VARIABLE CROSS REFERENCE LIST

<u>Symbol</u>	<u>Fortran Variable</u>
A_E/A_O	AEDVAO
$(A_E/A_O)_{\min}$	AEAOMN
bldwt	WEIGHT
$C_{.75R}$	C75R
D	DIA
D_{\lim}	DIALIM
h_{cl}	HCL
J	RJ
K_Q	KQ
K_T	KT
noscrw	NOSCRW
N_P	N
P_E	PE
P_D	PD
P/D	PDIVD
p_{watvap}	PWATVA
p_{atm}	PATM
promat	PROMAT
Q_S	QS
$Rn^*_{.75R}$	R75R
S_c	SC
td	TD

APPENDIX A (CONT.)

<u>Symbol</u>	<u>Fortran Variable</u>
Temp	TEMP
$(t^*/c) .75R$	TC75R
V (ft/sec)	V
V (knots)	VK
wt	WT
Z	Z
η_o	ETAO
η_R	ETARR
ν	WATNU
ρ	WATRO

APPENDIX B

SUBROUTINE LISTINGS

```

SUBROUTINE ELDPRP
SUBROUTINE: BLDPRP
DATE OF LAST REVISION: APR 83
INPUT          OUTPUT
AEDVAD
DIA
Z
COMMON /THICD/

      COMMON/AREBLD/
      COMMON/CGX/

      COMMON/CGY/

      COMMON/CRDLNT/
      COMMON/VALU1/

      COMMON/VALW1/

      COMMON/VALU2/

      COMMON/VALW2/

```

EXPANDED AREA RATIO
PROPELLER DIAMETER (FEET)
NO. OF PROPELLER BLADES
BLADE SECTION MAXIMUM
THICKNESSES ALONG PROPELLER
RADIUS (FEET)
BLADE SECTION AREAS (FEET*2)
LOCATION OF BLADE CROSS-SEC-
TIONAL AREA CENTROID WITH
RESPECT TO GENERATOR LINE
(FEET)
LOCATION OF BLADE CROSS-SEC-
TIONAL AREA CENTROID WITH
RESPECT TO PITCH-REFERENCE
LINE (FEET)
BLADE SECTION CHORD LENGTHS
(FEET)
ORIGINATE OF CRITICAL POINT NO.
1 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ABSCISSA OF CRITICAL POINT NO.
1 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ORIGINATE OF CRITICAL POINT NO.
2 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ABSCISSA OF CRITICAL POINT NO.
2 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ORIGINATE OF CRITICAL POINT NO.
2 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ABSCISSA OF CRITICAL POINT NO.
2 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)

AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ORIGINATE OF CRITICAL POINT NO.
3 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ABSCISSA OF CRITICAL POINT NO.
3 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ORIGINATE OF CRITICAL POINT NO.
4 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
ABSCISSA OF CRITICAL POINT NO.
4 ON BLADE SECTION PERIPHERY
WITH RESPECT TO A SYSTEM OF
AXES PARALLEL AND NORMAL TO
PITCH-REFERENCE LINE WITH ORI-
GIN AT THE CENTROID OF THE
SECTION (FEET)
MOMENT OF INERTIA ABOUT NEU-
TRAL AXIS PARALLEL TO GENERA-
TOR LINE, PASSING THROUGH
BLADE CROSS SECTION CENTROID
(FEET**4)
MOMENT OF INERTIA ABOUT NEU-
TRAL AXIS PARALLEL TO PITCH
REFERENCE LINE, PASSING THROUGH
BLADE CROSS SECTION CENTROID
(FEET**4)

COMMON/VALU3/

COMMON/VALW3/

COMMON/VALU4/

COMMON/VALW4/

COMMON/A2MOMX/

COMMON/A2MOMY/

REAL*4
1 ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDI,VD,QS,TC75R,V,
2 RJC�L,RJCNU,R75RCL,R75RCU,AEADCL,AEACCU,TC75CL,TC75CU,
3 FCWBAL,DIACNU,AEADCV,TCSTRS,
4 VK,TC,WT,Z,WATKO,WATNU,TEMP,NOSCRW,HCL,PATM,
PATVA,PROMAT,DIALIM,ETARK,

CC


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5 REAL*4 RJ,C75R,R75R,KT,KQ
1 T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,T90R,
1 T100R,RAT
1 CR(1),AR(10),BR(10),PLF(10),PLA(10),IR(10),VIF(10,11),
2 V2F(10,11),VIA(10,10),V2A(10,10),YFACE(11),YBACK(11),
3 FA(10),Y(11),DELPA(10),AA(10),DLPASM,XA(10),YA(10),SUMAA,
4 SUMAXA,SUMAYA,SUMAX2A,SUMAY2A,FF(11),DELPE(11),AF(11),DLPFSM,
5 XFT(11),YF(11),SUMAF,SUMAXF,SUMAYF,SMAX2F,SMAY2F,AREA(10),
1 REAL*4 XMT(10),YPR(10),R1XNA(10),R1YNA(10),XCG(10),YCG(10),
1 U1(10),U2(10),U3(10),U4(10),W1(10),W2(10),W3(10),W4(10),
1 YBK,YFC,DN
1 INTEGER*4 I,IR,IPI
COMMON /GLOECM/ETA0,WEIGHT,AEDVA0,DIA,N,PE,FUIVD,QS,TC75R,V,RJCNL,
1RJCNNU,R75RCL,R75RCU,AEAOCL,AEAOCC,TC75CL,TC75CU,PQWBAL,DIACNU,
2AEACCV,TCSTFS,RJ
COMMON /PAR4M/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PROMAT,CLALIM,ETARR,AEADMN,TC75MN,SC
COMMON /THICD/TCOR,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,
1T90R,T100R,RAT
COMMON /AREELD/AREA
COMMON /CGX/XCG
COMMON /CGY/YCG
COMMON /CRDINT/CR
COMMON /VAL1/U1
COMMON /VAL2/U2
COMMON /VAL3/U3
COMMON /VAL4/U4
COMMON /VAL1/W1
COMMON /VAL2/W2
COMMON /VAL3/W3
COMMON /VAL4/W4
COMMON /A2MCMX/R1YNA
COMMON /A2MCMY/R1XNA

DETERMINE CFORD LENGTHS ALONG BLADE RADIUS USING TABLE (1), REF 2
IF(.NOT.(Z.EQ.3.0))GO TC 1
CR(1)=(1.5932*AEDVA0*DIA)/Z
GO TO 5
1 CONTINUE
IF(.NOT.(Z.EQ.4.0))GO TO 2
CR(1)=(1.5894*AEDVA0*DIA)/Z
GO TO 5
2 CONTINUE
IF(.NOT.(Z.EQ.5.0))GO TC 3
CR(1)=(1.5894*AEDVA0*DIA)/Z
GO TO 5
3 CONTINUE

```

C
C
C


```

IF(.NOT.(Z.EQ.6.0))GO TC 4
GO TO 5
4 CONTINUE
CR(1)=(1.6180*AEDVAO*DIA)/Z
5 CONTINUE
IF(.NOT.(Z.EQ.3.0))GO TC 6
CR(2)=(1.6330*AEDVAO*DIA)/Z
CR(3)=(1.8320*AEDVAO*DIA)/Z
CR(4)=(2.0000*AEDVAO*DIA)/Z
CR(5)=(2.1200*AEDVAO*DIA)/Z
CR(6)=(2.1860*AEDVAO*DIA)/Z
CR(7)=(2.1680*AEDVAO*DIA)/Z
CR(8)=(2.1270*AEDVAO*DIA)/Z
CR(9)=(1.6570*AEDVAO*DIA)/Z
CR(10)=(0.0000*AEDVAO*DIA)/Z
GO TO 7
6 CONTINUE
CR(2)=(1.6620*AEDVAO*DIA)/Z
CR(3)=(1.8820*AEDVAO*DIA)/Z
CR(4)=(2.0500*AEDVAO*DIA)/Z
CR(5)=(2.1520*AEDVAO*DIA)/Z
CR(6)=(2.1870*AEDVAO*DIA)/Z
CR(7)=(2.1440*AEDVAO*DIA)/Z
CR(8)=(1.9700*AEDVAO*DIA)/Z
CR(9)=(1.5820*AEDVAO*DIA)/Z
CR(10)=(0.0000*AEDVAO*DIA)/Z
7 CONTINUE
C
C
C
C
CALCULATE POSITION OF GENERATOR LINE (I.E., AR(I)) AND POSITION OF
MAXIMUM ELASE SECTION THICKNESS (I.E., ER(I)) USING FIGURE (1) AND
TABLE (1), REF 2
IF(.NOT.(Z.EQ.3.0))GO TC 8
AR(1)=C.617*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
8 CONTINUE
IF(.NOT.(Z.EQ.4.0))GO TC 9
AR(1)=C.61832*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
9 CONTINUE
IF(.NOT.(Z.EQ.5.0))GO TC 10
AR(1)=C.61832*CR(1)
BR(1)=C.350*CR(1)
GO TO 12
10 CONTINUE
APP01450
APP01460
APP01470
APP01480
APP01490
APP01500
APP01510
APP01520
APP01530
APP01540
APP01550
APP01560
APP01570
APP01580
APP01590
APP01600
APP01610
APP01620
APP01630
APP01640
APP01650
APP01660
APP01670
APP01680
APP01690
APP01700
APP01710
APP01720
APP01730
APP01740
APP01750
APP01760
APP01770
APP01780
APP01790
APP01800
APP01810
APP01820
APP01830
APP01840
APP01850
APP01860
APP01870
APP01880
APP01890
APP01900
APP01910
APP01920

```



```

IF(.NOT.(Z.EQ.6.0))GO TO 11
  AR(1)=C.61832*CR(1)
  BR(1)=C.350*CR(1)
  GO TO 12
11 CONTINUE
  AR(1)=C.6178*CR(1)
  BR(1)=C.350*CR(1)
  GO TO 13
12 CONTINUE
IF(.NOT.(Z.EQ.3.0))GO TO 13
  AR(2)=C.616*CR(2)
  AR(3)=C.611*CR(3)
  AR(4)=C.599*CR(4)
  AR(5)=C.583*CR(5)
  AR(6)=C.558*CR(6)
  AR(7)=C.526*CR(7)
  AR(8)=C.481*CR(8)
  AR(9)=C.400*CR(9)
  AR(10)=0.0
  BR(2)=C.350*CR(2)
  BR(3)=C.350*CR(3)
  BR(4)=C.350*CR(4)
  BR(5)=C.355*CR(5)
  BR(6)=C.385*CR(6)
  BR(7)=C.442*CR(7)
  BR(8)=C.478*CR(8)
  BR(9)=C.500*CR(9)
  BR(10)=0.0
  GO TO 14
13 CONTINUE
  AR(2)=C.617*CR(2)
  AR(3)=C.613*CR(3)
  AR(4)=C.601*CR(4)
  AR(5)=C.586*CR(5)
  AR(6)=C.561*CR(6)
  AR(7)=C.524*CR(7)
  AR(8)=C.463*CR(8)
  AR(9)=C.351*CR(9)
  AR(10)=0.0
  BR(2)=C.350*CR(2)
  BR(3)=C.350*CR(3)
  BR(4)=C.351*CR(4)
  BR(5)=C.355*CR(5)
  BR(6)=C.389*CR(6)
  BR(7)=C.443*CR(7)
  BR(8)=C.479*CR(8)
  BR(9)=C.500*CR(9)
  BR(10)=0.0
  GO TO 14
14 CONTINUE

```

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APP01530
APP01540
APP01550
APP01560
APP01570
APP01580
APP01590
APP02000
APP02010
APP02020
APP02030
APP02040
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APP02060
APP02070
APP02080
APP02090
APP02100
APP02110
APP02120
APP02130
APP02140
APP02150
APP02160
APP02170
APP02180
APP02190
APP02200
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APP02290
APP02300
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APP02330
APP02340
APP02350
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APP02370
APP02380
APP02390
APP02400

```


V1A(1,1,1)=0.1713
 V1A(1,1,2)=0.2098
 V1A(1,1,3)=0.2505
 V1A(1,1,4)=0.2715
 V1A(1,1,5)=0.2905
 V2F(1,1,1)=1.0000
 V2F(1,1,2)=0.9752
 V2F(1,1,3)=0.8860
 V2F(1,1,4)=0.8126
 V2F(1,1,5)=0.7213
 V2F(1,1,6)=0.6116
 V2F(1,1,7)=0.4680
 V2F(1,1,8)=0.3813
 V2F(1,1,9)=0.2749
 V2F(1,1,10)=0.1463
 V2F(1,1,11)=0.0000
 V2A(1,1,1)=1.0000
 V2A(1,1,2)=0.9414
 V2A(1,1,3)=0.7915
 V2A(1,1,4)=0.6909
 V2A(1,1,5)=0.5744
 V2A(1,1,6)=0.4438
 V2A(1,1,7)=0.2986
 V2A(1,1,8)=0.1406
 V2A(1,1,9)=0.0602
 V2A(1,1,10)=0.0000

GO TO 17

C
C
C

... OR BEGIN AT R=167R FOR PROPELLERS WITH 4,5,OR 6 BLADES

16 CONTINUE

V1F(1,1,1)=0.0000
 V1F(1,1,2)=0.0075
 V1F(1,1,3)=0.0357
 V1F(1,1,4)=0.0529
 V1F(1,1,5)=0.0881
 V1F(1,1,6)=0.1274
 V1F(1,1,7)=0.1808
 V1F(1,1,8)=0.2152
 V1F(1,1,9)=0.2541
 V1F(1,1,10)=0.3033
 V1F(1,1,11)=0.3756
 V1A(1,1,1)=0.0000
 V1A(1,1,2)=0.0282
 V1A(1,1,3)=0.0817
 V1A(1,1,4)=0.1134
 V1A(1,1,5)=0.1457
 V1A(1,1,6)=0.1812

APP02890
 APP02900
 APP02910
 APP02920
 APP02930
 APP02940
 APP02950
 APP02960
 APP02970
 APP02980
 APP02990
 APP03000
 APP03010
 APP03020
 APP03030
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 APP03050
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 APP03090
 APP03100
 APP03110
 APP03120
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 APP03200
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 APP03250
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 APP03270
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 APP03290
 APP03300
 APP03310
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 APP03330
 APP03340
 APP03350
 APP03360


```

V1A(1,1)=0.2184
V1A(1,8)=0.2565
V1A(1,9)=0.2764
V1A(1,10)=0.2946
V2F(1,1)=1.0000
V2F(1,2)=0.9754
V2F(1,3)=0.8847
V2F(1,4)=0.8056
V2F(1,5)=0.7167
V2F(1,6)=0.6065
V2F(1,7)=0.4613
V2F(1,8)=0.3751
V2F(1,9)=0.2686
V2F(1,10)=0.1395
V2F(1,11)=0.0000
V2A(1,1)=1.0000
V2A(1,2)=0.9391
V2A(1,3)=0.7868
V2A(1,4)=0.6850
V2A(1,5)=0.5676
V2A(1,6)=0.4372
V2A(1,7)=0.2936
V2A(1,8)=0.1372
V2A(1,9)=0.0576
V2A(1,10)=0.0000

```

17 CONTINUE

C
C
C

... CONTINUE FOR K=.2R

```

V1F(2,1)=0.000
V1F(2,2)=0.0049
V1F(2,3)=0.0304
V1F(2,4)=0.0502
V1F(2,5)=0.0804
V1F(2,6)=0.1180
V1F(2,7)=0.1685
V1F(2,8)=0.2000
V1F(2,9)=0.2353
V1F(2,10)=0.2821
V1F(2,11)=0.3560
V1A(2,1)=0.000
V1A(2,2)=0.0172
V1A(2,3)=0.0592
V1A(2,4)=0.0880
V1A(2,5)=0.1207
V1A(2,6)=0.1570
V1A(2,7)=0.1967
V1A(2,8)=0.2400

```

```

APP03370
APP03380
APP03390
APP03400
APP03410
APP03420
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APP03480
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APP03500
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APP03580
APP03590
APP03600
APP03610
APP03620
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APP03640
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APP03660
APP03670
APP03680
APP03690
APP03700
APP03710
APP03720
APP03730
APP03740
APP03750
APP03760
APP03770
APP03780
APP03790
APP03800
APP03810
APP03820
APP03830
APP03840

```



```

V1A(2,9)=0.2630
V1A(2,10)=0.2826
V2F(2,1)=1.0000
V2F(2,2)=0.9750
V2F(2,3)=0.9875
V2F(2,4)=0.9170
V2F(2,5)=0.9277
V2F(2,6)=0.9190
V2F(2,7)=0.9777
V2F(2,8)=0.9905
V2F(2,9)=0.9840
V2F(2,10)=0.9560
V2A(2,1)=1.0000
V2A(2,2)=0.9446
V2A(2,3)=0.9584
V2A(2,4)=0.9995
V2A(2,5)=0.9842
V2A(2,6)=0.9535
V2A(2,7)=0.9060
V2A(2,8)=0.9455
V2A(2,9)=0.9640
V2A(2,10)=0.9000

```

```

... CONTINUE FOR R=.3R

```

```

V1F(3,1)=0.0000
V1F(3,2)=0.0027
V1F(3,3)=0.0148
V1F(3,4)=0.0300
V1F(3,5)=0.0503
V1F(3,6)=0.0709
V1F(3,7)=0.1191
V1F(3,8)=0.1445
V1F(3,9)=0.1760
V1F(3,10)=0.2189
V1A(3,1)=0.0000
V1A(3,2)=0.0033
V1A(3,3)=0.0202
V1A(3,4)=0.0376
V1A(3,5)=0.0623
V1A(3,6)=0.0943
V1A(3,7)=0.1333
V1A(3,8)=0.1790
V1A(3,9)=0.2040
V2F(3,1)=1.0000

```

CC

```

APP03850
APP03860
APP03870
APP03880
APP03890
APP03900
APP03910
APP03920
APP03930
APP03940
APP03950
APP03960
APP03970
APP03980
APP03990
APP04000
APP04010
APP04020
APP04030
APP04040
APP04050
APP04060
APP04070
APP04080
APP04090
APP04100
APP04110
APP04120
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APP04140
APP04150
APP04160
APP04170
APP04180
APP04190
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APP04210
APP04220
APP04230
APP04240
APP04250
APP04260
APP04270
APP04280
APP04290
APP04300
APP04310
APP04320

```


V2F(3,2)=C.5750
V2F(3,3)=C.5920
V2F(3,4)=C.5315
V2F(3,5)=C.7520
V2F(3,6)=C.5505
V2F(3,7)=C.5130
V2F(3,8)=C.4265
V2F(3,9)=C.3197
V2F(3,10)=C.1890
V2F(3,11)=C.0000
V2A(3,1)=C.5833
V2A(3,2)=C.5265
V2A(3,3)=C.4335
V2A(3,4)=C.3195
V2A(3,5)=C.1985
V2A(3,6)=C.3360
V2A(3,7)=C.1670
V2A(3,8)=C.0800
V2A(3,9)=C.0000
V2A(3,10)=C.0000

CCC

... CCNTINUE FOR R=.4R

V1F(4,1)=C.0000
V1F(4,2)=C.0000
V1F(4,3)=C.0333
V1F(4,4)=C.0090
V1F(4,5)=C.0189
V1F(4,6)=C.0357
V1F(4,7)=C.0637
V1F(4,8)=C.0833
V1F(4,9)=C.1088
V1F(4,10)=C.1467
V1F(4,11)=C.2181
V1A(4,1)=C.0000
V1A(4,2)=C.0000
V1A(4,3)=C.0044
V1A(4,4)=C.0116
V1A(4,5)=C.0214
V1A(4,6)=C.0395
V1A(4,7)=C.0630
V1A(4,8)=C.0972
V1A(4,9)=C.1200
V1A(4,10)=C.1467
V2F(4,1)=C.0000
V2F(4,2)=C.5725
V2F(4,3)=C.5933
V2F(4,4)=C.5345

APP04330
APP04340
APP04350
APP04360
APP04370
APP04380
APP04390
APP04400
APP04410
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APP04430
APP04440
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APP04460
APP04470
APP04480
APP04490
APP04500
APP04510
APP04520
APP04530
APP04540
APP04550
APP04560
APP04570
APP04580
APP04590
APP04600
APP04610
APP04620
APP04630
APP04640
APP04650
APP04660
APP04670
APP04680
APP04690
APP04700
APP04710
APP04720
APP04730
APP04740
APP04750
APP04760
APP04770
APP04780
APP04790
APP04800


```

V2F(4,5)=0.7593
V2F(4,6)=0.6590
V2F(4,7)=0.5220
V2F(4,8)=0.4335
V2F(4,9)=0.3235
V2F(4,10)=0.1935
V2A(4,11)=0.0000
V2A(4,12)=1.6645
V2A(4,13)=0.8415
V2A(4,14)=0.7525
V2A(4,15)=0.6353
V2A(4,16)=0.5040
V2A(4,17)=0.3500
V2A(4,18)=0.1810
V2A(4,19)=0.0905
V2A(4,20)=0.0000
    .. CCNTINUE FCR R=.5R
V1F(5,1)=0.0000
V1F(5,2)=0.0000
V1F(5,3)=0.0000
V1F(5,4)=0.0008
V1F(5,5)=0.0034
V1F(5,6)=0.0085
V1F(5,7)=0.0211
V1F(5,8)=0.0328
V1F(5,9)=0.0500
V1F(5,10)=0.0778
V1F(5,11)=0.1278
V1A(5,12)=0.0000
V1A(5,13)=0.0000
V1A(5,14)=0.0000
V1A(5,15)=0.0012
V1A(5,16)=0.0040
V1A(5,17)=0.0100
V1A(5,18)=0.0190
V1A(5,19)=0.0330
V1A(5,20)=0.0420
V1A(5,21)=0.0522
V2F(5,22)=1.0000
V2F(5,23)=0.8800
V2F(5,24)=0.8275
V2F(5,25)=0.7478
V2F(5,26)=0.6430
V2F(5,27)=0.5039

```

```

APP04810
APP04820
APP04830
APP04840
APP04850
APP04860
APP04870
APP04880
APP04890
APP04900
APP04910
APP04920
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APP04940
APP04950
APP04960
APP04970
APP04980
APP04990
APP05000
APP05010
APP05020
APP05030
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APP05070
APP05080
APP05090
APP05100
APP05110
APP05120
APP05130
APP05140
APP05150
APP05160
APP05170
APP05180
APP05190
APP05200
APP05210
APP05220
APP05230
APP05240
APP05250
APP05260
APP05270
APP05280

```

CCC


```

V2F(5,8)=0.4135
V2F(5,9)=0.3056
V2F(5,10)=0.1750
V2F(5,11)=0.0000
V2A(5,1)=1.0000
V2A(5,2)=0.3635
V2A(5,3)=0.3456
V2A(5,4)=0.7580
V2A(5,5)=0.6439
V2A(5,6)=0.5140
V2A(5,7)=0.3565
V2A(5,8)=0.1865
V2A(5,9)=0.0550
V2A(5,10)=0.0000

```

CC

```

.. CCATINUE FOR K=.6R

```

```

V1F(6,1)=0.0000
V1F(6,2)=0.0000
V1F(6,3)=0.0000
V1F(6,4)=0.0000
V1F(6,5)=0.0000
V1F(6,6)=0.0006
V1F(6,7)=0.0022
V1F(6,8)=0.0067
V1F(6,9)=0.0165
V1F(6,10)=0.0382
V1A(6,1)=0.0000
V1A(6,2)=0.0000
V1A(6,3)=0.0000
V1A(6,4)=0.0000
V1A(6,5)=0.0000
V1A(6,6)=0.0000
V1A(6,7)=0.0000
V1A(6,8)=0.0000
V1A(6,9)=0.0000
V1A(6,10)=0.0000
V2F(6,1)=1.0000
V2F(6,2)=0.3690
V2F(6,3)=0.3790
V2F(6,4)=0.3090
V2F(6,5)=0.6060
V2F(6,6)=0.4620
V2F(6,7)=0.3775
V2F(6,8)=0.3720
V2F(6,9)=0.1485

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APP05290
APP05300
APP05310
APP05320
APP05330
APP05340
APP05350
APP05360
APP05370
APP05380
APP05390
APP05400
APP05410
APP05420
APP05430
APP05440
APP05450
APP05460
APP05470
APP05480
APP05490
APP05500
APP05510
APP05520
APP05530
APP05540
APP05550
APP05560
APP05570
APP05580
APP05590
APP05600
APP05610
APP05620
APP05630
APP05640
APP05650
APP05660
APP05670
APP05680
APP05690
APP05700
APP05710
APP05720
APP05730
APP05740
APP05750
APP05760

```



```

V2F(6,11)=0.0000
V2A(6,1)=1.0000
V2A(6,2)=C.5613
V2A(6,3)=0.5426
V2A(6,4)=C.1530
V2A(6,5)=0.5415
V2A(6,6)=0.5110
V2A(6,7)=C.5585
V2A(6,8)=0.1885
V2A(6,9)=0.0965
V2A(6,10)=0.0000
... CONTINUE FOR R=.7R
V1F(7,1)=0.0000
V1F(7,2)=0.0000
V1F(7,3)=0.0000
V1F(7,4)=0.0000
V1F(7,5)=0.0000
V1F(7,6)=0.0000
V1F(7,7)=0.0000
V1F(7,8)=0.0000
V1F(7,9)=0.0000
V1F(7,10)=0.0000
V1F(7,11)=0.0000
V1A(7,1)=0.0000
V1A(7,2)=0.0000
V1A(7,3)=0.0000
V1A(7,4)=0.0000
V1A(7,5)=0.0000
V1A(7,6)=0.0000
V1A(7,7)=0.0000
V1A(7,8)=0.0000
V1A(7,9)=0.0000
V1A(7,10)=0.0000
V1A(7,11)=1.0000
V2F(7,1)=0.0000
V2F(7,2)=C.5675
V2F(7,3)=0.5660
V2F(7,4)=C.1850
V2F(7,5)=C.5840
V2F(7,6)=0.5615
V2F(7,7)=0.5140
V2F(7,8)=C.5330
V2F(7,9)=0.2337
V2F(7,10)=0.1240
V2F(7,11)=0.0000
V2A(7,1)=1.0000
V2A(7,2)=C.5600

```

CCC

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APP05770
APP05780
APP05790
APP05800
APP05810
APP05820
APP05830
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APP05870
APP05880
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APP05900
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APP06220
APP06230
APP06240

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```

V2A(7,3)=C.E400
V2A(7,4)=C.E500
V2A(7,5)=C.E400
V2A(7,6)=C.E100
V2A(7,7)=C.E600
V2A(7,8)=C.E900
V2A(7,9)=C.E975
V2A(7,10)=C.E0000
.. CONTINUE FOR R=.8R
V1F(8,1)=0.C000
V1F(8,2)=0.C000
V1F(8,3)=0.C000
V1F(8,4)=0.C000
V1F(8,5)=0.C000
V1F(8,6)=0.C000
V1F(8,7)=0.C000
V1F(8,8)=0.C000
V1F(8,9)=0.C000
V1F(8,10)=0.C000
V1F(8,11)=0.C000
V1A(8,1)=0.C000
V1A(8,2)=0.C000
V1A(8,3)=0.C000
V1A(8,4)=0.C000
V1A(8,5)=0.C000
V1A(8,6)=0.C000
V1A(8,7)=0.C000
V1A(8,8)=0.C000
V1A(8,9)=0.C000
V1A(8,10)=0.C000
V1A(8,11)=1.C000
V2F(8,1)=0.C.E635
V2F(8,2)=0.C.E520
V2F(8,3)=0.C.E635
V2F(8,4)=0.C.E545
V2F(8,5)=0.C.E265
V2F(8,6)=0.C.E765
V2F(8,7)=0.C.E925
V2F(8,8)=0.C.E028
V2F(8,9)=0.C.E1050
V2F(8,10)=0.C000
V2A(8,1)=1.C000
V2A(8,2)=0.C.E600
V2A(8,3)=0.C.E400
V2A(8,4)=0.C.E750
V2A(8,5)=0.C.E400

```

CCC

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APP06250
APP06260
APP06270
APP06280
APP06290
APP06300
APP06310
APP06320
APP06330
APP06340
APP06350
APP06360
APP06370
APP06380
APP06390
APP06400
APP06410
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APP06630
APP06640
APP06650
APP06660
APP06670
APP06680
APP06690
APP06700
APP06710
APP06720

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```

V2A(8,6)=C.5100
V2A(8,7)=C.5600
V2A(8,8)=C.1900
V2A(8,9)=C.5975
V2A(8,10)=C.0000
... CONTINUE FOR R=.9R
V1F(9,1)=C.0000
V1F(9,2)=C.0000
V1F(9,3)=C.0000
V1F(9,4)=C.0000
V1F(9,5)=C.0000
V1F(9,6)=C.0000
V1F(9,7)=C.0000
V1F(9,8)=C.0000
V1F(9,9)=C.0000
V1F(9,10)=C.0000
V1F(9,11)=C.0000
V1A(9,1)=C.0000
V1A(9,2)=C.0000
V1A(9,3)=C.0000
V1A(9,4)=C.0000
V1A(9,5)=C.0000
V1A(9,6)=C.0000
V1A(9,7)=C.0000
V1A(9,8)=C.0000
V1A(9,9)=C.0000
V1A(9,10)=C.0000
V2F(9,1)=C.0000
V2F(9,2)=C.5600
V2F(9,3)=C.5400
V2F(9,4)=C.5500
V2F(9,5)=C.5400
V2F(9,6)=C.5100
V2F(9,7)=C.5600
V2F(9,8)=C.5775
V2F(9,9)=C.1900
V2F(9,10)=C.0975
V2F(9,11)=C.0000
V2A(9,1)=C.0000
V2A(9,2)=C.5600
V2A(9,3)=C.5400
V2A(9,4)=C.5500
V2A(9,5)=C.5400
V2A(9,6)=C.5100
V2A(9,7)=C.5600
V2A(9,8)=C.5775
V2A(9,9)=C.1900

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C
C
C

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APP06730
APP06740
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APP06990
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APP07100
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APP07120
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APP07170
APP07180
APP07190
APP07200

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```

V2A(9,5)=0.0975
V2A(9,10)=0.0000
... CONTINUE FOR R=1.0R
V1F(10,1)=0.0000
V1F(10,2)=0.0000
V1F(10,3)=0.0000
V1F(10,4)=0.0000
V1F(10,5)=0.0000
V1F(10,6)=0.0000
V1F(10,7)=0.0000
V1F(10,8)=0.0000
V1F(10,9)=0.0000
V1F(10,10)=0.0000
V1F(10,11)=0.0000
V1A(10,1)=0.0000
V1A(10,2)=0.0000
V1A(10,3)=0.0000
V1A(10,4)=0.0000
V1A(10,5)=0.0000
V1A(10,6)=0.0000
V1A(10,7)=0.0000
V1A(10,8)=0.0000
V1A(10,9)=0.0000
V1A(10,10)=0.0000
V2F(10,1)=1.0000
V2F(10,2)=0.9600
V2F(10,3)=0.8400
V2F(10,4)=0.7500
V2F(10,5)=0.6400
V2F(10,6)=0.5100
V2F(10,7)=0.3600
V2F(10,8)=0.2775
V2F(10,9)=0.1900
V2F(10,10)=0.0975
V2A(10,1)=1.0000
V2A(10,2)=0.9600
V2A(10,3)=0.8400
V2A(10,4)=0.7500
V2A(10,5)=0.6400
V2A(10,6)=0.5100
V2A(10,7)=0.3600
V2A(10,8)=0.1900
V2A(10,9)=0.0975
V2A(10,10)=0.0000

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C C C

C

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APP07220
APP07230
APP07240
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APP07300
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APP07360
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APP07470
APP07480
APP07490
APP07500
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APP07590
APP07600
APP07610
APP07620
APP07630
APP07640
APP07650
APP07660
APP07670
APP07680

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18 CC      CCNT INUE
    IF(.NCT. ((I.EQ.8).OR.(I.EQ.9)))GO TO 19
    DELPA(I)=(PLA(IR)/10.0)*C.5
19 GC TO 20
    CCNT INUE
    DELPA(I)=(PLA(IR)/10.0)*1.0
20 CCNT INUE
    AA(I)=(0.5*(HA(IP1)+HA(I)))*DELP A(I)
    DLPASM=DLPASM+DELP A(I)
    XA(I)=DLPASM-(DELP A(I)/2.0)
    YA(I)=(Y(IP1)+Y(I))/2.0
    SMAA=SUMAA+AA(I)
    SLMAXA=SUMAXA+(AA(I)*XA(I))
    SLMAXA=SUMAXA+(AA(I)*YA(I))
    SMAZA=SMAZA+(AA(I)*((XA(I))**2))+
    ((1.0/12.0)*((HA(I)+HA(IP1))/2.0)*(DELP A(I)**3))
    SMAYZA=SMAYZA+(AA(I)*((YA(I))**2))+
    ((1.0/12.0)*(DELP A(I))*((HA(I)+HA(IP1))/2.0)**5))
21 CCNT INUE
    ::CONTINUE WITH "FORWARD" PORTION (P=0 TO P=+1) OF BLADE
    SECTION AS DEPICTED IN FIGURE (1), REF 2
DC 25 I=1,10
    IF1=I+1
    YFACE(I)=(V1F(IR,I))*TR(IR)-(U.1*TR(IR))
    YEACK(I)=((V1F(IR,I)+V2F(IR,I))*(TR(IR)-(O.1*TR(IR))))
    HF(I)=YBACK(I)-YFACE(I)
    Y(I)=(YBACK(I)+YFACE(I))/2.0
    YFACE(IP1)=(V1F(IR,IP1))*TR(IR)-(O.1*TR(IR))
    YEACK(IP1)=((V1F(IR,IP1)+V2F(IR,IP1))
    HF(IP1)=YBACK(IP1)-YFACE(IP1)
    Y(IP1)=(YBACK(IP1)+YFACE(IP1))/2.0
    IF(.NCT. ((I.EQ.1).OR.(I.EQ.2)))GO TO 22
    DELPF(I)=(PLF(IR)/10.0)*2.0
    GC TO 24
    CCNT INUE
    IF(.NCT. ((I.EQ.7).OR.(I.EQ.8).OR.(I.EQ.9).OR.(I.EQ.10)))
    GC TO 23
    DELPF(I)=(PLF(IR)/10.0)*C.5
    GC TO 24
    CCNT INUE
    DELPF(I)=(PLF(IR)/10.0)*1.0
23 CCNT INUE
    AF(I)=(0.5*(HF(IP1)+HF(I)))*DELPF(I)
    DLPFSM=DLPFSM+DELPF(I)
24

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APP086650
APP086660
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APP086680
APP086690
APP086700
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APP086970
APP086980
APP086990
APP090000
APP090010
APP090020
APP090030
APP090040
APP090050
APP090060
APP090070
APP090080
APP090090
APP090100
APP090110
APP090120

1  XF(I)=DLPFISM-(DELPF(I)/2.0)
1  YF(I)=(Y(IP1)+Y(I))/2.0
25 CC SLMAF=SUMAF+AF(I)
CC SLMAXF=SUMAXF+(AF(I)*XF(I))
CC SLMAXYF=SUMAYF+(AF(I)*YF(I))
CC SMAX2F=SMAX2F+(AF(I)*((XF(I))*2))+
1 ((1.0/12.0)*((HF(I)+HF(IP1))/2.0)*(DELPF(I))*3))
1 SMAY2F=SMAY2F+(AF(I)*((YF(I))*2))+
1 ((1.0/12.0)*(DELPF(I))*((HF(I)+HF(IP1))/2.0))*3))
25 CC CONTINUE

CC DETERMINE BLADE SECTION PROPERTIES, I.E., CROSS-SECTION AREA,
CC ANCL AREA CENTROID LOCATION WITH RESPECT TO PITCH-REFERENCE
CC LINE AND LINE OF MAXIMUM BLADE SECTION THICKNESS AS DEPICTED
CC IN FIGURE (1), REF 2

AREA(IR)=SLMAA+SUMAF
IF(.NOT.(AREA(IR).LE.0.0))GO TO 26
XMT(IR)=0.0
YFRL(IR)=(DIA*(0.0030))/2.0
GC TO 27
26 CC CONTINUE
XMT(IR)=(SUMAXF)-(SUMAXA)/AREA(IR)
YFRL(IR)=(SUMAYF)+(SUMAYA)/AREA(IR)
27 CC CONTINUE

CC CALCULATE MOMENT OF INERTIA WITH RESPECT TO NEUTRAL AXES
CC USING PARALLEL AXIS THEOREM

R1YYNA(IR)=((SMAX2F)+(SMAX2A))-
1 ((XMT(IR))*2)*AREA(IR)
R1XXNA(IR)=((SMAY2F)+(SMAY2A))-
1 ((YFRL(IR))*2)*AREA(IR)

CC DETERMINE LOCATION OF "CG" (I.E., NEUTRAL AXES ORIGIN) WITH
CC RESPECT TO PITCH-REFERENCE LINE AND GENERATOR LINE DEPICTED
CC IN FIGURE (1), REF2

IF(.NOT.(XMT(IR).LT.0.0))GO TO 28
DN=BR(IR)+ABS(XMT(IR))
GC TO 30
28 CC CONTINUE
IF(.NOT.(XMT(IR).GT.0.0))GO TO 29
DN=BR(IR)-ABS(XMT(IR))
GC TO 30
29 CC CONTINUE
DN=BR(IR)
30 CC CONTINUE

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```

31 IF(.NOT.(DN.LT.AR(IR)))GO TO 31
   XCG(IR)=AR(IR)-DN
   GC TC 33
   CCNTINLE
32 IF(.NOT.(DN.GT.AR(IR)))GO TO 32
   XCG(IR)=- (DN-AR(IR))
   GC TO 33
   CCNTINLE
33 XCG(IR)=0.0
   CCNTINLE
   YCG(IR)=YPRL(IR)

CC DETERMINE "CRITICAL PCINTS" OF A BLADE SECTION WITH RESPECT
CC TO NEUTRAL AXES WHERE STRESSES ARE LIKELY TO BE A MAXIMUM
CC
CC ** CALCULATE ORDINATE (U2) AND ABCISSA (W2) OF CRITICAL POINT
CC NO. 2 DEPICTED IN FIGURE (10), REF 8
CC
W2(IR)=(AR(IR)-BR(IR))-XCG(IR)
U2(IR)=((V1A(IR,1)+V2A(IR,1))* (TR(IR)-(0.1*TR(IR))))+
      (0.1*TR(IR))-YPRL(IR)
1
CC
CC ** CALCULATE ORDINATE (U4) AND ABCISSA (W4) OF CRITICAL POINT
CC NO. 4 DEPICTED IN FIGURE (10), REF 8
CC
W4(IR)=W2(IR)
U4(IR)=((V1A(IR,1))* (TR(IR)-(0.1*TR(IR))))-YPRL(IR)
CC
CC ** CALCULATE ORDINATE (U1) AND ABCISSA (W1) OF CRITICAL POINT
CC NO. 1 DEPICTED IN FIGURE (10), REF 8
CC
W1(IR)=-((CR(IR)-AK(IR))+XCG(IR))
YBK=((V1A(IR,10)+V2A(IR,10))* (TR(IR)-(0.1*TR(IR))))
YFC=((V1A(IR,10))* (TR(IR)-(0.1*TR(IR))))
IF(.NOT.(YCG(IR).GT.YBK))GC TC 40
   U1(IR)=YFC-YCG(IR)
   GC TC 45
   CCNTINLE
40 IF(.NOT.(YCG(IR).LE.YBK).AND.(YCG(IR).GE.YFC)))GO TO 44
   IF(.NOT.(ABS(YCG(IR)-YFC).GT.(ABS(YBK-YCG(IR))))
   GC TO 41
   U1(IR)=YFC-YCG(IR)
   GC TO 43
   CCNTINLE
41 IF(.NOT.(ABS(YCG(IR)-YFC).LT.(ABS(YBK-YCG(IR))))
   GC TO 42
   U1(IR)=YBK-YCG(IR)
   GC TO 43
1

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42      CCNT INUE
43      U1( IR)=0.0
44      CCNT INUE
45      GC TO 45
      CCNT INUE
      U1( IR)=YBK-YCG( IR)
      CCNT INUE
      :: CALCULATE ORDINATE (U3) AND ABSCISSA (W3) OF CRITICAL POINT
      NC= 3 DEPICTED IN FIGURE (10), REF 8
      W3( IR)=CR( IR)-ABS(W1( IR))
      YBK=(V1F( IR,11)+V2F( IR,11))*( TR( IR)-(0.1*TR( IR)))
      YFC=(V1F( IR,11))*( TR( IR)-(0.1*TR( IR)))
      IF (.NOT.( YCG( IR).GT.YBK)) GC TC 46
      U3( IR)=YFC-YCG( IR)
      GC TC 51
      CCNT INUE
      IF (.NOT.( ( YCG( IR).LE.YBK).AND.( YCG( IR).GE.YFC))) GO TO 50
      IF (.NOT.( (ABS(YCG( IR)-YFC)).GT.(AES(YBK-YCG( IR))))))
      GC TO 47
      U3( IR)=YFC-YCG( IR)
      GC TO 49
      CCNT INUE
      IF (.NOT.( (ABS(YCG( IR)-YFC)).LT.(AES(YBK-YCG( IR))))))
      GC TO 48
      U3( IR)=YBK-YCG( IR)
      GC TC 49
      CCNT INUE
      U3( IR)=0.0
      CCNT INUE
      GC TO 51
      CCNT INUE
      U3( IR)=YBK-YCG( IR)
      CCNT INUE
      RETURN
      END

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APP05610
 APP05620
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 APP05710
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 APP05960
 APP05970
 APP05980


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SUBROUTINE ELDVCL(VCLBLD)
SUBROUTINE: BLDVOL
DATE OF LAST REVISION: APR 63
INPUT      OUTPUT      DEFINITION
DIA
Z          COMMON/AREBLD/ VOLBLD
COMMON/AREBLD/ VOLBLD
REAL*4 EIAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
1 RJCNU,RJCNU,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,
2 PCWBAL,DIACNU,AEACCU,TCSTRS,
3 VK,IT,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,
4 PWATVA,PRMAT,DIALIM,ETARR,
5 RJ,C75R,R75R,KT,KQ
REAL*4 AREA(10),WIDTH,VCL1,VOL2,VOLBLD
COMMON /GLOECM/ETAC,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,RJCNU,
1 RJCNU,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2 AEACCU,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1 PRMAT,DIALIM,ETARR,AEACMN,TC75MN,SC
COMMON /AREELD/AREA
DETERMINE BLADE VOLUME BY SIMPSON INTEGRATION SCHEME OF BLADE
CROSS-SECTIONAL AREAS FROM 2/10 RADIUS OUTWARD TO TIP
VOL1=((DIA/2.0)*0.1)/3.0)*(AREA(2)+(4.0*AREA(3))+(2.0*AREA(4))+(
1 (4.0*AREA(5))+(2.0*AREA(6))+(
2 (4.0*AREA(7))+(2.0*AREA(8))+(
3 (4.0*AREA(9))+AREA(10))
DETERMINE BLADE VOLUME FROM 2/10 RADIUS INWARD TO R=.18R FOR 3
BLADE PROPELLERS OR R=.167R FOR 4,5,6 BLADE PROPELLERS USING
SIMPLE TRAPEZOIDAL INTEGRATION SCHEME
IF(.NOT.(Z.EQ.3.0))GO TO 1
GO TO 5
WICLF=(DIA/2.0)*(0.2-0.18)
1 CONTINUE
IF(.NOT.(Z.EQ.4.0))GO TO 2
GO TO 5
WICLF=(DIA/2.0)*(0.2-0.167)
2 CONTINUE
IF(.NOT.(Z.EQ.5.0))GO TO 3

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APPLIC490
APPLIC500
APPLIC510
APPLIC520
APPLIC530
APPLIC540
APPLIC550
APPLIC560
APPLIC570
APPLIC580
APPLIC590
APPLIC600
APPLIC610
APPLIC620
APPLIC630
APPLIC640

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      WIC1F=(DIA/2.0)*(0.2-0.167)
      GO TO 5
3 CONTINUE
      IF(.NOT.(Z.EQ.6.0))GO TO 4
      WIC1F=(DIA/2.0)*(0.2-0.167)
      GO TO 5
4 CONTINUE
      WIC1F=(DIA/2.0)*(0.2-0.18)
5 CONTINUE
      VOL2=((AREA(1)+AREA(2))/2.0)*WIDTH
      CALCULATE PROPELLER BLADE VOLUME
      VOLBLD=VOL1+VOL2
      RETURN
      END

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SUBROUTINE ELPCW1(KT,SCWBAL)

SUBROUTINE: BLPCW1

INPUT OUTPUT

PE
KT
DIA
N
WATRO
V
WT
TD
ETARR

SCWBAL

DATE OF LAST REVISION: FEB 83

DEFINITION

HULL EFFECTIVE HORSEPOWER(HP)
THRUST COEFFICIENT
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
WATER DENSITY (LBF-SEC²/FT⁴)
SHIP SPEED (FT/SEC)
WAKE FRACTION
THRUST DEDUCTION
RELATIVE ROTATIVE EFFICIENCY
CONSTRAINT VARIABLE FOR PRO-
PELLER-DEVELOPED EFFECTIVE
HORSEPOWER CONSTRAINT
(SCWBALCO)

REAL*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
1 RJCNU,RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,
2 FOWBAL,DIA,NU,AEACCU,V,TCSTRS,RJ,
3 VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,
4 PWATVA,PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5 KI,PEDEV,THRST,FT,SCWBAL
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,RJCNU,
1 RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,PWMBAL,DIA,NU,
2 AEACCU,TCSTRS,RJ
COMMON /PAR4M/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1 PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

CALCULATE THRUST DEVELOPED BY EACH PROPELLER

THRST=(KT*WATRO*(DIA**4)*((N/60.0)**2))

DETERMINE HULL EFFECTIVE POWER DEVELOPED BY PROPELLER(S)

PT=(THRST*((1.0-WT)*V))/550.0

PEDEV=((1.0-TC)/(1.0-WT))*ETARR*PT)*NCSCRW)

CALCULATE CONSTRAINT VARIABLE

SCWEAL=1.0-(PEDEV/PE)

RETURN
END

APPLC670
APPLC680
APPLC690
APPLC700
APPLC710
APPLC720
APPLC730
APPLC740
APPLC750
APPLC760
APPLC770
APPLC780
APPLC790
APPLC800
APPLC810
APPLC820
APPLC830
APPLC840
APPLC850
APPLC860
APPLC870
APPLC880
APPLC890
APPLC900
APPLC910
APPLC920
APPLC930
APPLC940
APPLC950
APPLC960
APPLC970
APPLC980
APPLC990
APPL1000
APPL1010
APPL1020
APPL1030
APPL1040
APPL1050
APPL1060
APPL1070
APPL1080
APPL1090
APPL1100
APPL1110
APPL1120
APPL1130


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SUBROUTINE ELPOW2(KG,SCWBAL)
SUBROUTINE: BLPCW2
DATE OF LAST REVISION: FEB 83
INPT      OUTPUT
QS
KQ
DIA
N
WATRO
SOWBAL
DELIVERED TORQUE (FT-LBF)
TORQUE COEFFICIENT
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
WATER DENSITY (LBF-SEC2/FT4)
CONSTRAINT VARIABLE FOR PRO-
PELLER REQUIRED TORQUE
(SCWBAL<0)
REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVL,QS,TC75R,V,
1      FJCNL,RJCNL,R75FCL,R75RCU,AEADCL,TC75CL,TC75CU,
2      SCWBAL,DIA,CNU,AEADCV,TCSTRS,
3      VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,
4      PWATVA,PRGMAT,DIALIM,ETARK,
5      KC,QSABD,SOWBAL
COMMON /GLOBE/ETAC,WEIGHT,AEDVAO,DIA,N,PE,PDIVL,QS,TC75R,V,RJCNL,
1RJCNL,R75FCL,R75RCU,AEADCL,AEADCV,TC75CL,TC75CU,PDWBAL,DIA,CNU,
2AEADCV,TCSTRS,RJ
COMMON /PARM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRGMAT,DIALIM,ETARK,AEADMN,TC75MN,SC
CALCULATE THE TORQUE ABSORBED BY PROPELLER
QSABD=(KC*WATRO*(DIA**5)*((N/60.0)**2))
CALCULATE CONSTRAINT VARIABLE
SOWBAL=1.0-(QSABD/CS)
RETURN
END

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CCCC

SUBROUTINE ELPCW3(KT,KC,SOWBLP,SOWBLQ)

SUBROUTINE: BLPCW3

DATE OF LAST REVISION: FEB 83

INPLT OUTPUT

DEFINITION

KT
KQ
PE
N
QS
NDCSRW
WT
TD
ETARR

THRUST COEFFICIENT
TCRQUE COEFFICIENT
HULL EFFECTIVE HORSEPOWER (HP)
PROPELLER REVOLUTION RATE (RPM)
DELIVERED TORQUE (FT-LBF)
NUMBER OF PROPELLERS
WAKE FRACTION
THRUST DEDUCTION
RELATIVE ROTATIVE EFFICIENCY
CONSTRAINT VARIABLE FOR
EFFECTIVE HORSEPOWER
CONSTRAINT (SOWBLP<0)
CONSTRAINT VARIABLE FOR
DELIVERED POWER
CONSTRAINT (SOWBLQ<0)

SOWBLP

SOWBLQ

REAL*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
1 RJCNU,RJCNU,R75RCL,R75RCL,AEAOCL,AEAOCL,TC75CL,TC75CU,
2 FCWBLAL,DIA,CNU,AEAOCL,AEAOCL,TC75CL,TC75CU,
3 VK,TC,WT,Z,WATRC,WATNU,TEMP,NDCSRW,HCL,PATM,
4 PWATVA,PRCMT,DIAPIM,ETARK,AEADMA,TC75MN,SC,KT,KQ,
5 PII,ETAUSP,THRST,PEDEV,CPABD,PC,SOWBLP,SOWBLQ
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,KJCNU,
1 RJCNU,R75RCL,R75RCU,AEAOCL,AEAOCL,TC75CL,TC75CU,PWBLAL,DIA,CNU,
2 AEAOCL,TC75R,RJ
COMMON /FARAW/VK,TC,WT,Z,WATRC,WATNU,TEMP,NDCSRW,HCL,PATM,PWATVA,
1 PRCMT,CIALIM,ETARR,AEADMA,TC75MN,SC
PII=3.141592654

CALCULATE THRUST DEVELOPED BY EACH PROPELLER

THRST=(KT*WATRO*(DIA**4)*((N/60.0)**2))

DETERMINE EFFECTIVE POWER DEVELOPED BY PROPELLER(S)

PT=(THRST*((1.0-WT)*V))/550.0

PEDEV=((1.0-TC)/(1.0-WT))*ETARR*PT)*NDCSRW)

CALCULATE THE TORQUE ABSORBED BY PROPELLER

CPABD=(KC*WATRO*(DIA**5)*((N/60.0)**2))

C
C
C
C
C
C
C
C

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CALCULATE THE POWER ABSORBED BY THE PROPELLER
PDABD=(2.0*FII*CPABD*N)/33000.0
CALCULATE THE PCWER DELIVERED TO THE PROPELLER
PD=(2.0*FII*QS*N)/33000.0
DETERMINE CONSTRAINT VARIABLES
SOWELP=1.0-(PEDEV/PE)
SOWBLC=(PDABD/PC)-1.0
RETURN
END
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APP12020
APP12030
APP12040
APP12050
APP12060
APP12070
APP12080
APP12090
APP12100
APP12110
APP12120
APP12130
APP12140
APP12150
APP12160


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SUBROUTINE CALCKQ(J,PDIVD,AEDVAC,Z,REYN,KQ)
SUBROUTINE: CALCKQ      DATE OF LAST REVISION: FEB 83
INPUT      OUTPUT      DEFINITION
AEDVAO     EXPANDED AREA RATIO
PDIVD      PITCH-DIAMETER RATIO
Z          NO. OF BLADES
J          ADVANCE RATIO
REYN       REYNOLDS NO. 3/4 RADIUS
          (CORRECTED FOR T/C 3/4 RADIUS
          THRUST COEFFICIENT)

      KQ

REAL*4 J,PDIVD,AEDVAO,Z,REYN,KQ,KQ1,KQ2,KQ3,KQ4,KQ5,KQ6,KQ7,KQ8,
1 KQ9,KQ10,DELKQ,DELKQ1,DELKQ2,DELKQ3,REYFAC

FIRST, CALCULATING "KQ1" THRU "KQ10".....
KQ1=((0.379368)+((0.0886523)*(J**2))+((-0.32241)*J*PDIVD)+
1 ((-0.0344778)*(PCIVD**2))+((-0.040811)*PCIVD*AEDVAO)+
KQ2=(((-1.08009)*J*PDIVD*AEDVAO)+((-0.085381)*(J**2)*PDIVD*AEDVAC)+
1 ((-1.88561)*(PDIVD**2)*AEDVAO)+((-0.00370871)*J**2)+
2 ((-0.00513696)*PCIVD**2))
KQ3=(((-0.0209449)*J*PDIVD**2))+((-0.00474319)*(J**2)*PDIVD**2)+
1 ((-0.00723408)*(J**2)*AEDVAO**2))+((-0.00438388)*J*PDIVD*AEDVAO**2)+
2 ((-0.0269403)*(PDIVD**2)*AEDVAO**2))
KQ4=(((-0.0558082)*(J**3)*AEDVAO)+((-0.0161886)*(PDIVD**3)*AEDVAO)+
1 ((-0.00318086)*J*(PDIVD**3)*AEDVAC)+((-0.015896)*(AEDVAO**2))+
2 ((-0.0471129)*J*(AEDVAO**2)))
KQ5=(((-0.0196283)*(J**3)*(AEDVAO**2))+((-0.0502782)*PDIVD*
1 (AEDVAC**2))+((-0.030055)*(J**3)*PCIVD*(AEDVAO**2))+
2 ((-0.0417122)*(J**2)*PDIVD**2)*(AEDVAO**2))+
3 ((-0.0397722)*(PCIVD**3)*(AEDVAO**2)))
KQ6=(((-0.00350024)*(J**3)*(PDIVD**6)*(AEDVAC**2))+((-0.0106854)*(J**3)*
1 (Z)+((-0.00110903)*(J**3)*PDIVD**6)*(AEDVAO**2))+((-0.000313912)*(PDIVD**6)*
2 (Z)+((-0.0035595)*(J**3)*AEDVAO**2)))
KQ7=(((-0.0014211)*(PDIVD**6)*AEDVAC**2))+((-0.00383637)*J*
1 (AEDVAC**2)*Z)+((-0.0126803)*(PDIVD**2)*(AEDVAO**2)*Z)+
2 ((-0.00318278)*(J**2)*(PDIVD**3)*(AEDVAC**2)*Z))+
3 ((-0.00334268)*(PCIVD**6)*(AEDVAO**2)*Z))
KQ8=(((-0.00183491)*J*PDIVD**2)*Z)+((-0.000112451)*(J**3)*(PDIVD**2)*
1 (Z**2))+((-0.000257228)*(J**3)*(PCIVD**6)*(Z**2))+
2 ((-0.000269551)*J*AEDVAO*(Z**2))+((-0.00083265)*(J**2)*AEDVAO*
3 (Z**2))
KQ9=(((-0.00155334)*(PCIVD**2)*AEDVAO*(Z**2))+((-0.000302683)*
1 (PCIVD**6)*AEDVAC*(Z**2))+((-0.0001843)*(AEDVAO**2)*(Z**2))+
2 ((-0.000425359)*(PDIVD**3)*(AEDVAC**2)*(Z**2))+

```


APP12670
 APP12680
 APP12690
 APP12700
 APP12710
 APP12720
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 APP12920
 APP12930
 APP12940

```

3      ((.CC00E69243)*(J**3)*(PDIVD**3)*(AEDVAC**2)*(Z**2))
KQ10=(((-.00C4655)*(PDIVD**6)*(AEDVAC**2)*(Z**2))+
1      ((.000C554194)*J*(PDIVD**6)*(AEDVAC**2)*(Z**2))
NEXT,CALCULATE LOGARITHMIC REYNOLDS NUMER FACTOR "REYFAC"
WRITE(6,*)REYNO,REYNO,REYNO
REYFAC=ALOG10(REYNO)-0.301
THEN,CALCULATE "DELKQ=DELKQ1+DELKQ2+DELKQ3".....
DELKQ1=((-.0C0591412)+((.00696858)*PDIVD)+
1      ((-.000666654)*Z*(PDIVD**6))+((.0160818)*(AEDVAC**2))
DELKQ2=(((-.00538051)*REYFAC*PDIVD)+((-0.00059593)*REYFAC*
1      (PDIVD**2))+((.000782099)*(REYFAC**2)*(PDIVD**2))+
2      ((.0C00052159)*REYFAC*Z*(AEDVAC*(J**2)))
DELKQ3=(((-.0006088528)*(REYFAC**2)*Z*(AEDVAC*PDIVD**J))+
1      ((.0C00230171)*REYFAC*Z*(PDIVD**6))+((-0.00000184341)*
2      (REYFAC**2)*Z*(PDIVD**6))+((-0.00400252)*REYFAC*
3      (AEDVAC**2))+((-0.00220515)*(REYFAC**2)*(AEDVAC**2))
DELKQ=DELKQ1+DELKQ2+DELKQ3
FINALLY,CALCULATE TRUST COEFFICIENT "KC" WHERE
"KQ=KC1+KQ2+KQ3+KQ4+KC5+KQ6+KQ7+KQ8+KQ9+KQ10+DELKQ"
KQ=KQ1+KQ2+KQ3+KQ4+KQ5+KQ6+KQ7+KQ8+KQ9+KQ10+DELKQ
RETURN
END

```

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```

SUBROUTINE CALCKT(J,PDIVD,AEDVAC,Z,REYNC,KT)
SUBROUTINE: CALCKT      DATE OF LAST REVISION: FEB 83
INPUT      OUTPUT      DEFINITION
AEDVAC      EXPANDED AREA RATIO
PDIVD      PITCH-DIAMETER RATIO
Z      NC. OF BLADES
J      ADVANCE RATIO
REYNO      REYNOLDS NO. 3/4 RADIUS
          (CORRECTED FOR T/C 3/4 RADIUS
          THRUST COEFFICIENT

      KT

REAL*4 J,PDIVD,AEDVAC,Z,REYNO,KT,KT1,KT2,KT3,KT4,KT5,KT6,KT7,KT8,
      DELKT,DELKT1,DELKT2,DELKT3,REYFAC

      FIRST, CALCULATING "KT1" THRU "KT8".....1.
      KT1=.0080456+((-204554)*J)+((-166351)*PDIVD)+
      ((.158114)*(PDIVD**2))+((-147581)*(J**2)*AEDVAC)+
      ((-.481497)*J*PDIVD*AEDVAC)+((.415437)*(PDIVD**2)*AEDVAC)+
      ((.0144043)*Z)+((-0530054)*(J**2)+((-0143481)*PDIVD**2)
      KT3=((-.0606826)*J*PDIVC*Z)+((-0125894)*AEDVAC*Z)+
      ((.0109689)*J*AEDVAC*Z)+((-0133698)*(PDIVD**3))+((-00638407)*
      (PDIVD**6))
      KT4=((-.00132716)*(J**2)*(PDIVD**6))+((-168496)*(J**3)*AEDVAC)+
      ((-.0507214)*(AEDVAC**2))+((-0854555)*(J**2)*(AEDVAC**2))+
      ((-.0504475)*(J**3)*(AEDVAC**2))
      KT5=((-.010465)*J*(PDIVC**6)*(AEDVAC**2))+((-00841728)*(PDIVD**3)*Z)+
      ((.0168424)*J*(PDIVD**3)*Z)+((-00102296)*(J**3)*(PDIVD**3)*Z)
      KT6=((-.0317791)*(PDIVD**3)*AEDVAC*Z)+((-018604)*J*(AEDVAC**2)*Z)+
      ((-.00410798)*(PDIVD**2)*(AEDVAC**2)*Z)+((-000606848)*(Z**2)
      +((-0045815)*J*(Z**2))
      KT7=((-.0025583)*(J**2)*(Z**2))+((-000560528)*(J**3)*(Z**2))+
      ((-.00163652)*J*(PDIVD**2)*(Z**2))+((-000328787)*J*
      (PDIVD**6)*(Z**2))+((-000116502)*(J**2)*(PDIVD**6)*(Z**2))
      KT8=((.00060904)*AEDVAC*(Z**2))+((-00421749)*(PDIVD**3)*AEDVAC*
      (Z**2))+((-0000565229)*(J**3)*(PDIVD**6)*AEDVAC*(Z**2))+
      ((-.00146564)*(PDIVD**3)*(AEDVAC**2)*(Z**2))

      NEXT, CALCULATE LOGARITHMIC REYNOLDS NUMBER FACTOR "REYFAC"
      WRITE(6,*)REYNC,REYNO
      REYFAC=ALOG10(REYNO)-0.501

      THEN, CALCULATE "DELKT=DELKT1+DELKT2+DELKT3".....

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APPL33450
 APPL33460
 APPL33470
 APPL33480
 APPL33490
 APPL33500
 APPL33510
 APPL33520
 APPL33530
 APPL33540
 APPL33550
 APPL33560
 APPL33570
 APPL33580
 APPL33590
 APPL33600
 APPL33610

```

DELKT1=((.00C353485)+( (-.00333758)*AEDVAC*(J**2)))+
1  ((-.C04781251)*AEDVAC*PD1VD*J)
DELKT2=((.00C0257792)*(REYFAC**2)*AE[VAQ*(J**2)))+
1  ((.00C00643152)*(REYFAC*(PD1VD**6)*(J**2)))+
2  ((-.C000110636)*(REYFAC**2)*(PD1VD**6)*(J**2))
DELKT3=(((-.C000276305)*(REYFAC**2)*7*AEDVAC*(J**2)))+
1  ((.00C00954)*REYFAC*Z*AEDVAC*PD1VD*J)
2  ((.00C0032049)*REYFAC*(Z**2)*AEDVAC*(PD1VD**3)*J)
DELKT=DELKT1+DELKT2+DELKT3
FINALLY,CALCULATE TRUST COEFFICIENT "KT" WHERE
"KT=KT1+KT2+KT3+KT4+KT5+KT6+KT7+KT8+DELKT".....
KT=KT1+KT2+KT3+KT4+KT5+KT6+KT7+KT8+DELKT
RETURN
END

```

C

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```

SUBROUTINE CALCPE(KT,SPE)
SUBROUTINE: CALCPE
DATE OF LAST REVISION: FEB 83
INPUT      OUTPUT
KT
DIA
N
WATRO
TD
WT
V
SPE
REAL*4  EIAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
1      RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2      FCWBAL,DIACNU,AEACCV,TCSTRS,RJ,
3      VK,TC,WT,Z,WATRO,WATNU,TEMP,NOSCROW,HCL,PATM,
4      PWATVA,PROMAT,DIALIM,ETARK,AEADON,TC75MN,SC,
5      KT,SFE,T
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,RJCNU,
1      RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POMBAL,DIACNU,
2      AEACCV,TCSTRS,RJ
COMMON /PARM/VK,TC,WT,Z,WATRO,WATNU,TEMP,NOSCROW,HCL,PATM,PWATVA,
1      PROMAT,DIALIM,ETARR,AEADON,TC75MN,SC
CALCULATE THRUST DEVELOPED BY PROPELLER
T=(KT*WATRO*(DIA**4)*((N/60.0)**2))
CALCULATE EFFECTIVE HORSEPOWER DEVELOPED BY PROPELLER
SPE=((1.0-TC)/(1.0-WT))*((T*(1.0-WT)*V)/550.0)
RETURN
END

```

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SUBROUTINE CAVCNA(KT,SEACCV)

SUBROUTINE: CAVCNA

DATE OF LAST REVISION: FEB 83

INPUT OUTPUT

DEFINITION

PATM
WATRO
PWATVA

ATMOSPHERIC PRESSURE (PSIA)
WATER DENSITY (LBF-SEC²/FT⁴)
VAPORIZATION PRESSURE FOR WATER (PSIA)

HCL

DEPTH OF PROPELLER SHAFT

KT
DIA
N

CENTERLINE (FT)
THRUST COEFFICIENT
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE(RPM)

Z
NOSCRW
AEDVAO

NO. CF BLADES
NO. CF PROPELLERS
PROPELLER EXPANDED AREA RATIO
CONSTRAINT VARIABLE FOR PRO-
PELLER EXPANDED AREA RATIO
CONSTRAINT (SEACCV<0)

SEACCV

```
REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,FOI,VC,QS,TC75R,V,  
1      RJCNU,RJCNU,R75RCL,R75RCU,AEACCV,AEACCU,TC75CL,TC75CU,  
2      FCWBAL,DIA,ACNU,AEACCV,TCSTRS,RJ  
3      VK,IL,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,  
4      FWATVA,PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC,  
5      KI,PCMN,TPV,THRST,SEACMN,SEACCV  
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FOI,VD,QS,TC75R,V,KJCNU,  
1      RJCNU,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,POWBAL,DIA,ACNU,  
2      AEACCV,TCSTRS,RJ  
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,  
1      PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC
```

CALCULATE DIFFERENCE BETWEEN STATIC PRESSURE AT SHAFT CENTERLINE
& WATER VAPORIZATION PRESSURE

POMNPV=(PATM*144.0)+(WATRO*32.174*HCL)-(PWATVA*144.0)

DETERMINE MINIMUM REQUIRED EXPANDED AREA RATIO FOR PROPELLER(S)
USING "KELLER" CRITERIA FROM RELATION (13), REF 2

THRST=(KI*WATRO*(DIA**4)*((N/60.0)**2))
SEACMN=((1.3+(0.3*Z))*THRST)/((DIA**2)*FOMNFV)

CORRECT ACCORDING TO NUMBER OF PROPELLERS

IF(.NOT.(NOSCRW.EQ.1.0))GO TO 1

APPI4330
APPI4340
APPI4350
APPI4360
APPI4370
APPI4380
APPI4390
APPI4400
APPI4410
APPI4420
APPI4430
APPI4440
APPI4450
APPI4460
APPI4470
APPI4480
APPI4490
APPI4500
APPI4510
APPI4520
APPI4530
APPI4540
APPI4550
APPI4560
APPI4570
APPI4580
APPI4590
APPI4600
APPI4610
APPI4620
APPI4630
APPI4640
APPI4650
APPI4660
APPI4670
APPI4680
APPI4690
APPI4700
APPI4710
APPI4720
APPI4730
APPI4740
APPI4750
APPI4760
APPI4770
APPI4780
APPI4790
APPI4800

APP14810
 APP14820
 APP14830
 APP14840
 APP14850
 APP14860
 APP14870
 APP14880
 APP14890
 APP14900
 APP14910
 APP14920
 APP14930
 APP14940
 APP14950
 APP14960

```

      SEACMN=SEACMN+0.2
      GO TO 3
3 CONTINUE
1 IF(.NOT.(NO$CRW.EQ.2.0))GO TO 2
      SEACMN=SEACMN+0.1
      GO TO 3
2 CONTINUE
      SEACMN=SEACMN+0.0
3 CONTINUE
      AEACMN=SEACMN
      DETERMINE CONSTRAINT VARIABLE
      SEACOV=(SEACMN/AEDVAD)-1.0
      RETURN
      END
  
```

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SUBROUTINE CH75RA(C75R)

SUBROUTINE: CH75RA

DATE OF LAST REVISION: FEB 83

INPLT C75R OUTPUT

AEDVAO
DIA
Z

EXPANDED AREA RATIO
PROPELLER DIAMETER (FEET)
NO. OF BLADES
CHORD LENGTH AT 3/4 RADIUS
(FEET)

REAL*4 ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,

1

2

3

4

5

COMMON /GLDECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1RJCNL,RJCNL,R75RCL,R75RCL,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,CIACNU,
2AEACCU,TCSTFS,RJ
COMMON /PARAM/VK,TC,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1PRMAT,CIALIM,ETARR,AEACMN,TC75MN,SC

CALCULATE CHORD LENGTH AT 3/4 RADIUS USING RELATION (17), REF 2

C75R=(2.073*AEDVAO*CIA)/Z

RETURN
END

APPL14590
APPL15000
APPL15010
APPL15020
APPL15030
APPL15040
APPL15050
APPL15060
APPL15070
APPL15080
APPL15090
APPL15100
APPL15110
APPL15120
APPL15130
APPL15140
APPL15150
APPL15160
APPL15170
APPL15180
APPL15190
APPL15200
APPL15210
APPL15220
APPL15230
APPL15240
APPL15250
APPL15260
APPL15270


```

4 5 REAL*4 FPA1A,PRCMAT,DIALIM,ETARR,
REAL*4 RJ,C75R,R75R,KT,KQ
REAL*4 AREA(10),XCG(10),YCG(10),RIXNA(10),RIYNA(10)
REAL*4 SUMS, SUMSV, SUMSVT, SUMSVA
REAL*4 SMALLI(10),SMALLA(10),SMALLT(10),SMAIMA(10),
SINBET(10),CCSBET(10),SUMI(10),SUMA(10),
SUM(10),SUMR(10),SUMT(10),SMPO(10),SMQO(10)
1 2 VCL0(10),XBRCCG0(10),BIGTO(10),BIGAO(10),RADIUS
3 CMCBN(10),CMCBL(10),BIGNO(10),RADIUS
REAL*4 INTEGER*4 IF,IRPI,KCUNT,I
COMMON /GLOECM/ETA0,WEIGHT,AEDVA0,DIA,N,PE,PDIVD,QS,TC75R,V,RJUNL,
1RJCN0,R75RCL,R75RCL,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,CIA0NU,
2AEACCV,TCSTFS,RJ,TC,WI,Z,WATRU,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PR0MAT,LIALIM,ETARR,AEACMN,TC75MN,SC
COMMON /AREELD/AREA
COMMON /CGX/XCG
COMMON /CGY/YCG
COMMON /A2MCMX/RIYNA
COMMON /A2MCMY/RIXNA
COMMON /CFGEMN/CMCBN
COMMON /CFGEML/CMCBL
COMMON /CFGFD/BIGNO
PII=3.141592654
CC C
CALCULATE RAKE ANGLE IN RADIAN
ETA=15.C*(PII/180.0)
CC C
SPECIFY WEIGHT DENSITY OF MATERIAL SELECTED WHERE...
PRCMAT MATERIAL WEIGHT DENSITY (LBF/IN**3)
1 CAST IRON .260
2 CAST STEEL .284
3 TYPE 2 BRONZE .305
4 TYPE 4 NI-AL BRONZE .278
5 STAINLESS STEEL .283
IF (PRCMAT.EC.1.C)WD=.260
IF (PRCMAT.EC.2.C)WD=.284
IF (PRCMAT.EC.3.C)WD=.305
IF (PRCMAT.EC.4.C)WD=.278
IF (PRCMAT.EC.5.C)WD=.283
CC C
DETERMINE VALUES FOR...
SMALLI(IR) DISTANCE TO THE PITCH REFERENCE LINE, ALONG THE
PROPELLER RADII, WITH RESPECT TO A LINE NORMAL
APPI15760
APPI15790
APPI15800
APPI15810
APPI15820
APPI15830
APPI15840
APPI15850
APPI15860
APPI15870
APPI15880
APPI15890
APPI15900
APPI15910
APPI15920
APPI15930
APPI15940
APPI15950
APPI15960
APPI15970
APPI15980
APPI15990
APPI16000
APPI16010
APPI16020
APPI16030
APPI16040
APPI16050
APPI16060
APPI16070
APPI16080
APPI16090
APPI16100
APPI16110
APPI16120
APPI16130
APPI16140
APPI16150
APPI16160
APPI16170
APPI16180
APPI16190
APPI16200
APPI16210
APPI16220
APPI16230
APPI16240
APPI16250

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SMALLA(IR)
TC THE PROPELLER SHAFT AXIS AND PASSING THROUGH
THE INTERSECTION OF THE GENERATOR LINE AND THE
PROPELLER SHAFT AXIS (FEET)
DISTANCE, BLADE SECTION TO THE PROPELLER SHAFT AXIS,
OF EACH BLADE SECTION NEUTRAL AXES ORIGIN, GIVEN
BY COORDINATES (XCG(IR),YCG(IR)), WITH RESPECT
TO THE INTERSECTION OF THE PITCH REFERENCE LINE
AND THE GENERATOR LINE (FEET)
DISTANCE, NORMAL TO THE PROPELLER SHAFT AXIS,
OF EACH BLADE SECTION NEUTRAL AXES ORIGIN, GIVEN
BY COORDINATES (XCG(IR),YCG(IR)), WITH RESPECT
TO THE INTERSECTION OF THE PITCH REFERENCE LINE
AND THE GENERATOR LINE (FEET)

SMALLT(IR)

... FOR EACH BLADE SECTION ALONG THE PROPELLER RADIUS
DO 9 IR=2,10
  IF(.NOT.(Z.EQ.4.0))GO TO 7
  IF(.NOT.(IR.EQ.2))GO TO 2
  RF=0.822
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.3))GO TO 3
  RF=0.887
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.4))GO TO 4
  RF=0.950
  GC TO 6
  CCNT INUE
  IF(.NOT.(IR.EQ.5))GO TO 5
  RF=0.992
  GC TO 6
  CCNT INUE
  RF=1.00
  CCNT INUE
  DENOM=SQRT((RF*PDIVD)**2)+(PII**2)
  SINBET(IR)=(RF*PDIVD)/DENOM
  CCSBET(IR)=(PII)/DENOM
  GC TO 6
  CCNT INUE
  DENOM=SQRT((PDIVC**2)+(PII**2))
  SINBET(IR)=(PDIVC)/DENOM
  CCSBET(IR)=(PII)/DENOM
  CCNT INUE
  SMALLI(IR)=(DIA/2.0)*(FLOAT(IR)/10.0)*(TAN(ETA))
  SMALLA(IR)=(XCG(IR)*SINBET(IR))+(YCG(IR)*CCSBET(IR))
  SMALLT(IR)=(XCG(IR)*CCSBET(IR))-(YCG(IR)*SINBET(IR))

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APP16260
 APP16270
 APP16280
 APP16290
 APP16300
 APP16310
 APP16320
 APP16330
 APP16340
 APP16350
 APP16360
 APP16370
 APP16380
 APP16390
 APP16400
 APP16410
 APP16420
 APP16430
 APP16440
 APP16450
 APP16460
 APP16470
 APP16480
 APP16490
 APP16500
 APP16510
 APP16520
 APP16530
 APP16540
 APP16550
 APP16560
 APP16570
 APP16580
 APP16590
 APP16600
 APP16610
 APP16620
 APP16630
 APP16640
 APP16650
 APP16660
 APP16670
 APP16680
 APP16690
 APP16700
 APP16710
 APP16720
 APP16730


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C      SMAIMA(IR)=SMALLI(IR)-SMALLA(IR)
C      5 CONTINUE
C      CALCULATE TABULAR SUMMATIONS FOR INTEGRATION ALONG THE PROPELLER
C      RADIUS
C      DO 10 IR=2,5
C          IRP1=IR+1
C          SUM(IR)=AREA(IR)+AREA(IRP1)
C          SUMR(IR)=(FLOAT(IR)*AREA(IR))+(FLCAT(IRP1)*AREA(IRP1))
C          SUMT(IR)=(SMALLT(IR)*AREA(IR))+(SMALLT(IRP1)*AREA(IRP1))
C          SUMA(IR)=(SMAIMA(IR)*AREA(IR))+(SMAIMA(IRP1)*AREA(IRP1))
C      10 CONTINUE
C      INTEGRATE ALONG THE PROPELLER RADIUS TO DETERMINE THE FORCE AND
C      BENDING MOMENT COMPONENTS, ACTING ON A BLADE SECTION AT ITS NEUTRAL
C      AXES ORIGIN, WHICH ARE IMPOSED BY CENTRIFUGAL LOADING OF THE BLADE
C      ELEMENT VOLUME ABOVE THE BLADE SECTION UNDER CONSIDERATION
C      DO 12 IR=2,5
C          KCOUNT=IR
C          SUMSV=C.O
C          SUMSVR=O.O
C          SUMSVT=O.O
C          SUMSVA=O.O
C          DO 11 J=KCOUNT,5
C              SUMSV=SUMSV+SUM(I)
C              SUMSVR=SUMSVR+SUMR(I)
C              SUMSVT=SUMSVT+SUMT(I)
C              SUMSVA=SUMSVA+SUMA(I)
C          11 CONTINUE
C          DETERMINE BLADE ELEMENT VOLUME (FT**3) ABOVE THE BLADE
C          SECTION UNDER CONSIDERATION
C          VOLC(IF)=O.5*(CIA/2.O)*(O.I)*(SUMSV)
C          DETERMINE RADIAL FRACTION OF THE BLADE ELEMENT VOLUME'S CG
C          WITH RESPECT TO THE PROPELLER SHAFT AXIS
C          XRCGO(IR)=(SUMSVR/SUMSV)/10.O
C          DETERMINE DISTANCE (FEET) FROM THE GENERATOR LINE, NORMAL TO
C          THE PROPELLER SHAFT AXIS, OF THE BLADE ELEMENT VOLUME'S CG
C          WITH RESPECT TO THE PROPELLER SHAFT AXIS
C          BIGTO(IR)=(SUMSVT/SUMSV)

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APPL6740
APPL6750
APPL6760
APPL6770
APPL6780
APPL6790
APPL6800
APPL6810
APPL6820
APPL6830
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APPL6850
APPL6860
APPL6870
APPL6880
APPL6890
APPL6900
APPL6910
APPL6920
APPL6930
APPL6940
APPL6950
APPL6960
APPL6970
APPL6980
APPL6990
APPL7000
APPL7010
APPL7020
APPL7030
APPL7040
APPL7050
APPL7060
APPL7070
APPL7080
APPL7090
APPL7100
APPL7110
APPL7120
APPL7130
APPL7140
APPL7150
APPL7160
APPL7170
APPL7180
APPL7190
APPL7200
APPL7210

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CC      DETERMINE DISTANCE (FEET) FROM THE GENERATOR LINE, PARALLEL
CC      TO THE PROPELLER SHAFT AXIS, OF THE BLADE ELEMENT VOLUME'S CG
CC      WITH RESPECT TO THE PROPELLER SHAFT AXIS
CC
CC      BIGAO(IR)=(SUMSVA/SUMSV)
CC
CC      DETERMINE DISTANCE (FEET) FROM THE GENERATOR LINE, NORMAL TO
CC      THE PROPELLER SHAFT AXIS, OF THE BLADE ELEMENT VOLUME'S CG
CC      WITH RESPECT TO THE BLADE SECTION UNDER CONSIDERATION
CC
CC      SMFO(IF)=(FLOAT(IR)/10.0)/XBRCGO(IR)*BIGTO(IR)
CC
CC      DETERMINE DISTANCE (FEET) FROM THE GENERATOR LINE, PARALLEL
CC      TO THE PROPELLER SHAFT AXIS, OF THE BLADE ELEMENT VOLUME'S CG
CC      WITH RESPECT TO THE BLADE SECTION UNDER CONSIDERATION
CC
CC      SMC0(IF)=BIGAO(IR)-(((DIA/2.0)*(FLCAT(IR)/10.0)*IAN(ETA))
CC
CC      CALCULATE FORCE AND BENDING MOMENT COMPONENTS WHERE...
CC
CC      BIGNO(IR)      CENTRIFUGAL FORCE ACTING AT BLADE SECTION
CC      CMCBN(IR)      BENDING MOMENT COMPONENT, IMPOSED BY CENTRI-
CC                      FUGAL LOADING, PARALLEL TO PITCH REFERENCE
CC                      (CHORD) LINE OF A BLADE SECTION
CC      CMCBL(IR)      BENDING MOMENT COMPONENT, IMPOSED BY CENTRI-
CC                      FUGAL LOADING, NORMAL TO PITCH REFERENCE
CC                      (CHORD) LINE OF A BLADE SECTION
CC
CC      RADIUS=(FLCAT(IR)/10.0)*(CJA/2.0)
CC      BIGNO(IR)=(WCD#1728.0*VLO0(IR)*((2.0*PII*(N/60.0)**2)*
CC                      (DIA/2.0)*XBRCGO(IR))/32.174)*
CC                      (RADIUS/(SQRT((RADIUS**2)+((SMPO(IR)**2))))
CC      CMCBN(IR)=BIGNO(IR)*((SMPO(IR)*SINEET(IR))+
CC                      (SMQO(IR)*COSBET(IR))+YCG(IR))
CC      CMCBL(IR)=BIGNO(IR)*((XCG(IR)+
CC                      ((SMQO(IR)*SINBET(IR))- (SMPO(IR)*COSBET(IR))))
CC
CC      12 CONTINUE
CC      RETURN
CC      END

```



```

SUBROUTINE COEFS(SJ,R75R,KT,KQ)
SUBROUTINE: COEFS
INPLT      OUTPUT
AEDVAD
PDIVD
Z
SJ
R75R

      REAL*4  ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDIVC,QS,TC75R,V,
      RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
      FOWBAL,DIACNU,AEACCU,TCSTRS,RJ,
      VK,TC,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,
      PWATVA,PRMAT,DIALIM,ETARK,AEADMN,TC75MN,SC,
      R75R,KT,KQ,SJ
      COMMON /GLOECM/ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNU,
      RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
      AEACCU,TCSTRS,RJ
      COMMON /PAR4M/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
      PRMAT,DIALIM,ETARK,AEADMN,TC75MN,SC

      CALCULATE THRUST & TORQUE COEFFICIENTS USING THE WAGENINGEN SERIES
      POLYNOMIALS GIVEN IN TABLES (5) AND (6), REF 2

      CALL CALCKT(SJ,PDIVC,AEDVAC,Z,R75R,KT)
      CALL CALCKQ(SJ,PDIVC,AEDVAC,Z,R75R,KQ)
      RETURN
      END

```

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APPL17640
APPL17650
APPL17660
APPL17670
APPL17680
APPL17690
APPL17700
APPL17710
APPL17720
APPL17730
APPL17740
APPL17750
APPL17760
APPL17770
APPL17780
APPL17790
APPL17800
APPL17810
APPL17820
APPL17830
APPL17840
APPL17850
APPL17860
APPL17870
APPL17880
APPL17890
APPL17900
APPL17910
APPL17920
APPL17930
APPL17940
APPL17950
APPL17960
APPL17970

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CCCCCCCCCCCC

CCCC


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SUBROUTINE DICNUA(SIACNU)
SUBROUTINE: DICNUA
INPUT      OUTPUT
DIA
DIALIM

      SIACNU
REAL*4  ETAO,WEIGHT,AEDVAU,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1      RJCNU,R75RCL,R75RCU,AEAOCL,AEAGCU,TC75CL,TC75CU,POWBAL,DIACNU,
2      FCWBAL,DIACNU,AEACCV,TCSTRS,RJ,
3      VK,TC,WT,Z,WATRC,WATNU,TEMP,NOSCRW,HCL,PATM,
4      PWATVA,PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5      SIACNU
COMMON /GLOECM/ETAC,WEIGHT,AEDVAU,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1RJCNU,R75RCL,R75RCU,AEAOCL,AEAGCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,PWATVA,
1PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC
      DETERMINE CCNSTRNT VARIABLE CF PROPELLER DIAMETER'S UPPER BUUND
      CCNSTRNT
      SIACNU=(DIA/DIALIM)-1.0
      RETURN
      END

```

```

APPI18000
APPI18010
APPI18020
APPI18030
APPI18040
APPI18050
APPI18060
APPI18070
APPI18080
APPI18090
APPI18100
APPI18110
APPI18120
APPI18130
APPI18140
APPI18150
APPI18160
APPI18170
APPI18180
APPI18190
APPI18200
APPI18210
APPI18220
APPI18230
APPI18240
APPI18250
APPI18260
APPI18270
APPI18280

```

DATE OF LAST REVISION: FEB 83

DEFINITION

PROPELLER DIAMETER (FT)
 MAXIMUM DIAMETER PERMITTED BY
 HULL'S AFTERBODY (FT)
 CCNSTRNT VARIABLE (UPPER
 BUUND) ON DIAMETER (DIACNU<0)
 BDIVC,QS,TC75R,V,
 AEAOCL,AEAGCU,TC75CL,TC75CU,
 NOSCRW,HCL,PATM,
 AEADMN,TC75MN,SC,
 DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
 TC75CL,TC75CU,POWBAL,DIACNU,
 TCSTRS,RJ,
 TEMP,NOSCRW,HCL,PATM,PWATVA,
 PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

DETERMINE CCNSTRNT VARIABLE CF PROPELLER DIAMETER'S UPPER BUUND
 CCNSTRNT

SIACNU=(DIA/DIALIM)-1.0
 RETURN
 END

APPI18310
APPI18320
APPI18330
APPI18340
APPI18350
APPI18360
APPI18370
APPI18380
APPI18390
APPI18400
APPI18410
APPI18420
APPI18430
APPI18440
APPI18450
APPI18460
APPI18470
APPI18480
APPI18490
APPI18500
APPI18510
APPI18520
APPI18530
APPI18540
APPI18550
APPI18560
APPI18570
APPI18580
APPI18590
APPI18600
APPI18610
APPI18620
APPI18630
APPI18640
APPI18650
APPI18660
APPI18670
APPI18680
APPI18690
APPI18700
APPI18710
APPI18720
APPI18730
APPI18740
APPI18750
APPI18760
APPI18770
APPI18780

SUBROUTINE EXTCCN(2, AEDVAO, TC75K, AEAUCL, AEACCU, TC75CL, TC75CU)
SUBROUTINE: EXTCCN DATE OF LAST REVISION: FEB 83
INPUT OUTPUT DEFINITION
Z AEDVAO NC, OF BLADES
TC75R AEAUCL EXPANDED AREA RATIO
 AEACCU BLADE THICKNESS-TO-CHORD RATIO
 TC75CL EXPANDED AREA RATIO CONSTRAINT
 TC75CU VARIABLE (LOWER BOUND) (AEAUCU<0)
 EXPANDED AREA RATIO CONSTRAINT
 VARIABLE (UPPER BOUND) (AEACCU<0)
 BLADE THICKNESS-TO-CHORD RATIO
 CONSTRAINT VARIABLE (LOWER BOUND)
 (TC75CL<0)
 BLADE THICKNESS-TO-CHORD RATIO
 CONSTRAINT VARIABLE (UPPER BOUND)
 (TC75CU<0)
REAL*4 Z, AEDVAO, TC75K, AEAUCL, AEACCU, TC75CL, TC75CU,
1 AEAUCL, AEACCU, TC75CL, TC75CU
SET LIMITS ON EXPANDED AREA RATIO BASED ON NUMBER OF BLADES
IF (.NOT. (Z.EQ.3.0)) GO TO 1
 AEAUCL=0.35
 AEACCU=0.80
GO TO 6
1 CONTINUE
IF (.NOT. (Z.EQ.4.0)) GO TO 2
 AEAUCL=0.40
 AEACCU=1.00
GO TO 6
2 CONTINUE
IF (.NOT. (Z.EQ.5.0)) GO TO 3
 AEAUCL=0.45
 AEACCU=1.05
GO TO 6
3 CONTINUE
IF (.NOT. (Z.EQ.6.0)) GO TO 4
 AEAUCL=0.50
 AEACCU=0.80
GO TO 6
4 CONTINUE
IF (.NOT. (Z.EQ.7.0)) GO TO 5
 AEAUCL=0.55
 AEACCU=0.85
GO TO 6
5 CONTINUE

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CCCC

APP18790
 APP18800
 APP18810
 APP18820
 APP18830
 APP18840
 APP18850
 APP18860
 APP18870
 APP18880
 APP18890
 APP18900
 APP18910
 APP18920
 APP18930
 APP18940
 APP18950
 APP18960
 APP18970
 APP18980
 APP18990

```

5  GO TO 6
   CONTINUE
   AEACCL=0.35
   AEACCU=1.05
6  CONTINUE

   DETERMINE CCNSTRAINT VARIABLES FOR EXPANDED AREA RATIO CCNSTRAINT
   AEACCL=AEACCL-AEDVAC
   AEACCU=AEACCU-AEADUP

   DETERMINE CCNSTRAINT VARIABLES FOR BLADE THICKNESS-TU-CHORD RATIO
   CCNSTRAINT

   TC75LQ=(0.5)*(((0.0185-0.00125*Z)*Z)/((2.073*AEADUP)))
   TC75UP=(4.0)*(((0.0185-0.00125*Z)*Z)/((2.073*AEADUP)))
   TC75CL=TC75LQ-TC75R
   TC75CU=TC75F-TC75UP
   RETURN
   END
  
```



```

SUBROUTINE HYDLC(KC,KT)
SUBROUTINE: HYDLD
INPLT      OUTPUT
DIA
N
PDIVD
KQ
KT
WATRO
Z

DATE OF LAST REVISION: APR 83
DEFINITION
PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
PITCH-DIAMETER RATIO
TORQUE COEFFICIENT
THRUST COEFFICIENT
WATER DENSITY (LBF-SEC2/FT4)
NC. CF PROPELLER BLADES
HYDRODYNAMIC MOMENTS PARALLEL
TO BLADE SECTION PITCH REFER-
ENCE (CHCRD) LINE (FT-LBF)
HYDRODYNAMIC MOMENTS NORMAL
TO BLADE SECTION PITCH REFER-
ENCE (CHCRD) LINE (FT-LBF)

COMMON/HYDMMMN/
COMMON/HYDMOML/

REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,
1  RJCNL,RJCNV,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2  FGWBAL,DIACNU,AEAOCV,TCSTRS,
3  VK,TI,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,
4  FWATVA,PRCMAT,DIALIM,ETARK,
5  RJ,CT5R,R75R,KT,KQ
REAL*4  FMPN(10),HMPN(10),XO,PHI(10),PHI2(10),GAM2(10),T,Q,
1  FMT(10),BMP(10),DENOM,KF,PII
INTEGER*4  IR
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1  RJCNV,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2  AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,FWATVA,
1  PRCMAT,DIALIM,ETARK
COMMON /HYMCMN/HMPN
COMMON /HYMCML/HMPL
PII=3.141592654

DETERMINE VALUES ALONG THE PROPELLER RADIUS OF VARIOUS FUNCTIONS
OF XO AND XF DEFINED BY BRACKETED EXPRESSIONS IN RELATIONS (4),
(9) AND (19), REF E
UG 1  IR=2,1C
XO=(FLCAT(IR))/10.0
PHI1(IR)=(1.0/15.0)*
1  (8.0+(4.0*XO)+(3.0*(XC**2))-(15.0*(XO**3)))
2  (SQR(1.0-XO))
PHI2(IR)=(2.0/45.0)*

```

SUBROUTINE HYDLC(KC,KT)

SUBROUTINE: HYDLD

INPLT OUTPUT

DIA
N
PDIVD
KQ
KT
WATRO
Z

COMMON/HYDMMMN/

COMMON/HYDMOML/

```

REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,
1  RJCNL,RJCNV,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2  FGWBAL,DIACNU,AEAOCV,TCSTRS,
3  VK,TI,Z,WATRO,WATNU,TEMP,NOSCRW,HCL,PATM,
4  FWATVA,PRCMAT,DIALIM,ETARK,
5  RJ,CT5R,R75R,KT,KQ
REAL*4  FMPN(10),HMPN(10),XO,PHI(10),PHI2(10),GAM2(10),T,Q,
1  FMT(10),BMP(10),DENOM,KF,PII
INTEGER*4  IR
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1  RJCNV,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2  AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,FWATVA,
1  PRCMAT,DIALIM,ETARK
COMMON /HYMCMN/HMPN
COMMON /HYMCML/HMPL
PII=3.141592654

```

DETERMINE VALUES ALONG THE PROPELLER RADIUS OF VARIOUS FUNCTIONS
OF XO AND XF DEFINED BY BRACKETED EXPRESSIONS IN RELATIONS (4),
(9) AND (19), REF E

```

UG 1  IR=2,1C
XO=(FLCAT(IR))/10.0
PHI1(IR)=(1.0/15.0)*
1  (8.0+(4.0*XO)+(3.0*(XC**2))-(15.0*(XO**3)))
2  (SQR(1.0-XO))
PHI2(IR)=(2.0/45.0)*

```


APP15570
 APP15580
 APP15590
 APP20000
 APP20010
 APP20020
 APP20030
 APP20040
 APP20050
 APP20060
 APP20070
 APP20080
 APP20090
 APP20100
 APP20110
 APP20120
 APP20130
 APP20140
 APP20150

```

6  CCNTINUE
7  RF=1.00
   CCNTINUE
   DENOM=SQRT(((RF*PCIVC)**2)+(PII**2))
   HAPN(IR)=(BMT(IR)*((PII/DENOM)))+(
1    BMT(IR)*((RF*PCIVD)/DENCM))-
1    BMT(IR)*((RF*PCIVD)/DENCM))-
   HAPL(IR)=(BMT(IR)*((PII/DENOM))
      GC TC 5
8  CCNTINUE
   DENOM=SQRT((PCIVC**2)+(PII**2))
   HAPN(IR)=(BMT(IR)*((PII/DENOM)))+(
1    BMT(IR)*((PCIVD/DENCM)))-
1    BMT(IR)*((PCIVD/DENCM)))-
   HAPL(IR)=(BMT(IR)*((PII/DENOM))
      CCNTINUE
9  CCNTINUE
10 RETURN
   ENC

```



```

SUBROUTINE JCNA(RJ,RJCNL,RJGNU)
SUBROUTINE: JCNA
INPLT      OUTPUT
RJ
      RJCNL
      RJGNU
REAL*4 RJ,RJCON,RJCNL,RJGNU
DETERMINE CCNSTRANT VARIABLE FOR ADVANCE CCEFFICIENT CCNSTRANT
RJCCN=RJ/1.6
RJCNL=C.C-RJCON
RJGNU=RJCCN-1.0
RETURN
END

```

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```

DATE OF LAST REVISION: FEB 83
DEFINITION
ADVANCE COEFFICIENT
CCNSTRANT VARIABLE FOR AD-
VANCE COEFFICIENT(LOWER BOUND
(RJCNL<0)
CCNSTRANT VARIABLE FOR AD-
VANCE COEFFICIENT(UPPER BOUND
(RJGNU)

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APP2C180
APP2C190
APP2C200
APP2C210
APP2C220
APP2C230
APP2C240
APP2C250
APP2C260
APP2C270
APP2C280
APP2C290
APP2C300
APP2C310
APP2C320
APP2C330
APP2C340
APP2C350
APP2C360
APP2C370
APP2C380
APP2C390


```

SUBROUTINE CPWEFF(RJ,KT,KQ,ETAO)
SUBROUTINE: OPWEFF
      INPUT      OUTPUT
      RJ
      KT
      KQ
      ETAO
      REAL*4 RJ,K1,KQ,ETAC,PII
      PII=3.141592654
      CALCULATE GFEN WATER EFFICIENCY
      IF (KT.LE.0.C)KT=0.0
      IF (KQ.LE.0.C)KQ=0.0001
      ETAC=(RJ*KT)/(2.0*PII*KQ)
      RETURN
      END

```

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C C C C C C C C C C

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APP20420
APP20430
APP20440
APP20450
APP20460
APP20470
APP20480
APP20490
APP20500
APP20510
APP20520
APP20530
APP20540
APP20550
APP20560
APP20570
APP20580
APP20590
APP20600
APP20610
APP20620

```

```

DATE OF LAST REVISION: FEB 83
DEFINITION
ADVANCE RATIO
THRUST COEFFICIENT
TORQUE COEFFICIENT
OPEN WATER EFFICIENCY

```



```

SUBROUTINE FDCAL(RCIA)
SUBROUTINE: RDCAL
INPUT      OUTPUT
V          VELCCIT, (FT/SEC)
N          PROPELLER REVOLUTION RATE(RPM)
RJ         ADVANCE RATIO
WT         WAKE FRACTION
          PROPELLER DIAMETER (FEET)
          RCIA
          REAL*4  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
1              RJCNU,RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2              FOWBAL,DIACNU,AEACCU,TC75R,RJ,TCSTRS,RJ,TC75CL,TC75CU,POWBAL,DIACNU,
3              VK,TC,WT,Z,WATRC,WATNU,TEMP,NOS,CRW,HCL,PATM,
4              PWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
5              RCIA
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FUIVU,QS,TC75R,V,RJCNL,
1RJCNL,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCU,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRU,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRCMAT,DIALIM,ETARR,AEADMN,TC75MN,SC
          CALCULATE PROPELLER DIAMETER
          RDIA=(V*(1.0-WT))/((N/60.0)*RJ)
          RETURN
          END

```

CCCCCCCCCCCC

CCC

SUBROUTINE FEYCNA(R75R,R75RCL,R75RCU)

SUBROUTINE: REYCNA

DATE OF LAST REVISION: FEB 83

INPUT

OUTPUT

DEFINITION

R75R

R75RCL

R75RCU

REAL*4 F75R,R75RCN,R75RCL,R75RCU

DETERMINE CONSTRAINT VARIABLE FOR CORRECTED REYNOLDS NO. CONSTRAINT

R75RCN=F75R/200000.0

R75RCL=1.0-F75RCN

R75RCU=R75RCN-1000.0

RETURN

END

APP20970
APP20980
APP20990
APP21000
APP21010
APP21020
APP21030
APP21040
APP21050
APP21060
APP21070
APP21080
APP21090
APP21100
APP21110
APP21120
APP21130
APP21140
APP21150
APP21160
APP21170
APP21180
APP21190
APP21200

SUBROUTINE FEY75K(C75R,R75R)

SUBROUTINE: REY75R

DATE OF LAST REVISION: FEB 83

INPUT OUTPUT

C75R

CHORD LENGTH AR 3/4 RADIUS

N

PROPELLER REVOLUTIONS (RPM)

DIA

PROPELLER DIAMETER (FEET)

Z

NO. OF BLADES

TC75R

BLADE SECTION THICKNESS-TO-

AEDVAO

CHORD RATIO AT 3/4 RADIUS

WATNU

EXPANDED AREA RATIO

R 75R

KINEMATIC VISCOSITY

REAL*4

REYNOLDS NO. AT 3/4 RADIUS

1

BLADE THICKNESS EFFECTS

2

BLADE THICKNESS

3

POIIVD, QS, TC75R, V,

4

AEACCU, TC75CL, TC75CU,

5

AEACCU, TC75CL, TC75CU,

COMMON

TC75R, PII, VA, R75RUN, TC75WS, NUM, DENOM, R75R

1

REYNOLDS NO. AT 3/4 RADIUS

2

AEACCU, TC75R, PII, VA, R75RUN, TC75WS, NUM, DENOM, R75R

1

COMMON /PAR4M/VK, TC, WT, Z, WATRO, WATNU, TEMP, NCSCRW, HCL, PATM, PWATVA,

PII=3.141592654

CALCULATE SPEED OF ADVANCE

VA=V*(1.0-C-W1)

CALCULATE UNCORRECTED REYNOLDS NUMBER AT 3/4 PROPELLER RADIUS

USING RELATION (10), REF 2

R75RUN=(1.0/WATNU)*C75R*SQR((VA**2)+((.75*PII*(N/60.0)*DIA)**2))

DETERMINE BLADE THICKNESS-TO-CHORD RATIO BASED ON A SPECIFIC "Z"

"AEDVAC" USING RELATION (11), REF 2

TC75WS=((0.0185-(0.00125*Z))*Z)/(2.073*AEDVAC)

DETERMINE REYNOLDS NUMBER AT 3/4 RADIUS, CORRECTED FOR BLADE

THICKNESS-TC-CHORD RATIO "TC75R", USING RELATION (12), REF 2

APP21230
APP21240
APP21250
APP21260
APP21270
APP21280
APP21290
APP21300
APP21310
APP21320
APP21330
APP21340
APP21350
APP21360
APP21370
APP21380
APP21390
APP21400
APP21410
APP21420
APP21430
APP21440
APP21450
APP21460
APP21470
APP21480
APP21490
APP21500
APP21510
APP21520
APP21530
APP21540
APP21550
APP21560
APP21570
APP21580
APP21590
APP21600
APP21610
APP21620
APP21630
APP21640
APP21650
APP21660
APP21670
APP21680
APP21690
APP21700

APP21710
APP21720
APP21730
APP21740
APP21750

```
NUM=1.0+(2.0*TC75WS)  
DENCM=1.0+(2.0*TC75R)  
R75R=EXP(4.6052+((SQRT(NUM/DENUM))*(ALCG(R75RUN)-4.6052)))  
RETURN  
END
```



```

SUBROUTINE FJCAL(J)
SUBROUTINE: RJCAL
INPLT      CLTPLT      DATE OF LAST REVISION: FEB 83
V          J
DIA
N
WT
REAL*4     ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDIVC,QS,TC75R,V,
1          RJCNU,RJCNL,RJCNL,R75RCU,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
2          POWBAL,DIACNU,AEACCV,TCSTRS,RJ,
3          VK,TC,WT,Z,WATRO,WATNU,TEMP,NGSCRW,HCL,PATM,
4          PWATVA,PRMAT,CIALIM,ETARK,AEADMA,TC75MN,SC,
5          J
COMMON /GLOECM/ETAC,WEIGHT,AEDVAD,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNL,
1RJCNU,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NGSCRW,HCL,PATM,PWATVA,
1PRMAT,CIALIM,ETARR,AEACMN,TC75MN,SC
CALCULATE ADVANCE RATIO
J=(V*(1.0-WT))/(DIA*(N/60.0))
RETURN
END

```

```

APP21780
APP21790
APP21800
APP21810
APP21820
APP21830
APP21840
APP21850
APP21860
APP21870
APP21880
APP21890
APP21900
APP21910
APP21920
APP21930
APP21940
APP21950
APP21960
APP21970
APP21980
APP21990
APP22000
APP22010
APP22020
APP22030
APP22040
APP22050
APP22060
APP22070

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CCC


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SUBROUTINE FNCAL(RN)
SUBROUTINE: RNCAL
DATE OF LAST REVISION: FEB 83
INPUT          OUTPUT          DEFINITION
V              DIA              VELOCITY (FT/SEC)
RJ             DIA              PROPELLER DIAMETER (FEET)
WT            ADVANCE          ADVANCE RATIO
              WAKE             WAKE FRACTION
              PROPELLER        PROPELLER REVOLUTION RATE(RPM)
              RN
1  REAL*4      ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,
2  RJCNU,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
3  POWBAL,DIA,CNU,AEACCU,V,TCSTRS,RJ,TC75CL,TC75CU,POWBAL,DIA,CNU,
4  VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,
5  PWATVA,PROMAT,DIALIM,ETARK,AEACMN,TC75MN,SC,
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVD,QS,TC75R,V,KJCNU,
1RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIA,CNU,
2AEACCU,V,TCSTRS,RJ
COMMON /PAR/M/VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PROMAT,DIALIM,ETARK,AEACMN,TC75MN,SC
CALCULATE PROPELLER REVOLUTION RATE
RN=60.C*((V*(1.0-WT))/((DIA*KJ))
RETURN
END

```

CCCCCCCCCCCC

CCC

SUBROUTINE SIGNES	DATE OF LAST REVISION: APR 83	DEFINITION	APP22420
SUBROUTINE: SIGNES			APP22430
INPUT			APP22440
COMMON /AREELD/ COMMON /VAL11/		BLADE SECTION AREAS (FEET**2)	APP22450
		ORIGINATE OF CRITICAL POINT NO.	APP22460
		1 ON BLADE SECTION PERIPHERY	APP22470
		WITH RESPECT TO A SYSTEM OF	APP22480
		AXES PARALLEL AND NORMAL TO	APP22490
		PITCH-REFERENCE LINE WITH ORI-	APP22500
		GIN AT THE CENTROID OF THE	APP22510
		SECTION (FEET)	APP22520
COMMON /VAL11/		ABSCISSA OF CRITICAL POINT NO.	APP22530
		1 ON BLADE SECTION PERIPHERY	APP22540
		WITH RESPECT TO A SYSTEM OF	APP22550
		AXES PARALLEL AND NORMAL TO	APP22560
		PITCH-REFERENCE LINE WITH ORI-	APP22570
		GIN AT THE CENTROID OF THE	APP22580
		SECTION (FEET)	APP22590
COMMON /VAL12/		ORIGINATE OF CRITICAL POINT NO.	APP22600
		2 ON BLADE SECTION PERIPHERY	APP22610
		WITH RESPECT TO A SYSTEM OF	APP22620
		AXES PARALLEL AND NORMAL TO	APP22630
		PITCH-REFERENCE LINE WITH ORI-	APP22640
		GIN AT THE CENTROID OF THE	APP22650
		SECTION (FEET)	APP22660
COMMON /VAL12/		ABSCISSA OF CRITICAL POINT NO.	APP22670
		2 ON BLADE SECTION PERIPHERY	APP22680
		WITH RESPECT TO A SYSTEM OF	APP22690
		AXES PARALLEL AND NORMAL TO	APP22700
		PITCH-REFERENCE LINE WITH ORI-	APP22710
		GIN AT THE CENTROID OF THE	APP22720
		SECTION (FEET)	APP22730
COMMON /VAL13/		ORIGINATE OF CRITICAL POINT NO.	APP22740
		3 ON BLADE SECTION PERIPHERY	APP22750
		WITH RESPECT TO A SYSTEM OF	APP22760
		AXES PARALLEL AND NORMAL TO	APP22770
		PITCH-REFERENCE LINE WITH ORI-	APP22780
		GIN AT THE CENTROID OF THE	APP22790
		SECTION (FEET)	APP22800
COMMON /VAL13/		ABSCISSA OF CRITICAL POINT NO.	APP22810
		3 ON BLADE SECTION PERIPHERY	APP22820
		WITH RESPECT TO A SYSTEM OF	APP22830
		AXES PARALLEL AND NORMAL TO	APP22840
		PITCH-REFERENCE LINE WITH ORI-	APP22850
		GIN AT THE CENTROID OF THE	APP22860
		SECTION (FEET)	APP22870
COMMON /VAL13/		ABSCISSA OF CRITICAL POINT NO.	APP22880
		3 ON BLADE SECTION PERIPHERY	APP22890
		WITH RESPECT TO A SYSTEM OF	
		AXES PARALLEL AND NORMAL TO	
		PITCH-REFERENCE LINE WITH ORI-	
		GIN AT THE CENTROID OF THE	

CC

SECTION (FEET) APP222500
ORIGINATE OF CRITICAL POINT NO. APP222510
4 ON BLADE SECTION PERIPHERY APP222520
WITH RESPECT TO A SYSTEM OF APP222530
AXES PARALLEL AND NORMAL TO APP222540
PITCH-REFERENCE LINE WITH ORI- APP222550
GIN AT THE CENTROID OF THE APP222560
SECTION (FEET) APP222570
ABSCISSA OF CRITICAL POINT NC. APP222580
4 ON BLADE SECTION PERIPHERY APP222590
WITH RESPECT TO A SYSTEM OF APP222600
AXES PARALLEL AND NORMAL TO APP222610
PITCH-REFERENCE LINE WITH ORI- APP222620
GIN AT THE CENTROID OF THE APP222630
SECTION (FEET) APP222640
MOMENT OF INERTIA ABOUT NEU- APP222650
TRAL AXIS PARALLEL TO GENERA- APP222660
TOR LINE, PASSING THROUGH APP222670
BLADE CROSS SECTION CENTROID APP222680
(FEET**4) APP222690
MOMENT OF INERTIA ABOUT NEU- APP222700
TRAL AXIS PARALLEL TO PITCH APP222710
REFERENCE LINE, PASSING THROUGH APP222720
BLADE CROSS SECTION CENTROID APP222730
HYDRODYNAMIC MOMENTS PARALLEL APP222740
TO BLADE SECTION PITCH REFER- APP222750
ENCE (CFORU) LINE (FT-LBF) APP222760
HYDRODYNAMIC MOMENTS NORMAL APP222770
TO BLADE SECTION PITCH REFER- APP222780
ENCE (CFORD) LINE (FT-LBF) APP222790
CENTRIFUGAL FORCE COMPONENTS, APP222800
ALONG THE PROPELLER RADIUS, APP222810
PARALLEL TO GENERATOR LINE, APP222820
WHICH ACT ON A BLADE SECTION APP222830
AT ITS NEUTRAL AXES ORIGIN APP222840
(LBF) APP222850
CENTRIFUGAL BENDING MOMENT COM- APP222860
PONENTS, ALONG THE PROPELLER APP222870
RADIUS, PARALLEL TO THE PITCH APP222880
REFERENCE (CHORD) LINE, WHICH APP222890
ACT ON A BLADE SECTION AT ITS APP222900
NEUTRAL AXES ORIGIN (FT-LBF) APP222910
CENTRIFUGAL BENDING MOMENT COM- APP222920
PONENTS, ALONG THE PROPELLER APP222930
RADIUS, NORMAL TO THE PITCH APP222940
REFERENCE (CHORD) LINE, WHICH APP222950
ACT ON A BLADE SECTION AT ITS APP222960
NEUTRAL AXES ORIGIN (FT-LBF) APP222970

COMMON /VAL L4/

COMMON /VAL V4/

COMMON /A2MCMX/

COMMON /A2MCMY/

COMMON /HYDQMOMN/

COMMON /HYDQMCL/

COMMON /CFGFD/

COMMON /CMCEN/

COMMON /CMCEL/

CC


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C
C
C
C
1 CONTINUE
CALCULATE TCTAL BENDING MOMENT COMPONENT (FT-LBF), PARALLEL TO
PITCH REFERENCE (CHORD) LINE, IMPOSED BY HYDRODYNAMIC AND
CENTRIFUGAL LOADING
DO 2 IR=2,9
TCTEMN(IR)=HMPN(IR)+CMCBN(IR)
2 CONTINUE
CALCULATE TCTAL BENDING MOMENT COMPONENT (FT-LBF), NORMAL TO
PITCH REFERENCE (CHORD) LINE, IMPOSED BY HYDRODYNAMIC AND
CENTRIFUGAL LOADING
DO 3 IR=2,9
TCTEML(IR)=HMPN(IR)+CMCBL(IR)
3 CONTINUE
CALCULATE STRESSES (LBF/FT**2) AT CRITICAL POINTS 1, 2, 3 AND 4
ALONG THE PROPELLER RADIUS USING RELATION (105), REF 8, WHERE...
SIG1(IR) DIRECT STRESS AT CRITICAL POINT 1 SPECIFIED BY
BLADE SECTION COORDINATES (W1, U1)
SIG2(IR) BLADE STRESS AT CRITICAL POINT 2 SPECIFIED BY
BLADE SECTION COORDINATES (W2, U2)
SIG3(IR) DIRECT STRESS AT CRITICAL POINT 3 SPECIFIED BY
BLADE SECTION COORDINATES (W3, U3)
SIG4(IR) DIRECT STRESS AT CRITICAL POINT 4 SPECIFIED BY
BLADE SECTION COORDINATES (W4, U4)
DO 4 IR=2,9
SIG1(IF)=SIGCF(IR)+((TOTBML(IR)*(-W1(IR)))/RIYVNA(IR))+
((TOTBMN(IR)*(-U1(IR)))/RIXNA(IR))+
1 SIG2(IF)=SIGCF(IR)+((TOTBML(IR)*(-W2(IR)))/RIYVNA(IR))+
((TOTBMN(IR)*(-U2(IR)))/RIXNA(IR))+
1 SIG3(IF)=SIGCF(IR)+((TOTBML(IR)*(-W3(IR)))/RIYVNA(IR))+
((TOTBMN(IR)*(-U3(IR)))/RIXNA(IR))+
1 SIG4(IF)=SIGCF(IR)+((TOTBML(IR)*(-W4(IR)))/RIYVNA(IR))+
((TOTBMN(IR)*(-U4(IR)))/RIXNA(IR))+
1
4 CONTINUE
RETURN
END

```

```

APP23360
APP23370
APP23380
APP23390
APP23500
APP23510
APP23520
APP23530
APP23540
APP23550
APP23560
APP23570
APP23580
APP23590
APP24000
APP24010
APP24020
APP24030
APP24040
APP24050
APP24060
APP24070
APP24080
APP24090
APP24100
APP24110
APP24120
APP24130
APP24140
APP24150
APP24160
APP24170
APP24180
APP24190
APP24200
APP24210
APP24220
APP24230
APP24240
APP24250
APP24260
APP24270

```


SUBROUTINE STRCNA(KC,C75R,SCSTRS)

INPUT OUTPUT

PRCMAT

DEFINITION

PROPELLER MATERIAL IDENTIFIER

1: CAST IRON
2: CAST STEEL
3: TYPE 2 BRONZE
4: TYPE 4 NI-AL
5: STAINLESS STEEL

PROPELLER DIAMETER (FT)
PROPELLER REVOLUTION RATE (RPM)
PITCH-DIAMETER RATIO
TORQUE COEFFICIENT
CHORD LENGTH 3/4 RADIUS (FT)
NO. OF PROPELLERS
BLADE THICKNESS-TU-CHORD RATIO
3/4 RADIUS
CONSTRAINT VARIABLE FOR BLADE
THICKNESS-TU-CHORD RATIO 3/4
RADIUS CONSTRAINT (TCSTRS<0)

DIA

PDIVD

KQ

C75R

NO SCRW

TC75R

SCSTRS

REAL*4

1 RJCNL,RJCNL,R75RCL,R75RCU,DIA,N,PE,PDIVD,QS,TC75R,V,

2 FCWBAL,DIA,CNU,AEACCV,TCSTRS,RJ,CRW,HCL,PATM,

3 VK,TC,WI,Z,WATRU,WATNU,TEMP,NO,SCRM,TC75MN,SCSTRS,

4 PWATVA,PRMAT,DIALIM,ETARR,AEACCV,TC75R,KQ,PII,SC,CABSRB,PABSRB,FACID,FACID,FACIN,175MIN

5 C75R,KQ,PII,SC,CABSRB,PABSRB,FACID,FACID,FACIN,175MIN

COMMON R75RCL,R75RCU,AEACCV,AEACCU,TC75CL,TC75CU,POWBAL,DIA,CNU,

1RJCNL,R75RCL,R75RCU,AEACCV,AEACCU,TC75CL,TC75CU,POWBAL,DIA,CNU,

2AEACCV,TCSTRS,RJ

1PRCMAT,FARIM/VK,TC,WI,Z,WATRU,WATNU,TEMP,NO,SCRM,HCL,PATM,PWATVA,

1PRCMAT,DIALIM,ETARR,AEACCV,TC75MN,SC

PII=3.141592654

DETERMINE MAXIMUM ALLOWABLE STRESS (PSI) BASED ON NUMBER OF PRO-

PELLERS AND MATERIAL IDENTIFIER

IF(.NOT.(NO SCRW.EQ.1.0)) GO TO 1

IF(FRCMAT.EQ.1.0) SC=3600.0

IF(FRCMAT.EQ.2.0) SC=5915.0

IF(FRCMAT.EQ.3.0) SC=7200.0

IF(FRCMAT.EQ.4.0) SC=8910.0

IF(FRCMAT.EQ.5.0) SC=5400.0

GO TO 2

1 CONTINUE

IF(FRCMAT.EQ.1.0) SC=3950.0

IF(FRCMAT.EQ.2.0) SC=6265.0

APP24300
APP24310
APP24320
APP24330
APP24340
APP24350
APP24360
APP24370
APP24380
APP24390
APP24400
APP24410
APP24420
APP24430
APP24440
APP24450
APP24460
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APP24480
APP24490
APP24500
APP24510
APP24520
APP24530
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APP24550
APP24560
APP24570
APP24580
APP24590
APP24600
APP24610
APP24620
APP24630
APP24640
APP24650
APP24660
APP24670
APP24680
APP24690
APP24700
APP24710
APP24720
APP24730
APP24740
APP24750
APP24760
APP24770


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      IF (PROMAT.EQ.3.0) SC=7585.0
      IF (PROMAT.EQ.4.0) SC=9430.0
      IF (PROMAT.EQ.5.0) SC=5500.0
2 CONTINUE
      DETERMINE AESORBED POWER FOR EACH PROPELLER
      QABSRB=ABS((KQ*WATRC*(DIA**5)*((N/60.0)**2))
      PABSRB=(2.0*PI*I*QABSRB*N)/33000.0
      CALCULATE MINIMUM REQUIRED BLADE THICKNESS-TO-CHORD RATIO AT 3/4
      BLADE RADIUS USING RELATION (16), REF 2
      FACID=SC+(((DIA*N)**2)/(12.788))
      FAC2D=4.123*N*(CIA**3)
      FACIN=(2375.0-(1125.0*PUIVDI))*PABSRB
      T75MIN=(((FACIN/(FAC2D*FACID))**0.33333)*0.21)+0.0028)*DIA
      TC75MN=175MIN/C75R
      DETERMINE CCNSTRANT VARIABLE FOR BLADE THICKNESS-TO-CHORD RATIO
      CCNSTRANT
      SCSTRS=(TC75MN/TC75R)-1.0
      RETURN
      END

```

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APP24780
APP24790
APP24800
APP24810
APP24820
APP24830
APP24840
APP24850
APP24860
APP24870
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APP24890
APP24900
APP24910
APP24920
APP24930
APP24940
APP24950
APP24960
APP24970
APP24980
APP24990
APP25000
APP25010
APP25020

SUBROUTINE STRCNK(KQ,KT,C75R,SCSTRS)

SUBROUTINE: STRCNK

DATE OF LAST REVISION: APR 83

INPUT OUTPUT

DEFINITION

PKCMAT

PROPELLER MATERIAL IDENTIFIER

1: CAST IRON

2: CAST STEEL

3: TYPE 2 BRONZE

4: TYPE 4 NI-AL BRONZE

5: STAINLESS STEEL

DIA

PROPELLER DIAMETER (FT)

PDIVD

PROPELLER REVOLUTION RATE (RPM)

AECVAD

PITCH-DIAMETER RATIO

KQ

EXPANDED AREA RATIO

KT

TORQUE COEFFICIENT

C75R

THRUST COEFFICIENT

NOSCRW

CHORD LENGTH 3/4 RADIUS (FT)

TC75R

NO. CF PROPELLERS

ELADE THICKNESS-TO-CHORD RATIO

SCSTRS

3/4 RADIUS

CONSTRAINT VARIABLE FOR BLADE

THICKNESS-TO-CHORD RATIO 3/4

RADIUS CCNSTRANT (SCSTRSCO)

REAL*4

ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDIVD,QS,TC75R,V,

1

RJCNL,RJCNL,R75RCL,R75RCU,AEACCU,TC75CL,TC75CU,

2

FCWBZL,CIACNU,AEACCU,TCSTRS,

3

VK,TC,WT,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,

4

FWATVA,PRCMAT,DIALIM,ETARR,

REAL*4

FJ,C75R,R75R,KT,KQ

1

T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,T90R,

REAL*4

T100F,RAT

1

CR(10),AR(10),BR(10),PLF(10),PLA(10),TR(10),VIF(10,11),

1

V2F(10,11),VIA(10,10),V2A(10,10),YFACE(11),YBACK(11),

2

FA(10),Y(11),DELPA(10),AA(10),DLFASM,XA(10),YA(10),SUMAA,

3

SUMAXA,SUMAYA,SUMAXF,SUMAY2A,HF(11),DELPF(11),AF(11),DLPFM,

4

XF(11),YF(11),SUMAF,SUMAXF,SUMAY2F,SMAY2F,AREA(10),

REAL*4

XMT(10),YPRLL(10),R1XXNA(10),R1YNA(10),XCG(10),YCG(10),

1

U1(10),U2(10),U3(10),U4(10),W1(10),W2(10),W3(10),W4(10),

REAL*4

YBK,YFC

1

FMPN(10),HMPL(10),CMCBN(10),CMCEL(10),BIGNU(10),SIGI(10),

INTEGER*4

SIG2(10),SIG3(10),SIG4(10),TC75MN,TC75WS,SCSTRS,SC

LOGICAL

CKAY1,CKAY2,CKAY3,CKAY4

COMMON

/GLOECM/ETAO,WEIGHT,AEDVAD,DIA,N,PE,PDIVD,QS,TC75R,V,RJCNL,

1

RJCNL,R75RCL,AEACCU,TC75CL,TC75CU,PUMBAL,LIACNU,

APP25050
APP25060
APP25070
APP25080
APP25090
APP25100
APP25110
APP25120
APP25130
APP25140
APP25150
APP25160
APP25170
APP25180
APP25190
APP25200
APP25210
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APP25230
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APP25690
APP25700
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APP25890
APP25900
APP25910
APP25920
APP25930
APP25940
APP25950
APP25960
APP25970
APP25980
APP25990
APP26000

2AEACCV,7CSTFS,RJ
COMMON /PARAM/VK,TD,WI,Z,WATRO,WATNL,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRCMAT,LIALIM,ETARR,AEACMN,IC75MN,SC
COMMON /IHCID/TCOR,IIXR,I20R,I30R,I40R,I50R,I60R,I70R,I75R,I80R,
IT90R,I1CCR,FAT
COMMON /AREELD/AREA
COMMON /CGX/XCG
COMMON /CGY/YCG
COMMON /CRDINT/CR
COMMON /VAL1/U1
COMMON /VAL2/U2
COMMON /VAL3/U3
COMMON /VAL4/U4
COMMON /VAL1/W1
COMMON /VAL2/W2
COMMON /VAL3/W3
COMMON /VAL4/W4
COMMON /A2MCX/RIXYNA
COMMON /A2MCY/RIXXNA
COMMON /HYMCN/HMPN
COMMON /HYCML/HMPL
COMMON /CFGEMN/CMCBN
COMMON /CFGEML/CMCBL
COMMON /CFGFL/BIGNO
COMMON /STRES1/SIG1
COMMON /STRES2/SIG2
COMMON /STRES3/SIG3
COMMON /STRES4/SIG4

DETERMINE MAXIMUM ALLOWABLE STRESS (PSI) BASED UPON THE NUMBER OF PROPELLERS AND TYPE OF MATERIAL

IF(.NOT.(NOSCKW.EQ.1.0)) GO TO 1
IF(PROMAT.EQ.1.0)SC=3600.0
IF(PROMAT.EQ.2.0)SC=5515.0
IF(PROMAT.EQ.3.0)SC=7200.0
IF(PROMAT.EQ.4.0)SC=8510.0
IF(PROMAT.EQ.5.0)SC=5400.0

GO TO 2
1 CONTINUE

IF(PROMAT.EQ.1.0)SC=3950.0
IF(PROMAT.EQ.2.0)SC=6265.0
IF(PROMAT.EQ.3.0)SC=7585.0
IF(PROMAT.EQ.4.0)SC=9430.0
IF(PROMAT.EQ.5.0)SC=5500.0

2 CONTINUE

INITIALIZE MINIMUM REQUIRED BLADE THICKNESS-TC-CHORD RATIO AT 3/4


```

DC 8 IF=2,5
IF(.NOT.(ABS(SIG1(IR)).GE.(SC*144.0)))GO TO 4
  CKA Y1=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG2(IR)).GE.(SC*144.0)))GO TO 5
  CKA Y2=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG3(IR)).GE.(SC*144.0)))GO TO 6
  CKA Y3=.FALSE.
CCNT INUE
IF(.NOT.(ABS(SIG4(IR)).GE.(SC*144.0)))GO TO 7
  CKA Y4=.FALSE.
CCNT INUE
CCNT INLE
IF(.NOT.((OKAY1).AND.(CKAY2).AND.(OKAY3).AND.(OKAY4)))GC TO 3
IF(.NOT.(((CKAY1).AND.(CKAY2).AND.(CKAY3).AND.(CKAY4))).OR.
  (KCUNT.EQ.100)))GC TO 3
1
HAVING DETERMINED MINIMUM REQUIRED ELASE THICKNESS-TO-CHORD
RATIO AT 3/4 PROPELLER RADIUS, CALCULATE CCNSTRANT VARIABLE
SCSTRS=(TC75MN/TC75R)-1.0
RETURN
END

```



```

SUBROUTINE TDIST(C75R,TC75R,DIA,Z,AEDVAC)
SUBROUTINE: TDIST      DATE OF LAST REVISION: MAR 83
INPUT      OUTPUT      DEFINITION
C75R      CHCRD LENGTH AT 3/4 RADIUS
TC75R      (FEET)
DIA      BLADE SECTION THICKNESS-TO-
Z      CHCRD RATIO AT 3/4 RADIUS
AEDVAC    PROPELLER DIAMETER (FEET)
          NO. OF PROPELLER BLADES
          EXPANDED AREA RATIO
          BLADE SECTION MAXIMUM
          THICKNESSES ALONG PROPELLER
          RADIUS (FEET)
REAL*4 T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,T90R,
1 T100R,RAT,C75R,TC75R,DIA,Z,AEDVAC,T75RWS
COMMON /THICD/T00R,T1XR,T20R,T30R,T40R,T50R,T60R,T70R,T75R,T80R,
1 T90R,T100R,RAT
          CCMCN /THICD/
CALCULATE MAXIMUM THICKNESS AT 3/4 RADIUS
T75R=TC75R*(75R
CALCULATE MAXIMUM THICKNESS AT 3/4 RADIUS FOR SERIES PROPELLER
USING RELATION (11), REF 2
T75RWS=DIA*(0.0185-(0.00125*Z))
CALCULATE RATIO OF MAXIMUM THICKNESSES AT 3/4 RADIUS
RAT=T75R/T75RWS
GENERATE A DISTRIBUTION OF BLADE SECTION MAXIMUM THICKNESSES ALONG
THE PROPELLER RADIUS BASED ON THE RATIO CALCULATED ABOVE
IF(.NOT.(Z.EQ.3.0))GO TC 1
TC0R=0.050C*DIA
T1XR=RAT*DIA*(0.05364-(0.0041*Z))
GO TO 5
1 CONTINUE
IF(.NOT.(Z.EQ.4.0))GO TO 2
TC0R=0.045*DIA
T1XR=RAT*DIA*(0.054646-(0.004165*Z))
GO TO 5
2 CONTINUE

```

CCCCCCCCCCCCCCCC

CCCCCCCCCCCCCCCC


```

IF(.NOT.(Z.EQ.5.0))GO TO 3
  TCCR=0.040*DIA
  T1XR=RAT*DIA*(0.054646-(0.004165*Z))
  GO TO 5
3 CONTINUE
IF(.NOT.(Z.EQ.6.0))GO TO 4
  TCCR=0.035*DIA
  T1XR=RAT*DIA*(0.054646-(0.004165*Z))
  GO TO 5
4 CONTINUE
TCCR=0.035*DIA
T1XR=RAT*DIA*(0.05384-(0.0041*Z))
5 CONTINUE
T2OR=RAT*(0.0526-(0.0040*Z))
T3OR=RAT*(0.0464-(0.0035*Z))
T4OR=RAT*(0.0402-(0.0030*Z))
T5OR=RAT*(0.0340-(0.0025*Z))
T6OR=RAT*(0.0278-(0.0020*Z))
T7OR=RAT*(0.0216-(0.0015*Z))
T8OR=RAT*(0.0154-(0.0010*Z))
T9OR=RAT*(0.0092-(0.0005*Z))
T10OR=RAT*DIA*(0.0030-(0.0000*Z))
RETURN
END

```

```

APP27230
APP27240
APP27250
APP27260
APP27270
APP27280
APP27290
APP27300
APP27310
APP27320
APP27330
APP27340
APP27350
APP27360
APP27370
APP27380
APP27390
APP27400
APP27410
APP27420
APP27430
APP27440
APP27450
APP27460

```


SUBROUTINE WGTCL(C75R)

SUBROUTINE: WGTCL

DATE OF LAST REVISION: MAR 83

INPUT OUTPUT

AEDVAO
C75R

DIA
PRCMAT

TC75R

Z

WEIGHT

REAL*4

1

2

3

4

5

REAL*4

1

REAL*4

1

2

3

4

5

REAL*4

1

REAL*4

COMMON

1RJCNV

2AEACCV

COMMON

1PRCMAT

COMMON

1T90R

COMMON

COMMON

COMMON

COMMON

COMMON

COMMON

COMMON

ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,
RJCNV,RJCNV,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,
FCWBAL,DIA,AEACCU,V,TCSTRS,
VK,TL,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,
FWATVA,PRCMAT,DIALIN,ETARR,
RJ,C75R,R75R,KT,KQ
TCOR,T1XR,T2OR,T3OR,T4OR,T5OR,T6CR,T7OR,T75R,T8OR,T9OR,
TICOR,RAT
CR(10),AR(10),BR(10),PLF(10),PLA(10),TR(10),VIF(10,11),
V2F(10,11),VIA(10,10),V2A(10,10),YFACE(11),YBACK(11),
FA(10),Y(11),DELP(10),AA(10),DLFASM,XA(10),YA(10),SUMAA,
SUMAXA,SUMAYA,SUMAX2A,SUMAY2A,HF(11),DELPF(11),AF(11),DLPFSM,
XF(11),YF(11),SUMAF,SUMAXF,SUMAYF,SMAY2F,AREAF(10),
XMT(10),YPR(10),RI,XNA(10),RIYNA(10),XCG(10),YCG(10),
LI(10),U2(10),U3(10),U4(10),W1(10),W2(10),W3(10),W4(10),
YBK,YFC
WC,VCLBLD
COMMON/GLOECM/ETAC,WEIGHT,AEDVAO,DIA,N,PE,PDI,VC,QS,TC75R,V,RJCNV,
1RJCNV,R75RCL,R75RCL,AEACCU,TC75CL,TC75CU,PWBAL,DIA,AEACCU,
2AEACCV,TCSTRS,RJ
COMMON/PAR/VM/VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRCMAT,DIALIN,ETARR,AEADMN,TC75MN,SC
COMMON/THICD/TOUR,T1XR,T2OR,T3OR,T4OR,T5OR,T6OR,T7OR,T75R,T8OR,
1T90R,T1CCR,FAT
COMMON/AREELD/AREA
COMMON/CGX/XCG
COMMON/CGY/YCG

APP27490
APP27500
APP27510
APP27520
APP27530
APP27540
APP27550
APP27560
APP27570
APP27580
APP27590
APP27600
APP27610
APP27620
APP27630
APP27640
APP27650
APP27660
APP27670
APP27680
APP27690
APP27700
APP27710
APP27720
APP27730
APP27740
APP27750
APP27760
APP27770
APP27780
APP27790
APP27800
APP27810
APP27820
APP27830
APP27840
APP27850
APP27860
APP27870
APP27880
APP27890
APP27900
APP27910
APP27920
APP27930
APP27940
APP27950
APP27960

APPENDIX C

ANALIZ CODES--DESIGN CASE NO. 1

```

SUBROUTINE ANALIZ(ICALC)
  INTEGER*4 ICALC
  REAL*4
1  ETAO,WEIGHT,AEDVAO,DIA,N,PE,PDIVC,QS,TC75R,V,
2  RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
3  FQWBAL,DIACNU,AEACCV,TCSTRS,RJ,
4  VK,TC,WI,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,
5  PWATVA,PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
  COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJCNU,
1  RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,PWBAL,DIACNU,
2  AEACCV,TCSTRS,RJ
  COMMON /PARAM/VK,TC,WI,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1  PROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

  THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
  METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

  INPUT-INITIALIZATION PHASE

  PII=3.14159264
  IF(.NOT.(ICALC.EQ.1))GO TO 1

  SET "DESIGN CASE 1" PARAMETERS

  ENVIRONMENTAL

  TEMP=59.0
  WATRC=1.9384
  WATNU=.000012850
  PATM=14.7
  PWATVA=.247

  PROPELLER PARAMETERS

  Z=5.0
  PROMAT=5.0

  HULL PARAMETERS

  WT=C.22
  TC=C.14
  ETARR=1.025
  NCSCRW=1.0
  HCL=19.0
  DIALIM=22.0

  SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 1"

```


APP00C490
 APP00C500
 APP00C510
 APP00C520
 APP00C530
 APP00C540
 APP00C550
 APP00C560
 APP00C570
 APP00C580
 APP00C590
 APP00C600
 APP00C610
 APP00C620
 APP00C630
 APP00C640
 APP00C650
 APP00C660
 APP00C670
 APP00C680
 APP00C690
 APP00C700
 APP00C710
 APP00C720
 APP00C730
 APP00C740
 APP00C750
 APP00C760
 APP00C770
 APP00C780
 APP00C790
 APP00C800
 APP00C810
 APP00C820
 APP00C830
 APP00C840
 APP00C850
 APP00C860
 APP00C870
 APP00C880
 APP00C890
 APP00C900
 APP00C910
 APP00C920
 APP00C930
 APP00C940
 APP00C950
 APP00C960

PE=18153.0
 VK=24.C
 V=1.68E*VK
 AECVAG=0.85
 DIA=22.0
 TC75R=((0.0185-U.C0125*Z)*2)/(2.073*AECVAG)

END OF INPUT-INITIALIZATION PHASE

GO TO 3

EXECUTION PHASE

1 CONTINUE
 IF(.NOT.(ICALC.EQ.2))GO TO 2
 CALL RNCAL(N)
 CALL CF75RA(C75R)
 CALL REV75R(C75R,R75R)
 CALL CCEFSA(RJ,R75R,KT,KQ)
 CALL OFWEFF(RJ,KT,KQ,ETAU)
 CALL CALCCS(KC,QS)
 CALL JCNA(RJ,RJCNL,RJCNH)
 CALL REYCNA(K75R,R75RCL,R75RCU)
 CALL EXTCCN(Z,AECVAG,TC75R,AEACCL,AEACCU,TC75CL,TC75CU)
 CALL BLPOWI(KT,POWBAL)
 CALL DICNUA(DIACNU)
 CALL CAVCNA(KT,AEACCV)
 CALL STRCNA(KC,C75R,TCSTRS)

END OF EXECUTION PHASE

GO TO 3

2 CONTINUE

OUTPUT-RESULT PHASE

PD=(2.C*PII*QS*N)/33000.0
 WRITE(6,90C0)
 WRITE(6,9001)
 WRITE(6,9002) TEMP,WATRO,WAINU,PATN,PWATVA
 WRITE(6,9003) WT,TD,ETAKR,NUSCRH,HCL,DIALIM
 WRITE(6,9004) Z
 IF(.NOT.(PRCMAT.EQ.1.C))GO TO 81
 WRITE(6,9005) SC
 GO TO 86
 CCNTINUE
 IF(.NOT.(PRCMAT.EQ.2.0))GO TO 82
 WRITE(6,9006) SC

81


```

1X,25X,'ALL QWABLE STRESS (PSI)',12X,'=',F10.4,/,/
1X,25X,'MATERIAL TYPE',21X,'=',STAINLESS STEEL,/,/
1X,25X,'ALL QWABLE STRESS (PSI)',12X,'=',F10.4,/,/
1X,25X,'MATERIAL TYPE',21X,'=',NCT,CCNSIDERED,/,/
1X,25X,'SELECTION VALUES',12X,'PE (HP)',27X,'=',F10.4,/,/
1X,25X,'V (FT/SEC)',23X,'=',F10.4,/,/
1X,25X,'N (RPM)',23X,'=',F10.4,/,/
1X,25X,'QS (FT-LBF)',23X,'=',F12.1,/,/
1X,25X,'PD (HP)',23X,'=',F10.2,/,/
1X,25X,'J',23X,'=',F10.4,/,/
1X,25X,'KT',23X,'=',F10.4,/,/
1X,25X,'KG',23X,'=',F10.4,/,/
1X,25X,'ETA C',23X,'=',F10.4,/,/
1X,25X,'REY 75R',23X,'=',F10.4,/,/
1X,25X,'DIA (FT)',23X,'=',F10.4,/,/
1X,25X,'P/D',23X,'=',F10.4,/,/
1X,25X,'AE/AO',23X,'=',F10.4,/,/
1X,25X,'T/C',23X,'=',F10.4,/,/
1X,25X,'CON STRAIN T/C',11X,'MAX DIA (FT)',22X,'=',F10.4,/,/
1X,25X,'MIN AE/AO',25X,'=',F10.4,/,/
1X,25X,'MIN T/C',22X,'=',F10.6)
END

```



```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 ICALC
REAL*4
1  RJCNU,RJCNLU,R75RCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,
2  FGWBAL,DIACNU,AEACCU,V,TCSTRS,RJ,
3  VK,TC,WT,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PATM,
4  PWATVA,PROMAT,DIALIM,ETARR,AEACMN,TC75MN,SC,
5  C75R,R75R,KI,KC,PD
COMMON /GLOECM/ETAG,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,KJCNL,
1RJCNU,R75RCL,R75RCU,AEACCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCU,TCSTRS,RJ
COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PATM,PWATVA,
1PRCMAT,DIALIM,ETARR,AEACMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/CUMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 1" PROPELLER SELECTION PROBLEMS

INPUT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GC TO 1

SET "DESIGN CASE 1" PARAMETERS

ENVIRONMENTAL

TEMP=55.0
WATRO=1.9384
WATNU=.000012285
PATM=14.7
PWATVA=.247

PROPELLER PARAMETERS

Z=5.0
PRCMAT=5.0

HULL PARAMETERS

WT=C.22
TC=C.15
ETARR=1.025
NCSCRW=1.0
HCL=19.0
DIALIM=22.0

SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 1"

```


PE=18153.0
VK=24.0
V=1.68E*VK

END CF INPUT-INITIALIZATION PHASE

GO TO 3

EXECUTION PHASE

```

1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GO TO 2
  CALL RNCAL(N)
  CALL CF75RA(C75R,R75R)
  CALL REY75R(C75R,R75R)
  CALL CCEFFSA(RJ,K75R,KT,KQ)
  CALL OFWEFF(RJ,KT,KQ,ETAO)
  CALL CALCCQS(KC,QS)
  CALL JCNA(RJ,RJCNL,RJCNL)
  CALL REYCNA(R75R,R75RCL,R75RCU)
  CALL EXTCN(Z,AE,VAO,TC75R,AEACCL,AEADCU,TC75CL,TC75CU)
  CALL BLPOWI(KT,POWBAL)
  CALL DIGNUA(DIACNU)
  CALL CFVCNA(KT,AEACCV)
  CALL STRCNA(KC,C75R,TCSTRS)

```

END OF EXECUTION PHASE

GO TO 3

2 CONTINUE

OUTPUT-RESULT PHASE

```

PC=(2.0*(PII*QS*N)/33000.0)
WRITE(6,9000)
WRITE(6,9001)
WRITE(6,9002) TEMP,WATRU,WATNU,PATM,PWATVA
WRITE(6,9003) WT,TD,ETAKR,NUSCRH,HCL,DIALIM
WRITE(6,9004) Z
IF(.NOT.(PROMAT.EG.1.0))GC TO 81
  WRITE(6,9005) SC
GC TO 86
CONTINUE
IF(.NOT.(PROMAT.EG.2.0))GC TO 82
  WRITE(6,9006) SC
GC TO 86
CONTINUE
IF(.NOT.(PROMAT.EG.3.0))GC TO 83

```

81

82

APP03850
APP03860
APP03870
APP03880
APP03890
APP03900
APP03910
APP03920
APP03930
APP03940
APP03950
APP03960
APP03970
APP03980
APP03990
APP04000
APP04010
APP04020
APP04030
APP04040
APP04050
APP04060
APP04070
APP04080
APP04090
APP04100
APP04110
APP04120
APP04130
APP04140
APP04150
APP04160
APP04170
APP04180
APP04190
APP04200
APP04210
APP04220
APP04230
APP04240
APP04250
APP04260
APP04270
APP04280
APP04290
APP04300
APP04310
APP04320

APP04330
APP04340
APP04350
APP04360
APP04370
APP04380
APP04390
APP04400
APP04410
APP04420
APP04430
APP04440
APP04450
APP04460
APP04470
APP04480
APP04490
APP04500
APP04510
APP04520
APP04530
APP04540
APP04550
APP04560
APP04570
APP04580
APP04590
APP04600
APP04610
APP04620
APP04630
APP04640
APP04650
APP04660
APP04670
APP04680
APP04690
APP04700
APP04710
APP04720
APP04730
APP04740
APP04750
APP04760
APP04770
APP04780
APP04790
APP04800

```

      WRITE(6,9007)SC
      GC TO E6
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.4.0))GO TO 84
      WRITE(6,5C08)SC
      GC TO E6
      CCNTINLE
      IF(.NOT.(PRCMAT.EQ.5.0))GO TO 85
      WRITE(6,5C09)SC
      GC TO E6
      CCNTINLE
      WRITE(6,5010)
      CCNTINLE
      WRITE(6,9011)PE,V,N,QS,PD,KJ,KT,KC,ETAC,R75R,DIA,PCDIVD,
      AEDVAC,TC75R
      WRITE(6,9012)DIALIM,AEADOMN,IC75MN
      1
      2 CONTINUE
      RETURN
C
C MISCELLANEOUS FORMAT STATEMENTS
C
9000 FORMAT(1X,OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1,/,
1X,DESIGN VARIABLES SPECIFIED: PE,V,/,
9001 FORMAT(1X,ENVIRONMENTAL PARAMETERS: TEMP (DEG F),22X,=,/,
1X,4,/,
1X,25X,DENSITY (LBF-SEC2/FT4),,12X,=,F10.4,/,
1X,25X,VISCOSITY (FT2/SEC),,15X,=,E16.9,/,
1X,25X,ATMOSPHERIC PRESSION (PSIA),,7X,=,F10.4,/,
1X,25X,WATER VAPORIZATION PRESSURE (PSIA)=,F10.4,/,
9003 FORMAT(1X,PARAMETERS:,13X,WAKE FRACTION,21X,=,F10.4,/,
1X,25X,FULL THRUST DEDUCTION FRACTION,9X,=,F10.1,/,
1X,25X,RELATIVE ROTATIVE EFFICIENCY,6X,=,F10.4,/,
1X,25X,NUMBER OF PROPELLERS,14X,=,F10.1,/,
1X,25X,DEPTH TO SHAFT CENTERLINE (FT),4X,=,F10.4,/,
1X,25X,DIAMETER LIMIT (FT),15X,=,F10.4,/,
9004 FORMAT(1X,PROPELLER PARAMETERS:,8X,NUMBER OF BLADES,18X,=,/,
1X,1),
9005 FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST IRON,/,
1X,25X,ALLCABLE STRESS (PSI),,12X,=,F10.1,/,
9006 FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST STEEL,/,
1X,25X,ALLCABLE STRESS (PSI),,12X,=,F10.1,/,
9007 FORMAT(1X,25X,MATERIAL TYPE,21X,=,BRONZE,/,
1X,25X,ALLCABLE STRESS (PSI),,12X,=,F10.1,/,
9008 FORMAT(1X,25X,MATERIAL TYPE,21X,=,NI-AL BRONZE,/,
1X,25X,ALLCABLE STRESS (PSI),,12X,=,F10.1,/,
9009 FORMAT(1X,25X,MATERIAL TYPE,21X,=,STAINLESS STEEL,/,
1X,25X,ALLCABLE STRESS (PSI),,12X,=,F10.1,/)

```



```

9010 FORMAT(1X,25X,MATERIAL TYPE:,21X,PE,12X,PE,NCT CCNSIDERED,/)
9011 FORMAT(1X,25X,SELECTION VALUES:,12X,PE,10.4,/,
1X,25X,V,23X,PE,10.4,/,
1X,25X,AN,23X,PE,10.4,/,
1X,25X,CS,23X,PE,10.4,/,
1X,25X,PD,23X,PE,10.4,/,
1X,25X,J,23X,PE,10.4,/,
1X,25X,KT,23X,PE,10.4,/,
1X,25X,KQ,23X,PE,10.4,/,
1X,25X,ETA0,23X,PE,10.4,/,
1X,25X,REY75R,23X,PE,10.4,/,
1X,25X,DIA,23X,PE,10.4,/,
1X,25X,P/D,23X,PE,10.4,/,
1X,25X,AE/AC,23X,PE,10.4,/,
1X,25X,T/C,23X,PE,10.4,/,
9012 FORMAT(1X,CONSTRAINT VALUES:,11X,MAX DIA (FT),22X,PE,10.4,/,
1X,25X,MIN AE/AC,25X,PE,10.4,/,
1X,25X,MIN T/C,25R,22X,PE,10.6)
2
END

```

```

APP04810
APP04820
APP04830
APP04840
APP04850
APP04860
APP04870
APP04880
APP04890
APP04900
APP04910
APP04920
APP04930
APP04940
APP04950
APP04960
APP04970
APP04980
APP04990

```


PE=18153.0
VK=24.C
V=1.68E*VK

END OF INPUT-INITIALIZATION PHASE

GO TO 3

EXECUTION PHASE

```

1 CONTINUE
IF(.NOT.(ICALC.EQ.2))GC TO 2
CALL RNCAL(N)
CALL CF75RA(C75R,R75R)
CALL REY75R(C75R,R75R)
CALL CEFSA(RJ,R75R,KT,KQ)
CALL QWEEFF(RJ,KT,KQ,ETA0)
CALL CALCCS(KQ,QS)
CALL JCNA(RJ,RJCNL,KJCNL)
CALL REYCNNA(R75R,R75RCL,R75RCL)
CALL EXTCCN(Z,AEDVA0,TC75R,AEACCL,AEADCU,TC75CL,TC75CL)
CALL BLPOWI(KT,PCWBAI)
CALL DIGNUA(DIACNU)
CALL CAVCNNA(KT,AEACCVI)
CALL STRCNK(KQ,KT,C75R,TCSTRS)

```

END OF EXECUTION PHASE

GO TO 3

2 CONTINUE

OUTPUT-RESULT PHASE

```

PD=(2.C*PII*QS*N)/33000.0
WRITE(6,9000)
WRITE(6,9001)
WRITE(6,9002) TEMP,WATRO,WATNU,PATM,PWATVA
WRITE(6,9003)WT,TD,ETARR,NLSCRW,HCL,DIALIM
IF(.NOT.(PRCMAT.EC.1.0))GC TO 81
WRITE(6,9005)SC
GC TO 86
CCCONTINUE
IF(.NOT.(PRCMAT.EC.2.0))GC TO 82
WRITE(6,9006)SC
GC TO 86
CCCONTINUE
IF(.NOT.(PRCMAT.EC.3.0))GC TO 83

```

81

82

APP05500
APP05510
APP05520
APP05530
APP05540
APP05550
APP05560
APP05570
APP05580
APP05590
APP05600
APP05610
APP05620
APP05630
APP05640
APP05650
APP05660
APP05670
APP05680
APP05690
APP05700
APP05710
APP05720
APP05730
APP05740
APP05750
APP05760
APP05770
APP05780
APP05790
APP05800
APP05810
APP05820
APP05830
APP05840
APP05850
APP05860
APP05870
APP05880
APP05890
APP05900
APP05910
APP05920
APP05930
APP05940
APP05950
APP05960
APP05970

APPENDIX D

CONTROL CARD IMAGES--DESIGN CASE NO. 1

\$A	TITLE	E-SERIES	PROPELLER	CPTIMIZATION	NXAPR X	IPNPUT	IPDBG
\$B	WAGENINGEN	NDV	NSV	N2VAR		0	
\$C	NCALC	2				LINGBJ	NACMX1
\$D1	IPRINT	ITMAX	ICNDR	NSCAL	ITRM		15
\$D2	FDCH	ICOO	CT	CTMIN	CTL		THETA
\$E	NDV	FDCHM	ALPFA	ABGBJI			
\$F	VDOT	CABFUN					
\$G	VDOT	ICBJ	SGNPT				
\$H	VDOT	1	1.0				
\$I1	VDOT	VUB	0.1	SCAL			
\$I2	VDOT	1.8	0.5	1.0			
-1.0	VDOT	1.4	AMULT	1.0			
-1.0	VDOT	IDSGN	1.0				
-1.0	VDOT	23	1.0				
-1.0	VDOT	7					
-1.0	VDOT	JCON	LCCN				
-1.0	VDOT	SCAL1	BU	SCAL2			
-1.0	VDOT	11	0.0	1.0			
-1.0	VDOT	12	0.0	1.0			
-1.0	VDOT	13	0.0	1.0			
-1.0	VDOT	14	0.0	1.0			
-1.0	VDOT	15	0.0	1.0			
-1.0	VDOT	16	0.0	1.0			
-1.0	VDOT	17	0.0	1.0			
-1.0	VDOT	18	0.0	1.0			
-1.0	VDOT	19	0.0	1.0			
-1.0	VDOT	10	0.0	1.0			
\$V	VDOT	16	0.0	1.0			
END	VDOT	16	0.0	1.0			

\$A	TITLE	E-SERIES	PROPELLER	CPTIMIZATION	IPNPUT	IPDUG
\$B	WAGENINGEN	NDV	NSV	NZVAR	LINUBJ	NACMXI
\$C	NCALC	ITPAX	ICNDR	NSCAL	CTLMIN	THETA
\$D1	IPRINT	ICOO	CT	CTMIN		
\$D2	FDCH	FDCHM				
	0.0001	0.0001				
	CELFUN	CABFUN	ALPHAX	ABCBJI		
\$E	NDVTGT	ICBJ	SGNCPT			
\$F	VLB	VUB	1.0 X	SCAL		
	0.2	1.1	0.30	1.0		
	1.0	50.0	30.0	10.0		
	0.01	1.6	0.1	1.0		
	0.4	1.4	0.5	1.0		
	0.003	0.5	0.300	0.010		
\$G	NDSSGN	IDSGN	AMULT			
	1	3	1.0			
	2	4	1.0			
	3	23	1.0			
	4	7	1.0			
	5	9	1.0			
\$H	NCON	JCON	LCCN	SCAL2		
\$I1	ICCN	SCAL1	BU	1.0		
\$I2	EL	11	0.0	1.0		
-1.0	+11	12	0.0	1.0		
-1.0	+12	13	0.0	1.0		
-1.0	+13	14	0.0	1.0		
-1.0	+14	15	0.0	1.0		
-1.0	+15	16	0.0	1.0		
-1.0	+16	17	0.0	1.0		
-1.0	+17	18	0.0	1.0		
-1.0	+18	19	0.0	1.0		
-1.0	+19	20	0.0	1.0		
-1.0	+20	16	0.0	1.0		

-1.0	21	0.0	1.0
-1.0	1.0	0.0	1.0
\$V	22	0.0	1.0
END	1.0		

TITLE
MAGENAGEA B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS
CALCULATION CONTROL
NUMBER OF DESIGN VARIABLES: NCALC = 3000000
NUMBER OF CONSTRAINTS: NSC = 1
NUMBER OF CONSTRAINTS IN TWO-SPACE: NS2V = 1
NUMBER OF APPROXIMATE FUNCTION SPACE: NS2V = 1
NUMBER OF APPROXIMATE FUNCTION SPACE: NS2V = 1
INPUT IDENTIFICATION CODE: IPDBL = 1
DEBUG IDENTIFICATION CODE: IPDBL = 1

CALCULATION CONTROL, NCALC
VALUE
1. ANALYSIS
2. OPTIMIZATION
3. SENSITIVITY
4. TWO-VARIABLE FUNCTION SPACE
5. APPROXIMATE OPTIMIZATION
6

* OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01

CONSTRAINT PARAMETERS (IF ZERO, CONSTRAINT DEFAULT WILL OVER-RIDE)

IPRINT ITHAX ILCNR NSCAL ITHM LINDBJ NACHXI MFDG
1 1000 0 15 0 0
FCHM FCHM 0.1000E+03 0.0 0.0
CTL CTLMIN 0.0 0.0
DEFUN DAFUN 0.0 0.0
ALPHA ALPTAX 0.0 0.0
ABOJJI

DESIGN VARIABLE INFORMATION
NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT
C. V. BOUND LOWER BOUND UPPER BOUND SCALE
1 0.1000E+00 0.1000E+01 0.1000E+00 0.1000E+01
2 0.1000E+00 0.1000E+01 0.1000E+00 0.1000E+01

DESIGN VARIABLES
ID 0. V. GLOBAL MULTIPLYING
1 1 1 1 1
2 2 2 2 2
3 3 3 3 3
4 4 4 4 4
5 5 5 5 5
6 6 6 6 6
7 7 7 7 7
8 8 8 8 8
9 9 9 9 9

CONSTRAINT INFORMATION

THERE ARE 5 CONSTRAINT SETS
ID AFF. GLOBAL LINEAR
1 1 1 1 1
2 2 2 2 2
3 3 3 3 3
4 4 4 4 4
5 5 5 5 5
6 6 6 6 6
7 7 7 7 7
8 8 8 8 8
9 9 9 9 9
TOTAL ALPHEX OF CONSTRAINED PARAMETERS = 9

NORMALIZATION
FAC TOR
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01

UPPER
BOUND
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0

LOWER
BOUND
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01

* ESTIMATED DATA STORAGE REQUIREMENTS


```

51      450      10000      50      50      10000
OPTIMIZATICA RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRONA"

DESIGN VARIABLES SPECIFIED:  PE,VIA,ELVAU,DIA,TC,DR
ENVIRONMENTAL PARAMETERS:
TYPE (DEG F) = 59.0000
DENSITY (LB/FT3) = 0.122897E-04
VISCOSITY (CP) = 1.0000
ATMOSPHERIC PRESSURE (PSIA) = 14.7000
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470
WAKE FRICTION COEFFICIENT = 0.2200
PROPELLER EFFICIENCY = 0.8500
NUMBER OF PROPELLERS = 1.0000
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000
DIAMETER LIMIT (FT) = 22.0000

FULL PARAMETERS:
NUMBER OF BLADES = 5.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 24000.0
PE (HP) = 16193.0
V (FT/SEC) = 40.5120
N (RPM) = 861.8005
QS (FT-LBF) = 29.3374E+0
PD (HP) = 484685.00
J = 6.1000
KO = 6.1735
ETA = 0.143
REYNOLDS = 6.1527
DIA (FT) = 0.5409
P/C = 22.0000
AE/AC = 0.5000
T/C = 0.750
T/C = 0.8500
T/C = 0.8340

CONSTRAINT VALUES:
MAX CIA (FT) = 22.0000
MIN AE/AC = 0.143
MIN T/C = 0.750
  
```



```

*****
* C C P M I N *
* F O R T R A N P R O G R A M F O R *
* C O N S T R A I N E C F U N C T I O N M I N I M I Z A T I O N *
*****

```

INITIAL FUNCTION INFORMATION

CBJ = -C.152694E+00

DECISION VARIABLES (X-VECTOR)

1) C.10000E+00 0.5000E+00

CONSTRAINT VALUES (G-VECTOR)

1) -0.5220E+01 -0.2218E+00 -0.2373E+02 -0.7648E+03 -0.4000E+00 -0.2000E+00


```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C9073E+0C
LEGISLA VARIABLES (X-VECTOR)
11 C.73112E+00 0.10032E+01
CONSTRAINT VALUES (G-VECTOR)
11 -C.46070E+00 -0.53936E+00 -0.32395E+02 -0.96660E+03 -0.40600E+00 -0.20000E+00
11 -C.26691E-01 -0.22788E+00 0.12219E-04
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ)-ABS(OBJ11) LESS THAN DLEFUN FOR 3 ITERATIONS
ABS(OBJ11)-OBJ11111 LESS THAN DABFUN FOR 3 ITERATIONS
NUMBER OF ITERATIONS = 14
OBJECTIVE FUNCTION WAS EVALUATED 69 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 69 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.70507E+00

DESIGN VARIABLES

ID	D. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.1000E+01	0.7371E+00	0.1000E+01
2	2	2	0.4000E+00	0.1003E+01	0.1400E+01

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	-0.1000E+16	-0.4075E+00	0.0
2	1	-0.1000E+16	-0.4233E+02	0.0
3	1	-0.1000E+16	-0.4233E+02	0.0
4	1	-0.1000E+16	-0.4233E+02	0.0
5	1	-0.1000E+16	-0.4233E+02	0.0
6	1	-0.1000E+16	-0.4233E+02	0.0
7	1	-0.1000E+16	-0.4233E+02	0.0
8	1	-0.1000E+16	-0.4233E+02	0.0
9	1	-0.1000E+16	-0.4233E+02	0.0

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRONA"

TESTION VARIABLES SPECIFIED:	PE,VA,DELTA,UA,IC,75K
TEMP (DEG F)	5.0000
DENSITY (LBF-SECZ/FT ³)	1.5384
VISCOSITY (LBFZ/SEC)	1.289997E-04
ATMOSPHERIC PRESSURE (PSIA)	14.7000
WATER VAPORIZATION PRESSURE (PSIA)	0.2470
WAKE FRACTION	0.2000
THRUST COEFFICIENT	0.9000
RELATIVE ROTATIONAL EFFICIENCY	1.0250
NUMBER OF PROPELLERS	1.00
DEFT OF SHAFT CENTERLINE (FT)	15.0000
DIAPYTED CLIP (IN)	22.0000
NUMBER OF BLADES	5
MATERIAL TYPE	STAINLESS
ALLOWABLE STRESS (PSI)	5400.0
SELECTION VALUES:	
VE (FT/SEC)	18.1720
N (RPM)	112.1342
QS (FT-LBF)	116.0000
PD (HP)	24251.21
JT	0.3771
KT	0.0000
KV	0.0000
WAC	0.0000
WDA	0.0000
DIA (FT)	2.0000
P/C	1.0036
AC/AC	1.0000
T/C .75F	0.0340
MAX LIA (FT)	22.0000
MIN A/Z	0.5250
MIN T/C .75F	0.021260
CONSTRAINT VALUES:	

. PROGRAM CALLS TO ANALYZ

ICALL	CALLS
1	1
2	72
3	2

—

TABLE WAGENINGEN D-SERIES PROPELLER OPTIMIZATION

CARD IMAGES OF CONTROL DATA

CARD		IMAGE									
1	2	3	4	5	6	7	8	9	10	11	12
1	1	1	1	1	1	1	1	1	1	1	1
2	2	2	2	2	2	2	2	2	2	2	2
3	3	3	3	3	3	3	3	3	3	3	3
4	4	4	4	4	4	4	4	4	4	4	4
5	5	5	5	5	5	5	5	5	5	5	5
6	6	6	6	6	6	6	6	6	6	6	6
7	7	7	7	7	7	7	7	7	7	7	7
8	8	8	8	8	8	8	8	8	8	8	8
9	9	9	9	9	9	9	9	9	9	9	9
10	10	10	10	10	10	10	10	10	10	10	10
11	11	11	11	11	11	11	11	11	11	11	11
12	12	12	12	12	12	12	12	12	12	12	12
13	13	13	13	13	13	13	13	13	13	13	13
14	14	14	14	14	14	14	14	14	14	14	14
15	15	15	15	15	15	15	15	15	15	15	15
16	16	16	16	16	16	16	16	16	16	16	16
17	17	17	17	17	17	17	17	17	17	17	17
18	18	18	18	18	18	18	18	18	18	18	18
19	19	19	19	19	19	19	19	19	19	19	19
20	20	20	20	20	20	20	20	20	20	20	20
21	21	21	21	21	21	21	21	21	21	21	21
22	22	22	22	22	22	22	22	22	22	22	22
23	23	23	23	23	23	23	23	23	23	23	23
24	24	24	24	24	24	24	24	24	24	24	24
25	25	25	25	25	25	25	25	25	25	25	25
26	26	26	26	26	26	26	26	26	26	26	26
27	27	27	27	27	27	27	27	27	27	27	27
28	28	28	28	28	28	28	28	28	28	28	28
29	29	29	29	29	29	29	29	29	29	29	29
30	30	30	30	30	30	30	30	30	30	30	30
31	31	31	31	31	31	31	31	31	31	31	31
32	32	32	32	32	32	32	32	32	32	32	32
33	33	33	33	33	33	33	33	33	33	33	33
34	34	34	34	34	34	34	34	34	34	34	34
35	35	35	35	35	35	35	35	35	35	35	35
36	36	36	36	36	36	36	36	36	36	36	36
37	37	37	37	37	37	37	37	37	37	37	37
38	38	38	38	38	38	38	38	38	38	38	38
39	39	39	39	39	39	39	39	39	39	39	39
40	40	40	40	40	40	40	40	40	40	40	40
41	41	41	41	41	41	41	41	41	41	41	41
42	42	42	42	42	42	42	42	42	42	42	42

TITLE
MAGNETRON B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:
CALCULATED CONTROL NO. 3
NUMBER OF DESIGN VARIABLES 3
NUMBER OF CONSTRAINTS IN TWO SPACE 0
NUMBER OF CONSTRAINTS IN ONE SPACE 0
NUMBER OF CONSTRAINTS IN TWO SPACE 0
NUMBER OF CONSTRAINTS IN ONE SPACE 0
INPUT DATA PRINT CODE 0
LEBUC PRINT CODE 0

CALCULATION CONTROL, NCALC
VALUE 0
CALCULATING ANALYSIS
1
2
3
4
5
6

• • OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01
CONSTRAINT PARAMETERS (IF ZERO, CONSTRAINT DEFAULT WILL OVER-RIDE)
IPRINT 1000 LINEAR NSCALE 10 LINEAR MAGNIFY 10
FOCUS 0.1000E+03 FOCUS 0.0 CTMIN 0.0
CTL 0.0 CTMIN 0.0 META 0.0
DELEUN 0.0 DAFEM 0.0 ALPHA 0.0
NON-ZERO INITIAL VALUE WILL OVER-RIDE MULTIPLE INPUT
D.V. 1 0.1000E+01 0.1000E+01 0.1000E+01 0.1000E+01
2 0.1000E+01 0.1000E+01 0.1000E+01 0.1000E+01

DESIGN VARIABLES GLOBAL MULTIPLYING
ID 0.0 VAR NO. 23
1 1 0.1000E+01

CONSTRAINT INFORMATION

THERE ARE 5 CONSTRAINT SETS
ID 1 2 3 4 5
VAR 1 2 3 4 5
GLOBAL 1 2 3 4 5
LINEAR 1 2 3 4 5
LOWER 1 2 3 4 5
BOUND 1 2 3 4 5
UPPER 1 2 3 4 5
BOUND 1 2 3 4 5
NORMALIZATION 1 2 3 4 5
FAC 1 2 3 4 5
TOR 1 2 3 4 5
SCALE 1 2 3 4 5

TOTAL NUMBER OF CONSTRAINED PARAMETERS = 5

• • ESTIMATED DATA STORAGE REQUIREMENTS


```

57      420      10000      30      94      1000
OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:  PE,V,REC,VAC,DIA,TCT5R
ENVIRONMENTAL PARAMETERS:
TEMP (DEG F) 5002.1741      = 50.0000
DENSITY (LB/FT**3) 0.0000      = 0.0000
WATER VAPOR PRESSURE (PSIA) 0.0000      = 0.0000
WATER VAPORIZATION PRESSURE (PSIA) 0.0000      = 0.0000
FULL PARAMETERS:
WAKE FRACTION      = 0.2200
THRUST REDUCTION FRACTION      = 0.1900
RELATIVE ROCKET EFFICIENCY      = 1.0000
DEVELOPMENTAL FRACTION      = 0.0000
DIAGNOSTIC SHOT CENTERLINE (FT)      = 15.0000
DIAGNOSTIC LIFT (FT)      = 2.0000
PROPELLER PARAMETERS:
NUMBER OF BLADES      = 5.0
MATERIAL TYPE      = STAINLESS STEEL
ALLOWABLE STRESS (PSI)      = 24000.0
SELECTION VALUES:
PE (PSI)      = 18153.0
V (FT/SEC)      = 40.5120
N (RPM)      = 861.8003
WS (FT-LBF)      = 2933344.0
PD (HP)      = 4846085.00
KT      = 0.1000
KQ      = 0.125
KTAC      = 0.1927
REV75R      = 0.5E+09
DIA (FT)      = 2.0000
P/C      = 0.5000
AE/AC      = 0.8500
T/C .75R      = 0.0348
CONSTRAINT VALUES:
MAX LIA (FT)      = 24.0000
MIN AE/AC      = 24.7882
MIN T/C .75R      = 0.134760

```



```

*****
*               *
*   C C P M I N   *
*               *
*   F O R T R A N   P R O G R A M   F O R   *
*               *
*   C O N S T R A I N E D   F U N C T I O N   M I N I M I Z A T I O N   *
*               *
*****

```

INITIAL FUNCTION INFORMATION

```

GBJ = -C.152694E+0C
DECISION VARIABLES (X-VECTOR)
1)  C.16000E+00  C.50000E+00
CONSTRAINT VALUES (G-VECTOR)
1)  -C.92500E-01 -C.93750E+00 -C.23915E+03 -C.76485E+03 -C.40600E+00 -C.20000E+00
2)  -C.2691E-01 -C.22788E+00 -C.13755E+02

```



```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C9073E+00
DESIGN VARIABLES (X-VECTOR)
11 C.73712E+00 0.10036E+01
CONSTRAINT VALUES (G-VECTOR)
11 -C.44070E+00 -0.53937E+00 -0.32355E+02 -0.96660E+03 -C.40C00E+00 -0.20000E+00
11 -C.21691E-01 -0.22788E+00 0.12219E-04
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(1-CBJ1)-1/OBJ(1) LESS THAN DELFUN FOR 3 ITERATIONS
ABS(CBJ1-OBJ(1-1)) LESS THAN DABFUN FOR 3 ITERATIONS
NUMBER OF ITERATIONS = 14
OBJECTIVE FUNCTION WAS EVALUATED 69 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 69 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 6.70907E+00
GLOBAL LOCATION

DESIGN VARIABLES

ID	GLOBAL VAR. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.0000E+00	0.73712E+00	0.1E+00E+01
2	2	2	0.4000E+00	0.10036E+01	0.14000E+01

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.0000E+00	0.0000E+00	0.0
2	2	2	0.0000E+00	0.0000E+00	0.0
3	3	3	0.0000E+00	0.0000E+00	0.0
4	4	4	0.0000E+00	0.0000E+00	0.0
5	5	5	0.0000E+00	0.0000E+00	0.0
6	6	6	0.0000E+00	0.0000E+00	0.0
7	7	7	0.0000E+00	0.0000E+00	0.0
8	8	8	0.0000E+00	0.0000E+00	0.0
9	9	9	0.0000E+00	0.0000E+00	0.0


```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
                               SUBROUTINE "STRONK"

DESIGN VARIABLES SPECIFIED:  PE,V,REC,VAC,DIA,TCTSR
                               = 5.0000
                               = 1.9384
                               = 0.12849997E-04
                               = 1.0000
                               = 1.2477

ENVIRONMENTAL PARAMETERS:
TEMP (DEG F)                  =
DENSITY (LBF-SEC2/FT4)       =
VISCOSITY (LBF2/SEC)         =
ATMOSPHERIC PRESSURE (PSIA)  =
WATER VAPORIZATION PRESSURE (PSIA) =

FULL PARAMETERS:
WAKE FRACTION                 =
THRUST COEFFICIENT FRACTION  =
RELATIVE ROTATIVE EFFICIENCY =
NUMBER OF PROPELLERS         =
DEPTHTO SHATT CENTERLINE (FT) =
DIAPETER LPII (FT)          =

PROPELLER PARAMETERS:
NUMBER OF BLADES              =
MATERIAL TYPE                 =
ALLOWABLE STRESS (PSI)       =

SELECTION VALUES:
PE (FT) SEC                  =
V (FT) SEC                   =
N (RPM)                      =
QD (FT-LBF)                  =
PD (HP)                       =
J                             =
KI                             =
KVAC                          =
FEFTSR                        =
DIA (FT)                     =
P/D                           =
AE/AC                         =
T/C .75F                     =

CONSTRAINT VALUES:
MAX CIA (FT)                 =
MIN AE/AD                    =
MIN T/C .75R                 =

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PROGRAM CALLS TO ANALYZE
LOCAL CALLS
1 2 3
1 2


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CCCCC  CCCCCC  CCCCCC  P P P P P P  EEEEE  SSSSSS
C C C C C  C C C C C  C C C C C  P P P P P  EEEEE  SSSSSS
C C C C C  C C C C C  C C C C C  P P P P P  EEEEE  SSSSSS
C C C C C  C C C C C  C C C C C  P P P P P  EEEEE  SSSSSS

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C C T R O L P R O G R A M
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 T I T L E
 W A G E N I N G E N B - S E R I E S P R O P E L L E R O P T I M I Z A T I O N

TITLE:
HAGENIERA B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:
CALCULATION CONTROL: NCALC = 3
NUMBER OF GLOBAL DESIGN VARIABLES, NDV = 0
NUMBER OF SENSITIVITY VARIABLES, NSV = 0
NUMBER OF FUNCTIONS IN TWO-SPACE, NZVAR = 0
NUMBER OF APPROXIMATING VARS, NXAPRX = 0
INPUT INCORPORATION PRINT CODE, IPAPRI = 0
DEBUG PRINT CODE, IPDBG = 0

CALCULATION CONTROL, NCALC
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* * OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01
CONFIN PARAMETERS (IF ZERO, CONFIN DEFAULT WILL OVER-RIDE)

IPRINT ITMAX IONLIN NSCAL ITRM LINGBJ NACMAX NFDG
1 1 0 0 15 0
FOLHM 0.1000E-03 CT 0.0 CTMIN 0.0
CTL 0.0 CTLMIN 0.0 PHI 0.0
DELFIN 0.0 DABFIN 0.0 ALPHAX 0.0
ALPHAJ 0.0

DESIGN VARIABLE INFORMATION
NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT
NO. 1 2 3 4 5
C.2000E+00 0.1000E+01 0.3000E+02 0.3000E+00 0.1000E+01
C.1000E+01 0.3000E+02 0.3000E+00 0.3000E+00 0.1000E+01
C.0000E+00 0.3000E+00 0.3000E+00 0.3000E+00 0.1000E+01
C.3000E-02 0.3000E+00 0.3000E-01 0.3000E-01 0.1000E-01

DESIGN VARIABLES
O.V. GLOBAL
NO. 1 2 3 4 5
C.2000E+00 0.1000E+01 0.3000E+02 0.3000E+00 0.1000E+01
C.1000E+01 0.3000E+02 0.3000E+00 0.3000E+00 0.1000E+01
C.0000E+00 0.3000E+00 0.3000E+00 0.3000E+00 0.1000E+01
C.3000E-02 0.3000E+00 0.3000E-01 0.3000E-01 0.1000E-01

CONSTRAINT INFORMATION

THERE ARE 12 CONSTRAINT SETS
GLOBAL GLOBAL LINEAR
ID VAR 1 2 3 4 5
1 1 1 1 1 1 1 1 1 1 1 1
2 1 1 1 1 1 1 1 1 1 1 1
3 1 1 1 1 1 1 1 1 1 1 1
4 1 1 1 1 1 1 1 1 1 1 1
5 1 1 1 1 1 1 1 1 1 1 1
6 1 1 1 1 1 1 1 1 1 1 1
7 1 1 1 1 1 1 1 1 1 1 1
8 1 1 1 1 1 1 1 1 1 1 1
9 1 1 1 1 1 1 1 1 1 1 1
10 1 1 1 1 1 1 1 1 1 1 1
11 1 1 1 1 1 1 1 1 1 1 1
12 1 1 1 1 1 1 1 1 1 1 1

NORMALIZATION
FACTOR
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01

UPPER
BOUND
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0

NORMALIZATION
FACTOR
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01

LOWER
BOUND
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01
-0.1000E+01

TOTAL NUMBER OF CONSTRAINED PARAMETERS = 12

* * ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	REAL	EXECUTION	AVAILABLE	INTEGER	EXECUTION	AVAILABLE
87	270		10000	37	124	1000

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRUNA"

DESIGN VARIABLES SPECIFIED: PE,V

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F)	=	55.0000
DENSITY (LBF-SEC ² /FT ⁴)	=	0.1284997E-04
VISCOSITY (FT ² /SEC)	=	1.87000
ATMOSPHERIC PRESSURE (PSIA)	=	14.7000
WATER VAPORIZATION PRESSURE (PSIA)	=	0.2470

FULL PARAMETERS:

WAKE FRACTION	=	0.2200
THRUST COEFFICIENT	=	1.0000
RELATIVE ROTATIVE EFFICIENCY	=	1.0250
NUMBER OF PROPELLERS	=	1.0
DEPTH TO SHAFT CENTERLINE (FT)	=	15.0000
DIAPETER LIMIT (FT)	=	22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES	=	5.0
MATERIAL TYPE	=	STAINLESS STEEL
ALLOWABLE STRESS (PSI)	=	54000.0

SELECTION VALUES:

PE (FT/SEC)	=	14152.0
V (RPM)	=	40.5120
NS (RPM)	=	631.9898
PD (MP)	=	0.31622.0
J	=	0.0000
KT	=	0.1879
KQ	=	0.0160
ETAL	=	0.1865
REYSR	=	0.7E+09
PIA (FT)	=	30.0000
PEC	=	0.0000
AECAC	=	0.3000
TTC.75R	=	0.0300

CONSTRAINT VALUES:

MAX CIA (FT)	=	26.0000
MIN AECAC	=	31.1615
MIN TTC.75R	=	0.157532


```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.710857E+0C
DECISION VARIABLES (X-VECTOR)
1) C.62045E+00 0.21995E+02 0.73433E+00 0.99813E+00 0.33004E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.47895E+00 -0.54105E+00 -0.38016E+02 -0.96098E+04 -0.37049E+00 -0.24951E+00
   -0.16935E-01 -0.22663E+00 -0.16354E-02 -0.31233E-04 -C.35786E+00 -0.33350E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5 10
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ(1)-OBJ(1-1)) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 83
OBJECTIVE FUNCTION WAS EVALUATED 548 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 548 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION GLOBAL LOCATION 1 FUNCTION VALUE 0.71086E+00

DESIGN VARIABLES

ID	D. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.2000E+00	0.91093E+02	0.1000E+01
2	2	2	0.0000E+00	0.57343E+00	0.1000E+02
3	3	3	0.4000E+00	0.99813E+00	0.1000E+01
4	4	4	0.3000E-02	0.33004E-01	0.5000E+00

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1.000E+00	0.91093E+02	0.0
2	2	0.000E+00	0.57343E+02	0.0
3	3	0.000E+00	0.57343E+02	0.0
4	4	0.000E+00	0.57343E+02	0.0
5	5	0.000E+00	0.57343E+02	0.0
6	6	0.000E+00	0.57343E+02	0.0
7	7	0.000E+00	0.57343E+02	0.0
8	8	0.000E+00	0.57343E+02	0.0
9	9	0.000E+00	0.57343E+02	0.0
10	10	0.000E+00	0.57343E+02	0.0
11	11	0.000E+00	0.57343E+02	0.0
12	12	0.000E+00	0.57343E+02	0.0

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRONA"

DESIGN VARIABLES SPECIFIED: PE,V

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F)	=	55.0000
DENSITY (LBF-SEC2/FT4)	=	1.9384
VISCOSITY (FT2/SEC)	=	0.1494997E-04
ATMOSPHERIC PRESSURE (PSIA)	=	14.7000
WATER VAPORIZATION PRESSURE (PSIA)	=	0.2470

HULL PARAMETERS:

WAKE FRACTION	=	0.2200
THRUST REDUCTION FRACTION	=	0.1900
RELATIVE ROTATIVE EFFICIENCY	=	1.0250
NUMBER OF PROPELLERS	=	1.0
DEPTH (FT)	=	15.0000
DIAMETER (FT)	=	22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES	=	5.0
MATERIAL TYPE	=	STAINLESS STEEL
ALLOWABLE STRESS (PSI)	=	2400.0

SELECTION VALUES:

PE (HP)	=	1E+3.0
V (FPM)	=	1.0E+2.0
QS (FT-LBF)	=	1.0E+3.0
PD (HP)	=	2.4E+3.0
J	=	0.7343
K1	=	0.1713
K2	=	0.0182
REVSR	=	0.86108
DIA (FT)	=	21.9981
P/C	=	0.9981
AE/AQ	=	0.8205
T/C	=	0.0330

CONSTRAINT VALUES:

MAX CIA (FT)	=	22.0000
MIN AE/AQ	=	0.9289
MIN T/C	=	0.0198

PROGRAM CALLS TO ANALYZ
ICALL CALLS
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2 2
3 2


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CCCCC  DCCCC  P P P P P  E E E E E  S S S S S
C C C C C  C C C C C  P P P P P  E E E E E  S S S S S
C C C C C  C C C C C  P P P P P  E E E E E  S S S S S
C C C C C  C C C C C  P P P P P  E E E E E  S S S S S

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 E N G I N E E R I N G S Y N T H E S I S

T I T L E
 W A G E N I N G E N B - S E R I E S P R O P E L L E R O P T I M I Z A T I O N

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IMAGE

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TOTAL ALPHEP OF UNCONSTRAINED PARAMETERS = 14

* * ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
87	EXECUTION	10000	57	EXECUTION	1000
	570			124	

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRUNK"

DESIGN VARIABLES SPECIFIED: PE,V

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F)	5627.4	=	55.0000
DENSITY (LBF/FT ³)		=	0.129384
VISCOSITY (CENTIPOISE)		=	1.29384E-04
AIR CRYSTALLINE PRESSURE (PSIA)		=	15.0000
WATER VAPORIZATION PRESSURE (PSIA)		=	2.2470

FULL PARAMETERS:

WAKE FRACTION		=	0.2200
THRUST REDUCTION FRACTION		=	0.1500
RELATIVE ROTOR EFFICIENCY		=	1.0500
RELATIVE STATOR EFFICIENCY		=	1.0500
DEPTH (FT)		=	15.0000
DIAPHRAGM CENTERLINE (FT)		=	22.0000
DIAPHRAGM LIMIT (FT)		=	22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES		=	5.0
MATERIAL TYPE		=	STAINLESS STEEL
ALLOWABLE STRESS (PSI)		=	2400.0

SELECTION VALUES:

PE (HP)		=	18123.0
V (FT/SEC)		=	40.5120
N (RPM)		=	631.9868
QS (FT-LBF)		=	8312672.0
PD (HP)		=	*****
JT		=	0.1000
KT		=	0.0700
KQ		=	0.0600
REYN		=	0.1865
REV75R		=	0.7E+09
DIA (FT)		=	30.0000
P/C		=	0.5000
AE/AC		=	0.3000
T/C .75F		=	0.0300

CONSTRAINT VALUES:

MAX CIA (FT)		=	24.0000
MIN AE/AD		=	31.1615
MIN T/C .75R		=	0.198487


```

*****
C C P M I N
*****
FORTRAN PROGRAM FOR
*****
CONSTRAINED FUNCTION MINIMIZATION
*****

```

INITIAL FUNCTION INFORMATION

OBJ = -C.16489E+00

DECISION VARIABLES (X-VECTOR)

1) C.30000E+00 0.30000E+02 0.10000E+00 0.50000E+00 C.30000E-01

CONSTRAINT VALUES (G-VECTOR)

1) -C.62500E-01 -0.93750E+00 -0.33074E+03 -0.96826E+03 C.15000E+00 -0.75000E+00
2) -C.15930E-01 -0.23264E+00 -0.10917E+03 0.36368E+00 C.10287E+03 0.56102E+01


```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.7C657E+00
DECISION VARIABLES (X-VECTOR)
1) C.8148E+00 0.2196E+02 0.7393E+00 C.10071E+01 C.64152E-01
CONSTRAINT VALUES (G-VECTOR)
1) -C.42219E+00 -0.1275E+00 -0.1538E+02 -0.87161E+03 -0.3668E+00 -0.2301E+00
1) -C.5082E-01 -0.1584E+00 0.1733E-02 -0.1530E-02 -0.3538E+00 -0.1880E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
10
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(1-CBJ(1)-1)/OBJ(1) LESS THAN DLFUN FOR 30 ITERATIONS
AEC1CBJ(1)-OBJ(1-1) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 73
OBJECTIVE FUNCTION WAS EVALUATED 491 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 491 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.70258E+00
GLOBAL LOCATION

DESIGN VARIABLES

ID	D. V. NC.	GLOBAL VAR. NC.	LOWER BOUND	VALUE	UPPER BOUND
1	1	3	0.2000E+00	0.31488E+00	0.1000E+01
2	2	4	0.1000E+01	0.21595E+02	0.1000E+02
3	3	5	0.0000E+00	0.73538E+00	0.1000E+00
4	4	6	0.3000E+02	0.1000E+01	0.1000E+01
5	5	7	0.3000E+02	0.84132E-01	0.3000E+00

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NC.	LOWER BOUND	VALUE	UPPER BOUND
1	1	-0.1000E+16	-0.52790E+00	0.0
2	2	-0.1000E+16	-0.53388E+02	0.0
3	3	-0.1000E+16	-0.53368E+00	0.0
4	4	-0.1000E+16	-0.53518E+00	0.0
5	5	-0.1000E+16	-0.53518E+00	0.0
6	6	-0.1000E+16	-0.53518E+00	0.0
7	7	-0.1000E+16	-0.53518E+00	0.0
8	8	-0.1000E+16	-0.53518E+00	0.0
9	9	-0.1000E+16	-0.53518E+00	0.0
10	10	-0.1000E+16	-0.53518E+00	0.0
11	11	-0.1000E+16	-0.53518E+00	0.0
12	12	-0.1000E+16	-0.53518E+00	0.0

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 1
SUBROUTINE "STRONK"

DESIGN VARIABLES SPECIFIED: PE,V

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F) 55.0000
DENSITY (LBF/FT³) 1.9286
WATER VAPOR PRESSURE (PSIA) 0.122997E-04
WATER VAPORIZATION PRESSURE (PSIA) 1.47043
WATER VAPORIZATION PRESSURE (PSIA) 1.47043

FULL PARAMETERS:

WAKE FRACTION 0.2200
THRUST REDUCTION FRACTION 0.1900
RELATIVE ROTATIONAL VELOCITY 1.0250
NUMBER OF PROPELLERS 0
DIPPER SHOT LENGTH (FT) 15.0000
DIAPER LENGTH (FT) 22.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES 5.0
MATERIAL TYPE 5
ALLOWABLE STRESS (PSI) 3400.0
STAINLESS STEEL

SELECTION VALUES:

PE (HP) 18153.0
V (FT/SEC) 40.5120
N (RPM) 116.7419
QS (FT-LBF) 1684003.0
PD (HP) 24094.79
JT 0.7396
KT 0.1736
KTAL 0.0289
REV7 0.7066
REV7 0.5608
DIA (FT) 21.5659
P/D 1.0071
AE/AC 0.8149
T/C .756 0.0642

CONSTRAINT VALUES:

MAX CIA (FT) 22.0000
MIN AE/AC 0.5267
MIN T/C .756 0.03259

PROGRAM CALLS TO ANALIZ
ICALC CALLS
1 452
2
3

APPENDIX F

ANALIZ CODES--DESIGN CASE NO. 2

```

SUBROUTINE ANALIZ(ICALC)
  INTEGER*4 ICALC
  REAL*4
1  ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVL,GS,TC75R,V,
2  RJC�L,RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
3  FCWBAL,DIA,CNU,AEAOCCV,TCSTRS,RJ,
4  VKITL,WT,Z,WATRC,WATNU,TEMP,NOSCRL,HCL,PATM,
5  PWATVA,PRMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
  COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVL,GS,TC75R,V,RJC�L,
1  RJCNU,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIA,CNU,
2  AEACCV,TCSTRS,RJ
  COMMON /PARFM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NOSCRL,HCL,PATM,PWATVA,
1  IPROMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

  THIS SUBROUTINE, COUPLED WITH COUPES/COMMIN, CONSTITUTES ANALYSIS
  METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

  INPLT-INITIALIZATION PHASE

  PI=3.14159264
  IF(.NOT.(ICALC.EQ.1))GC TO 1

  SET "DESIGN CASE 2" PARAMETERS

  ENVIRONMENTAL

  TEMP=64.4
  WATRO=1.9852
  WATNU=.000011500
  PATM=14.697
  PWATVA=.295435251

  PROPELLER PARAMETERS

  Z=6.0
  PRCMAT=5.0

  HULL PARAMETERS

  WT=C.22
  TC=C.14
  ETARR=1.025
  NOSCRL=1.0
  HCL=21.9827
  DIALIM=30.0

  SET DESIGN VARIABLES FELD FIXED FOR "DESIGN CASE 2"

```



```

CC      QS=121C129.835
CC      N=110.C
CC      VK=20.C641C256
CC      V=1.68E*VK
CC
CC      END OF INPUT-INITIALIZATION PHASE
CC
CC      GO TO 3
CC
CC      EXECUTION PHASE
CC
1 CONTINUE
  IF(.NOT.(ICALC.EC.2))GC TO 2
  TC75R=((0.C185-0.C0125*Z)+Z)/(2.073*AEDVAD)
  CALL RCCAL(DIA)
  CALL CF75RA(C75R,R75R)
  CALL REY75R(C75R,R75R)
  CALL CCEFSR(RJ,R75R,KI,KQ)
  CALL OFWEFF(RJ,KI,KQ,EIAD)
  CALL CALCPE(KI,PE)
  CALL JCNA(RJ,RJCNL,RJCNH)
  CALL REYCNA(R75R,R75RCL,R75RCU)
  CALL EXTCCN(Z,AEDVAD,TC75R,AEADCL,AEADCU,TC75CL,TC75CU)
  CALL BLPOW2(KC,PCWBL)
  CALL DIGNUA(DIACNU)
  CALL CAVCNA(KI,AEACCV)
  CALL STRCNA(KQ,C75R,TCSTRS)
CC
CC      END OF EXECUTION PHASE
CC
CC      GO TO 3
CC
2 CONTINUE
  OUTPUT-RESULT PHASE
  VK=V/1.688
  VAK=(1.0-WI)*VK
  PC=(2.C*PI)*QS*N/330CU.0
  WRITE(6,9000)
  WRITE(6,9001) TEMP,WATKO,WATNU,PAIM,PWATVA
  WRITE(6,9002) WI,TD,ETARR,NOSCRW,HCL,DIALIM
  WRITE(6,9003) Z
  IF(.NOT.(PROMAT.EC.1.0))GO TO 81
  GC TO 86
  CCNTINLE
  IF(.NOT.(PROMAT.EC.2.0))GC TO 82
CC
CC      81

```



```

      GC TC E6
      CCNTINLE
      IF(.NOT.(PRMAT.EC.3.0))GC TO 83
      WRITE(6,9007)SC
      GC TO E6
      CCNTINLE
      IF(.NOT.(PRMAT.EC.4.0))GC TO 84
      WRITE(6,9008)SC
      GC TC E6
      CCNTINLE
      IF(.NOT.(PRMAT.EC.5.0))GC TO 85
      WRITE(6,9009)SC
      GC TC E6
      CCNTINLE
      CCNTINLE
      WRITE(6,9010)PE,V,VK,VAK,N,QS,PD,R,KT,KQ,ETAO,K75R,DIA,
      WRITE(6,9012)DIALIM,AEADOMN,TC75MN
      3 CONTINUE
      RETURN
C
C
C MISCELLANEOUS FORMAT STATEMENTS
9000 FORMAT('1','OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2',/,
1X,/,
1X,/,DESIGN VARIABLES SPECIFIED: QS,N,V,TC75R,/,
9001 FORMAT(1X,/,ENVIRONMENTAL PARAMETERS:
9002 FORMAT(FIC,4,/,
1X,2SX,DENSITY (LBF-SEC2/FT4),12X,=,F10.4,/,
1X,2SX,VISCOSITY (FT2/SEC),15X,=,E16.9,/,
1X,2SX,ATMOSPHERIC PRESSURE (PSIA),7X,=,F10.4,/,
1X,2SX,WATER VAPORIZATION PRESSURE (PSIA)=,F10.4,/,
9003 FORMAT(1X,2SX,HULL PARAMETERS:,13X,WAKE FRACTION,21X,=,F10.4,/,
1X,2SX,THRUST DEDUCTION FRACTION,9X,=,F10.4,/,
1X,2SX,RELATIVE ROTATIVE EFFICIENCY,6X,=,F10.4,/,
1X,2SX,NUMBER OF PROPELLERS,14X,=,F10.1,/,
1X,2SX,DEPTH TO SHAFT CENTERLINE (FT),4X,=,F10.4,/,
1X,2SX,DIAMETER LIMIT (FT),15X,=,F10.4,/,
9004 FORMAT(1X,/,PROPELLER PARAMETERS:,8X,NUMBER OF BLADES,18X,=,
FIC,1)
9005 FORMAT(1X,2SX,MATERIAL TYPE,21X,= CAST IRON,/,
1X,2SX,ALLOWABLE STRESS (PSI),12X,=,F10.1,/)
9006 FORMAT(1X,2SX,MATERIAL TYPE,21X,= CAST STEEL,/,
1X,2SX,ALLOWABLE STRESS (PSI),12X,=,F10.1,/)
9007 FORMAT(1X,2SX,MATERIAL TYPE,21X,= BRONZE,/,
1X,2SX,ALLOWABLE STRESS (PSI),12X,=,F10.1,/)

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APPCC570
 APPCC580
 APPCC590
 APP01000
 APP01010
 APP01020
 APP01030
 APP01040
 APP01050
 APP01060
 APP01070
 APP01080
 APP01090
 APP01100
 APP01110
 APP01120
 APP01130
 APP01140
 APP01150
 APP01160
 APP01170
 APP01180
 APP01190
 APP01200
 APP01210
 APP01220
 APP01230
 APP01240
 APP01250
 APP01260
 APP01270
 APP01280
 APP01290
 APP01300
 APP01310
 APP01320
 APP01330
 APP01340
 APP01350
 APP01360
 APP01370
 APP01380
 APP01390
 APP01400
 APP01410
 APP01420
 APP01430
 APP01440


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9008 FORMAT(1X,25X,MATERIAL TYPE,21X,PSI,NIAL BRGNZE,1,1,
1 1X,25X,ALLOWABLE STRESS,21X,PSI,12X,STEEL,1,1,
9009 1X,25X,MATERIAL TYPE,21X,PSI,12X,LESS STEEL,1,1,
1 1X,25X,ALLOWABLE STRESS,21X,PSI,12X,STEEL,1,1,
9010 1X,25X,MATERIAL TYPE,21X,PSI,12X,STEEL,1,1,
9011 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
1 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
2 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
2 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
3 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
4 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
5 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
6 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
7 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
8 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
9 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
A 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
B 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
C 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
D 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
E 1X,25X,SELECTED MATERIAL VALUES,12X,PSI,12X,STEEL,1,1,
9012 1X,25X,CONSTRINT VALUES,11X,PSI,12X,STEEL,1,1,
1 1X,25X,CONSTRINT VALUES,11X,PSI,12X,STEEL,1,1,
2 1X,25X,CONSTRINT VALUES,11X,PSI,12X,STEEL,1,1,
2 END

```

```

APP01450
APP01460
APP01470
APP01480
APP01490
APP01500
APP01510
APP01520
APP01530
APP01540
APP01550
APP01560
APP01570
APP01580
APP01590
APP01600
APP01610
APP01620
APP01630
APP01640
APP01650
APP01660
APP01670
APP01680
APP01690

```



```

CC      QS=121(129.835
CC      N=110.C
CC      VK=20.(6410256
CC      V=1.68E*VK
CC
CC      END OF INPUT-INITIALIZATION PHASE
CC
CC      GO TO 3
CC
CC      EXECUTION PHASE
CC
CC      1 CONTINUE
CC      IF(.NOT.(ICALC.EQ.2))GO TO 2
CC      TC75R=((0.0185-0.00125*Z)*Z)/(2.073*AELEVAO)
CC      CALL RICAL(DIA)
CC      CALL CF75RA(C75R)
CC      CALL REY75R(C75R,R75R)
CC      CALL CCEFFSA(RJ,R75R,KT,KQ)
CC      CALL OFWEFF(KJ,KT,KQ,ETAO)
CC      CALL CALCPE(KT,PE)
CC      CALL JCNA(RJ,RJCNL,RJCNL)
CC      CALL REYCNAL(R75R,R75RCL,R75RCU)
CC      CALL EXITCCN(Z,AELEVAO,TC75R,AEACCL,AEAOOU,TC75CL,TC75CU)
CC      CALL BLPOW2(KC,PONBAL)
CC      CALL DICNUA(DIACNU)
CC      CALL CAVCNA(KT,AEAOOU)
CC      CALL STRCNK(KC,KT,C75R,TCSTKS)
CC
CC      END OF EXECUTION PHASE
CC
CC      GO TO 3
CC
CC      2 CONTINUE
CC
CC      OUTPUT-RESULT PHASE
CC
CC      VK=V/1.688
CC      VAK=(1.0-WT)*VK
CC      PC=(2.C#PII*QS*N)/33000.0
CC      WRITE(6,9000)
CC      WRITE(6,9001)
CC      WRITE(6,9002) TEMP,WATFO,WATNU,PATM,PWATVA
CC      WRITE(6,9003) WT,TD,ETARR,NGSCRH,HCL,DIALIM
CC      WRITE(6,9004) Z
CC      IF(.NOT.(PRGMAT.EQ.1.0))GO TO 81
CC      WRITE(6,9005)ISC
CC      TC 86
CC      CCNTINLE
CC      IF(.NOT.(PRGMAT.EQ.2.0))GO TO 82
CC
CC      81
CC
CC      82

```



```

9008 FFORMAT(IX,25X,'MATERIAL TYPE:',21X,'= NI-AL BRONZE',/,
1X,25X,'ALLCABLE STRESS (PSI)',12X,'=',F10.4,/,
9009 FFORMAT(IX,25X,'MATERIAL TYPE:',21X,'= STAINLESS STEEL',/,
1X,25X,'ALLCABLE STRESS (PSI)',12X,'=',F10.4,/,
9010 FFORMAT(IX,25X,'MATERIAL TYPE:',21X,'= NCT CCONSIDERED',/,
1X,25X,'SELECTION VALUES:',12X,'PE (HP)',27X,'=',F10.1,/,
9011 FFORMAT(IX,25X,'V',12X,'= F10.4,/,
1X,25X,'VA',12X,'= F10.4,/,
1X,25X,'VN',12X,'= F10.4,/,
1X,25X,'CS',12X,'= F12.1,/,
1X,25X,'PD',12X,'= F10.2,/,
1X,25X,'J',12X,'= F10.4,/,
1X,25X,'KT',12X,'= F10.4,/,
1X,25X,'KQ',12X,'= F10.4,/,
9012 FFORMAT(IX,25X,'ETA Q',12X,'= F10.4,/,
1X,25X,'REV 75R',12X,'= E10.1,/,
1X,25X,'DIA (FT)',12X,'= F10.4,/,
1X,25X,'P/D',12X,'= F10.4,/,
1X,25X,'AE/AC',12X,'= F10.4,/,
1X,25X,'T/C',12X,'= F10.4,/)
9012 FFORMAT(IX,25X,'STRAIN AE/AO',11X,'MAX DIA (FT)',22X,'=',F10.4,/,
1X,25X,'MIN T/C',25X,'= F10.4,/,
1X,25X,'MIN T/C',25X,'= F10.6)
END
```



```

SUBROUTINE ANALIZ(ICALC)
INTEGER*4 IALC
REAL*4
1  ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,
2  RJC�L,RJCNJ,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,
3  FGWBAL,DIACNU,AEAOCV,TCSTRS,RJ,
4  VK,TC,WT,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PAIM,
5  PWATVA,PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC,
COMMON /GLOECM/ETAO,WEIGHT,AEDVAO,DIA,N,PE,FDIVD,QS,TC75R,V,RJC�L,
1RJCNJ,R75RCL,R75RCU,AEAOCL,AEACCU,TC75CL,TC75CU,POWBAL,DIACNU,
2AEACCV,TCSTRS,RJ
COMMON /PARAM/VK,TC,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PAIM,PWATVA,
1PRGMAT,DIALIM,ETARR,AEADMN,TC75MN,SC

THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
METHOD FOR "DESIGN CASE 2" PROPELLER SELECTION PROBLEMS

INPUT-INITIALIZATION PHASE

PII=3.14159264
IF(.NOT.(ICALC.EQ.1))GO TO 1

SET "DESIGN CASE 2" PARAMETERS

ENVIRONMENTAL

TEMP=64.4
WATRO=1.9852
WATNU=.000011900
PAIM=14.657
PWATVA=.255435291

PROPELLER PARAMETERS

Z=6.0
PRCMAT=5.0

HULL PARAMETERS

WT=0.22
TC=C.15
ETARR=1.025
NCSCRW=1.0
HCL=21.5827
DIALIM=30.0

SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 2"

```



```

CC      QS=121(129.835
CC      N=110.C
CC      ENC OF INPUT-INITIALIZATION PHASE
C
C      GO TO 3
C      EXECUTION PHASE
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GC TO 2
    CALL RICCAL(DIA)
    CALL CF75RA(C75R,R75R)
    CALL REY75R(C75R,R75R,KT,KQ)
    CALL CCEFFS(RJ,KT,KQ,ETAU)
    CALL OFWEFF(RJ,KT,KQ,ETAU)
    CALL CALCPE(KT,PE)
    CALL JCNA(RJ,RJCNL,RJCNL)
    CALL REYCNNA(R75R,R75RCL,R75RCL)
    CALL EXTCCN(Z,AEDVAO,TC75R,AEACCL,AEADCU,TC75CL,TC75CU)
    CALL BLPOW2(KC,POWBAL)
    CALL DICNUA(DIACNU)
    CALL CAVCNA(KT,AEADCV)
    CALL STRCNA(KC,C75R,TCSTRS)
  ENC OF EXECUTION PHASE
CC
CC      GO TO 3
CC      2 CONTINUE
C      OUTPUT-RESULT PHASE
    VK=V/1.688
    VAK=(1.0-WT)*VK
    PC=(2.C*PI*I*QS*N)/33000.0
    WRITE(6,9000)
    WRITE(6,9001)
    WRITE(6,9002) TEMP,WAIRQ,WAIRNU,PATM,PWAIVA
    WRITE(6,9003) WT,TD,ETARR,NUSCRW,HCL,DIALIM
    WRITE(6,9004) Z
    IF(.NOT.(PROMAT.EQ.1.0))GC TO 81
      WRITE(6,9005)SC
      GC TO 86
      CCATINLE
      IF(.NOT.(PROMAT.EQ.2.0))GC TO 82
      WRITE(6,9006)SC
      GC TO 86
      CCATINLE
81
82

```



```

83      IF(.NOT.(PROMAT.EQ.3.0))GO TO 83
        WRITE(6,9007)SC
        GC TC E6
        CCNTINLE
84      IF(.NOT.(PROMAT.EQ.4.0))GO TO 84
        WRITE(6,9008)SC
        GC TC E6
        CCNTINLE
85      IF(.NOT.(PROMAT.EQ.5.0))GO TC 85
        WRITE(6,9009)SC
        GC TC E6
        CCNTINLE
86      WRITE(6,9010)
        CCNTINLE
        WRITE(6,9011)PE,V,VK,VAK,N,GS,PD,RJ,KT,KQ,ETAO,R75R,DIA,
        WRITE(6,9012)DIALIM,AEADOMN,IC75MN
        3 CONTINUE
        RETURN
C
C
C      MISCELLANEOUS FORMAT STATEMENTS
9000  FORMAT('1','OPTIMIZATION RESULTS -----
1X,
9001  FORMAT(1X,'DESIGN VARIABLES SPECIFIED:
9002  FORMAT(1X,'ENVIRONMENTAL PARAMETERS:
        FIC.4,/,
        1X,25X,/, DENSITY (LBF-SEC2/FT4), 12X, '=',F10.4,/,
        1X,25X,/, VISCOSITY (FT2/SEC), 15X, '=',E16.9,/,
        1X,25X,/, ATMOSPHERIC PRESSURE (PSIA), 7X, '=',F10.4,/,
        1X,25X,/, WATER VAPORIZATION PRESSURE (PSIA), 7X, '=',F10.4,/,
        9003  FORMAT(1X,'FULL PARAMETERS: 13X, WAKE FRACTION, 21X, '=',F10.4,/,
        1X,25X,/, THRUST DEDUCTION FRACTION, 9X, '=',F10.4,/,
        1X,25X,/, RELATIVE ROTATIVE EFFICIENCY, 6X, '=',F10.4,/,
        1X,25X,/, NUMBER OF PROPELLERS, 14X, '=',F10.1,/,
        1X,25X,/, DEPTH TO SHAFT CENTERLINE (FT), 4X, '=',F10.4,/,
        1X,25X,/, DIAMETER LIMIT (FT), 15X, '=',F10.4,/,
        9004  FORMAT(1X,'PROPELLER PARAMETERS: 8X, NUMBER OF BLADES, 18X, '=',
        F10.1),
        9005  FORMAT(1X,25X,/, MATERIAL TYPE, 21X, '=', CAST IRON,/,
        1X,25X,/, ALLCABLE STRESS (PSI), 12X, '=',F10.1,/,
        9006  FORMAT(1X,25X,/, MATERIAL TYPE, 21X, '=', CAST STEEL,/,
        1X,25X,/, ALLCABLE STRESS (PSI), 12X, '=',F10.1,/,
        9007  FORMAT(1X,25X,/, MATERIAL TYPE, 21X, '=', BRONZE,/,
        1X,25X,/, ALLCABLE STRESS (PSI), 12X, '=',F10.1,/,
        9008  FORMAT(1X,25X,/, MATERIAL TYPE, 21X, '=', NI-AL BRONZE,/,
        1X,25X,/, ALLCABLE STRESS (PSI), 12X, '=',F10.1,/,
        9009  FORMAT(1X,25X,/, MATERIAL TYPE, 21X, '=', STAINLESS STEEL,/,

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APP04390
APP04400
APP04410
APP04420
APP04430
APP04440
APP04450
APP04460
APP04470
APP04480
APP04490
APP04500
APP04510
APP04520
APP04530
APP04540
APP04550
APP04560
APP04570
APP04580
APP04590
APP04600
APP04610
APP04620
APP04630
APP04640
APP04650
APP04660
APP04670
APP04680
APP04690
APP04700
APP04710
APP04720
APP04730
APP04740
APP04750
APP04760
APP04770
APP04780
APP04790
APP04800
APP04810
APP04820
APP04830
APP04840
APP04850
APP04860

```



```

1
9010 FORMAT(1X,25X,ALL CABLE STRESS (PSI),12X,*,F10.1,/)
9011 FORMAT(1X,25X,MATERIAL TYPE:,12X,*,PE (HP),CCNSIDERED:,/)
1
2
3
4
5
6
7
8
9
A
B
C
D
E
F
9012 FORMAT(1X,25X,CONSTR INT AE/AQ,25X,*,F10.4,/,
1
2
END
1X,25X,SELECTION VALUES:,12X,*,27X,*,F10.1,/,
1X,25X,V (FT/SEC),23X,*,F10.4,/,
1X,25X,V (KNOTS),23X,*,F10.4,/,
1X,25X,V (KNOTS),23X,*,F10.4,/,
1X,25X,V (RPM),23X,*,F12.1,/,
1X,25X,V (FT-LBF),23X,*,F10.2,/,
1X,25X,PD (HP),23X,*,F10.4,/,
1X,25X,JT,23X,*,F10.4,/,
1X,25X,KQ,23X,*,F10.4,/,
1X,25X,ETAC,23X,*,F10.4,/,
1X,25X,REV75R,23X,*,E10.1,/,
1X,25X,CIA (FT),23X,*,F10.4,/,
1X,25X,P/D,23X,*,F10.4,/,
1X,25X,AE/AQ,23X,*,F10.4,/,
1X,25X,T/C,23X,*,F10.4,/,
1X,25X,CONSTR INT AE/AQ,25X,*,F10.4,/,
1X,25X,MIN T/C,25X,*,F10.6)

```

```

APP04870
APP04880
APP04890
APP04900
APP04910
APP04920
APP04930
APP04940
APP04950
APP04960
APP04970
APP04980
APP04990
APP05000
APP05010
APP05020
APP05030
APP05040
APP05050
APP05060
APP05070
APP05080

```



```

CC      QS=121C129.835
CC      N=110.C
CC
C      ENC OF INPUT-INITIALIZATION PHASE
C
C      GO TO 3
C
C      EXECUTION PHASE
C
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GC TO 2
  CALL RECAL(DIAL)
  CALL CF75RA(C75R)
  CALL REY75R(C75R,R75R)
  CALL CCEESA(RJ,R75R,KI,KQ)
  CALL OFWEFF(RJ,KI,KQ,ETAO)
  CALL CALCPE(KI,PE)
  CALL JCNA(RJ,RJCNL,RJCNL)
  CALL REVCNA(R75R,R75RCL,R75RCU)
  CALL EXTCCN(Z,AEDVAO,TC75R,AEACCL,AEAOOU,TC75CL,TC75CU)
  CALL BLPOW2(KC,POWBAL)
  CALL DIGNUA(DIACNU)
  CALL CAVCNA(KI,AEAOOU)
  CALL SIRCNK(KC,KI,TC75R,TC75R)
  CC
  CC      ENC OF EXECUTION PHASE
  CC
C      GO TO 3
C
2 CONTINUE
  CC
  CC      OUTPUT-RESULT PHASE
  CC
    VK=V/1.688
    VAK=(1.0-WT)*VK
    PC=(2.C*PI)*QS*N/35000.0
    WRITE(6,9000)
    WRITE(6,9001)
    WRITE(6,9002) TEMP,WATRU,WATNU,PATM,PWATVA
    WRITE(6,9003) WT,TD,ETARR,NGSCRH,HCL,DIALIM
    WRITE(6,9004) Z
    IF(.NOT.(PRGMAT.EQ.1.0))GC TO 81
      WRITE(6,9005)SC
    GC TO 86
    CC CONTINUE
  IF(.NOT.(PRGMAT.EQ.2.0))GC TO 82
    GC TO 86
    CC CONTINUE
  81
  82

```



```

IF(.NOT.(PRGMAT.EQ.3.0))GC TO E3
WRITE(6,5007)SC
GC TO E6
CCCONTINUE
IF(.NOT.(PRGMAT.EQ.4.0))GC TO E4
WRITE(6,5008)SC
GC TO E6
CCCONTINUE
IF(.NOT.(PRGMAT.EQ.5.0))GC TO E5
WRITE(6,5009)SC
GC TO E6
CCCONTINUE
WRITE(6,5010)
CCCONTINUE
WRITE(6,9011)PE,V,VK,VAK,N,QS,PD,RJ,KT,KQ,ETAO,R75R,DIA,
PLIVC,AE,CVAO,TC75R
1
WRITE(6,9012)DIALIM,AEADMA,TC75MN
3 CONTINUE
RETURN
C
C MISCELLANEOUS FORMAT STATEMENTS
C
9000 FORMAT(1X,OPTIMIZATION RESULTS -----
1X,
1X,DESIGN VARIABLES SPECIFIED:
9001 FORMAT(1X,ENVIRONMENTAL PARAMETERS:
9002 FORMAT(1X,
1X,4,
1X,25X,DENSITY (LBF-SEC2/FT4),12X,=,F10.4,/,
1X,25X,VISCOSITY (FT2/SEC),15X,=,E16.9,/,
1X,25X,ATMOSPHERIC PRESSURE (PSIA),7X,=,F10.4,/,
1X,25X,WATER VAPORIZATION PRESSURE (PSIA)=,F10.4,/,
9003 FORMAT(1X,FULL PARAMETER:13X,WAKE FRACTION,21X,=,F10.4,/,
1X,25X,THRUST DEDUCTION EFFICIENCY,9X,=,F10.4,/,
1X,25X,RELATIVE ROTATIVE ELLIPS,14X,=,F10.4,/,
1X,25X,NUMBER OF PROPELLERS,14X,=,F10.4,/,
1X,25X,DEPTH TO SHAFT CENTERLINE (FT),4X,=,F10.4,/,
1X,25X,DIAMETER LIMIT (FT),15X,=,F10.4,/,
9004 FORMAT(1X,PROPELLER PARAMETERS:,8X,NUMBER OF BLADES,18X,=,
F10.1)
9005 FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST IRON,/,
1X,25X,ALLCWAIBLE STRESS (PSI),12X,=,F10.1,/,
9006 FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST STEEL,/,
1X,25X,ALLCWAIBLE STRESS (PSI),12X,=,F10.1,/,
9007 FORMAT(1X,25X,MATERIAL TYPE,21X,=,BRONZE,/,
1X,25X,ALLCWAIBLE STRESS (PSI),12X,=,F10.1,/,
9008 FORMAT(1X,25X,MATERIAL TYPE,21X,=,NI-AL BRCNZE,/,
1X,25X,ALLCWAIBLE STRESS (PSI),12X,=,F10.1,/,
9009 FORMAT(1X,25X,MATERIAL TYPE,21X,=,STAINLESS STEEL,/,

```

```

APP06070
APP06080
APP06090
APP06100
APP06110
APP06120
APP06130
APP06140
APP06150
APP06160
APP06170
APP06180
APP06190
APP06200
APP06210
APP06220
APP06230
APP06240
APP06250
APP06260
APP06270
APP06280
APP06290
APP06300
APP06310
APP06320
APP06330
APP06340
APP06350
APP06360
APP06370
APP06380
APP06390
APP06400
APP06410
APP06420
APP06430
APP06440
APP06450
APP06460
APP06470
APP06480
APP06490
APP06500
APP06510
APP06520
APP06530
APP06540

```



```

1 1X,25X, 'ALLQWABLE STRESS (PSI)', 12X, '==', F10.1, /)
901G FORMAT(1X,25X, 'MATERIAL TYPE', 12X, '==', NCT, CCONSIDEREC, /)
9011 FORMAT(1X,25X, 'SELECTION VALUES:', 12X, '==', (HP), 27X, '==', F10.1, /,
1X,25X, 'V', 23X, '==', F10.4, /,
1X,25X, 'V', 23X, '==', F10.4, /,
1X,25X, 'VA', 23X, '==', F10.4, /,
1X,25X, 'VA', 23X, '==', F10.4, /,
1X,25X, 'NCS', 23X, '==', F12.1, /,
1X,25X, 'CS', 23X, '==', F10.2, /,
1X,25X, 'PD', 23X, '==', F10.4, /,
1X,25X, 'J', 23X, '==', F10.4, /,
1X,25X, 'KT', 23X, '==', F10.4, /,
1X,25X, 'KQ', 23X, '==', F10.4, /,
1X,25X, 'ETAC', 23X, '==', F10.4, /,
1X,25X, 'REV75R', 23X, '==', F10.4, /,
1X,25X, 'DIA (FT)', 23X, '==', F10.4, /,
1X,25X, 'P/D', 23X, '==', F10.4, /,
1X,25X, 'AE/AG', 23X, '==', F10.4, /,
1X,25X, 'T/C', 23X, '==', F10.4, /,
9012 FORMAT(1X,25X, 'CON STRAIN VALUES:', 11X, 'MAX DIA (FT)', 22X, '==', F10.4, /,
1X,25X, 'MIN AE/AG', 25X, '==', F10.4, /,
1X,25X, 'MIN T/C', 22X, '==', F10.6)
2 END
APP06550
APP06560
APP06570
APP06580
APP06590
APP06600
APP06610
APP06620
APP06630
APP06640
APP06650
APP06660
APP06670
APP06680
APP06690
APP06700
APP06710
APP06720
APP06730
APP06740
APP06750
APP06760

```


CONTROL CARD IMAGES--DESIGN CASE NO. 2

288

\$A	TITLE	B-SERIES	PROPELLER	OPTIMIZATION	IPINPUT	IPDBG
\$B	WAGENINGEN	NDV	NSV	N2VAR	LINOBJ	NACMX1
\$C	IPRINT	ITMAX	ICNDR	NSCAL	CTLMIN	THETA
\$D1	FDCH	ICOO	CT	CTMIN		
\$D2	DELFUN	FDCHM	ALPHAX	ABCBJI		
\$E	NDVTOT	ICBJ	SGNGPT			
\$F	VLB	VUB	1.0	SCAL		
	0.4	1.1	0.40	1.0		
	0.01	1.1	0.1	1.0		
	0.4	1.4	1.0	1.0		
	10.0	100.0	50.0	10.0		
\$G	NDSSGN	CD5	0.0500	0.01		
	1	IDSGN	AMULT			
	2	23	1.0			
	3	7	1.0			
	4	10	1.0			
	5	9	1.0			
\$H	NCON	JCON	LCCN	SCAL2		
\$I1	ICCN	SCAL1	BU	1.0		
\$I2	11	11	0.0	1.0		
-1.0	+16	12	0.0	1.0		
-1.0	+12	13	0.0	1.0		
-1.0	+13	14	0.0	1.0		
-1.0	+14	15	0.0	1.0		
-1.0	+15	16	0.0	1.0		
-1.0	+16	17	0.0	1.0		
-1.0	+17	18	0.0	1.0		
-1.0	+18	19	0.0	1.0		
-1.0	+19	20	0.0	1.0		
-1.0	+20	1.0	0.0	1.0		

-1.0	21	0.0	1.0
-1.0	+16	0.0	
\$V	22	0.0	1.0
END	+16		

COPEs OUTPUT--DESIGN CASE NO. 2

```

SSSSS  EEEEE  OOOO  CCCC
SSSS  EE  OO  CC
SSSS  EE  OO  CC
SSSSS  EEEEE  OOOO  CCCC

```

CENTRAL PROGRAM

FOR

ENGINEERING SYNTHESES

3712

WAGENINGEN B-SERIES PROPELLER OPTIMIZATION

CARC IMAGES OF CONTROL DATA

CARD	IMAGE	PROPPELLER OPTIMIZATION									
		\$A TITLE	H-SERIES	NSV	NSCAL	ITRM	IPNPLT	IPDBG			
1		\$E ACALC	NOV								
2		\$E ACALC	NOV								
3		\$E ACALC	NOV								
4		\$E ACALC	NOV								
5		\$E ACALC	NOV								
6		\$E ACALC	NOV								
7		\$E ACALC	NOV								
8		\$E ACALC	NOV								
9		\$E ACALC	NOV								
10		\$E ACALC	NOV								
11		\$E ACALC	NOV								
12		\$E ACALC	NOV								
13		\$E ACALC	NOV								
14		\$E ACALC	NOV								
15		\$E ACALC	NOV								
16		\$E ACALC	NOV								
17		\$E ACALC	NOV								
18		\$E ACALC	NOV								
19		\$E ACALC	NOV								
20		\$E ACALC	NOV								
21		\$E ACALC	NOV								
22		\$E ACALC	NOV								
23		\$E ACALC	NOV								
24		\$E ACALC	NOV								
25		\$E ACALC	NOV								
26		\$E ACALC	NOV								
27		\$E ACALC	NOV								
28		\$E ACALC	NOV								
29		\$E ACALC	NOV								
30		\$E ACALC	NOV								
31		\$E ACALC	NOV								
32		\$E ACALC	NOV								
33		\$E ACALC	NOV								
34		\$E ACALC	NOV								
35		\$E ACALC	NOV								
36		\$E ACALC	NOV								
37		\$E ACALC	NOV								
38		\$E ACALC	NOV								
39		\$E ACALC	NOV								
40		\$E ACALC	NOV								
41		\$E ACALC	NOV								
42		\$E ACALC	NOV								
43		\$E ACALC	NOV								
44		\$E ACALC	NOV								
45		\$E ACALC	NOV								
46		\$E ACALC	NOV								
47		\$E ACALC	NOV								
48		\$E ACALC	NOV								
49		\$E ACALC	NOV								
50		\$E ACALC	NOV								
51		\$E ACALC	NOV								
52		\$E ACALC	NOV								
53		\$E ACALC	NOV								
54		\$E ACALC	NOV								
55		\$E ACALC	NOV								
56		\$E ACALC	NOV								
57		\$E ACALC	NOV								
58		\$E ACALC	NOV								
59		\$E ACALC	NOV								
60		\$E ACALC	NOV								
61		\$E ACALC	NOV								
62		\$E ACALC	NOV								
63		\$E ACALC	NOV								
64		\$E ACALC	NOV								
65		\$E ACALC	NOV								
66		\$E ACALC	NOV								
67		\$E ACALC	NOV								
68		\$E ACALC	NOV								
69		\$E ACALC	NOV								
70		\$E ACALC	NOV								
71		\$E ACALC	NOV								
72		\$E ACALC	NOV								
73		\$E ACALC	NOV								
74		\$E ACALC	NOV								
75		\$E ACALC	NOV								
76		\$E ACALC	NOV								
77		\$E ACALC	NOV								
78		\$E ACALC	NOV								
79		\$E ACALC	NOV								
80		\$E ACALC	NOV								
81		\$E ACALC	NOV								
82		\$E ACALC	NOV								
83		\$E ACALC	NOV								
84		\$E ACALC	NOV								
85		\$E ACALC	NOV								
86		\$E ACALC	NOV								
87		\$E ACALC	NOV								
88		\$E ACALC	NOV								
89		\$E ACALC	NOV								
90		\$E ACALC	NOV								
91		\$E ACALC	NOV								
92		\$E ACALC	NOV								
93		\$E ACALC	NOV								
94		\$E ACALC	NOV								
95		\$E ACALC	NOV								
96		\$E ACALC	NOV								
97		\$E ACALC	NOV								
98		\$E ACALC	NOV								
99		\$E ACALC	NOV								
100		\$E ACALC	NOV								

TITLE: WAGENMAN U-SERIES PROPELLER CUPPLIZATION

```
CONTRUL PARAMETERS:
COUNT          = 1000000
CUBALIST        = 1
DEVIATION       = 1E-6
LOW VARIABLES   = 1
NUMBER OF VARS = 10
CONSTRAINTS    = 10
NUMBER OF VARS = 10
NUMBER OF VARS = 10
APPROXIMATE     = 1
INPUT PRINT CODE = 1
DEBUG PRINT CODE = 1
```

CALCULATION CONTROL	NALG
PEAKING	
SINGLE ANALYSIS	
INITIATION	
SENSITIVITY	
TRAO-VARIABLES	FUNCTILA SPLE
OPTIMUM SENSITIVITY	
APPROXIMATE OPTIMIZATION	

• • OPTIMIZATION INFORMATION

```

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01

COLUMN HEADERS (IF ZERO, COLUMN DEFAULT WILL OVER-RIDE)
      LINEAR    NSCAL    30    LINCBJ    NALCAL    NFEQ
      1000      -1      10      15

```

FULL	PULCH	-I	CLEAR
0.0	0.000E+03	0.000E-02	0.0
CL	CLEAR	INCL	PH
0.0	0.0	0.0	0.0
DELTA	DALFA	ALPHA	ABUJA
0.0	0.0	0.0	0.0

[illegible]

DESIGN VARIABLES		GLOBAL		MULTIPLYING
ID	C.V.	AC.	VAR. NO.	
1	1	1	3	0.1000E+01
2	1	1	3	0.1000E+01
3	1	1	3	0.1000E+01

LUNSTRAAL INFUKMATION

LINE	FILE	CON	CONSTRAINT	STATUS	LOWER	UPPER	NORMALIZATION
ID	NAME	VAR	LINE	LINE	BOUND	BOUND	FACTOR
1	1	1	1	1	1	1	1
2	2	2	2	2	2	2	2
3	3	3	3	3	3	3	3
4	4	4	4	4	4	4	4
5	5	5	5	5	5	5	5
6	6	6	6	6	6	6	6
7	7	7	7	7	7	7	7
8	8	8	8	8	8	8	8
9	9	9	9	9	9	9	9
10	10	10	10	10	10	10	10
11	11	11	11	11	11	11	11
12	12	12	12	12	12	12	12
13	13	13	13	13	13	13	13
14	14	14	14	14	14	14	14
15	15	15	15	15	15	15	15
16	16	16	16	16	16	16	16
17	17	17	17	17	17	17	17
18	18	18	18	18	18	18	18
19	19	19	19	19	19	19	19
20	20	20	20	20	20	20	20
21	21	21	21	21	21	21	21
22	22	22	22	22	22	22	22
23	23	23	23	23	23	23	23
24	24	24	24	24	24	24	24
25	25	25	25	25	25	25	25
26	26	26	26	26	26	26	26
27	27	27	27	27	27	27	27
28	28	28	28	28	28	28	28
29	29	29	29	29	29	29	29
30	30	30	30	30	30	30	30
31	31	31	31	31	31	31	31
32	32	32	32	32	32	32	32
33	33	33	33	33	33	33	33
34	34	34	34	34	34	34	34
35	35	35	35	35	35	35	35
36	36	36	36	36	36	36	36
37	37	37	37	37	37	37	37
38	38	38	38	38	38	38	38
39	39	39	39	39	39	39	39
40	40	40	40	40	40	40	40
41	41	41	41	41	41	41	41
42	42	42	42	42	42	42	42
43	43	43	43	43	43	43	43
44	44	44	44	44	44	44	44
45	45	45	45	45	45	45	45
46	46	46	46	46	46	46	46
47	47	47	47	47	47	47	47
48	48	48	48	48	48	48	48
49	49	49	49	49	49	49	49
50	50	50	50	50	50	50	50
51	51	51	51	51	51	51	51
52	52	52	52	52	52	52	52
53	53	53	53	53	53	53	53
54	54	54	54	54	54	54	54
55	55	55	55	55	55	55	55
56	56	56	56	56	56	56	56
57	57	57	57	57	57		

TOTAL ALPHEA OF UNCONSTRAINED PARALLELS = 10

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
67	EXECUTION	10000	43	EXECUTION	1000
	490			104	

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2
SUBROUTINE "STRUNA"

DESIGN VARIABLES SPECIFIED: US.N.V.1C75H

ENVIRONMENTAL PARAMETERS:

TEMP (DEG F) = 64.6000
DENSITY (LBF/FT³) = 0.11899997E-04
VISCOUSITY (FT²/SEC) = 1.07E-05
ATMOSPHERIC PRESSURE (PSIA) = 14.6970
WATER VAPORIZATION PRESSURE (PSIA) = 1.2556

FULL PARAMETERS:

WAKE FRACTION = 0.2200
WAKE FRACTION ON FRACTION = 0.2200
REAR VIEW PROPELLER EFFICIENCY = 1.07E-05
NUMBER OF PROPELLER ELEMENTS = 1.0
DEPTH TO SHAFT CENTERLINE (FT) = 21.9827
DIAMETER LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:

NUMBER OF BLADES = 6
MAXIMUM ALLOWABLE STRESS (PSI) = 30000.0 STEEL

SELECTION VALUES:

PE (HP) = 5152.4400
V (FT/SEC) = 33.8682
VA (KNOTS) = 20.0641
VA (KNOTS) = 11.6500
XOS (FT/HP) = 11.0100
PO (HP) = 23344.90
JT = 0.1000
KT = 0.4001
K9 = 0.0539
EIAS = 0.1182
ED (FT) = 0.1697
PD/C = 1.0000
AE/AC = 1.0000
T/C = 0.756

CONSTRAINT VALUES:

MAX DIA (FT) = 30.0000
MIN DIA (FT) = 0.0000
MIN T/C = 0.756


```

*****
C C P M I N
*****
F O R T R A N   P R O G R A M   F O R
*****
C O N S T R A I N E L   F U N C T I O N   M I N I M I Z A T I O N
*****

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.118204E+00
DECISION VARIABLES (X-VECTOR)
1) C.40000E+00 0.10000E+00 0.10000E+01
CONSTRAINT VALUES (G-VECTOR)
1) -C.62500E-01 -0.57735E+00 -0.52076E+03 -0.67876E+03 C.10600E+00 -0.40000E+00
7) -C.53696E-01 -0.47751E-01 -0.59996E+00 0.12270E+03 C.10600E+00 -0.40000E+00

```



```

FINAL OPTIMIZATION INFORMATION
CBJ = -C.665992E+00
DECISION VARIABLES (X-VECTOR)
1 1 C.80177E+00 0.64752E+00 0.50364E+00
CONSTRAINT VALUES (G-VECTOR)
1 1 -C.5470E+00 -0.59236E+00 -0.34820E+02 -0.97419E+03 -C.30178E+00 0.17773E-02
1 1 -C.15810E-01 -0.87642E-01 -0.24307E-02 -0.36772E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABSD(OBJ11-OBJ11-11) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 45
OBJECTIVE FUNCTION WAS EVALUATED 230 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 230 TIMES

```



```

OPTIMIZATION RESULTS

OBJECTIVE FUNCTION
GLOBAL LOCATION 1      FUNCTION VALUE 0.66599E+00

DESIGN VARIABLES
ID   D. V.   GLOBAL   LOWER   VALUE   UPPER
    NC.   VAR. NO.   BOUND   BOUND   BOUND
1     1     23      0.4000E+00  0.8017E+00  0.1000E+01
2     2     24      0.4000E+00  0.6435E+00  0.1000E+01
3     3     25      0.4000E+00  0.9036E+00  0.1000E+01

DESIGN CONSTRAINTS
ID   GLOBAL   LOWER   VALUE   UPPER
    VAR. NO.   BOUND   BOUND   BOUND
1     1     1000E+16  0.5530E+00  0.0
2     2     1000E+16  0.2822E+02  0.0
3     3     1000E+16  0.3319E+02  0.0
4     4     1000E+16  0.3319E+02  0.0
5     5     1000E+16  0.3319E+02  0.0
6     6     1000E+16  0.3319E+02  0.0
7     7     1000E+16  0.3319E+02  0.0
8     8     1000E+16  0.3319E+02  0.0
9     9     1000E+16  0.3319E+02  0.0
10    10     1000E+16  0.3319E+02  0.0
21    21     1000E+16  0.3319E+02  0.0

```


OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2
SUBROUTINE "STRNA"

DESIGN VARIABLES SPECIFIED: QS,A,V,IC75H

ENVIRONMENTAL PARAMETERS: TEMP (DEG F) = 64.4000

ENVIRONMENTAL PARAMETERS:	
TEMP (DEG F)	64.000
DENSITY (G/CM ³)	1.892
VISCOSITY (CENTIPOISE)	1.89997E-04
VAPOR PRESSURE (PSIA)	0.870
WATER VAPOR FRACTION	0.894

HULL PARAMETERS: WAKE FRACTION = 0.2200

[illegible]

PROPELLER PARAMETERS:	NUMBER OF BLADES	6.0
<p> </p>		

PROPELLER PARAMETERS:	
NUMBER OF BLADES	6.0
MATERIAL TYPE	STAINLESS STEEL
ALLOWABLE STRESS (PSI)	2400.0

SELECTION VALUES:

PE (HP) • 14057.3

SELECTION VALUES:	
PE (HP)	1457.3
VV (SEC)	32.862
VV (KNO3S)	20.041
VA (KNO3S)	15.500
VN (KNO3S)	15.500
VS (BT-BF)	11.130
PD (HP)	23.490

CONSTRAINT VALUES:

	MAX C/A (FT)	
	= 36,000	

CONSTRAINT	VALUES:
MAX CIA (FT)	= 30.0000
MIN AE/AD	= 0.5070
MIN T/C	= 0.021297

PROGRAM CALLS TO ANALYZE

ICALL	CALLS
1	201
2	202
3	

CARD IMAGES OF CONTROL DATA

CARD	IMAGE										
1	1	8A	TITLE	B-SERIES	PROPPELLER	OPTIMIZATION	IPNPLI	IPDUG			
2	2	8B	WAGENINGEN	NOV	NSV	NZVAR	LINDEJ	NALMYI			
3	3	8C	RCALC	1000	ICADIR	NSCAL	ITRM	CTLMIN			
4	4	8D	JPRINT	1000	NSCAL	-1	30	CTLMIN			
5	5	8E	FUCH	0.0001	CT	CTMIN	CTL	CTLMIN			
6	6	8F	FUCH	0.0001	ALPHA	ABGBJ		THEIA			
7	7	8G	DELFIN								
8	8	8H	NEUTOI	IOBJ	SGADPT						
9	9	8I	VLB	VLB	1.0	SCAL					
10	10	8J	0.01	0.01	0.40	1.0					
11	11	8K	0.04	0.04	0.1	1.0					
12	12	8L	0.04	0.04	0.1	1.0					
13	13	8M	0.04	0.04	0.1	1.0					
14	14	8N	0.04	0.04	0.1	1.0					
15	15	8O	0.04	0.04	0.1	1.0					
16	16	8P	0.04	0.04	0.1	1.0					
17	17	8Q	0.04	0.04	0.1	1.0					
18	18	8R	0.04	0.04	0.1	1.0					
19	19	8S	0.04	0.04	0.1	1.0					
20	20	8T	0.04	0.04	0.1	1.0					
21	21	8U	0.04	0.04	0.1	1.0					
22	22	8V	0.04	0.04	0.1	1.0					
23	23	8W	0.04	0.04	0.1	1.0					
24	24	8X	0.04	0.04	0.1	1.0					
25	25	8Y	0.04	0.04	0.1	1.0					
26	26	8Z	0.04	0.04	0.1	1.0					
27	27	8A	0.04	0.04	0.1	1.0					
28	28	8B	0.04	0.04	0.1	1.0					
29	29	8C	0.04	0.04	0.1	1.0					
30	30	8D	0.04	0.04	0.1	1.0					
31	31	8E	0.04	0.04	0.1	1.0					
32	32	8F	0.04	0.04	0.1	1.0					
33	33	8G	0.04	0.04	0.1	1.0					
34	34	8H	0.04	0.04	0.1	1.0					
35	35	8I	0.04	0.04	0.1	1.0					
36	36	8J	0.04	0.04	0.1	1.0					
37	37	8K	0.04	0.04	0.1	1.0					
38	38	8L	0.04	0.04	0.1	1.0					
39	39	8M	0.04	0.04	0.1	1.0					
40	40	8N	0.04	0.04	0.1	1.0					
41	41	8O	0.04	0.04	0.1	1.0					
42	42	8P	0.04	0.04	0.1	1.0					
43	43	8Q	0.04	0.04	0.1	1.0					
44	44	8R	0.04	0.04	0.1	1.0					
45	45	8S	0.04	0.04	0.1	1.0					
46	46	8T	0.04	0.04	0.1	1.0					
47	47	8U	0.04	0.04	0.1	1.0					
48	48	8V	0.04	0.04	0.1	1.0					
49	49	8W	0.04	0.04	0.1	1.0					
50	50	8X	0.04	0.04	0.1	1.0					
51	51	8Y	0.04	0.04	0.1	1.0					
52	52	8Z	0.04	0.04	0.1	1.0					
53	53	8A	0.04	0.04	0.1	1.0					
54	54	8B	0.04	0.04	0.1	1.0					
55	55	8C	0.04	0.04	0.1	1.0					
56	56	8D	0.04	0.04	0.1	1.0					
57	57	8E	0.04	0.04	0.1	1.0					
58	58	8F	0.04	0.04	0.1	1.0					
59	59	8G	0.04	0.04	0.1	1.0					
60	60	8H	0.04	0.04	0.1	1.0					
61	61	8I	0.04	0.04	0.1	1.0					
62	62	8J	0.04	0.04	0.1	1.0					
63	63	8K	0.04	0.04	0.1	1.0					
64	64	8L	0.04	0.04	0.1	1.0					
65	65	8M	0.04	0.04	0.1	1.0					
66	66	8N	0.04	0.04	0.1	1.0					
67	67	8O	0.04	0.04	0.1	1.0					
68	68	8P	0.04	0.04	0.1	1.0					
69	69	8Q	0.04	0.04	0.1	1.0					
70	70	8R	0.04	0.04	0.1	1.0					
71	71	8S	0.04	0.04	0.1	1.0					
72	72	8T	0.04	0.04	0.1	1.0					
73	73	8U	0.04	0.04	0.1	1.0					
74	74	8V	0.04	0.04	0.1	1.0					
75	75	8W	0.04	0.04	0.1	1.0					
76	76	8X	0.04	0.04	0.1	1.0					
77	77	8Y	0.04	0.04	0.1	1.0					
78	78	8Z	0.04	0.04	0.1	1.0					
79	79	8A	0.04	0.04	0.1	1.0					
80	80	8B	0.04	0.04	0.1	1.0					
81	81	8C	0.04	0.04	0.1	1.0					
82	82	8D	0.04	0.04	0.1	1.0					
83	83	8E	0.04	0.04	0.1	1.0					
84	84	8F	0.04	0.04	0.1	1.0					
85	85	8G	0.04	0.04	0.1	1.0					
86	86	8H	0.04	0.04	0.1	1.0					
87	87	8I	0.04	0.04	0.1	1.0					
88	88	8J	0.04	0.04	0.1	1.0					
89	89	8K	0.04	0.04	0.1	1.0					
90	90	8L	0.04	0.04	0.1	1.0					
91	91	8M	0.04	0.04	0.1	1.0					
92	92	8N	0.04	0.04	0.1	1.0					
93	93	8O	0.04	0.04	0.1	1.0					
94	94	8P	0.04	0.04	0.1	1.0					
95	95	8Q	0.04	0.04	0.1	1.0					
96	96	8R	0.04	0.04	0.1	1.0					
97	97	8S	0.04	0.04	0.1	1.0					
98	98	8T	0.04	0.04	0.1	1.0					
99	99	8U	0.04	0.04	0.1	1.0					
100	100	8V	0.04	0.04	0.1	1.0					

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
87	EXECUTION	10000	43	EXECUTION	1000
	490			104	

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:

U.S.A.V. 7075H

TTC (UG F-12) SEC (174)

DONALD W. IVINS (PSIA)

VINCE M. HARRIS (PSIA)

WATER VAPORIZATION PRESSURE (PSIA) =

0.1692

0.18997E+04

6.0000

FULL PARAMETERS:

WAKE FRACTION	WAKE FRACTION
RELATIVE ROTATIONAL EFFICIENCY	RELATIVE ROTATIONAL EFFICIENCY
NUMBER OF PROPELLERS	NUMBER OF PROPELLERS
DEPT-ICE LIMIT (FT)	DEPT-ICE LIMIT (FT)
21.92	21.92
30.00	30.00

PROPELLER PARAMETERS:

NUMBER OF BLADES	MATERIAL TYPE	ALLOWABLE STRESS (PSI)
6.0	STAINLESS STEEL	1400.0

SELECTION VALUES:

[illegible]

CONSTRAINT VALUES:

MAX	LIA (FY)	36,000J
MIN	AE/AO	45,4788
MIN	T/C	0.779593


```

FINAL OPTIMIZATION INFORMATION
OBJ = -C.665992E+00
DECISION VARIABLES (X-VECTOR)
1) C.80177E+00 0.64132E+00 0.50364E+00
CONSTRAINT VALUES (C-VECTOR)
1) -C.46470E+00 -0.59530E+00 -0.24820E+02 -0.97418E+03 -0.30178E+00 0.17773E-02
2) -C.15810E-01 -0.87642E-01 -0.24307E-02 -0.36772E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
8
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(OBJ1)-OBJ11) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 45
OBJECTIVE FUNCTION WAS EVALUATED 230 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 230 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.66599E+00
GLOBAL LOCATION

DESIGN VARIABLES		GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
ID	D. V.				
1	NC	3	0.4000E+00	0.90177E+00	0.1000E+01
2		23	0.1000E-01	0.64452E+00	0.1000E+01
3		7	0.4000E+00	0.90364E+00	0.14000E+01

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.1000E+16	0.4000E+00	0.0
2	2	0.1000E+16	0.4000E+00	0.0
3	3	0.1000E+16	0.4000E+00	0.0
4	4	0.1000E+16	0.4000E+00	0.0
5	5	0.1000E+16	0.4000E+00	0.0
6	6	0.1000E+16	0.4000E+00	0.0
7	7	0.1000E+16	0.4000E+00	0.0
8	8	0.1000E+16	0.4000E+00	0.0
9	9	0.1000E+16	0.4000E+00	0.0
10	10	0.1000E+16	0.4000E+00	0.0

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 2
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:
USAPV,IC75R

ENVIRONMENTAL PARAMETERS:
TEMP (DEG F) SEC2/FT4) = 4.5000
DENSITY (LBF-SEC2/FT4) = 1.9852
VISCOSITY (FT2/SEC) = 0.1899997E-04
ATMOSPHERIC PRESSURE (PSIA) = 14.6970
WATER VAPORIZATION PRESSURE (PSIA) = 0.2994

HULL PARAMETERS:
WAKE FRACTION = 0.2200
THRUST REDUCTION FRACTION = 0.1900
RELATIVE ROTATIVE EFFICIENCY = 1.0250
NUMBER OF PROPELLERS = 1
DEPTH OF SHATT CENTERLINE (FT) = 21.9827
DIAPETER (FT) = 30.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 2400.0

SELECTION VALUES:
PE (HP) = 4057.3
V (FT/SEC) = 33.0842
VA (KNOTS) = 30.0661
N (RPM) = 21.6500
QS (FT-LBF) = 115.0000
PD (HP) = 23744.90
KT = 0.6415
KQ = 0.1719
KDELTA = 0.0266
ETAL = 0.6660
REV75R = 0.5E+08
DIA (FT) = 24.2533
P/C = 0.9036
AE/AC = 0.8088
T/C .75R = 0.0397

CONSTRAINT VALUES:
MAX CIA (FT) = 30.0000
MIN AE/AC = 0.5070
MIN T/C .75R = 0.064709

PROGRAM CALLS TO ANALYZ
 CALL CALLS
 1 2312
 2 3
 3


```

CCCCC  GGGGG  PPPPP  EEEEE  SSSSS
C      G      P      E      S
C      G      P      E      S
C      G      P      E      S
C      G      P      E      S
CCCCC  GGGGG  PPPPP  EEEEE  SSSSS

```

C N T R L P R G R A M
 F O R
 E N G I N E E R I N G S Y N T H E S I S

T I T L E
 N A G E N I N G E N B - S E R I E S P R O P E L L E R O P T I M I Z A T I O N

TITLE: MAGENJEN B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:
CALCULATION CONTROL: NCALC = 3
NUMBER OF GLOBAL DESIGN VARIABLES: NDSV = 3000000
NUMBER OF SENSITIVITY VARIABLES: NDSV = 3000000
NUMBER OF FUNCTIONS IN TWO-SPACE: N2VAR = 0
NUMBER OF APPROXIMATING VARIABLES: N2VAR = 0
INPUT OPTIMIZATION CODE: IPBUC = 0
DEBUG PRINT CODE:

CALCULATION CONTROL: NCALC
PEAKING ANALYSIS
SINGLE ANALYSIS
SENSITIVITY
N2O-VARIABLE FUNCTION SPACE
OPTIMUM SENSITIVITY
APPROXIMATE OPTIMIZATION

* * OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01
CONSTRAINT PARAMETERS (IF ZERO, COMMON DEFAULT WILL OVER-RIDE)
IPRINT 1000 ICHGR -1 NCAL 30 LINCBI N2MAX N2FCG 0

FOLCH 0.1000E-03 FUCHM 0.1000E-02 CTMIN 0.0
CIL 0.0 CILMIN 0.0 THETA 0.0 PHI 0.0
DELFUN 0.0 DAEFUN 0.0 ALPHAX 0.0 ARGBJ1 0.0

DESIGN VARIABLE INFORMATION
NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT
NO. LOWER BOUND UPPER BOUND INITIAL VALUE SCALE
1 0.4000E+00 0.1100E+01 0.4000E+00 0.1000E+01
2 0.1000E-01 0.1100E+01 0.1000E+00 0.1000E+01
3 0.1000E+03 0.1100E+01 0.1000E+01 0.1000E+01
4 0.3000E-02 0.1100E+01 0.3000E-01 0.1000E+01

DESIGN VARIABLES
D.V. GLOBAL MULTIPLYING
ID AC. VAR. NO. LOWER BOUND UPPER BOUND
1 1 1 0.1000E+01 0.1000E+01
2 2 0.1000E+01 0.1000E+01
3 3 0.1000E+01 0.1000E+01
4 4 0.1000E+01 0.1000E+01

CONSTRAINT INFORMATION

HERE ARE 12 CONSTRAINT SETS
ID VAR1 GLOBAL L1 LINEAR LOWER BOUND UPPER BOUND NORMALIZATION FACTOR
1 1 1 1 1 0.1000E+01 0.1000E+01
2 2 2 2 2 0.1000E+01 0.1000E+01
3 3 3 3 3 0.1000E+01 0.1000E+01
4 4 4 4 4 0.1000E+01 0.1000E+01
5 5 5 5 5 0.1000E+01 0.1000E+01
6 6 6 6 6 0.1000E+01 0.1000E+01
7 7 7 7 7 0.1000E+01 0.1000E+01
8 8 8 8 8 0.1000E+01 0.1000E+01
9 9 9 9 9 0.1000E+01 0.1000E+01
10 10 10 10 10 0.1000E+01 0.1000E+01
11 11 11 11 11 0.1000E+01 0.1000E+01
12 12 12 12 12 0.1000E+01 0.1000E+01

TOTAL NUMBER OF CONSTRAINING PARAMETERS = 14

• * ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
87	EXECUTION	10000	57	EXECUTION	1000
	570			124	


```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCNA"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
    Q5,N
    TEMP (DEG F) = 60.4000
    DENSITY (LBF-SEC2/FT4) = 1.9892
    VISCOSITY (FT2/SEC) = 0.1699997E-04
    ATMOSPHERIC PRESSURE (PSIA) = 14.6970
    WATER VAPORIZATION PRESSURE (PSIA) = 0.2994

FULL PARAMETERS:
    WAKE FRACTION = 0.2400
    THRUST COEFFICIENT FRACTION = 0.1900
    RELATIVE ROTATIVE EFFICIENCY = 1.0250
    NUMBER OF PROPELLERS = 1.0
    DEPTH OF SHaft CENTERLINE (FT) = 21.9827
    DIAPHRAGM LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:
    NUMBER OF BLADES = 6.0
    MATERIAL TYPE = STAINLESS STEEL
    ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
    PE (FT)/SEC = *****
    V (KNOTS) = 50.000
    VA (KNOTS) = 50.000
    N (RPM) = 23.1042
    US (FT-LBF) = 110.0000
    PD (HP) = 25344.50

    ZT = 0.1000
    KY = 0.002
    REYN = 0.184
    DIA (FT) = 0.4640
    P/C = 21.7272
    AE/AC = 1.0000
    T/C = 0.754

CONSTRAINT VALUES:
    MAX DIA (FT) = 30.0000
    MIN AE/AC = 107.8435
    MIN T/C = 0.62794

```



```

*****
*               *
*   C   P   M   I   N   *
*               *
*   FURTRAN PROGRAM FOR *
*               *
*   CONSTRAINED FUNCTION MINIMIZATION *
*               *
*****

```

INITIAL FUNCTION INFORMATION

```

OBJ = -C.118442E+0C
DECISION VARIABLES (X-VECTOR)
11  C.4C6000E+00  0.10000E+00  0.10010E+01  C.50000E+02  0.50000E-01
CONSTRAINT VALUES (G-VECTOR)
11  -C.62500E-01 -0.93750E+00 -0.17803E+03  0.78128E+03  C.10000E+00 -0.50000E+00
11  -C.3C101E-01 -0.77352E-01 -0.59998E+00  0.60909E+01  0.26661E+03  0.25585E+00

```



```

FINAL OPTIMIZATION INFORMATION
LBJ = -C.75045E+0C
LEJISIA VARIABLES (X-VECTOR)
1) C.75462E+00 0.99270E+00 0.11986E+01 0.52182E+02 C.49512E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.62054E+00 -0.37954E+00 -0.23017E+05 -0.97558E+03 -C.29442E+00 -0.23805E-02
2) -C.30014E-01 -0.74351E-01 -0.36015E-05 -0.25551E+00 -0.41855E+00 -0.45777E+00
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5
6
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(1-CBJ(1-1)/OBJ(1)) LESS THAN DELFUN FOR 30 ITERATIONS
ABS(OBJ(1)-OBJ(1-1)) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 80
OBJECTIVE FUNCTION WAS EVALUATED 544 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 544 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.7576E+00
GLOBAL LOCATION

DESIGN VARIABLES

ID	L.V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.0000E+00	0.7576E+00	0.1000E+01
2	2	2	0.0000E+00	0.9795E+00	0.1000E+01
3	3	3	0.0000E+00	0.1588E+00	0.1000E+01
4	4	4	0.0000E+00	0.5189E+02	0.4000E+03
5	5	5	0.0000E+02	0.4591E-01	0.0000E+00

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.0000E+00	0.0000E+00	0.00
2	2	0.0000E+00	0.3950E+00	0.00
3	3	0.0000E+00	0.2301E+02	0.00
4	4	0.0000E+00	0.0000E+00	0.00
5	5	0.0000E+00	0.0000E+00	0.00
6	6	0.0000E+00	0.0000E+00	0.00
7	7	0.0000E+00	0.0000E+00	0.00
8	8	0.0000E+00	0.0000E+00	0.00
9	9	0.0000E+00	0.0000E+00	0.00
10	10	0.0000E+00	0.0000E+00	0.00
11	11	0.0000E+00	0.0000E+00	0.00
12	12	0.0000E+00	0.0000E+00	0.00


```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCHA"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
Q5,N      = 64.0000
TEMP (DEG F) = 1.0000
DENSITY (LBF-SEC/FT**3) = 0.1800
VISCOSITY (CENTI-POISE) = 1.8000
WATER VAPORIZATION PRESSURE (PSIA) = 0.2000

FULL PARAMETERS:
WAKE FRACTION = 0.2000
THRUJET REDUCTION FRACTION = 0.1900
RELATIVE ROTATIVE EFFICIENCY = 1.0250
NUMBER OF PROPELLERS = 1
DIAPYJET LINE CENTERLINE (FT) = 21.0000
DIAPYJET LINE (FT) = 36.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
PE (HP) = 15995.5
V (FT/SEC) = 1182.5
VA (KNOTS) = 30.8138
N (RPM) = 24.1127
QS (FT-LBF) = 10.0000
PD (HP) = 2344.90
AL = 0.927
KT = 0.155
ETAL = 0.0326
REYN = 0.2570
DIA (FT) = 9.5608
P/D = 22.3645
AE/AL = 1.1986
T/C = 0.75F

CONSTRAINT VALUES:
MAX L/A (FT) = 30.0000
MIN AE/AD = 0.4622
MIN T/C = 0.75F

```


PROGRAM CALLS TO ANALYZ

ICALL	CALLS
1	1
2	2
3	3


```

CCCCC  CCCCCC  P P P P P P P  E E E E E  S S S S S
C C C C C  C C C C C  C C C C C  C C C C C  S S S S S
C C C C C  C C C C C  C C C C C  C C C C C  S S S S S
C C C C C  C C C C C  C C C C C  C C C C C  S S S S S
C C C C C  C C C C C  C C C C C  C C C C C  S S S S S

```

C C N T R C L P R O G R A M
 F L R
 E N C I A E K I N G S Y N T H E S I S

T I T L E
 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION

TITLE:
WAGENINCKEN B-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:
CALCULATION CONTROL DESIGN VARIABLES, NCALC = 3
NUMBER OF GLOBAL DESIGN VARIABLES, NDV = 3
NUMBER OF SENSITIVITY VARIABLES, NSV = 0
NUMBER OF FUNCTIONS IN TWO-SPACE, NSV = 0
NUMBER OF APPROXIMATING VARIABLES, NAPPX = 0
DEBUG PRINT CODE, 10000 = 0
DEBUG PRINT CODE, 10000 = 0

CALCULATION CONTROL, NCALC
VALUE
MEANING
1 SINGLE ANALYSIS
2 INITIALIZATION
3 TWO-SPACE
4 TWO-SPACE
5 TWO-SPACE
6 APPROXIMATE OPTIMIZATION

• OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = 0.1000E+01

COMPUTER PARAMETERS (IF ZERO, COMMON DEFAULT WILL OVER-RIDE)

IPRINT ITHX ICHXK NSCAL ITHM LINGBJ MAGN1 NFDG
1 1000 0 30 0 15 0

FOCH FDCM CT CTMIN
0.1000E-03 0.1000E-03 0.0 0.0
CTL CTMIN ITHETA PHI
0.0 0.0 0.0 0.0
DELFUN DAFUN ALPHAX ABCBJ
0.0 0.0 0.0 0.0

DESIGN VARIABLE INFORMATION
NON-ZERO INITIAL VALUE WILL OVER-RIDE MODULE INPUT
D. V. LOWER UPPER INITIAL SCALE
NO. 1 0.2000E+00 0.1000E+01 0.3000E+02 0.1000E+01
2 0.2000E+01 0.1000E+02 0.1000E+03 0.1000E+01
3 0.2000E+01 0.1000E+02 0.1000E+03 0.1000E+01
4 0.2000E+01 0.1000E+02 0.1000E+03 0.1000E+01
5 0.2000E+02 0.1000E+03 0.3000E+04 0.1000E+01

DESIGN VARIABLES
L.V. GLOBAL MULTIPLYING
NO. VAR. NO.
1 1 1 0.1000E+01
2 1 1 0.1000E+01
3 23 0.1000E+01
4 7 0.1000E+01
5 9 0.1000E+01

CONSTRAINT INFORMATION

THERE ARE 12 CONSTRAINT SETS

10 VAR. 1 GLOBAL LINEAR LOWER BOUND UPPER BOUND
1 1 1 0 0.1000E+16 0.1000E+16
2 1 1 0 0.1000E+16 0.1000E+16
3 1 1 0 0.1000E+16 0.1000E+16
4 1 1 0 0.1000E+16 0.1000E+16
5 1 1 0 0.1000E+16 0.1000E+16
6 1 1 0 0.1000E+16 0.1000E+16
7 1 1 0 0.1000E+16 0.1000E+16
8 1 1 0 0.1000E+16 0.1000E+16
9 1 1 0 0.1000E+16 0.1000E+16
10 1 1 0 0.1000E+16 0.1000E+16
11 1 1 0 0.1000E+16 0.1000E+16

NORMALIZATION
FACTORS
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01

UPPER
BOUND
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0
0.0

NORMALIZATION
FACTORS
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01
0.1000E+01

LOWER
BOUND
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16
0.1000E+16


```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCK"

DESIGN VARIABLES SPECIFIED: Q5,N
ENVIRONMENTAL PARAMETERS:
TEMP (DIG F) = 64.000
DEWPT (DIG F) = 50.000
AIR CRYSTALL PRESSURE (PSIA) = 0.189897E-04
WATER VAPORIZATION PRESSURE (PSIA) = 0.5994

FULL PARAMETERS:
WAKE FRACTION = 0.200
THRUST REDUCTION FRACTION = 0.1900
RELATIVE EFFICIENCY = 1.0750
DIPPERHEAD CENTERLINE (FT) = 31.987
DIAPETER LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 2400.0

SELECTION VALUES:
PE (HP/SEC) = *****
V (KNOTS) = 50.0000
VA (KNOTS) = 50.6208
N (RPM) = 23.1042
PS (FT-LBF) = 10.0000
PD (HP) = 2544.90
J = 0.1000
KQ = 0.4002
ETAC = 0.0538
REY75R = 0.1186
DIA (FT) = 24.4710
ACAC = 21.7272
T/C = 75R = 0.0000
T/C = 75R = 0.0500

CONSTRAINT VALUES:
MAX DIA (FT) = 30.0000
MIN REY75R = 0.1186
MIN T/C = 75R = 0.179593

```



```

C L A M I N
FURTHER PROGRAM FOR
CONSTRAINED FUNCTION MINIMIZATION

```

INITIAL FUNCTION INFORMATION

```
OBJ = -C.118+2E+0C
DECISICA VARIABLES (X-VECTOR)
C.4600E+00 0.1000E+00 0.1000E+02 0.5000E+01
CONSTRAINT VALUES (G-VECTOR)
1 -C.6250E+01 -0.7375E+00 0.7812E+03 C.1000E+00 -0.7000E+00
1 -C.3131E+01 -0.7735E+01 0.2609E+03 C.2286E+03 -0.2559E+01
```



```

FINAL OPTIMIZATION INFORMATION
CBJ = -C.722977E+0C
DECISION VARIABLES (X-VECTOR)
1) C.75858E+00 0.87525E+00 0.103C8E+C1 0.49656E+C2 0.63E38E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.54704E+00 -0.45296E+00 -0.22531E+C2 -0.97645E+C3 -C.29E59E+00 -0.14130E-04
   -C.43940E-01 -0.63513E-01 -0.66166E-01 -0.19217E+00 -0.46E66E+00 0.45176E-03
THERE ARE 2 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
8 12
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
ABS(CBJ(I)-OBJ(I-1)) LESS THAN DABFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 35
OBJECTIVE FUNCTION WAS EVALUATED 209 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 209 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 1 FUNCTION VALUE 0.73297E+00
GLOBAL LOCATION

DESIGN VARIABLES

ID	D. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	23	0.4000E+00	0.7855E+00	0.1000E+01
2	2	24	0.1000E-01	0.8552E+00	0.1000E+01
3	3	25	0.1000E+00	0.1030E+01	0.1000E+01
4	4	26	0.4000E+02	0.4985E+02	0.1000E+03
5	5	27	0.1000E-02	0.6883E-01	0.1000E+00

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.1000E+00	0.7855E+00	0.0
2	2	0.1000E+00	0.8552E+00	0.0
3	3	0.1000E+00	0.1030E+01	0.0
4	4	0.1000E+00	0.4985E+02	0.0
5	5	0.1000E+00	0.1030E+01	0.0
6	6	0.1000E+00	0.4985E+02	0.0
7	7	0.1000E+00	0.6883E-01	0.0
8	8	0.1000E+00	0.6883E-01	0.0
9	9	0.1000E+00	0.6883E-01	0.0
10	10	0.1000E+00	0.6883E-01	0.0
11	11	0.1000E+00	0.6883E-01	0.0
12	12	0.1000E+00	0.6883E-01	0.0
13	13	0.1000E+00	0.6883E-01	0.0
14	14	0.1000E+00	0.6883E-01	0.0
15	15	0.1000E+00	0.6883E-01	0.0
16	16	0.1000E+00	0.6883E-01	0.0
17	17	0.1000E+00	0.6883E-01	0.0
18	18	0.1000E+00	0.6883E-01	0.0
19	19	0.1000E+00	0.6883E-01	0.0
20	20	0.1000E+00	0.6883E-01	0.0
21	21	0.1000E+00	0.6883E-01	0.0
22	22	0.1000E+00	0.6883E-01	0.0


```

OPTIMIZATION RESULTS -----
DESIGN CASE NO. 2
SUBROUTINE "STRCKR"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
  Q5,A
  TDP (DEG F) = 64.4000
  DENSITY (LBF-SEC2/FT4) = 0.11899997E-04
  VISCOSITY (LBF/SEC) = 14.6970
  ATMOSPHERIC PRESSURE (PSIA) = 0.2994
  WATER VAPORIZATION PRESSURE (PSIA)

HULL PARAMETERS:
  WAKE FRACTION = 0.2200
  THRUST REDUCTION FRACTION = 0.1900
  RELATIVE ROTATIVE EFFICIENCY = 1.0250
  NUMBER OF PROPELLERS = 1.0
  DEPTH TO SHAFT CENTERLINE (FT) = 31.9827
  DIAPETER LIMIT (FT) = 30.0000

PROPELLER PARAMETERS:
  NUMBER OF BLADES = 6.0
  MATERIAL TYPE = STAINLESS STEEL
  ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
  PE (HP/SEC) = 2663.7
  V (KNOTS) = 21.8359
  VA (KNOTS) = 21.1077
  N (RPM) = 110.0000
  QS (FT-LBF) = 110130.0
  PD (HP) = 25344.50
  J = 0.753
  K = 0.723
  KYAC = 0.0322
  REYN75R = 0.7330
  DIA (FT) = 9.5E+08
  PZC = 24.2349
  AC/AD = 1.0308
  T/C .75F = 0.7986
  C.0638

CONSTRAINT VALUES:
  MAX DIA (FT) = 30.0000
  MIN ME/AD = 0.4266
  MIN T/C .75R = 0.063868

```


PROGRAM CALLS TO ANALYZE

ICALL	CALLS
1	1
2	2
3	2

ANALIZ CODES--DESIGN CASE NO. 3

```

SUBROUTINE ANALIZ(ICALC)
INTEGER#4 ICALC
REAL#4  ETAO,WEIGHT,RJCNLU,R75RCL,R75RCU,AEACCU,AEACCU,IC75CL,TC75CU,
1      FCWBAL,DIACNU,AEACCU,IC75CL,TC75CU,
2      VK,TC,WT,Z,WATRC,WATNU,TEMP,NCSCRW,HCL,PAIM,
3      PWATVA,PRCMAT,DIACNU,AEACCU,IC75CL,TC75CU,
4      C75R,R75R,KT,KC,PD,TUEV,PTDEV,PECEV,CREC,POREQ
5      COMMON /GLOBECM/ETAO,WEIGHT,AEACCU,IC75CL,TC75CU,POWBAL,DIACNU,
1RJCNLU,R75RCL,R75RCU,TC75CL,TC75CU,
2AEACCU,IC75CL,TC75CU,
1PRCMAT,DIACNU,ETARR,AEACCU,IC75CL,TC75CU,
      COMMON /PARAM/VK,TD,WT,Z,WATRO,WATNU,TEMP,NCSCRW,HCL,PAIM,PWATVA,
      CIALIM,ETARR,AEACCU,IC75CL,TC75CU,
      THIS SUBROUTINE, COUPLED WITH COPEX/CONMIN, CONSTITUTES ANALYSIS
      METHOD FOR "DESIGN CASE 3" PROPELLER SELECTION PROBLEMS

INPLT-INITIALIZATION PHASE
PII=3.14159264
IF(.NOT.(ICALC.EC.1))GO TO 1
      SET "DESIGN CASE 3" PARAMETERS
      ENVIRONMENTAL
      TEMP=55.0
      WATRC=1.9905
      WATNU=.000C12817
      PAIM=14.7
      PWATVA=.247
      PROPELLER PARAMETERS
      Z=6.0
      PRCMAT=5.0
      FULL PARAMETERS
      WT=C.22
      TC=0.1725
      ETARR=1.025
      NCSCRW=1.0
      HCL=19.0
      SET DESIGN VARIABLES HELD FIXED FOR "DESIGN CASE 3"
      PE=21252.6

```



```

VK=24.24
V=1.68E*VK
N=1C5.C
QS=150C606.75
DIA=22.0
CC
CC
CC
    END CF INPUT-INITIALIZATION PHASE
GO TO 3
C
C
C
    EXECUTION PHASE
1 CONTINUE
  IF(.NOT.(ICALC.EQ.2))GO TO 2
    CALL R-CAL(RJ)
    CALL CF75RA(C75R,R75R)
    CALL REY75R(C75R,R75R,KT,KQ)
    CALL CCEFSR(RJ,KT,KQ,ETA)
    CALL OFWEFF(RJ,KT,KQ,ETA)
    CALL JCN(RJ,RJCN,RJCN)
    CALL REYCN(R75R,R75RCL,R75RCL)
    CALL REYCCN(Z,AEQVAO,TC75R,AEACCL,AEACCU,TC75CL,TC75CL)
    CALL BLPOW3(KT,KQ,POWBAL,DIA)
    CALL CAVCNA(KT,AEACCV)
    CALL STRCNK(KC,KT,C75R,TCSTRS)
    CALL WGTAL(C75R)
    END OF EXECUTION PHASE
CC
CC
CC
    GO TO 3
2 CONTINUE
    OUTPUT-RESULT PHASE
C
C
C
    PC=(2.C*PII*QS*N)/33000.0
    TDEV=(KT*WATRC*(DIA**4)*((N/60.0)**2))
    PTDEV=(TDEV*((1.0-WT)*V))/550.C
    PEDEV=((((1.0-ID)/(1.0-WT)))*ETARR*PTDEV)*NOSCRW)
    QREC=(KQ*WATRC*(DIA**5)*((N/60.0)**2))
    PLFEQ=(2.0*PII*QREQ*N)/33000.0
    ETACSP=((((1.0-WT)/(1.0-ID))*((PE/NOSCRW)*33000.0)/ETARR))/
      (2.C*PII*QS*N)
    WRITE(6,9000)
    WRITE(6,9001)
    WRITE(6,9002) TEMP,WATRO,WATNU,PATM,PWATVA
    WRITE(6,9003) WT,TD,ETARR,NOSCRW,HCL,DIALIM
    WRITE(6,9004) Z
    IF(.NOT.(PRGMAT.EQ.1.0))GO TO 81
APP0C490
APP0C500
APP0C510
APP0C520
APP0C530
APP0C540
APP0C550
APP0C560
APP0C570
APP0C580
APP0C590
APP0C600
APP0C610
APP0C620
APP0C630
APP0C640
APP0C650
APP0C660
APP0C670
APP0C680
APP0C690
APP0C700
APP0C710
APP0C720
APP0C730
APP0C740
APP0C750
APP0C760
APP0C770
APP0C780
APP0C790
APP0C800
APP0C810
APP0C820
APP0C830
APP0C840
APP0C850
APP0C860
APP0C870
APP0C880
APP0C890
APP0C900
APP0C910
APP0C920
APP0C930
APP0C940
APP0C950
APP0C960

```



```

1X,25X,ALL CWABLE STRESS (PSI),12X,=,F10.1,/,
9006 1 FORMAT(1X,25X,MATERIAL TYPE,21X,=,CAST STEEL,/,F10.1,/,
1X,25X,ALL CWABLE STRESS (PSI),12X,=,BRONZE,/,F10.1,/,
9007 1 FORMAT(1X,25X,MATERIAL TYPE,21X,=,BRONZE,/,F10.1,/,
1X,25X,ALL CWABLE STRESS (PSI),12X,=,F10.1,/,
9008 1 FORMAT(1X,25X,MATERIAL TYPE,21X,=,NI-AL BRCNZE,/,F10.1,/,
1X,25X,ALL CWABLE STRESS (PSI),12X,=,F10.1,/,
9009 1 FORMAT(1X,25X,MATERIAL TYPE,21X,=,STAINLESS STEEL,/,F10.1,/,
1X,25X,ALL CWABLE STRESS (PSI),12X,=,F10.1,/,
9010 1 FORMAT(1X,25X,MATERIAL TYPE,21X,=,NCT CCNSIDEREC,/,F10.1,/,
9011 1 FORMAT(1X,25X,SELECTI CN VALUES,12X,=,PE (HP),27X,=,F10.1,/,
1X,25X,V (FT/SEC),23X,=,F10.4,/,
1X,25X,N (RPM),23X,=,F10.4,/,
1X,25X,G (FT-LBF),23X,=,F12.1,/,
1X,25X,PD (HP),23X,=,F10.2,/,
1X,25X,ETAG SPECIFIED,20X,=,F10.4,/,
1X,25X,J,23X,=,F10.4,/,
1X,25X,K,23X,=,F10.4,/,
1X,25X,KQ,23X,=,F10.4,/,
1X,25X,ETAG,23X,=,F10.4,/,
1X,25X,REV75R,23X,=,F10.1,/,
1X,25X,CIA (FT),23X,=,F10.4,/,
1X,25X,PD,23X,=,F10.4,/,
1X,25X,AE/AC,23X,=,F10.4,/,
1X,25X,T/C,23X,=,F10.4,/,
1X,25X,75R,23X,=,F10.1,/,
9012 1 FORMAT(1X,25X,BLADE WEIGHT (LBF),16X,DIA (FT),22X,=,F10.4,/,
1X,25X,CONSTR INT VALUES,11X,MAX DIA (FT),22X,=,F10.4,/,
1X,25X,MIN AE/AU,25X,=,F10.4,/,
1X,25X,MIN T/C,75R,22X,=,F10.6,/,
9013 1 FORMAT(1X,25X,FROPELLER "POINTS",10X,PE DEVELOPED (HP),17X,=,F10.1,/,
1X,25X,V (KNOTS),23X,=,F10.4,/,
1X,25X,G (FT-LBF),14X,=,F12.1,/,
1X,25X,N (RPM),23X,=,F10.4,/,
1X,25X,PD REQUIRED (HP),18X,=,F10.2,/,
END

```


CONTROL CARD IMAGES--DESIGN CASE NO. 3

334

\$V
END

COPEX OUTPUT--DESIGN CASE NO. 3

[illegible]

CONTROL PROGRAM FOR ENCIPHERING SYNTHESIS

TABLE WAGENINGEN 6-SERIES PROPELLER OPTIMIZATION

CARD IMAGES OF CONTROL DATA

LAKI.

IMAGE

12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22																																																																														

TITLE:
WAGENINCK D-SERIES PROPELLER OPTIMIZATION

CONTROL PARAMETERS:
CALCULATION CONTROL NCALC = 3
NUMBER OF GLOBAL DESIGN VARIABLES, NSV = 3
NUMBER OF SENSITIVITY VARIABLES, NSV = 0
NUMBER OF FUNCTIONS IN TWO-SPACE, NZVAR = 0
NUMBER OF APPROXIMATING VAR, NXAPRX = 0
INPUT INFORMATION PRINT CODE, IPNPUT = 0
DEBUG PRINT CODE, IPDBG = 0

CALCULATION CONTROL, NCALC
VALUE
MEANING
1 SINGLE ANALYSIS
2 OPTIMIZATION
3 SENSITIVITY
4 CONTROL VARIABLE
5 CONTROL SENSITIVITY
6 APPROXIMATE OPTIMIZATION

* * OPTIMIZATION INFORMATION

GLOBAL VARIABLE NUMBER OF OBJECTIVE
MULTIPLIER (NEGATIVE INDICATES MINIMIZATION) = -0.1000E+01
CONFIN PARAMETERS (IF ZERO, CONFIN DEFAULT WILL OVER-RIDE)

IPRINT	ITMAX	ICNCR	NSCAL	ITERM	LINDBJ	NACHXI	NFUG
1	1000	0	-1	30	0	15	0

FUCH	FUCHM	CF	CTMIN
0.1000E-02	0.1000E-02	-0.1000E-01	0.0

CTL	CTLMIN	THETA	PHI
0.0	0.0	0.0	0.0

DEFLU	DAEFUN	ALPAX	ABUJJI
0.0	0.0	0.0	0.0

DESIGN VARIABLE INFORMATION

NON-ZERO	INITIAL VALUE	WILL EVER-RIDE	MODULE	INPUT
D. V.	LCNCR	UPPER	INITIAL	VALUE
1	0.2000E+00	0.1100E+01	0.1000E+01	0.1000E+01
2	0.5000E+00	0.1400E+01	0.1300E+01	0.1000E+01
3	0.5000E+02	0.5000E+03	0.1000E-01	0.1000E+00

DESIGN VARIABLES

ID	D. V.	GLOBAL	MULTIPLYING
1	1	3	0.1000E+01
2	2	9	0.1000E+01
3	3	9	0.1000E+01

CONSTRAINT INFORMATION

THERE ARE 12 CONSTRAINT SETS

ID	GLOBAL	GLOBAL LINEAR	LOWER	UPPER	NORMALIZATION	NORMALIZATION
1	1	2	0	0	0.1000E+01	0.1000E+01
2	1	12	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
3	1	13	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
4	1	14	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
5	1	15	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
6	1	16	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
7	1	17	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
8	1	18	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
9	1	19	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
10	1	20	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
11	1	21	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01
12	1	22	-0.1100E+16	0.1000E+01	0.1000E+01	0.1000E+01

TOTAL NUMBER OF CONSTRAINT PARAMETERS = 12

* * ESTIPATED DATA STORAGE REQUIREMENTS			
REAL		INTEGER	
INPUT	EXECUTION	INPUT	EXECUTION
75	1000	49	112
	AVAILABLE		AVAILABLE
	10000		1000


```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:
ENVIRONMENTAL PARAMETERS:
PE,V,QS,N,DIA
TEMP IDEG F = 55.0000
DENSITY (LBF-SEC2/FT4) = 1.9905
VISCOSITY (FT2/SEC) = 0.12817004E-04
ATMOSPHERIC PRESSURE (PSIA) = 14.7000
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

FULL PARAMETERS:
WAKE FRACTION = 0.2200
THRUST REDUCTION FRACTION = 0.1725
RELATIVE ROTATIVE EFFICIENCY = 1.3250
NUMBER OF PROPELLERS = 1.0
DEPTH IC SHAF CENTERLINE (FT) = 15.0000
DIAPETER LIMIT (FT) = 0.0

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 5400.0

SELECTION VALUES:
PE (HP) SEC = 21292.0
N (RPM) = 105.0000
QS (FT-LBF) = 150060.0
PD (HP) = 30600.05
ETAC SPECIFIED = 0.6527
J = 0.0250
KI = 0.2500
KQ = 0.2500
KAC = 0.6505
REV75R = 0.8E+08
DIA (FT) = 21.0000
P/D = 1.3000
AE/AG75F = 1.0000
T/C75F = 0.0100
BLADE WEIGHT (LBF) = 2563.1

CONSTRAINT VALUES:
MAX CIA (FT) = 0.0
MIN AE/AG = 0.10200
MIN T/C75R = 0.154830

PROPELLER "FCINIS":
PE DEVELOPED (HP) = 24056.6
V (KNOTS) = 24.2400
QS REQUIRED (FT-LBF) = 1875936.0
N (RPM) = 105.0000
PD REQUIRED (HP) = 37503.62

```



```

*****
*                                     *
*      C L P M I N                  *
*      FORTRAN PROGRAM FOR          *
*      CONSTRAINED FUNCTION MINIMIZATION
*                                     *
*****

```

INITIAL FUNCTION INFORMATION

```

OBJ =  C.256313E+04
DECISION VARIABLES (X-VECTOR)
1)  C.16000E+01  0.13000E+01  0.10000E-01
CONSTRAINT VALUES (C-VECTOR)
1)  -C.51811E+00 -0.58185E+00 -0.36158E+02 -0.96224E+03 -C.50000E+00  0.40000E+00
1)  C.98987E-02 -0.24470E+00 -0.26131E+00  C.25012E+00  C.26567E-01  0.44833E+01

```


FINAL OPTIMIZATION INFORMATION

```

OBJ = C.128426E+05
DECISION VARIABLES (X-VECTOR)
1) C.75444E+00 0.11813E+01 0.75385E-01
CONSTRAINT VALUES (C-VECTOR)
1) -C.51811E+00 -0.48185E+00 -0.13450E+02 -0.98555E+02 -0.29444E+00 -0.55616E-02
7) -C.55491E-01 -0.17531E+00 0.58471E-01 -0.61652E-01 0.71621E-01 -0.12991E+00
THERE ARE 1 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
0
THERE ARE 2 VIOLATED CONSTRAINTS
CONSTRAINT NUMBERS ARE
9 11
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINATION CRITERION
TEN CONSECUTIVE ITERATIONS FAILED TO PRODUCE A FEASIBLE DESIGN
NUMBER OF ITERATIONS = 10
OBJECTIVE FUNCTION WAS EVALUATED 37 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 37 TIMES

```


OPTIMIZATION RESULTS

OBJECTIVE FUNCTION 2 FUNCTION VALUE 0.12843E+05
GLOBAL LOCATION

DESIGN VARIABLES		GLOBAL VAR. NO.		LOWER BOUND		VALUE		UPPER BOUND	
ID	D. V. NO.	VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND				
1	1	1	0.2000E+00	0.1817E+00	0.1400E+01				
2	2	2	0.3000E-02	0.1817E-01	0.1400E+01				
3	3	3	0.3000E-02	0.1817E-01	0.1400E+01				

DESIGN CONSTRAINTS		LOWER BOUND		VALUE		UPPER BOUND	
ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND			
1	1	-0.11000E+10	-0.51819E+00	0.0			
2	2	-0.11000E+10	-0.41819E+00	0.0			
3	3	-0.11000E+10	-0.31819E+00	0.0			
4	4	-0.11000E+10	-0.21819E+00	0.0			
5	5	-0.11000E+10	-0.11819E+00	0.0			
6	6	-0.11000E+10	-0.01819E+00	0.0			
7	7	-0.11000E+10	0.08181E+00	0.0			
8	8	-0.11000E+10	0.18181E+00	0.0			
9	9	-0.11000E+10	0.28181E+00	0.0			
10	10	-0.11000E+10	0.38181E+00	0.0			
11	11	-0.11000E+10	0.48181E+00	0.0			
12	12	-0.11000E+10	0.58181E+00	0.0			


```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3
SUBROUTINE "STRUNK"

DESIGN VARIABLES SPECIFIED:  P=0.05,N,DIA
ENVIRONMENTAL PARAMETERS:
TEMP (DEG F) = 55.0000
DENSITY (LBF-SEC/FT3) = 0.119900
VISCOSITY (FT2/SEC) = 0.119700e-04
ATMOSPHERIC PRESSURE (PSIA) = 14.7000
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

FULL PARAMETERS:
WAKE FRACTION = 0.2500
RELATIVE PROPELLER EFFICIENCY = 1.0150
NUMBER OF PROPELLERS = 1
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000
DIAPETER LIMIT (FT) = 0.00

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STEEL
ALLOWABLE STRESS (PSI) = 24000.0
SELECTION VALUES:
PE (HP) = 21294.6
V (FT/SEC) = 40.9171
N (RPM) = 105.0000
PS (FT-LBF) = 1500607.0
PD (HP) = 33600.05
ETAC SPECIFIED = 0.6927
J = 0.8290
KT = 0.2349
KW = 0.0448
ETAC = 0.6913
KE (HP) = 0.3808
P/D = 25.0000
A/C = 1.7811
T/C = 0.754
T/C = 0.054
BLADE WEIGHT (LBF) = 15843.6

CONSTRAINT VALUES:
MAX DIA (FT) = 0.0
MIN DIA (FT) = 0.0515
MIN T/C = 0.69016

PROPELLER "FCRHS":
PE DEVELOPED (HP) = 21168.1
V (KNOTS) = 24.2400
WS REQUIRED (FT-LBF) = 1408084.0
W REQUIRED (HP) = 1050000
PU REQUIRED (HP) = 281501.33

```


PROGRAM CALLS TO ANALYZ
ICALL CALLS
1 2
2 3


```

CCCCC  CCCCCC  PPPPPP  EEEEE  SSSSS
C      C      P      E      S
C      C      P      E      S
C      C      P      E      S
C      C      P      E      S
CCCCC  CCCCCC  P      EEEEE  SSSSS

```

C N T R L P R G R A M
 F O R
 E N G I N E E R I N G S Y N T H E S I S

T I T L E
 WAGENINGEN B-SERIES PROPELLER OPTIMIZATION

CARD IMAGES OF CONTROL DATA

CARL

IMAGE

[illegible]

* * ESTIMATED DATA STORAGE REQUIREMENTS

INPUT	REAL	AVAILABLE	INPUT	INTEGER	AVAILABLE
75	EXECUTION	10000	49	EXECUTION	1000
	504			112	


```

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3
                               SUBROUTINE "STRUKK"

DESIGN VARIABLES SPECIFIED:  PE,V,CS,M,DIA
ENVIRONMENTAL PARAMETERS:
TYPE (DUU,FI) = 55.0000
DENSITY (LBF/SEC) = 1.9905
VISCOSITY (LBF/SEC) = 0.181700E-0
ATMOSPHERIC PRESSURE (PSIA) = 14.7000
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

HULL PARAMETERS:
WAKE FRACTION COEFFICIENT = 0.2700
THRU-ROTOR EFFICIENCY = 0.9000
REVERSE ROTATION EFFICIENCY = 1.0250
NUMBER OF PROPELLERS = 1.0
DEPTH TO SHAFT CENTERLINE (FT) = 15.0000
DIAPETER LIMIT (FT) = 0.0

PROPELLER PARAMETERS:
NUMBER OF BLADES = 6.0
MATERIAL TYPE = STAINLESS STE
ALLOWABLE STRESS (PSI) = 24000.0

SELECTION VALUES:
PZ (FT/SEC) = 1700.0
N (RPM) = 36.8240
QS (FT-LBF) = 105.0000
PD (HP) = 100007.0
ETAC SPECIFIED = 0.5394
JT = 0.7800
KT = 0.3212
KV = 0.0638
ETAC = 0.6302
REYNOLDS = 0.88408
D/D0 = 2.0000
P/D = 1.3000
AE/AC = 1.0000
T/C = 0.100
BLADE HEIGHT (LBF) = 2563.1

CONSTRAINT VALUES:
MAX L/A (FT) = 0.0
MIN L/D = 0.0
MIN T/C = 0.75R

PROPELLER "FCINIS":
PE DEVELOPED (HP) = 27514.5
V (KNOTS) = 23.0000
QS REQUIRED (FT-LBF) = 204469.0
N (RPM) = 105000.0
PD REQUIRED (HP) = 40076.84

```



```

*****
C C A M I N
*****
FORTRAN PROGRAM FOR
*****
CONSTRAINED FUNCTION MINIMIZATION
*****

```

INITIAL FUNCTION INFORMATION

```

OBJ = C.256313E+04
DECISION VARIABLES (X-VECTOR)
11 C.10000E+01 0.13000E+01 0.10000E-01
CONSTRAINT VALUES (G-VECTOR)
11 -C.45160E+00 -0.50840E+00 -0.36545E+02 -0.96245E+03 -C.50100E+00 0:
71 C.9E987E-02 -0.24470E+00 -0.56066E+00 0.33589E+00 0.90555E-01 0:

```


FINAL OPTIMIZATION INFORMATION

```

C0J = C.1C6647E+05
DECISION VARIABLES (X-VECTOR)
11 C.77419E+00 0.10900E+01 0.66118E-01
CONSTRAINT VALUES (C-VECTOR)
11 -C.46220E+01 -0.5084E+00 -0.15063E+02 -0.98354E+03 -C.27419E+00 -0.
-C.46220E+01 -0.18658E+00 -0.24834E-02 -0.22189E+00 -0.25204E-02 0.
THERE ARE 3 ACTIVE CONSTRAINTS
CONSTRAINT NUMBERS ARE
5 11 12
THERE ARE 0 VIOLATED CONSTRAINTS
THERE ARE 0 ACTIVE SIDE CONSTRAINTS
TERMINAL CRITERION
ABS(C0J-11708J11) LESS THAN DLFUN FOR 30 ITERATIONS
ABS(C0J11-08J11-11) LESS THAN DLFUN FOR 30 ITERATIONS
NUMBER OF ITERATIONS = 46
OBJECTIVE FUNCTION WAS EVALUATED 102 TIMES
CONSTRAINT FUNCTIONS WERE EVALUATED 102 TIMES

```


OPTIMIZATION RESULTS

CONJECTIVE FUNCTION 2 FUNCTION VALUE 0.10465E+05
GLOBAL LOCATION

DESIGN VARIABLES

ID	C. V. NO.	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	1	0.2000E+00	0.7419E+00	0.1000E+01
2	2	2	0.4000E+00	0.1090E+01	0.1400E+01
3	3	3	0.3000E+02	0.6811E-01	0.5000E+00

DESIGN CONSTRAINTS

ID	GLOBAL VAR. NO.	LOWER BOUND	VALUE	UPPER BOUND
1	1	0.1000E+16	0.7419E+00	0.0
2	2	0.1000E+16	0.5084E+00	0.0
3	3	0.1000E+16	0.1090E+01	0.0
4	4	0.1000E+16	0.1090E+01	0.0
5	5	0.1000E+16	0.1090E+01	0.0
6	6	0.1000E+16	0.1090E+01	0.0
7	7	0.1000E+16	0.1090E+01	0.0
8	8	0.1000E+16	0.1090E+01	0.0
9	9	0.1000E+16	0.1090E+01	0.0
10	10	0.1000E+16	0.1090E+01	0.0
11	11	0.1000E+16	0.1090E+01	0.0

OPTIMIZATION RESULTS ----- DESIGN CASE NO. 3
SUBROUTINE "STRCHK"

DESIGN VARIABLES SPECIFIED:

ENVIRONMENTAL PARAMETERS:
PE, V, QS, N, DIA
TIME (DUG FT) = 55.0000
VELOCITY (DUG FT/SEC) = 16.703
WATER VAPORIZATION PRESSURE (PSIA) = 0.14333
WATER VAPORIZATION PRESSURE (PSIA) = 14.333
WATER VAPORIZATION PRESSURE (PSIA) = 0.2470

HULL PARAMETERS:

WAKE FRACTION = C.4200
THROUST FRACTION = C.4200
NUMBER OF PROPELLERS = 1.0000
DIP OF SHAFT CENTERLINE (FT) = 15.0000
DIAPETER LIMIT (FT) = C.0

PROPELLER PARAMETERS:

NUMBER OF BLADES = 6
MATERIAL TYPE = STAINLESS STE
ALLOWABLE STRESS (PSI) = 54000.0

SELECTION VALUES:

PE (HP) = 17030.0
V (FT/SEC) = 56.8240
N (RPM) = 105.0000
QS (FT-LBF) = 100607.0
PD (HP) = 30000.05

ETAC SPECIFIED

ETAC = C.5394
J = C.7669
KT = C.5063
KQ = C.0372
ETAC = C.6950
REV75R = 9.3608
DIA (FT) = 21.0000
REV75R = 1.0322
ETAC = C.0681
T/C 75R = 10464.7
BLADE WEIGHT (LBF)

CONSTRAINT VALUES:

MAX DIA (FT) = C.0
MIN AC/FO = C.7722
MIN T/C 75R = 0.68124

PROPELLER "FCINTS":

PE DEVELOPED (HP) = 17673.8
V (KNOTS) = 23.0000
QS REQUIRED (FT-LBF) = 107640.0
N (RPM) = 105.0000
PD REQUIRED (HP) = 23543.40

PROGRAM CALLS TO ANALYZ

ICALL	CALLS
1	103
2	2
3	2

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